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SOLAR FLAT PLATE

FINAL REPORT

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### FOREWORD

The analysis and computer program described herein was prepared under Contract Number NAS5-9167 for the National Aeronautics and Space Administration, Goddard Space Flight Center.

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## SUMMARY

Solar thermoelectric energy conversion panels may be constructed from modules, each of which consists of an absorber, a thermoelement or thermoelectric couple, and a radiator. The absorber collects solar heat which causes a high temperature to be established. The thermoelements convert a portion of this heat to electricity, and the unconverted heat is rejected to space by the radiator.

Analyses conducted prior to the work reported herein have been limited to mathematical models which could be handled using manual calculation techniques. Calculation accuracy has consequently not been as precise as desired. Therefore, an accurate analysis has been developed, reported herein, so that the performance characteristics of the solar flat plate may be predicted for expected operational conditions. The mathematical model obtained as a result of this study contains a number of non-linear second order differential equations which cannot be solved analytically. A digital computer program therefore has been written for the IBM-7094 so that a solution may be obtained.

Major emphasis has been placed in the writing of the program so that calculation accuracy and ease of use of the program will result. The calculation accuracy has been obtained through a careful selection of the assumptions necessary to define the mathematical model and by careful computation techniques. Ease of use has been obtained through the use of the simplified input routines and carefully presented data. The input routine which has been developed for this program is virtually independent of format and the required data may be read in almost any order. Parametric studies may be performed by reading data which have been changed from previous runs while data which have not been changed need not be reread. Automatic generation of all of the mesh information is taken care of by the computer program with an input definition of the number of points around the periphery of the solar flat plate module. Computer output options exist so that limited or detailed computed results may be obtained. If desired, the output may contain a map of the computer node points printed to a scale of one part in 100, as well as temperatures and electrical characteristics as functions of time.

A comparison of the computer program results with experimental and manual calculations has been obtained. These show good correlation with the limited experimental information which is available, and predict slightly lower performance characteristics than have been obtained using manual approaches. In general, the correlation is good.

A limited parametric study has been performed and reported herein which shows characteristics to be expected for earth orbit behavior and for an application close to the sun.

## I. INTRODUCTION

### A. Background

The satisfaction of auxiliary power requirements in space constitutes a difficult problem. To date, most of these requirements have been met by the direct conversion of solar radiation to electricity. Until recently, the only practical solid-state device available for this purpose was the photovoltaic cell. Recent developmental work has resulted in the availability of an alternate technique, the thermoelectric flat plate solar generator. This electrical supply has promise of more attractive cost per watt of electrical power produced, lower weight/watt of power produced, and has high resistance to nuclear radiation. The solar flat plate also appears attractive for close-in orbits to the sun where at present photovoltaic cells would be difficult to use.

The background technology and a number of preliminary calculations illustrating characteristics are discussed by Fuschillo,\* and will not be considered in detail in this report. The purpose of this report is to present an investigation of the theoretical performance of the device, to compare the results of the theoretical investigation with experimentally determined performance, and to present a parametric study which may be utilized to evaluate use of the device for various postulated missions.

### B. The Solar Flat Plate Thermoelectric Generator

The solar thermoelectric generator consists of three major items: an absorber, thermoelements and a radiator. The absorber consists of a thin plate coated to give a high absorptivity for solar radiation with a low emissivity. This is roughly sun-oriented and attains a relatively high temperature because of its absorption-emissive properties. This heats the hot end of the thermoelements which are attached to the back of the absorber. The other end of the thermoelement is attached to a radiator, which has an exposed surface with a high emissivity coating so that unconverted heat can be rejected to space. The internal surfaces which do not "see" space are normally highly polished to minimize radiant interchange between the absorber and radiator.

Normal practice is to construct the solar panel so that the radiator and absorber serve as electrical interconnections between the thermoelements. Any desired arrays of series or series-parallel arrangements may be immediately obtained. A photograph of a typical panel is shown in Figure I-1. Construction details may be seen in Figure I-2 which shows the components of a typical cell.

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\* Fuschillo, N., Gibson, R., Eggelston, F. K., and Epstein, J., "Flat Plate Solar Electric Generator for Nuclear Earth Orbits", Advanced Energy Conversion, Vol. 6, pp. 103-125, 1966.

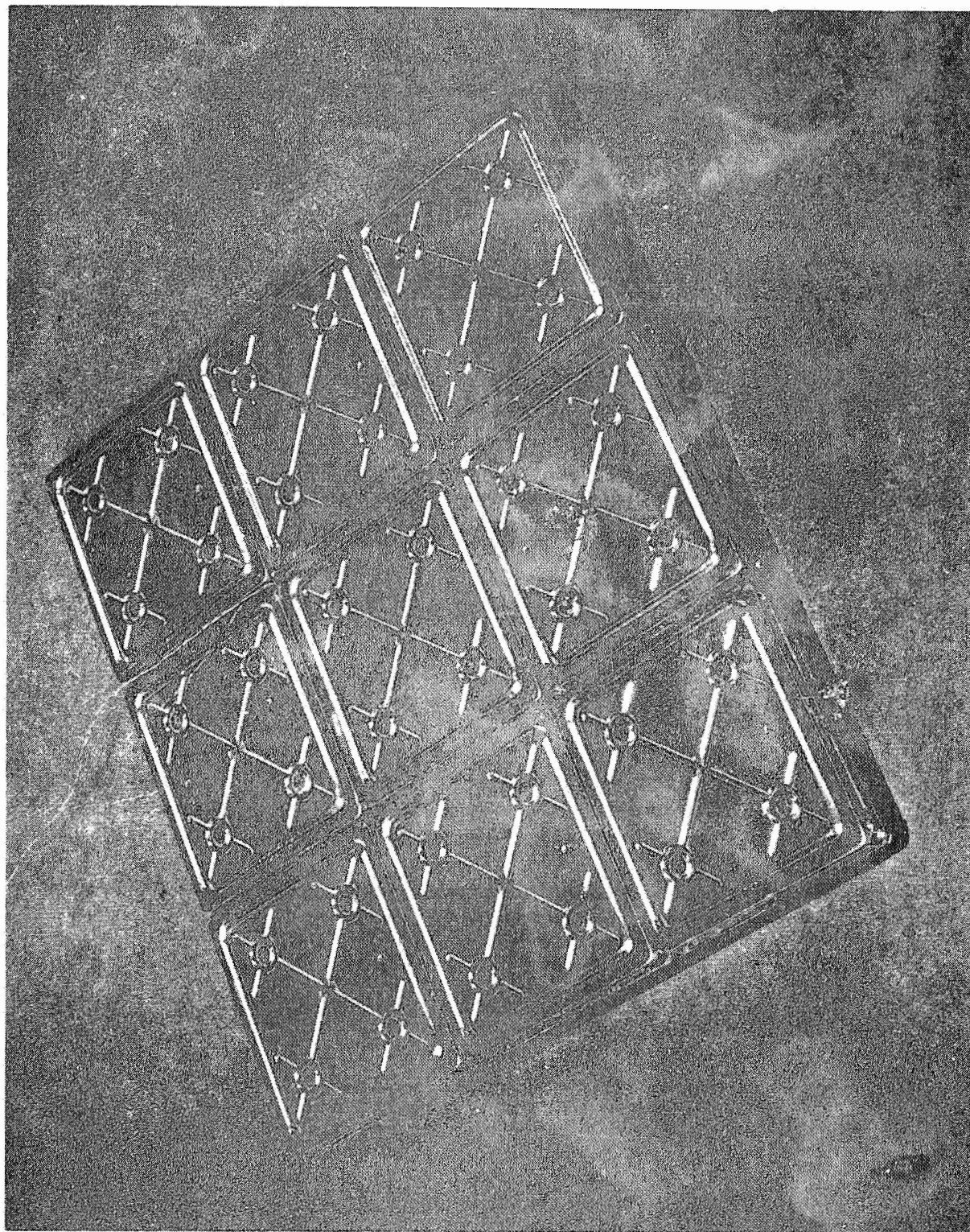


Figure I-1. Flat Plate Solar Thermoelectric Generator Panel

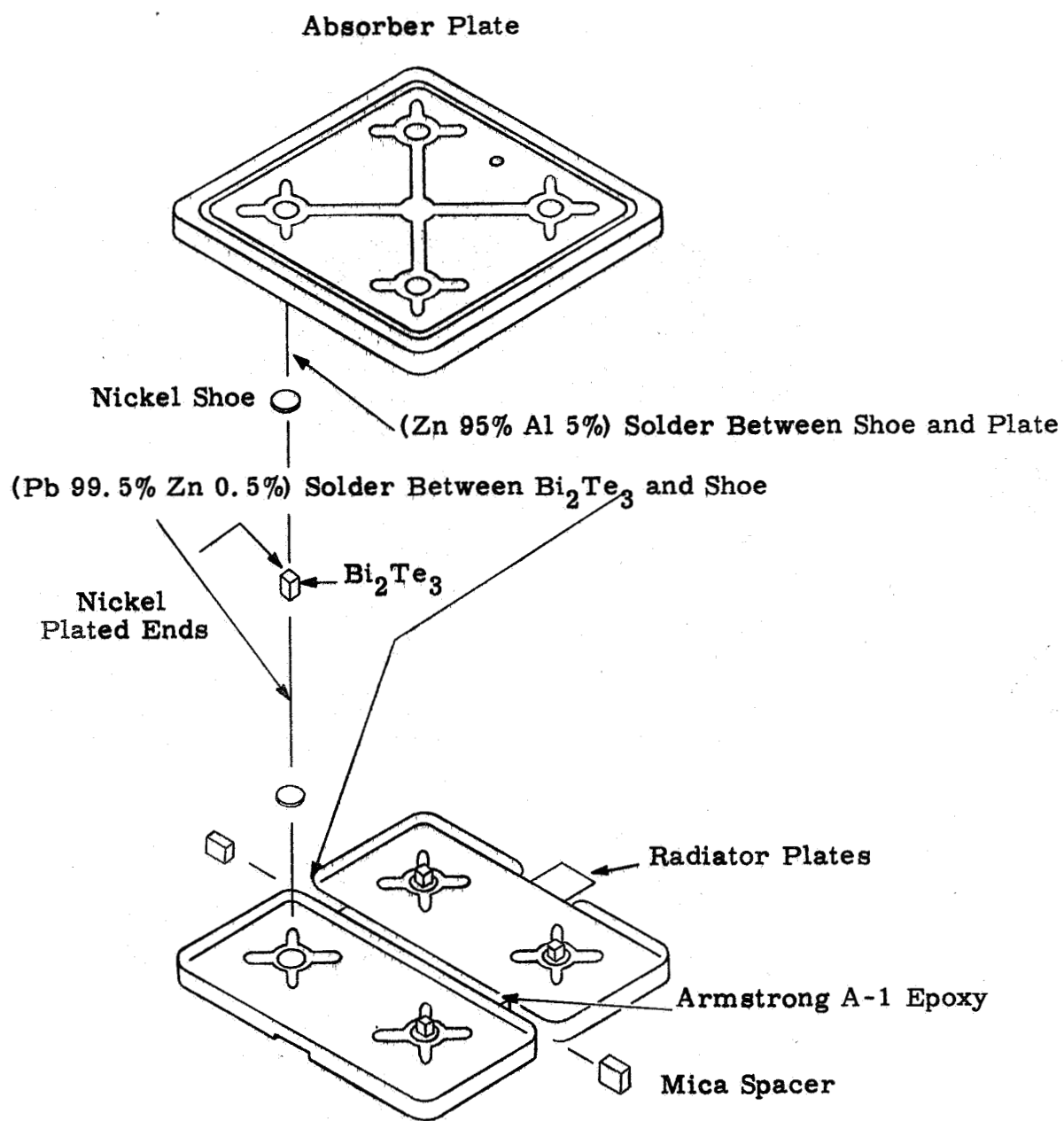


Figure I-2. Unit Couple Assembly Drawing

## II. PANEL ANALYSIS

Two basic types of analyses are necessary to fully investigate the solar flat plate capabilities. Long term behavior of the type associated with a constant solar flux and constant heat rejection characteristics are of interest so that the steady state performance may be investigated. Power generation behavior associated with transient type phenomena are required for such applications as a non-oriented array mounted on a spinning satellite or to predict the characteristics when a satellite moves from shade to sun conditions or vice versa. Either of these basic investigations may be performed by considering the three components comprising the system: the thermoelements, the absorber and the radiator.

Thermoelectric theory is relatively well known from the analysis or engineering standpoint and will accurately predict behavior provided suitable mathematical models are selected. The absorber and radiator may both be described utilizing transient heat conduction theory with appropriate incorporation of the radiant heat transfer phenomena. The basic assumptions required are as follows:

- (1) The radiator and absorber may be treated as transient two-dimensional heat conductors with a variable heat source and no temperature gradient perpendicular to the surface.
- (2) Heat input to the absorber is due to the solar flux, planet albedo, and an effective planet temperature with full recognition given to the various absorptivity and view factor terms.
- (3) Radiator input is treated in a manner similar to the absorber.
- (4) Thermal coupling between the absorber and radiator occurs due to the thermoelements and radiant heat transfer between the two plates.
- (5) The thermoelement can be described by a transient one-dimensional differential equation which contains terms for heat conduction, Joule heating, the Peltier, Seebeck and Thomson effects, and radiant interchange between the thermoelement, absorber and radiator.
- (6) The effect of resistance between a thermoelement and mounting hardware is included as a "contact" resistance (which must be determined experimentally).
- (7) Heat generation within the plate due to the flow of electricity and the effect of heat received by the plate from the thermoelements is small and may be neglected.

The justification for these assumptions and the mathematical model which results are discussed in the following sections.

### A. Thermoelectric Analysis

Analysis of thermoelectric elements is generally achieved by taking a heat balance over the entire thermoelectric couple to determine the heat required at the hot junction. If this is then divided into the output power, the efficiency is obtained. Suitable differentiations are then commonly used to obtain an optimum efficiency. It is relatively common practice to ignore the contact resistance in this approach, or to make assumptions pertaining to the behavior of contact resistance with cross-sectional area. If for the moment the correctness or incorrectness of these assumptions is neglected, one problem involved with this approach is the difficulty of understanding the errors involved in obtaining the heat balance. A further problem is that this approach may be used only for the steady state, and is inappropriate for use with transient analysis. One of the first analyses which considered the steady state thermoelectric behavior from a microscopic standpoint was presented by Lyon and Bustard.\* Although this analysis was derived on the basis of non-transient conditions, it is a relatively simple extension to utilize it for the transient representation. The approach will be used herein because it has been shown to be accurate.

Although the assumption of Lyon and Bustard that the variation of contact resistance with the inverse of cross sectional area is good for some analyses, it is not a legitimate assumption for all cases. The error is expected to become larger as thermoelectric element size is decreased, and since size will be of importance in some of the applications of the solar flat plate, the assumption is not considered legitimate herein.

To obtain a mathematical model which describes thermoelectric behavior, we first consider the thermoelectric phenomena which are involved. These are:

- (1) Thermal conduction: the transport of heat from one position in the thermoelectric material to another due to the temperature gradient.
- (2) Joule heating: the internal heating effect within the thermoelectric material due to resistance to the flow of electricity.
- (3) Seebeck effect: the phenomenon of the flow of electricity that occurs in a closed loop which is composed of two dissimilar materials when the junction between the materials is maintained at different temperatures.
- (4) Thomson effect: the phenomenon of heat generation or absorption which takes place when an electrical current flows in the presence of a temperature gradient.

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\* Lyon, W. C. and Bustard, T. S., "Digital Computation of Thermoelement Operational Parameters", Advanced Energy Conversion, Vol. 2, 1962, pp. 197-208.

- (5) Peltier effect: the heat generation or absorption phenomenon that occurs at a junction between dissimilar materials when an electrical current is caused to flow.

Incorporation of these well understood theoretical effects is sufficient to describe the thermoelectric behavior if no other perturbations occur. However, any mathematical analysis must include the effects of the surroundings to a greater or a lesser extent and must also consider the actual physical situation in which the thermoelements are manufactured or used. Consequently, further assumptions are necessary. These are:

- (1) The thermoelectric material is homogeneous.
- (2) Heat interchange between the lateral or exposed surface of the thermoelements may be represented by considering only radiant heat transfer between the thermoelement surface and the surroundings as seen by the surface.
- (3) There is no temperature gradient within the thermoelement normal to the axis.

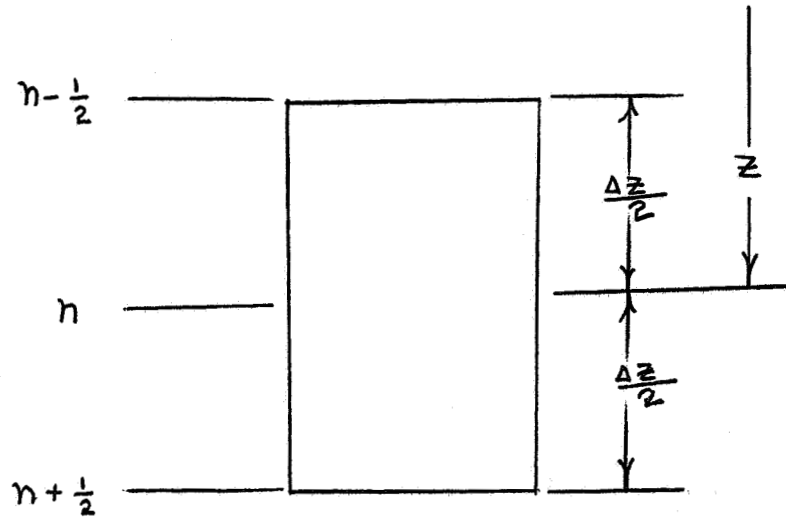
Homogeneity of material is generally obtained to the degree necessary for a valid mathematical analysis. This assumption is basically necessary at this time regardless of the degree of homogeneity or non-homogeneity in order that a mathematical representation can be obtained. However, a good thermoelectric material is homogeneous, and it is expected that little error will be involved with this assumption.

The assumption that the thermoelement exchanges heat from the exposed surface with its surroundings is in conflict with the assumption that no temperature gradient exists within the thermoelement normal to the axis. Obviously, if heat is going to be rejected from the lateral surface, there must be a gradient with respect to a normal drawn to that surface. This assumption is justified, however, because the quantity of heat which is conducted along the thermoelement is very much greater than that which is radiantly interchanged between the thermoelement and the absorber-radiator. The error involved is consequently small.

The calculation of the energy interchange between the thermoelement surface and its surroundings could become extremely complicated, with a consequent result that a tremendous amount of time would be used in obtaining numerical results. However, the thermoelement basically "sees" only the absorber and the radiator inside surfaces. If these are assumed to be represented by average temperatures, then an approximation of the heat transfer may be readily obtained. This approximation will not significantly perturb the results because it may be readily shown that the heat transfer from the surfaces is small in comparison to that which is conducted. This assumption therefore will be incorporated into the analysis since it will not affect the results.



Now consider a small section of thermoelement:



The following energy balance will describe the section behavior:

$$\left[ \begin{array}{l} \text{heat flow rate} \\ \text{into section at} \\ n-1/2 \end{array} \right] - \left[ \begin{array}{l} \text{heat flow rate} \\ \text{out of section} \\ \text{at } n+1/2 \end{array} \right] + \left[ \begin{array}{l} \text{rate of heat} \\ \text{generation} \\ \text{between } n-1/2 \\ \text{and } n+1/2 \end{array} \right] =$$

$$\left[ \begin{array}{l} \text{rate of heat accumulation} \end{array} \right] \quad (\text{II-1})$$

The rate at which heat flows into the section is:

$$-k(t)_{n-1/2} A \left( \frac{\partial T(t)}{\partial Z} \right)_{n-1/2} \quad (\text{II-2})$$

where it is assumed that there is no capacitance or impedance effect, and :

- k = thermal conductivity
- t = time
- A = cross sectional area normal to the temperature gradient
- T = temperature
- Z = position



The rate at which heat flows out of the section is:

$$-k(t)_{n+1/2} A \left( \frac{\partial T(t)}{\partial Z} \right)_{n+1/2} \quad (\text{II-3})$$

The rate of heat generation is:

$$\left[ I(t) \right]^2 R(t)_n + \gamma(t)_n I(t) \left( \frac{\partial T(t)}{\partial Z} \right)_n \Delta Z \quad (\text{II-4})$$

where:

- I = electrical current
- R = electrical resistance of the nth segment
- $\gamma$  = Thomson coefficient

and the first term represents the Joule heating, the second the Thomson effect.

A fictitious heat generation term can be used to represent the heat interchange between the thermoelement and its "surroundings". This term is:

$$- F(t) P \Delta Z \left[ T(t)_n^4 - \overline{T_s(t)^4} \right] \quad (\text{II-5})$$

where:

- F = view factor
- P = perimeter of thermoelement measured on a plane normal to the Z-axis
- $\overline{T_s^4}$  = surrounding temperature to the fourth power averaged

The rate of heat accumulation is:

$$C_p(t)_n \rho(t)_n A \Delta Z \frac{\partial T(t)_n}{\partial t} \quad (\text{II-6})$$

where:

- $C_p$  = heat capacity
- $\rho$  = density

Substituting these into Equation (II-1) yields:

$$\begin{aligned}
& -k(t)_{n-1/2} A \left( \frac{\partial T(t)}{\partial Z} \right)_{n-1/2} + k(t)_{n+1/2} A \left( \frac{\partial T(t)}{\partial Z} \right)_{n+1/2} + \\
& + [I(t)]^2 R(t)_n + \gamma(t)_n I(t) \left( \frac{\partial T(t)}{\partial Z} \right)_n \Delta Z - \\
& - F(t) P \Delta Z [T(t)_n^4 - \overline{T_s(t)^4}] = C_p(t)_n \rho(t)_n A \Delta Z \frac{\partial T(t)_n}{\partial t} \quad (\text{II-7})
\end{aligned}$$

The internal electrical resistance can be written:

$$R(t)_n = \frac{\rho(t)_n \Delta Z}{A} \quad (\text{II-8})$$

where  $\rho$  = electrical resistivity. Substituting this and taking the limit as  $\Delta Z$  approaches zero results in:

$$\begin{aligned}
& \frac{\partial}{\partial Z} \left[ k(Z, t) \frac{\partial T(Z, t)}{\partial Z} \right] + \frac{\rho(Z, t) [I(t)]^2}{A^2} - \frac{F(t) P}{A} \\
& [T(Z, t)^4 - \overline{T_s(t)^4}] + \left( \frac{\gamma(Z, t) I(t)}{A} \right) \frac{\partial T(Z, t)}{\partial Z} = \\
& = C_p(Z, t) \rho(Z, t) \frac{\partial T(Z, t)}{\partial t} \quad (\text{II-9})
\end{aligned}$$

The physical properties are actually known as functions of  $T(Z, t)$ . Further, the Thomson coefficient can be expressed as:

$$\gamma(Z, t) = -T(Z, t) \frac{\partial \alpha(Z, t)}{\partial T(Z, t)} \quad (\text{II-10})$$

where  $\alpha$  = Seebeck coefficient.

Making use of these, the differential equation becomes:

$$\begin{aligned}
& \frac{\partial}{\partial Z} \left[ k(T) \frac{\partial T(Z, t)}{\partial Z} \right] + \frac{\rho(T) [I(t)]^2}{A^2} - \\
& - \frac{F(t) P}{A} [T(Z, t)^4 - \overline{T_s(t)^4}] - \left[ \frac{T(Z, T) I(t)}{A} \right] \frac{\partial \alpha(T)}{\partial Z} =
\end{aligned}$$

$$= C_p (T) \rho (T) \frac{\partial T (Z, t)}{\partial t} \quad (\text{II-11})$$

where the subscripts on the T have been left out if T is used as a subscript, but it should be realized that the variation still exists.

There is little chance of solving Equation (II-11) in a useful analytic manner, so we re-write it in the difference form:

$$\begin{aligned} & \left[ k (T) \frac{\partial T (t)}{\partial Z} \right]_{n+1/2} - \left[ k (t) \frac{\partial T (t)}{\partial Z} \right]_{n-1/2} + \\ & + \frac{\rho (t)_n [I (t)]^2 \Delta Z}{A^2} - \frac{T (t)_n I (t)}{A} \left[ \alpha (t)_{n+1/2} - \right. \\ & \left. - \alpha (t)_{n-1/2} \right] - \frac{F (t) P}{A} \left[ T (t)_n^4 - \overline{T_s (t)^4} \right] = \\ & = C_p (t)_n \rho (t)_n \frac{dT (t)_n}{dt} \Delta Z \end{aligned} \quad (\text{II-12})$$

$$\begin{aligned} & k (t)_{n+1/2} \left[ \frac{T (t)_{n+1} - T (t)_n}{\Delta Z} \right] - k (t)_{n-1/2} \left[ \frac{T (t)_n - T (t)_{n-1}}{\Delta Z} \right] + \\ & + \frac{\rho (t)_n [I (t)]^2 \Delta Z}{A^2} - \frac{T (t)_n I (t)}{A} \left[ \alpha (t)_{n+1/2} - \alpha (t)_{n-1/2} \right] - \\ & - \frac{F (t) P}{A} \left[ T (t)_n^4 - \overline{T_s (t)^4} \right] = C_p (t)_n \rho (t)_n \frac{dT (t)_n}{dt} \Delta Z \end{aligned} \quad (\text{II-13})$$

In general:

$$\rho_{n+1/2} \approx \frac{\rho_{n+1} + \rho_n}{2} \quad (\text{II-14})$$

where  $\rho$  is some property, so that:

$$\begin{aligned}
& \frac{[k(t)_{n+1} + k(t)_n]}{2 \Delta Z} [T(t)_{n+1} - T(t)_n] - \\
& - \frac{[k(t)_n + k(t)_{n-1}]}{2 \Delta Z} [T(t)_n - T(t)_{n-1}] + \frac{\rho(t)_n [I(t)]^2 \Delta Z}{A^2} - \\
& - \frac{T(t)_n I(t)}{2 A} [\alpha(t)_{n+1} - \alpha(t)_{n-1}] - \\
& - \frac{F(t) P}{A} [T(t)_n^4 - \overline{T_s(t)^4}] = C_p(t)_n \rho(t)_n \frac{\partial T(t)_n \Delta Z}{\partial t} \quad (II-15)
\end{aligned}$$

which can be rewritten as:

$$\begin{aligned}
T_{n,j+1} &= T_{n,j} + \frac{\Delta t}{C_{pn,j} \rho_{n,j} \Delta Z} \left\{ \frac{k_{n+1,j} + k_{n,j}}{2 \Delta Z} \right. \\
& [T_{n+1,j} - T_{n,j}] - \frac{k_{n,j} + k_{n-1,j}}{2 \Delta Z} [T_{n,j} - T_{n-1,j}] + \\
& + \frac{\rho_{n,j} I_j^2 \Delta Z}{A^2} - \frac{T_{n,j} I_j}{2 A} [\alpha_{n+1,j} - \alpha_{n-1,j}] - \\
& \left. - \frac{F_j P}{A} [T_{n,j}^4 - \overline{T_{s,j}^4}] \right\} \quad (II-16)
\end{aligned}$$

where the time derivative has been changed to a forward finite difference approximation, indicated by the  $j$  subscript. Note that the only  $j+1$  term is on the left side of the equation, which greatly simplifies calculations.

This equation describes the thermoelectric performance within the thermoelement but is not applicable to the node at each end. Minor modifications allow this equation to also be applied at the end nodes. For example, at the hot end the  $n-1$  node is meaningless. However, the temperature represented by the  $n-1$  node may be represented approximately by the adjoining node in the absorber to which the thermoelement is attached. The Seebeck coefficient for the  $n-1$  node may be treated as the Seebeck coefficient for the  $n$ th node for this one point. This introduces negligible error because the boundary condition represented by the Peltier cooling is introduced separately as will be explained later. One of the thermal conductivity terms must also

be eliminated in a similar manner. This introduces very small error in the result since essentially a numerical average is being obtained. The same type of changes can be made at the other end of the thermoelement where it contacts the radiator. In this case, the  $n + 1$  node is meaningless, and is treated in a similar way.

The portion of the radiator and of the absorber which contacts the ends of the thermoelement is assumed to be a uniform temperature region with respect to position. This assumption is consistent with the previously stated assumption that there is no temperature gradient in a plane perpendicular to the thermoelement axis. It introduces only a very small error in the results because the volume involved is small and the temperature gradients over such a small volume will be minimal. The boundary conditions between the absorber and the thermoelement or between the radiator and the thermoelement are introduced during the treatment of this node. The rate at which heat is removed from the portion of the absorber in contact with the thermoelement can be described by:

$$Q_i(t) = -k(t)_1 A \left[ \frac{\partial T(Z, t)}{\partial Z} \right]_{Z=0} + \\ + \alpha(t)_1 I(t) T(t)_1 - [I(t)]^2 R_{ci}(t) \quad (\text{II-17})$$

where the first term is the heat conducted away by the thermoelement, the second is the Peltier cooling, the third is the Joule heating term at the contact, and:

$$Q_i = \text{heat flow into thermoelement} \\ R_{ci} = \text{contact resistance at the hot end}$$

The subscript one refers to the first node in the thermoelement (the hot end). The heat flow rate out of the cold end is:

$$Q_o(t) = -k(t)_N A \left[ \frac{\partial T(Z, t)}{\partial Z} \right]_{Z=Z} + \\ + \alpha(t)_N I(t) T(t)_N + [I(t)]^2 R_{co}(t) \quad (\text{II-18})$$

where the subscripts N and o refer to the heat out (cold) end and  $0 \leq Z \leq Z$ .

The derivative portions of these equations represent the change of temperature with respect to position within the thermoelement as the thermoelement end is approached from the inside. These terms may be immediately written in difference form and can therefore be readily evaluated using the previously presented difference equations. It is assumed that the contact resistances are known as functions of temperature from experimental data. Since the equations can be used to compute temperature, the contact resistances then can also be computed. The same is true of the other terms which appear in these equations. Again, the attractiveness of a forward difference technique is apparent because it is not necessary to iterate each time during the computation process. Were some other difference technique to be used, either iterative techniques or complicated approaches would be required which could be more time consuming. (The numerical approach will be expanded in later sections.)

The previously described equations are sufficient to enable thermoelectric performance to be calculated provided the electrical characteristics have been determined. However, the electrical characteristics depend upon the thermoelement temperatures and behavior. Consequently, these characteristics must be incorporated into the analysis so that a simultaneous solution may be obtained. To accomplish this, we first consider the following equation which gives the total internal resistance of a thermoelement:

$$R_i(t) = \int_0^Z \frac{\rho(Z, t)}{A} dZ \quad (\text{II-19})$$

The open circuit voltage is:

$$e(t) = \int_{T(t)_N}^{T(t)_1} \alpha(T) dT \quad (\text{II-20})$$

These two equations represent the behavior for a single thermoelement. In practice, the indicated operations will be performed for an N element and a P element, and the resistances and the voltages will then be added to obtain the behavior for a complete couple. Multiplication of these results by the number of couples immediately yields the overall characteristics of the complete system. The following equation immediately results from incorporating these answers into an analysis of the system circuit including a load resistance:

$$e_N(t) + e_P(t) - I(t) \left[ \frac{R_e(t)}{M} + R_t(t) \right] = 0 \quad (\text{II-21})$$

where:

N refers to the negative element

P refers to the positive element

M = number of couples

$R_e$  = external or load resistance

$R_t$  = total internal resistance per couple

$$= R_{iN}(t) + R_{iP}(t) + R_{CCN}(t) + R_{CCP}(t) + R_{CHN}(t) + R_{CHP}(t)$$

$R_{CC}$  = cold end contact resistance (including radiator)

$R_{CH}$  = hot end contact resistance (including absorber)

The output power is now:

$$P(t) = I(t)^2 R_e(t) \quad (\text{II-22})$$

The efficiency of the system can be defined in two ways:

$$\eta = \frac{\text{power delivered by thermoelement}}{\text{heat absorbed by thermoelement}} = \text{thermoelectric efficiency} \quad (\text{II-23})$$

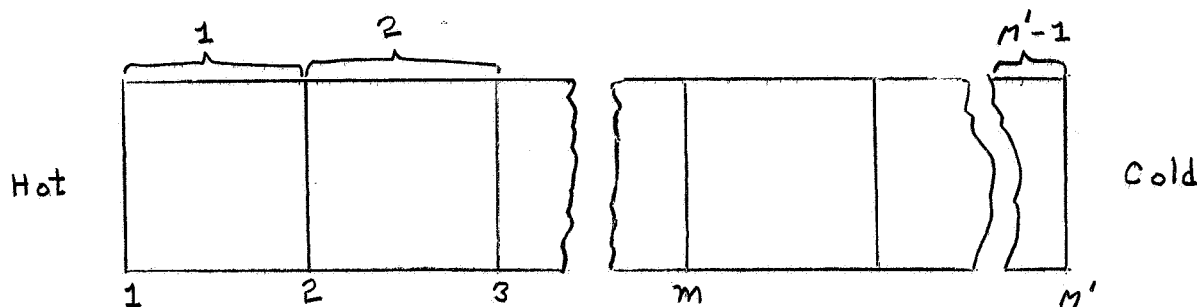
$$\eta = \frac{\text{useful power delivered to load}}{\text{heat incident on absorber}} = \text{overall efficiency} \quad (\text{II-24})$$

It is interesting to note that the latter expression will become infinite when the solar heat is cut off, but the generator continues to function for a short time because of stored thermal energy.

To obtain a mean efficiency for any particular situation, it is only necessary to integrate over the time span of interest.

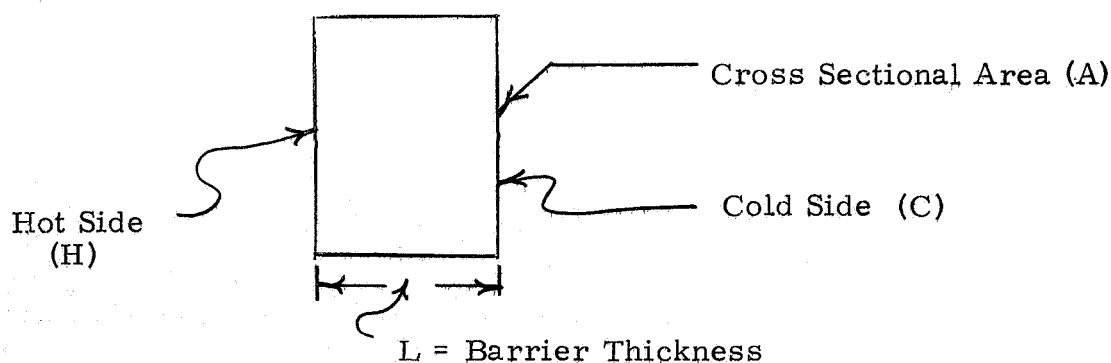
The same technique can be extended to a segmented thermoelement. However, it is necessary to include the effects of the discontinuity between the segments. These effects include contact resistances, inert material caused temperature drops, barrier losses, etc. Practical considerations of this type could cause an actual loss in switching to a segmented system as compared to a predicted gain for an ideal situation.

To gain further insight into the behavior of segmented systems, consider the following thermoelement leg:



The heat absorbed by this element at the hot end can be obtained by applying Equation (II-17) at this point. The heat given up at the cold end of segment 1 can be immediately determined by using Equation (II-18). The behavior within segment 1 is described by Equation (II-16). The same statements can be made for each of the  $M' - 1$  segments of which the thermoelement is composed.

The use of Equations (II-17) and (II-18) automatically includes the effect of contact resistance between the segments. In reality, a temperature drop could occur in a barrier material between the segments. Since this material will be thin, its behavior can be described by considering:



The rate of heat flow entering the hot side is the heat given up by the thermoelement to the left. The heat flow rate out is that absorbed by the thermoelement on the right. These terms are already described by Equations (II-17) and (II-18). Therefore, if  $T(t)$  is the average temperature of the barrier we may write:



$$\begin{aligned}
 \left[ \begin{array}{c} \text{rate of} \\ \text{heat in} \\ (Q_i(t)) \end{array} \right] - \left[ \begin{array}{c} \text{rate of} \\ \text{heat out} \\ (Q_o(t)) \end{array} \right] &= \left[ \begin{array}{c} \text{rate of} \\ \text{heat} \\ \text{accumulation} \end{array} \right] = \\
 &= \left[ C_p \rho_{AL} \frac{dT(t)}{dt} \right]
 \end{aligned} \tag{II-25}$$

The temperature drop across the barrier is approximately:

$$\frac{Q_i(t) + Q_o(t)}{2} \approx \frac{kA}{L} [T_i(t) - T_o(t)] \tag{II-26}$$

and further:

$$T(t) \approx \frac{T_i(t) + T_o(t)}{2} \tag{II-27}$$

so that the barrier behavior is now completely described.

If the barrier equations for each barrier (between segments) are combined with the equations and boundary conditions applicable to the segments, a set of difference equations is obtained similar to the set obtained for a non-segmented thermoelement, the basic difference being in the coefficients. Once these have been determined, the thermoelement points may be handled simultaneously with all of the other mesh points.

The electrical output and efficiency for the segmented system can now be obtained as easily as for the non-segmented system. The total electrical output is the summation of the electrical outputs of each segment minus the losses already discussed (with the additional assumption that the electrical resistance of the barriers is included in the contact resistance associated with those barriers). The efficiency is the net total electrical output divided by the heat into the hot end of each segment.

## B. Absorber and Radiator Analysis

The behavior of the radiator can be handled by identical equations as utilized to analyze the absorber provided proper care is taken in setting up the analysis. Consequently, this discussion will be limited to a consideration of basically the absorber. All assumptions and considerations which are made will be in sufficient depth that the applicable equations may be readily utilized for the radiator as well.

The absorber acts as a heat transfer fin in that it receives radiant energy principally from the sun and transmits a major portion of this energy to the thermoelement hot junction by conduction. The remaining energy is either reradiated from the surface or is radiated directly to the radiator. Since one objective in the design of this system is to minimize weight, it is readily apparent that the absorber thickness must be made as thin as possible consistent with good heat transfer characteristics. Exactly what comprises "good heat transfer characteristics" cannot be determined without a detailed investigation. Investigations performed prior to the analysis reported herein have not considered the temperature gradients which result in the absorber and have therefore been unable to fully evaluate the performance characteristics.

Since the absorber is thin relative to its other dimensions, it may be treated as having a negligible temperature gradient perpendicular to the surface. This reduces the problem to a two-dimensional conduction analysis in Cartesian coordinates since the absorber is a rectangular structure. However, the nature of the reradiation terms from the surface as well as the complexities of the radiant energy interchange effects result in a mathematical representation which cannot be handled using analytic techniques.

The absorber receives heat principally by radiation from the sun, although energy interchange will also result from the albedo (planet reflected solar radiation), from radiation emitted directly by planets which is intercepted by the collector and by interchange between the absorber and the radiator. Although the absorber input may be quite accurately calculated using only the direct solar radiation, the behavior of the radiator can be strongly effected by the surroundings it "sees". Since these surroundings must be incorporated into the radiator analysis, they will also be incorporated into the absorber analysis which will, of course, increase the accuracy of the absorber calculation.

The calculation of the solar energy intercepted by the absorber is relatively straightforward from the standpoint that the flux is relatively well-known and it is only necessary to determine the orientation of the absorber with respect to the incoming solar flux. Knowledge of the relative absorptivity to solar radiation immediately enables one to compute the fraction of the impinging energy which is absorbed. Since the relative absorptivity can be a function of temperature, this dependency will be an allowed variable.

Powers\* states that the calculation of "earthshine" is considerably less precise. He further states that fundamental assumptions are normally required to reduce the complexity of the computation. These normally consider that the earth is a diffusely emitting black body with a uniform surface temperature of  $450^{\circ}\text{R}$ . When this assumption has been made, it is immediately possible to compute the direct energy input impinging upon the

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\* Powers, Edward I., "Thermal Radiation to a Flat Surface Rotating About an Arbitrary Axis in an Elliptical Earth Orbit: Application to Spin-Stabilized Satellites", National Aeronautics and Space Administration, NASA TN D-2147, April 1964.

surface of the absorber from all portions of the earth which are "seen" from the absorber. Similar considerations are applicable to planets other than the earth.

It is immediately apparent that calculation of the effect of albedo requires a similar treatment. Powers in this case assumes that the earth can be represented as a diffusely reflecting sphere with the source intensity represented as a function of the absorber location relative to that portion of the earth which is lit by the sun.

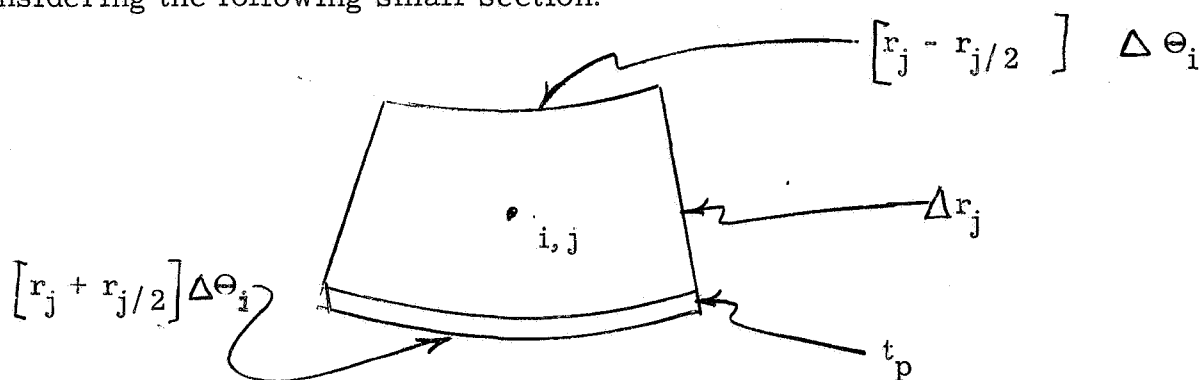
Heat will be reradiated from the surface of the absorber. This may be immediately determined if the temperature is being computed by utilizing the relative emissivity which can be represented as a function of temperature.

Energy interchange will also occur between the absorber and the radiator. Although this calculation may be readily made between two parallel flat plates if the surfaces are at uniform temperatures, the problem becomes more complex if temperature distributions occur, as is the case under consideration here. Since the surfaces of the absorber and radiator which "see" each other will be highly polished to minimize the energy interchange, the actual quantity of heat transmitted between the two surfaces will be small in comparison to the other heat transfer effects. Consequently, if some error occurs in this portion of the analysis, its effect on the overall performance will be minimized. As a result, the assumption will be made that the collector and the radiator can each be represented in terms of the fourth power of temperature integrated over the surface of the plate in question and hence averaged. This integration process is straightforward once the temperature distribution has been established. Immediately, the energy interchange between the two surfaces may now be treated as though the surfaces had a uniform temperature. The overall effect of this assumption is a slight perturbation in the temperature distribution on the surfaces. At the edges which are furthest removed from the thermoelement, the absorber and radiator will normally have the largest temperature difference, and the energy interchange at this point will be slightly greater than computed with the assumption. Conversely, the opposite effect will occur as the portion of the surface in question is considered as approaching the thermoelement.

The basic solar flat plate configuration as presently constructed and analyzed herein can be represented by a rectangular geometry. This would immediately imply that the basic analysis of the absorber should also be based on rectangular geometry. Indeed, initial approaches to the analysis of this problem were made utilizing this assumption. However, a consideration of the heat flux and hence the nature of the temperature gradients within the absorber resulted in a change from this geometry to the cylindrical coordinate system. This apparently contradictory decision was reached from consideration of two different standpoints. First, most of the energy which is absorbed by the absorber is transferred by conduction to the thermoelement. The heat fluxes in the vicinity of the hot end of the thermoelement are therefore high, with large temperature gradients, and these heat fluxes decrease as the distance from the hot end of the thermoelement is increased. This means that a numerical representation of the behavior of the absorber should be based upon selection of small volumes or nodes which are located closely

together. Since the temperature gradients become less as the distance from the thermoelement is increased, nodes which are located further away may be utilized to represent larger volumes with a similar accuracy. Secondly, it was desired that the computer program should be as easily used as possible. This essentially meant that the computer program should be caused to generate its own nodal arrangement based upon a minimum of input data. Both of these considerations immediately imply a cylindrical coordinate system as one which most readily allows the desired node arrangements to be attained. This assumption is discussed further and in more detail later in this report (see also Appendix A).

A heat balance may be taken over a small section of the absorber by considering the following small section:



The following heat balance can be written:

$$\begin{array}{ccccccc} \text{heating} & - & \text{rate of heat} & - & \text{radiant heat} & = & \text{rate of heat} \\ \text{rate} & & \text{accumulation} & & \text{loss rate} & & \text{conduction} \\ & & \text{in segment} & & & & \text{away from} \\ & & & & & & \text{segment} \end{array} \quad (\text{II-28})$$

The total heat flux per unit area absorbed by the exposed surface is given by:

$$\begin{aligned} Q_{i, j}(t) = & S \alpha(\delta, T_{i, j}) \beta(\delta) \cos(\delta(t)) + \sigma F_1(T_{i, j}, t) T_p(t)^4 + \\ & + S R F_2(T_{i, j}, t) \end{aligned} \quad (\text{II-29})$$

where:

$$\begin{aligned} Q &= \text{absorbed heat flux} \\ S &= \text{solar constant} \\ \alpha &= \text{absorptivity (for solar radiation)} \end{aligned}$$

$\delta$	=	incident angle for solar radiation measured from a line perpendicular to the plate surface
$T_{i,j}$	=	temperature at position, i, j
$\beta$	=	1 if $0 \leq \delta < \pi/2$
	=	0 otherwise
		} for collector
$\beta$	=	0 if $0 \leq \delta < \pi/2$
	=	1 otherwise
		} for radiator
$t$	=	time
$\sigma$	=	Stefan-Boltzmann constant
$F_1$	=	radiation view factor for heat transfer to plate from a planet
$T_p$	=	planet absolute temperature
$R$	=	reflectivity constant of planet (albedo)
$F_2$	=	radiation view factor for reflected (solar) heat transfer to plate from planet

Note that  $Q$  is not independent of location on the plate since the relative absorptivities are functions of temperature. The heat flow associated with the surface is referred to in the following way to simplify the algebra:

$$q = Q(T_{i,j}, t) r_j \Delta r \Delta \Theta \quad (\text{II-30})$$

where:

$q$  = the heat absorbed per unit time by the segment for this phenomenon

Heat will also be absorbed from the other plate (if we are considering the absorber, the other plate is the radiator). The net heat transferred can be described by:

$$q = \sigma F_{i,j} (T_{i,j}, T'_{i,j}) \left[ T_{i,j}^4(t) - \overline{T'^4(t)} \right] r_j \Delta r \Delta \Theta \quad (\text{II-31})$$

where:

$F_{i,j}$  = view factor  
 $T_{i,j}$  = temperature at (r,  $\Theta$ )  
 $\overline{T'^4}$  = effective temperature of other plate taken to the fourth power and averaged

Heat will be radiated from the exposed surface according to:

$$q = \sigma \epsilon_o (T_{i,j}) \Delta r r_j \Delta \Theta T_{i,j}^4 \quad (\text{II-32})$$

where:

$$\epsilon_o = \text{relative emissivity of exposed surface}$$

Heat conducted from the plate segment into other portions of the plate can be described by:

$$q = \sum (\text{rate of heat conduction out of segment through sides}) \quad (\text{II-33})$$

which can be written:

$$\begin{aligned} q = & -t_p \left\{ k(T_{i,j} - \frac{1}{2}) \frac{\Delta \Theta}{\Delta r} \left[ r_j - \frac{\Delta r}{2} \right] [T_{i,j-1}(t) - T_{i,j}(t)] + \right. \\ & + k(T_{i,j} + \frac{1}{2}) \frac{\Delta \Theta}{\Delta r} \left[ r_j + \frac{\Delta r}{2} \right] [T_{i,j+1}(t) - T_{i,j}(t)] + \\ & + \frac{k(T_{i+\frac{1}{2},j})}{r_j} \frac{\Delta r}{\Delta \Theta} [T_{i+1,j}(t) - T_{i,j}(t)] + \frac{k(T_{i-\frac{1}{2},j})}{r_j} \frac{\Delta r}{\Delta \Theta} \\ & \left. [T_{i-1,j}(t) - T_{i,j}(t)] \right\} \quad (\text{II-34}) \end{aligned}$$

Finally, heat will be accumulated within the segment due to a change in temperature with respect to time. This effect is:

$$q = C_p (T_{i,j}) \rho (T_{i,j}) t_p r_j \Delta \Theta \Delta r \frac{dT_{i,j}(T)}{dt} \quad (\text{II-35})$$

All other effects are assumed negligible, as previously discussed.

These effects can be combined accordingly to yield:

$$\begin{aligned} C_p (T_{i,j}) \rho (T_{i,j}) t_p r_j \Delta \Theta \Delta r \frac{dT_{i,j}(t)}{dt} = & Q (T_{i,j}, t) r_j \Delta r \Delta \Theta - \\ & - \sigma F_{i,j} (T_{i,j}, T'_{i,j}) [T_{i,j}(t)^4 - \overline{T'}^4] r_j \Delta \Theta \Delta r + S_{i,j}(t) t_p \Delta r r_j \Delta \Theta - \end{aligned}$$

$$\begin{aligned}
& - \epsilon_o (T_{i,j}) \Delta r r_j \Delta \Theta T_{i,j}(t)^4 + t_p k (T_{i,j} - \frac{1}{2}) \frac{\Delta \Theta}{\Delta r} \\
& \left[ r_j - \frac{\Delta r}{2} \right] \left[ T_{i,j-1}(t) - T_{i,j}(t) \right] + k (T_{i,j} + \frac{1}{2}) \frac{\Delta \Theta}{\Delta r} \\
& \left[ r_j + \frac{\Delta r}{2} \right] \left[ T_{i,j+1}(t) - T_{i,j}(t) \right] + \frac{k (T_{i,j} + \frac{1}{2})}{r_j} \frac{\Delta r}{\Delta \Theta} \\
& \left[ T_{i+1,j}(t) - T_{i,j}(t) \right] + \frac{k (T_{i,j} - \frac{1}{2})}{r_j} \frac{\Delta r}{\Delta \Theta} \left[ T_{i-1,j}(t) - \right. \\
& \left. - T_{i,j}(t) \right] \}
\end{aligned} \tag{II-36}$$

which may be transformed to the following differential equation by dividing by  $r_j t_p \Delta \Theta \Delta r$  and allowing  $\Delta r$  and  $\Delta \Theta$  to approach zero:

$$\begin{aligned}
& C_p (T) \rho (T) \frac{\partial T (\Theta, r, t)}{\partial t} = \frac{Q(T, t)}{t_p} - \frac{\sigma}{t_p} F (\Theta, r, T, T') \\
& \left[ T (\Theta, r, t)^4 - T'^4 \right] + S (\Theta, r, t) - \frac{\sigma}{t_p} \epsilon_o (T) T (\Theta, r, t)^4 + \\
& + \frac{1}{r_j} \frac{\partial}{\partial r} \left\{ r k (T) \frac{\partial T (\Theta, r, t)}{\partial r} \right\} + \frac{1}{r_j^2} \frac{\partial}{\partial \Theta} \left\{ k (T) \frac{\partial T (\Theta, r, t)}{\partial \Theta} \right\}
\end{aligned} \tag{II-37}$$

which is the governing differential equation describing heat transfer in the absorber. Note that the direct functional dependencies are indicated. Hence,  $F (\Theta, r, T, T')$  indicates that  $F$  is a function of the independent variables  $\Theta$  and  $r$ , and of the dependent variables  $T$  and  $T'$ .  $F$  is not directly a function of time, although it will vary with time because  $T$  and  $T'$  would be completely written as  $T (\Theta, r, t)$  and  $T' (t)$ .

The view factor  $F$  may be written as:

$$F_{i,j} = \frac{1}{\frac{1}{\epsilon_o(T)} + \frac{1}{\epsilon} - 1} \tag{II-38}$$

where  $\epsilon'$  is the average relative emissivity of the absorber.

Similar equations for the radiator may be obtained by exchanging the  $T$  and  $T'$  terms and applying the other parameters properly.

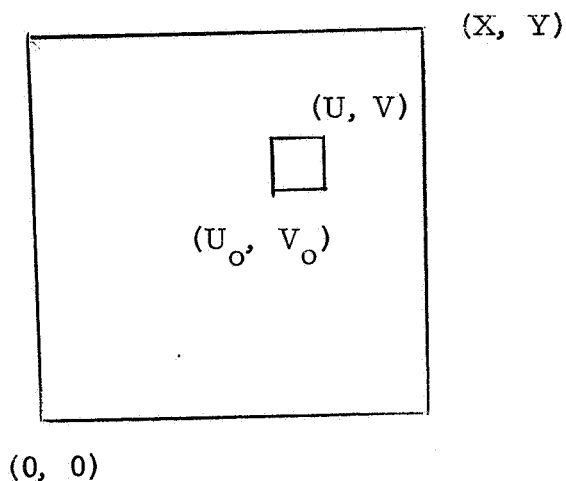
The boundary conditions necessary to obtain solutions to the equations which represent the absorber or radiator essentially consist of the previously discussed interconnections between these surfaces and the thermoelements as well as the additional assumptions that no heat is lost from the edges of the array which are exposed to the surroundings. This latter assumption is relatively good since the internal portions of the array do not "see" space to a significant degree. The view factor for the internal portions to space is further decreased by the lip that is formed around the edge of the collector which serves as a stiffening member. By the same token, a portion of the heat that is collected by the absorber will be perturbed by the presence of this lip, although this has not been considered in the analysis since the effect will be small.

### C. Mesh Set Up Technique

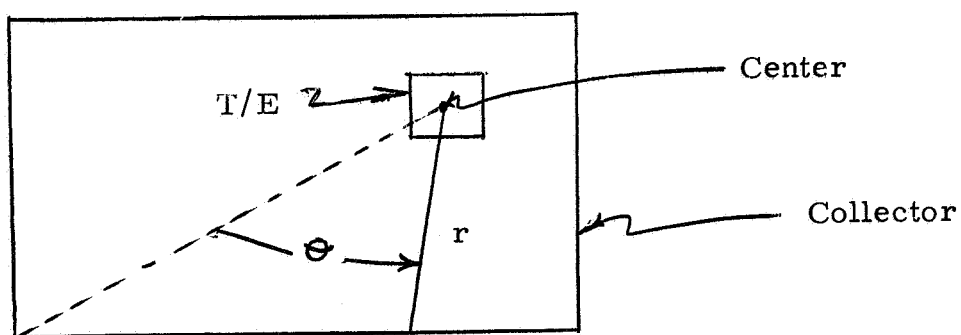
As previously discussed, heat conduction within either the absorber or the radiator may be described by taking a heat balance over a small representative section and then considering the equations which result as the section size is caused to approach zero. The resulting differential equation cannot be solved by present analytical techniques, and a numerical solution is therefore required. The solution selected basically consists of the expansion of the derivative terms through a Taylor series expansion with the rejection of higher order terms. This technique is commonly used for heat transfer calculations and yields excellent accuracy provided the increment sizes are properly selected. It further introduces relatively simple computation techniques which in turn speed up the solution through faster computation. A basic consideration is the selection of a suitable mesh arrangement which then forms the basis for the application of the differential equation.

For reference purposes the absorber will be considered as oriented in a Cartesian  $(x, y)$  coordinate system with the origin located at the lower left corner. Specification of the plate size may be immediately made by defining the point  $(X, Y)$  as the coordinate of the upper right corner. The lower left corner of the thermoelement will be specified by  $(U_o, V_o)$ , and the upper right corner will be specified by  $(U, V)$ . This defines the overall dimensions of the absorber, as seen from the following sketch:

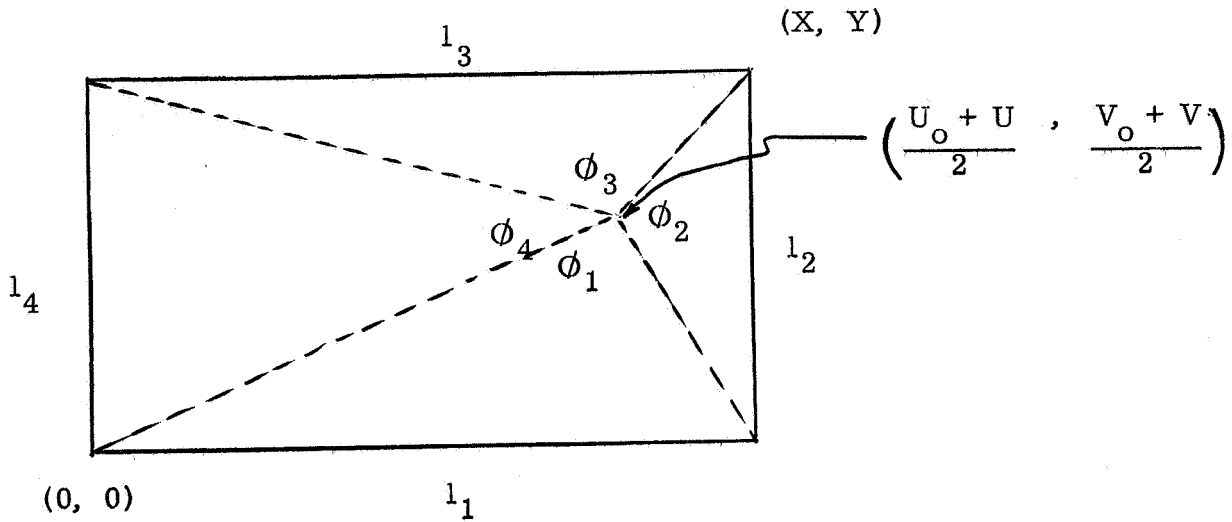




The calculations will be performed using radial geometry and selecting the radial increment,  $\Delta r$ , such that the controlling angular increment will approximately form a square. The controlling angular increment will be defined as the average value of  $\Delta \theta$  encountered in  $0 \leq \theta \leq 2\pi$  :



To determine the  $\Delta \theta$ 's we next consider:



where the  $l_i$  are the number of increments each of the  $i$  sides is to be divided into for calculation purposes. We can use the law of cosines to write:

$$\cos \phi_1 = \frac{4X^2 - (U_o + U)^2 - (2X - U_o - U)^2 - 2(V_o + V)^2}{-2\sqrt{(U_o + U)^2 + (V_o + V)^2} \sqrt{(2X - U_o - U)^2 + (V_o + V)^2}} \quad (\text{II-39})$$

$$\cos \phi_2 = \frac{4Y^2 - 2(2X - U_o - U)^2 - (V_o + V)^2 - (2Y - V_o - V)^2}{-2\sqrt{(2X - U_o - U)^2 + (V_o + V)^2} \sqrt{(2Y - V_o - V)^2 + (2X - U_o - U)^2}} \quad (\text{II-40})$$

$$\cos \phi_3 = \frac{4X^2 - 2(2Y - V_o - V)^2 - (2X - U_o - U)^2 - (U_o + U)^2}{-2\sqrt{(2Y - V_o - V)^2 + (2X - U_o - U)^2} \sqrt{(U_o + U)^2 + (2Y - V_o - V)^2}} \quad (\text{II-41})$$

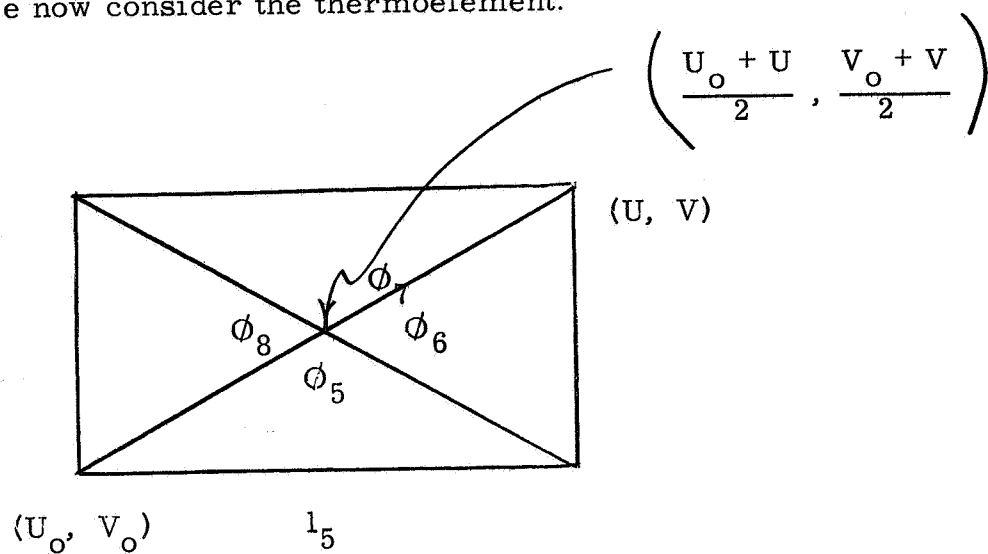
$$\phi_4 = 2\pi - \phi_1 - \phi_2 - \phi_3 \quad (\text{II-42})$$

Now define:

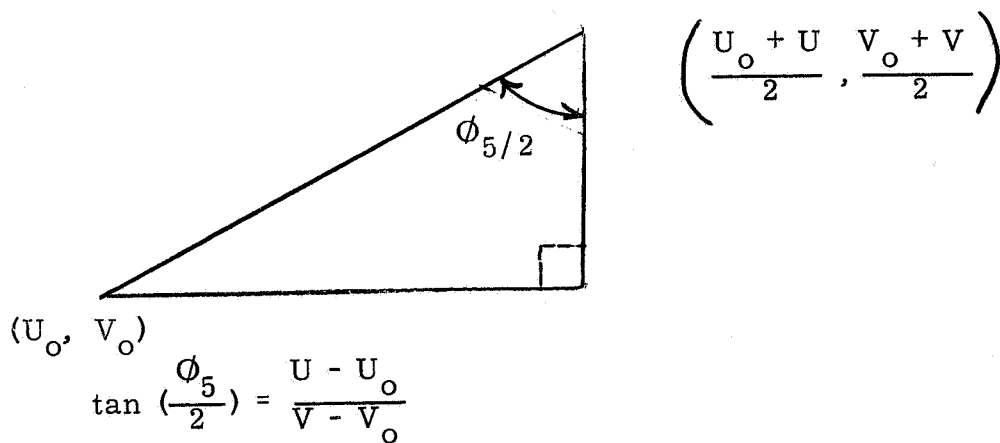
$$\Delta \Theta'_i = \frac{\phi_i}{l_i} \quad 1 \leq i \leq 4 \quad (\text{II-43})$$

These are the  $\Delta \Theta_i$  required by the  $l_i$ .

We now consider the thermoelement:



or, perhaps a little easier:



(II-44)

Similarly:

$$\tan\left(\frac{\phi_6}{2}\right) = \frac{V - V_o}{U - U_o} \quad (\text{II-45})$$

$$\phi_7 = \phi_5 \quad (\text{II-46})$$

$$\phi_8 = \phi_6 \quad (\text{II-47})$$

and:

$$\Delta\Theta'_i = \frac{\phi_i}{l_i} \quad 5 \leq i \leq 8 \quad (\text{II-48})$$

We now compare  $\Delta \Theta'_1$  and  $\Delta \Theta'_5$  and select the smaller of the two values to apply as  $\overline{\Delta \Theta}_1$  for  $0 \leq \Theta \leq \Phi_1$ . If  $\Delta \Theta'_5$  is selected, a new  $l_1$  corresponding to  $\Delta \Theta'_5$  will be computed. The new  $l_1$  will be truncated to the next larger integer if it is not already an integer, and the  $\Delta \Theta'_5$  will be recomputed using the truncated  $l_1$ . This  $\Delta \Theta'_5$  will become  $\overline{\Delta \Theta}_1$ .

Similar manipulations will be performed for  $\Phi_2$ ,  $\Phi_3$  and  $\Phi_4$ .

Typical of the type of angular arrangement which could result from this specification is the geometry shown in Figure II-1. Note that the size of the rays on each side of line "A" are different. If carried to an extreme, this would mean that adjoining segments constructed on this angular arrangement could be of drastically different size. There would immediately result an error in the representation of the differential equation by node arrangements of this type. As a consequence, it is desirable that the variation from ray to ray be minimized. This can be accomplished through an iterative averaging technique which will result in the same number of rays/side, but will arrange the size somewhat differently. To accomplish this, we first define:

$$n_0 = 0$$

$$n_1 = l_1$$

$$n_2 = n_1 + l_2 \tag{II-49}$$

$$n_3 = n_2 + l_3$$

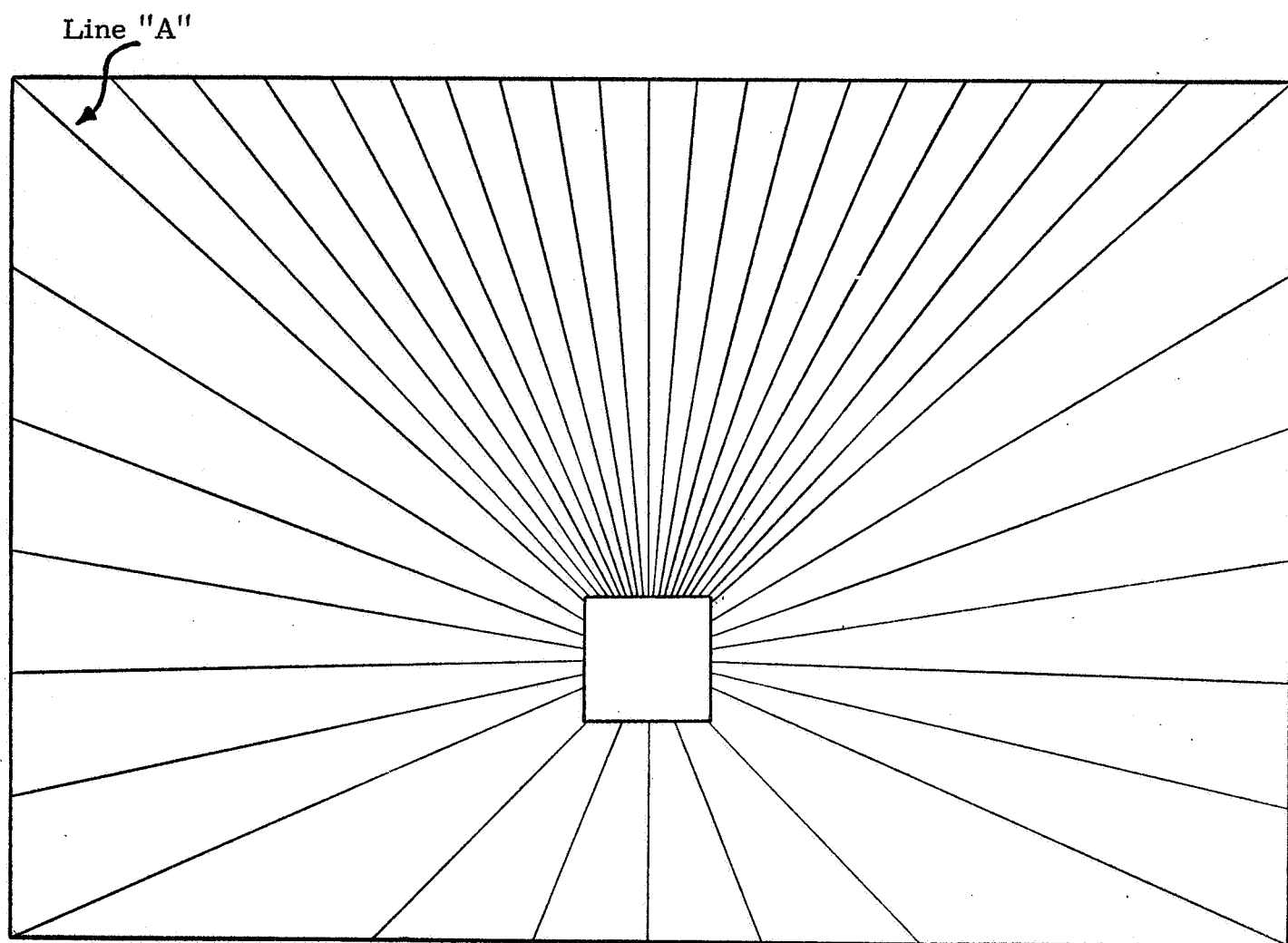
$$n_4 = n_3 + l_4$$

$$\Delta \Theta_i = \overline{\Delta \Theta}_j \quad 1 \leq j \leq 4, \quad n_{j-1} + 1 \leq i \leq n_j \tag{II-50}$$

Now we select  $j = 1$  and perform the following calculations:

$$\Delta \Theta_i = \frac{1}{2} [\Delta \Theta_{i+1} + \Delta \Theta_{i-1}] \tag{II-51}$$

for  $i = n_{j-1} + 1$ , then  $n_{j-1} + 2$ ,  $n_{j-1} + 3$ , . . . , to  $n_j$  (inclusive).



**Figure II-1. Non-Smoothed Angles**

A subscript zero will be taken to mean  $n_4$ , and a subscript  $n_4 + 1$  to mean one. We now compute:

$$\phi = \sum_{i = n_j - 1 + 1}^{n_j} (\Delta \Theta_i) \quad (\text{II-52})$$

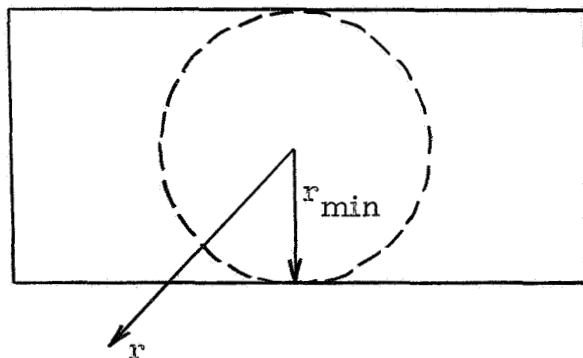
and we renormalize according to:

$$\Delta \Theta_i = \frac{\phi_j}{\phi} \Delta \Theta_i \quad n_j - 1 + 1 \leq i \leq n_j \quad (\text{II-53})$$

where the meaning of Equation (II-53) is to compute the right hand side and then replace  $\Delta \Theta_i$  by that value.

We now repeat Equations (II-51) through (II-53) for  $j = 2$ ,  $j = 3$ , and  $j = 4$ . Then we repeat for  $j = 1, 2, 3, 4$ , and we continue the process until convergence has been obtained. This could typically result in the behavior illustrated in Figure II-2. Note that the variation of size from ray to ray has been smoothed, and no large discontinuities exist. The desired number of rays per side is still the same.

This process will set up the  $\Delta \Theta_i$ . We next must compute the  $r_j$ . To do this we first compute an initial radius as the minimum of  $\frac{U - U_0}{2}$  and  $\frac{V - V_0}{2}$ . This is then the minimum value the radius can have and still be in the plate, but outside of the thermoelement.



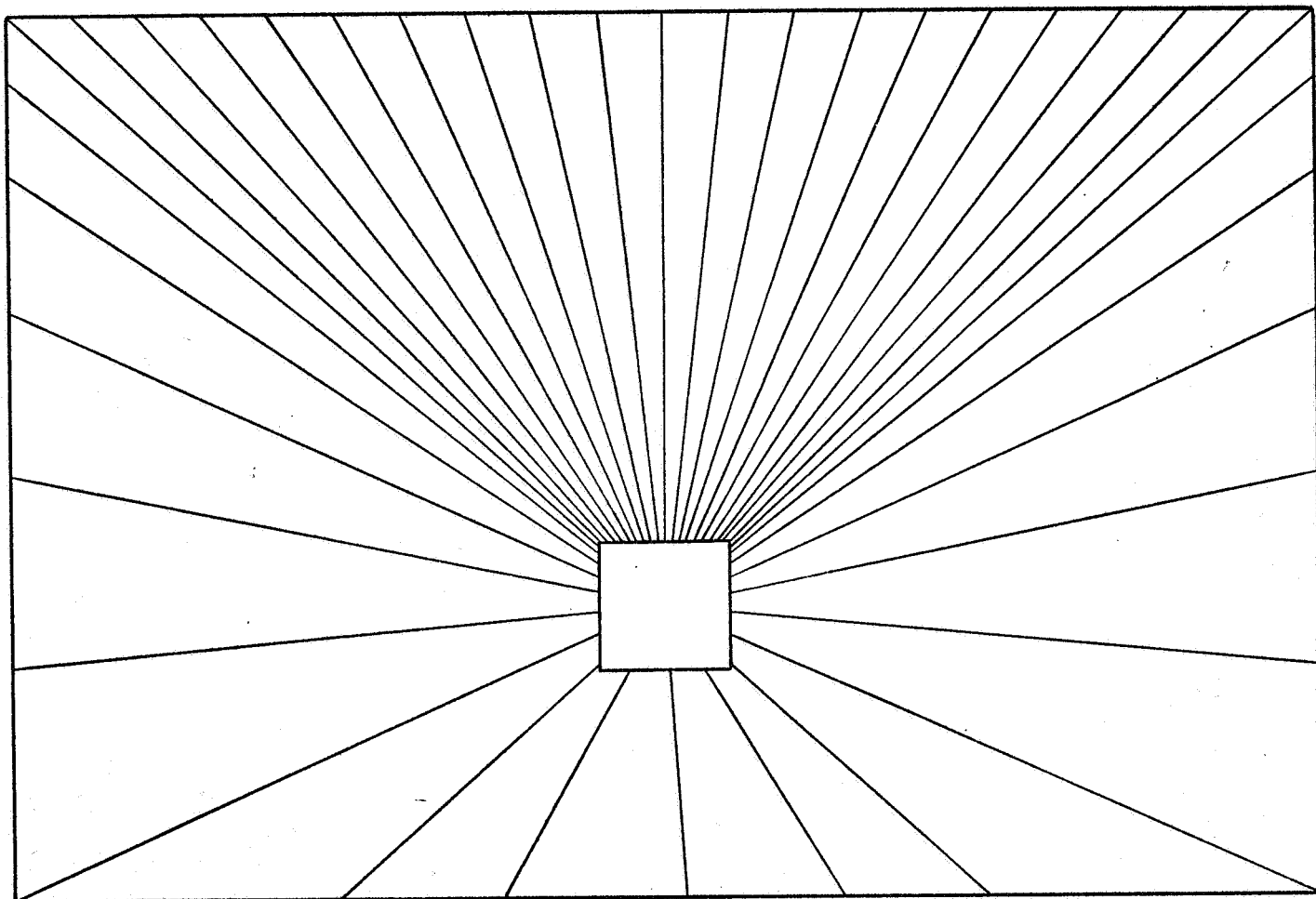


Figure II-2. Illustration of "Smoothed" Angles

We will base  $\Delta r_j$  on the average  $\Delta \Theta_i$ .  $\Delta r_j$  will be calculated by:

$$\Delta r_j = r_j \overline{\Delta \Theta} \quad (\text{II-54})$$

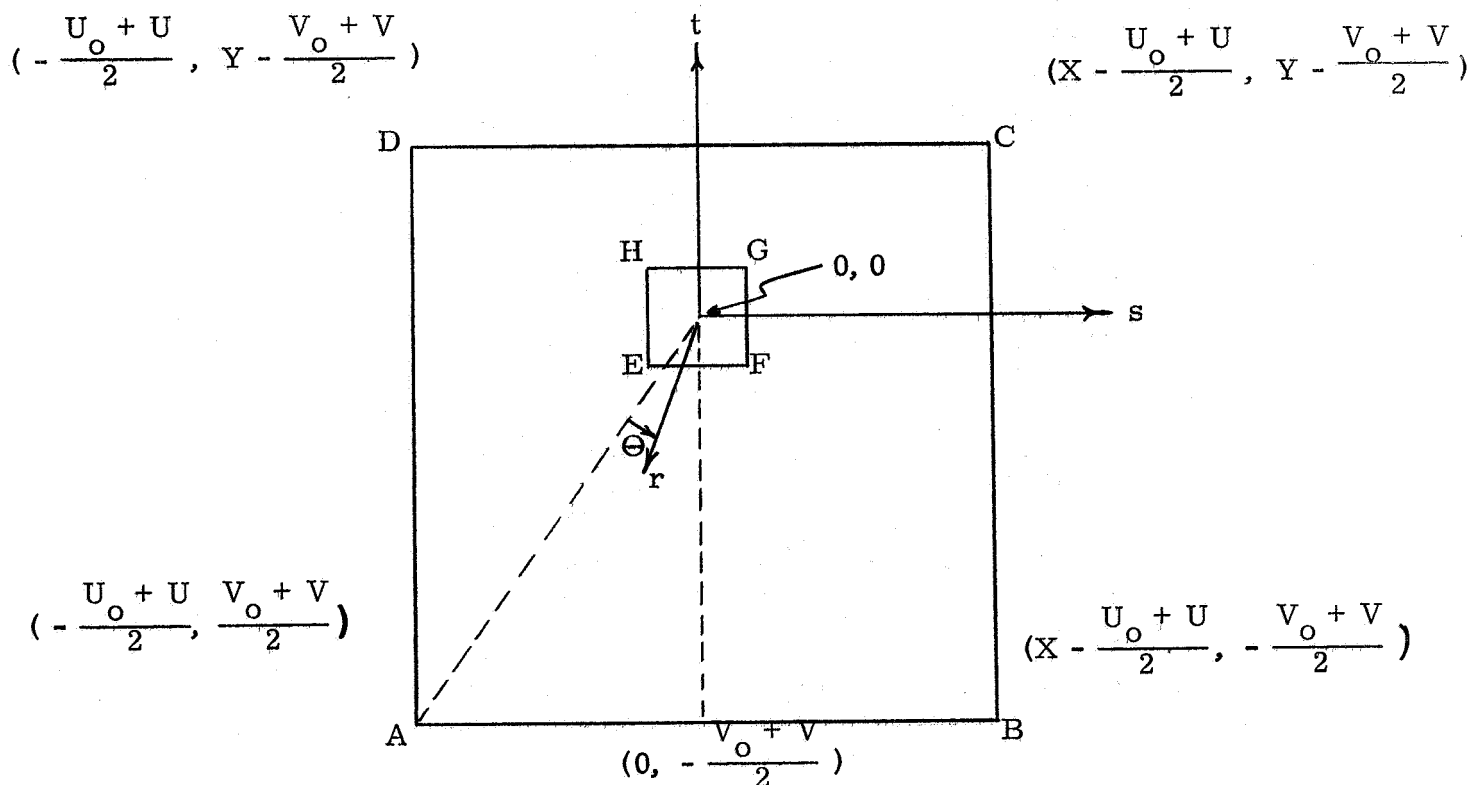
Thus, we see that  $\Delta r_j$  will increase as we move away from the thermoelement center. The initial  $\Delta r_j$  will be calculated from  $r_{\min}$ .

Then we can use:

$$r_{j+1} = r_j + \Delta r_j \quad (\text{II-55})$$

to compute all of the other  $r$ , of course recomputing  $\Delta r_j$  each time.

To set up the mesh and take into account the thermoelement and the plate edges requires a knowledge of where they are located relative to the center of rotation. This may be accomplished by a transformation of coordinates to  $(r, \Theta)$  geometry with the center at  $x = \frac{U_o + U}{2}$ ,  $y = \frac{V_o + V}{2}$ :





where the coordinates are given in terms of the (s, t) system, a rectangular system with the same zero point as the (r,  $\Theta$ ) system and otherwise oriented parallel to the (x, y) system. In (r,  $\Theta$ ) geometry, the values for points A - D are obvious. To obtain E - H, we first note that when the radius vector is straight down:

$$\tan \Theta = \frac{U_o + U}{V_o + V} \quad (\text{II-56})$$

Or:

$$\Theta = \tan^{-1} \left( \frac{U_o + U}{V_o + V} \right) \quad (\text{II-57})$$

But at this point, we also have a known  $\Phi_5/2$  (See Equation (II-44) ):

$$\frac{\Phi_5}{2} = \tan^{-1} \left( \frac{U - U_o}{V - V_o} \right) \quad (\text{II-58})$$

The value of  $\Theta$  corresponding to E may be obtained by subtracting:

$$\Theta_E = \tan^{-1} \left( \frac{U_o + U}{V_o + V} \right) - \frac{\Phi_5}{2} \quad (\text{II-59})$$

But we will not be concerned here with negative  $\Theta$ 's. Hence we will stipulate that if  $\Theta_E$  is negative as calculated, then corner E will be shifted to position F, F to G, etc. Immediately, corner F is located at  $\Theta = \Theta_E + \Phi_5$ , etc. Since the center of the coordinate system is located in the center of thermoelement, the radii for the four corners are identical and may be obtained immediately. These data are summarized in Table II-1.

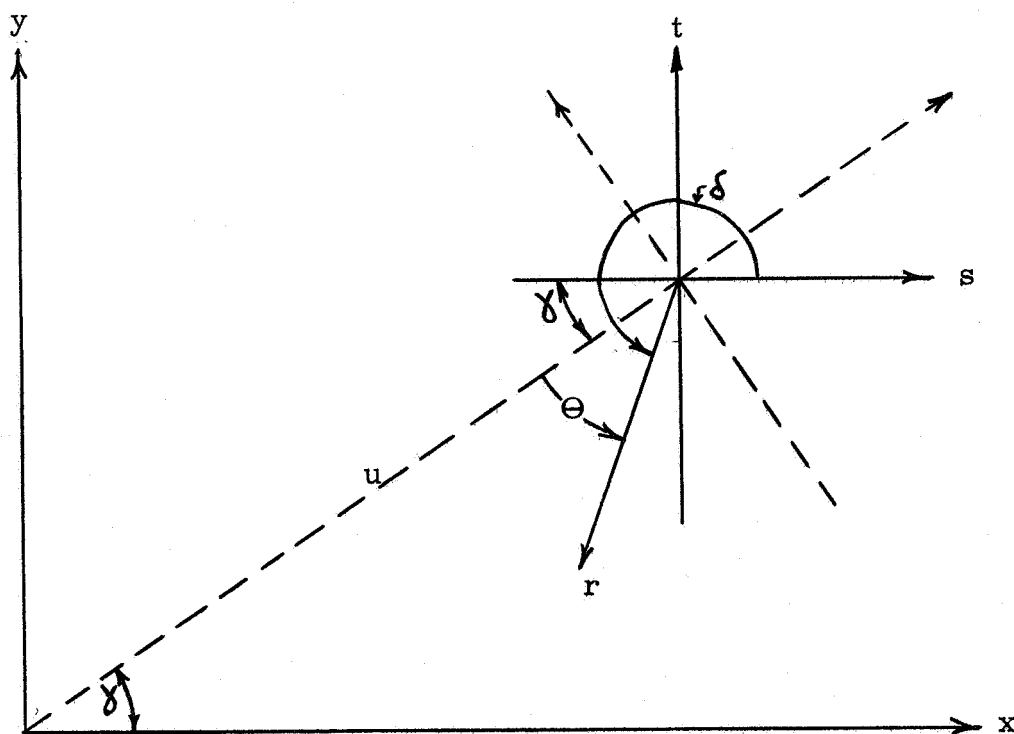
If (u, v) represents a rectangular coordinate system based upon the same zero as the polar coordinate system being considered here, then we may write:

$$u = r \cos \Theta \quad (\text{II-60})$$

$$v = r \sin \Theta \quad (\text{II-61})$$

TABLE II-1. POSITION OF CORNERS IN (r,  $\Theta$ ) GEOMETRY

<u>Corners</u>	<u><math>\Theta</math></u>	<u>r</u>
A	0	$\frac{1}{2} \sqrt{(U_o + U)^2 + (V_o + V)^2}$
B	$\phi_1$	$\sqrt{\left(X - \frac{U_o + U}{2}\right)^2 + \left(\frac{V_o + V}{2}\right)^2}$
C	$\phi_1 + \phi_2$	$\sqrt{\left(X - \frac{U_o + U}{2}\right)^2 + \left(Y - \frac{V_o + V}{2}\right)^2}$
D	$\phi_1 + \phi_2 + \phi_3$	$\sqrt{\left(\frac{U_o + U}{2}\right)^2 + \left(Y - \frac{V_o + V}{2}\right)^2}$
E	$\Theta_E$	$\frac{1}{2} \sqrt{(U - U_o)^2 + (V - V_o)^2}$
F	$\Theta_E + \phi_5$	$\frac{1}{2} \sqrt{(U - U_o)^2 + (V - V_o)^2}$
G	$\Theta_E + \phi_5 + \phi_6$	$\frac{1}{2} \sqrt{(U - U_o)^2 + (V - V_o)^2}$
H	$\Theta_E + 2\phi_5 + \phi_6$	$\frac{1}{2} \sqrt{(U - U_o)^2 + (V - V_o)^2}$



and if  $(s, t)$  represents a rectangular coordinate system based at the same zero as the polar coordinate system, but rotated clockwise from the  $(u, v)$  system by  $\pi + \gamma$ , then:

$$\delta = \Theta + \gamma + \pi \quad (\text{II-62})$$

but:

$$\tan \gamma = \frac{V_o + V}{U_o + U} \quad (\text{II-63})$$

$$\gamma = \tan^{-1} \left( \frac{V_o + V}{U_o + U} \right) \quad (\text{II-64})$$

and:

$$\delta = \Theta + \tan^{-1} \left( \frac{V_o + V}{U_o + U} \right) + \pi \quad (\text{II-65})$$

Further:

$$s = r \cos \delta \quad (\text{II-66})$$

$$t = r \sin \delta \quad (\text{II-67})$$

Or:

$$s = r \cos \left[ \Theta + \pi + \tan^{-1} \left( \frac{V_o + V}{U_o + U} \right) \right] \quad (\text{II-68})$$

$$t = r \sin \left[ \Theta + \pi + \tan^{-1} \left( \frac{V_o + V}{U_o + U} \right) \right] \quad (\text{II-69})$$

We may also write:

$$t = y - \frac{V_o + V}{2} \quad (\text{II-70})$$

$$s = x - \frac{U_o + U}{2} \quad (\text{II-71})$$

So that:

$$x = r \cos \left[ \Theta + \pi + \tan^{-1} \left( \frac{V_o + V}{U_o + U} \right) \right] + \frac{U_o + U}{2} \quad (\text{II-72})$$

$$y = r \sin \left[ \Theta + \pi + \tan^{-1} \left( \frac{V_o + V}{U_o + U} \right) \right] + \frac{V_o + V}{2} \quad (\text{II-73})$$

But:

$$\sin (u + v) = \sin (u) \cos (v) + \cos (u) \sin (v) \quad (\text{II-74})$$

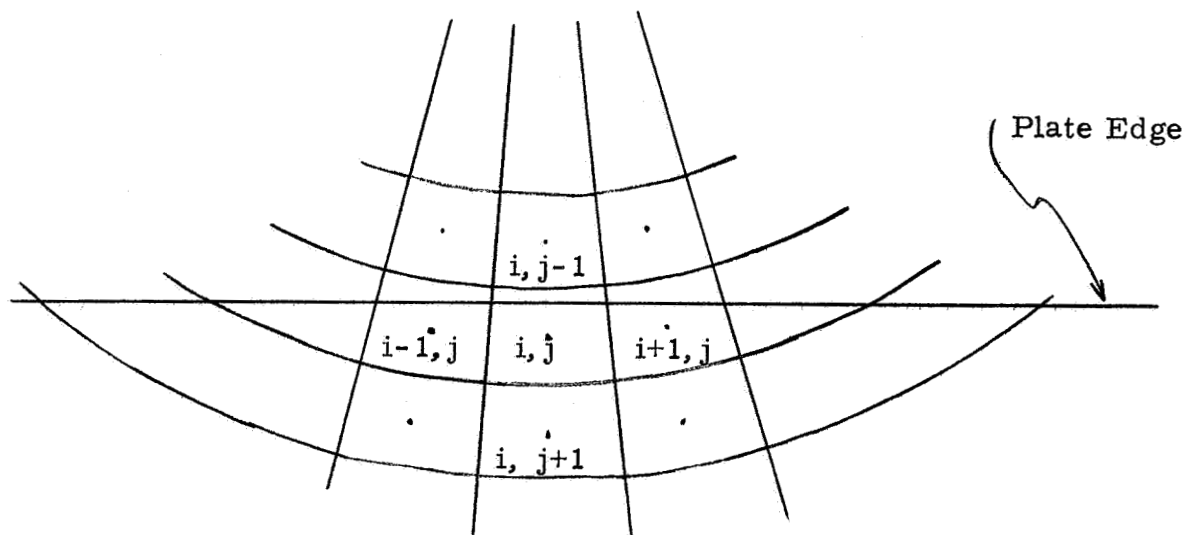
$$\cos (u + v) = \cos (u) \cos (v) - \sin (u) \sin (v) \quad (\text{II-75})$$

and:

$$\begin{aligned} x = & -r \left\{ \cos (\Theta) \cos \left( \tan^{-1} \left( \frac{V_o + V}{U_o + U} \right) \right) \right. \\ & \left. - \sin (\Theta) \sin \left( \tan^{-1} \left( \frac{V_o + V}{U_o + U} \right) \right) \right\} + \frac{U_o + U}{2} \end{aligned} \quad (\text{II-76})$$

$$y = -r \left\{ \sin(\Theta) \cos \left( \tan^{-1} \left( \frac{V_o + V}{U_o + U} \right) \right) + \right. \\ \left. + \cos(\Theta) \sin \left( \tan^{-1} \left( \frac{V_o + V}{U_o + U} \right) \right) \right\} \frac{V_o + V}{2} \quad (\text{II-77})$$

The plate edges will not fit into the mesh selected for analysis in a regular manner, and we must investigate these in more detail in order to generate a mesh which accurately represents the edge behavior as well as internal behavior. We first consider the following:



The plate edge is a straight line which, if horizontal, can be represented simply by:

$$y = C \quad (\text{II-78})$$

or, if vertical, by:

$$x = g \quad (\text{II-79})$$

Substituting these, we immediately obtain:

$$r = \frac{C - \frac{V_o + V}{2}}{\sin(\Theta) \cos \left( \tan^{-1} \left( \frac{V_o + V}{U_o + U} \right) \right) + \cos(\Theta) \sin \left( \tan^{-1} \left( \frac{V_o + V}{U_o + U} \right) \right)} \quad (\text{II-80})$$

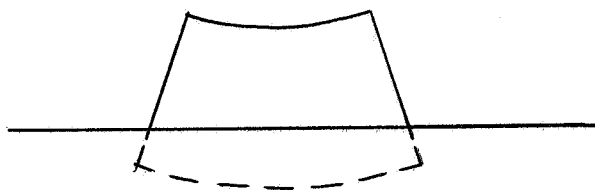
for a horizontal line and:

$$r = \frac{g - \frac{U_o + U}{2}}{\cos(\Theta) \cos\left(\tan^{-1}\left(\frac{V_o + V}{U_o + U}\right)\right) - \sin(\Theta) \sin\left(\tan^{-1}\left(\frac{V_o + V}{U_o + U}\right)\right)} \quad (\text{II-81})$$

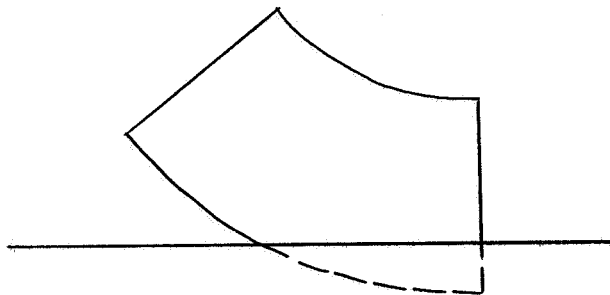
for a vertical line.

There are a number of possibilities or ways in which the horizontal line may intersect the mesh:

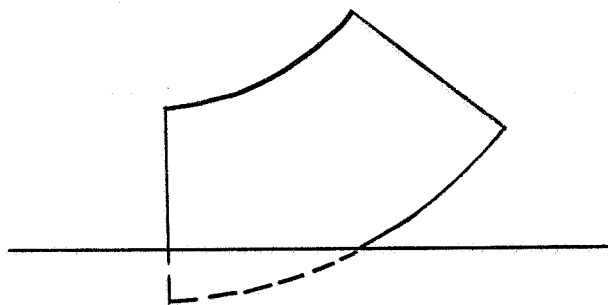
A)



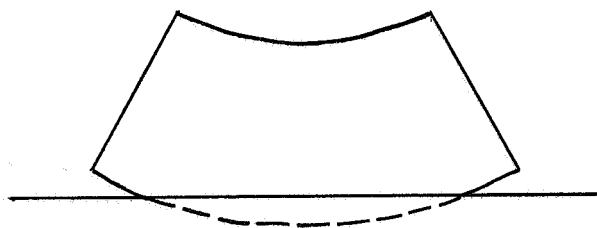
B)



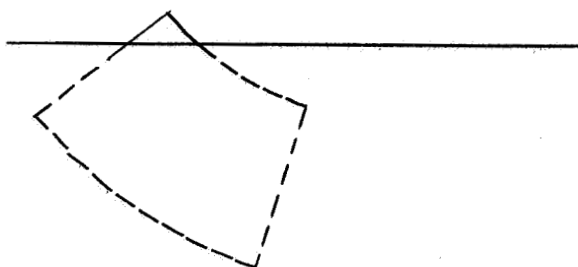
C)



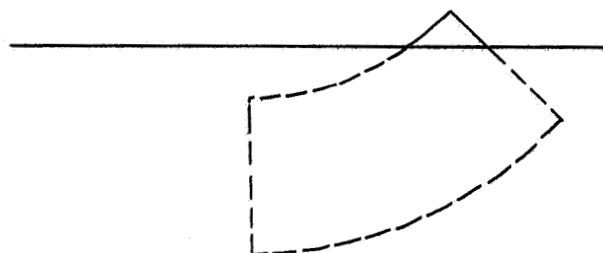
D)



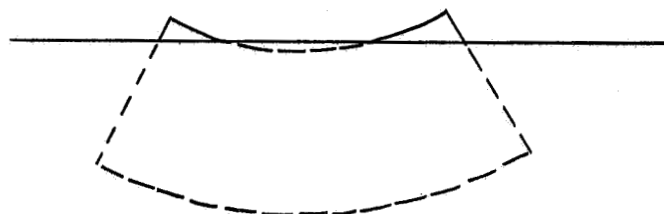
E)



F)



G)

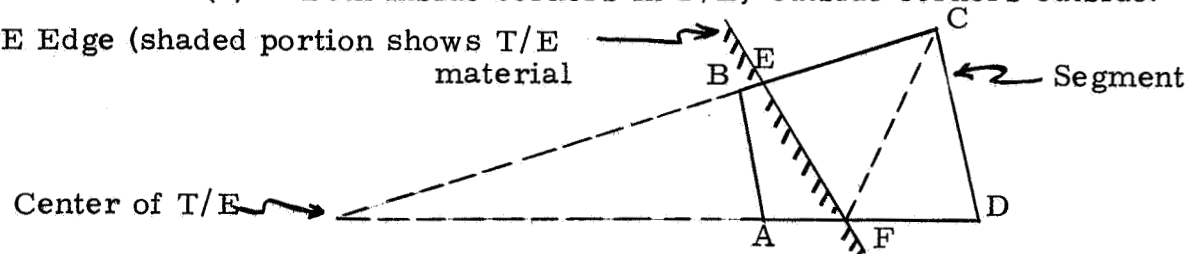


In each case the area enclosed by the solid line must be computed and included within the mesh. Although these areas can be computed exactly, the size of the increments is small and small errors in the size of several increments will have negligible effect on the overall plate behavior. We will therefore assume the segment shape to be trapezoidal. This immediately eliminates cases D and G as having a negligible effect on the mesh.

The following types of behavior can take place in fitting trapezoidal nodes to the plate-thermoelement intersection:

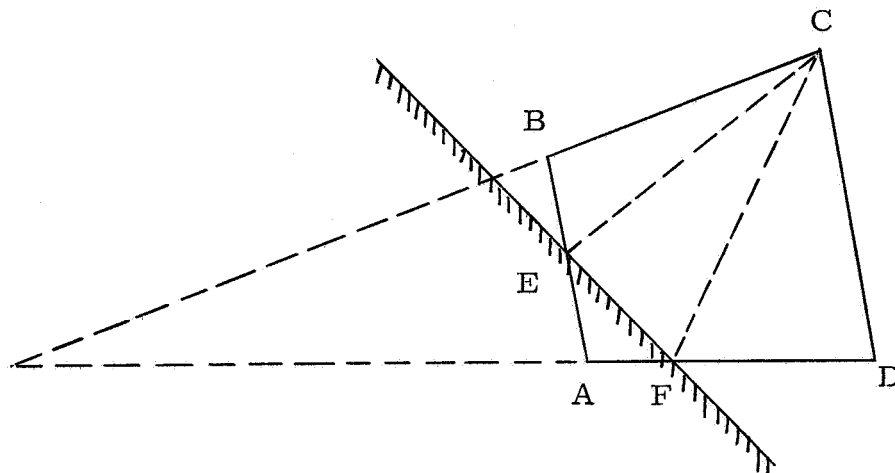
- (1) Both inside corners in T/E, outside corners outside:

T/E Edge (shaded portion shows T/E material)



Solution: Redefine points A, B; treat as case 8.

(2) Back inside corner in T/E, others outside:

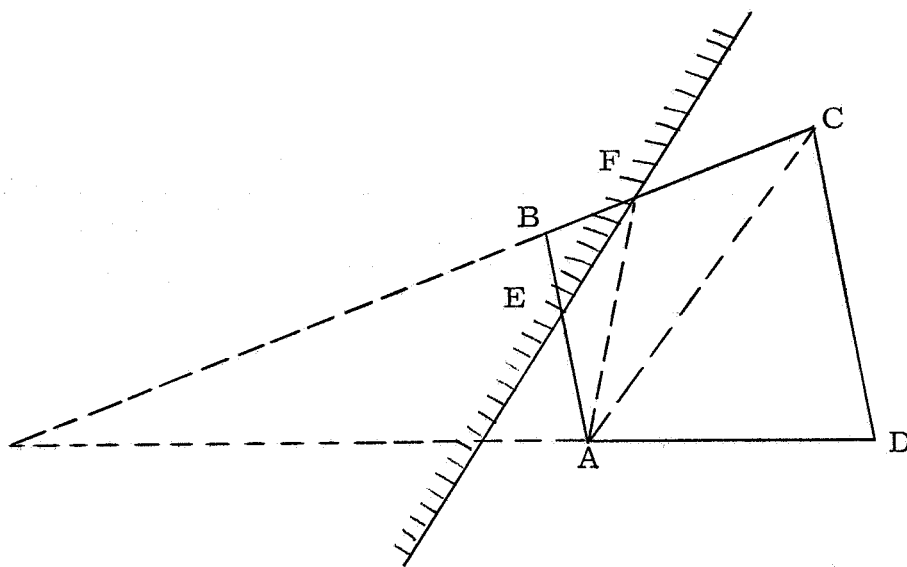


Solution: Compute area for several triangles, and add to obtain the total:

$$\text{area} = \text{BCE} + \text{CEF} + \text{CDF}$$

(II-82)

(3) Forward inside corner in T/E, others outside:

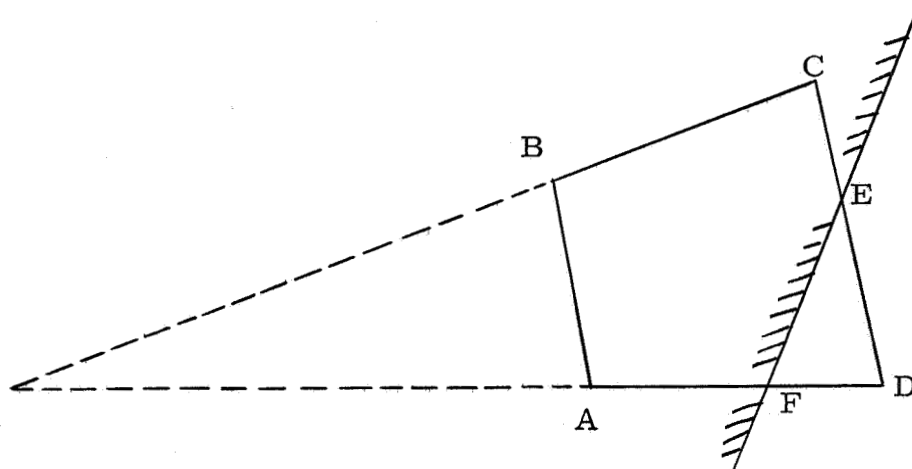


$$\text{area} = \text{AEF} + \text{ACF} + \text{ACD}$$

(II-83)



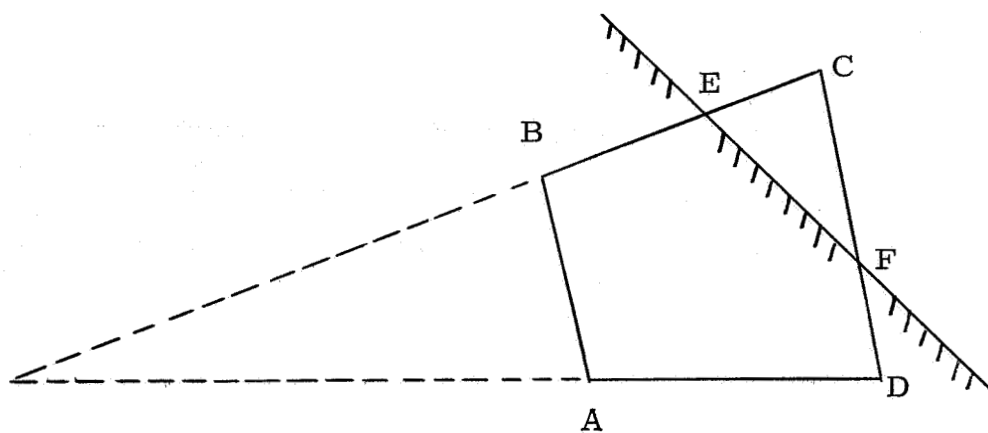
(4) Back outside corner outside T/E, others inside:



area = DEF

(II-84)

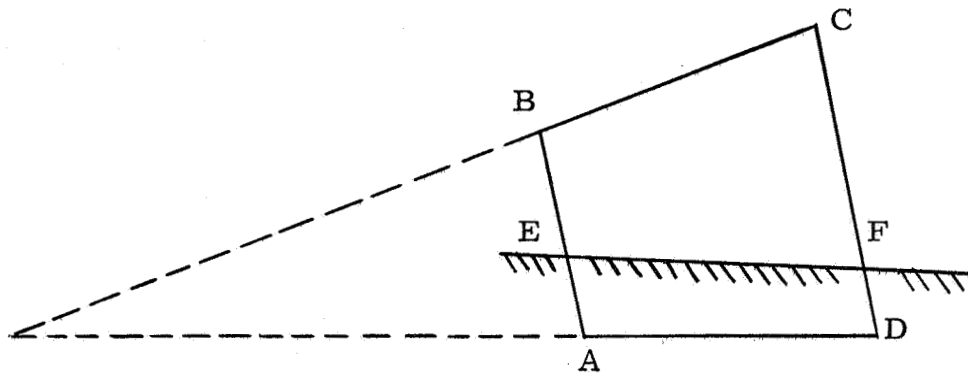
(5) Forward outside corner outside T/E, others inside:



area = CEF

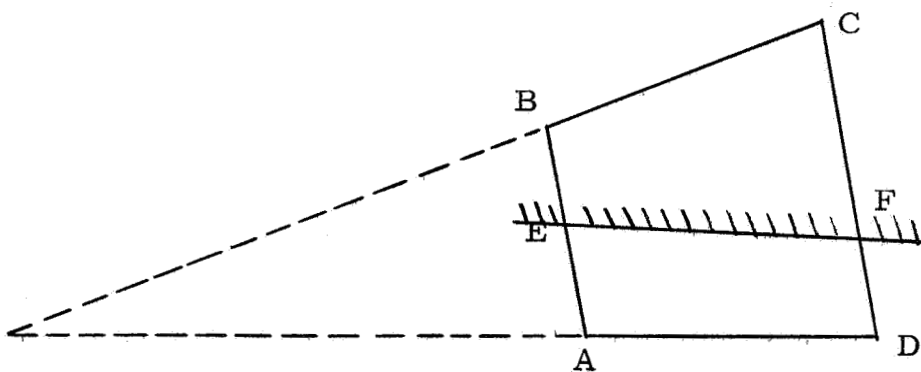
(II-85)

- (6) Backward corners in T/E, forward corners outside:



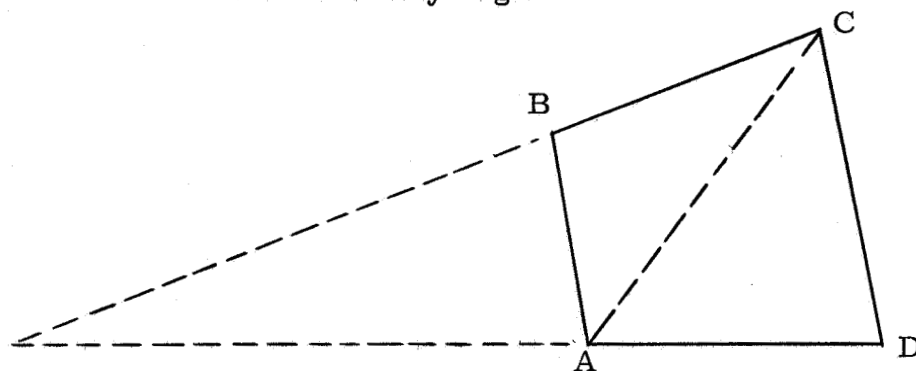
Redefine A, D; treat as case 8.

- (7) Backward corners outside, forward corners in T/E:



Redefine B, C; treat as case 8.

- (8) Finally, there is the simple case of a node that does not intersect any edges:



$$\text{area} = \text{ABC} + \text{ACD}$$

Similar construction techniques can be used to describe node behavior along the plate edge. First refer to the same sketches and apply them to the plate edge. If we assume that the shaded side is within the plate, then the following may be defined:

<u>Case</u>	<u>Area</u>
1'	Redefine C, D, Treat as 8
2'	AEF
3'	BEF
4'	ABC + ACE + AEF
5'	ABE + AEF + ADF
6'	-
7'	-

There additionally exist the possibilities that a node may be common to the plate edge and the T/E edge, or that the plate edge and the T/E edge correspond. If they correspond, there is no node at that location. If the node is common to both edges, then there exist possibilities in the plate edge treatment where neither 1 - 7 or 1' - 7' are useful. It may be more convenient to subtract the position outside the plate edge from a previously computed area than to compute the area inside the plate directly. These cases will be treated using a double prime with the understanding that the indicated area is to be subtracted. The area in each case is described by the non-prime cases, but with the edge being the edge of the plate, not the edge of the T/E. In a few of these cases, it will also be convenient to merely move the points A B C D to correspond to the plate and T/E edges. In these cases, the treatment is the same as though no edges were present after the redefinition of the points has been made.

Since all nodes may be considered by considering triangles, we next consider the characteristics which must be calculated. The area of a triangle is:

$$A = \sqrt{s [s - a] [s - b] [s - c]} \quad (\text{II-87})$$

$$\text{where: } s = \frac{1}{2} [a + b + c] \quad (\text{II-88})$$

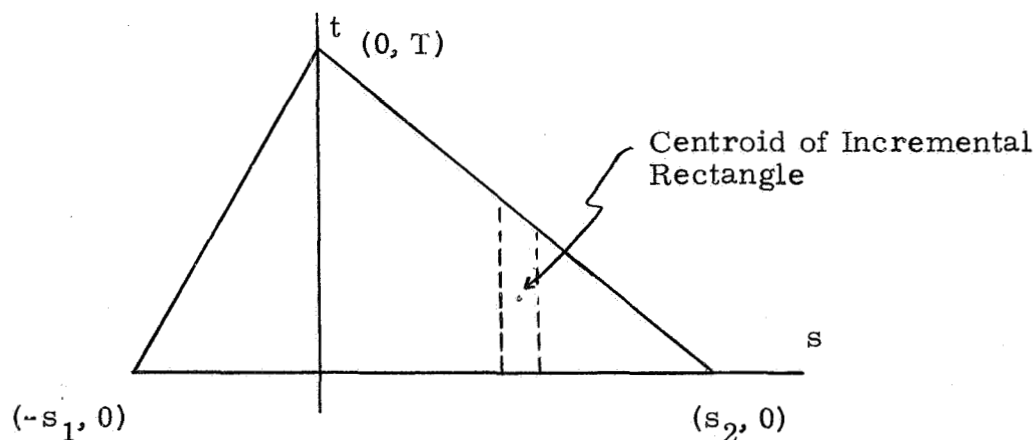
and a, b and c are the lengths of the sides. Once the coordinates of the corners have been obtained, the length of a line between the points may be determined from:

$$d = \sqrt{(x_2 - x_1)^2 + (y_2 - y_1)^2} \quad (\text{II-89})$$

where d is the line length, and the line ends are at  $(x_2, y_2)$  and  $(x_1, y_1)$ .

The selection of whether to treat the partial segment as a separate segment or as part of another segment may now be made. Some of these segments will be very small and if considered separately would require very small time steps. Consequently, we will make an arbitrary selection. If the area of a partial segment is less than one half that of the j-1 segment, then it will be attached to and made a part of the j-1 segment. This arbitrary selection is made with a small enough segment size that little error will be introduced by incorporating the segment into another segment.

The "center" of each of the segments modified by this procedure must be redetermined so that it is representative of the average behavior of the segment. We first determine the center of the partial segment without regard to its ultimate disposition. Since all areas were determined using triangles as a calculation base, we first determine the center of the triangle:



The moment of the incremental rectangle about the  $s$  axis is at  $\frac{t}{2} dA = \frac{t^2}{2} ds$ ; about the  $t$  axis it is at  $s dA = stds$ . The equation of the left line is:

$$t = \frac{T}{S_1} s + T \quad (\text{II-90})$$

and that of the right line:

$$t = +T - \frac{T}{S_2} s \quad (\text{II-91})$$

The area is:  $A = \frac{T}{2} [S_1 + S_2]$  (II-92)

The moment about the  $s$  axis is:

$$M_s = \int_{-S_1}^0 \frac{t^2}{2} ds + \int_0^{S_2} \frac{t^2}{2} ds \quad (\text{II-93})$$

Therefore:

$$M_s = \frac{1}{2} \int_{-S_1}^0 \left[ \frac{T}{S_1} s + T \right]^2 ds + \frac{1}{2} \int_0^{S_2} \left[ T - \frac{T}{S_2} s \right]^2 ds \quad (\text{II-94})$$

$$M_t = \int_{-S_1}^0 s t ds + \int_0^{S_2} s t ds \quad (\text{II-95})$$

$$M_t = \frac{T [S_2^2 - S_1^2]}{6} \quad (\text{II-96})$$

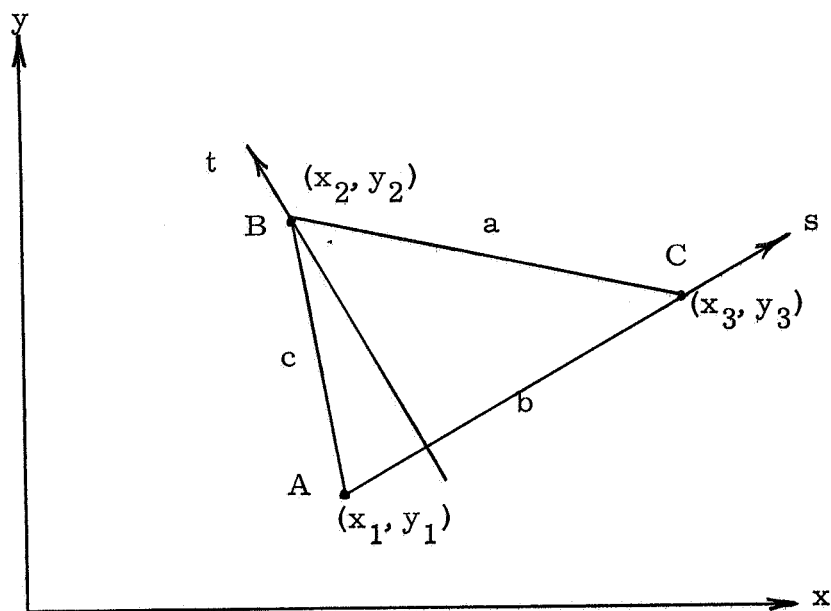
The centroids are now:

$$\bar{t} = \frac{2 T^2 [S_1 + S_2]}{6 T [S_1 + S_2]} = \frac{T}{3} \quad (\text{II-97})$$

$$\bar{s} = \frac{1}{3} [S_2 - S_1] \quad (\text{II-98})$$

These centroids are based upon a triangle with an orientation as shown in the previous sketch. The orientations necessary here will be located on a different coordinate system with a different zero point and in most cases with a different rotational orientation. These results will not be applied using the (s, t) system, but the results are simple and can be easily applied to the coordinate system under consideration.

Now suppose the corners of the triangle are located at  $(x_1, y_1)$ ,  $(x_2, y_2)$  and  $(x_3, y_3)$ . We will arbitrarily choose  $x_2$  as corresponding to  $s = 0$  and  $y_2$  as corresponding to  $t = T$ . Further,  $(x_1, y_1)$  will correspond to  $(-S_1, 0)$  and  $(x_3, y_3)$  to  $(S_2, 0)$ .



from the law of cosines:

$$\cos A = \frac{b^2 + c^2 - a^2}{2bc} \quad (\text{II-99})$$

Further:

$$a = \sqrt{(x_2 - x_3)^2 + (y_2 - y_3)^2} \quad (\text{II-100})$$

$$b = \sqrt{(x_1 - x_3)^2 + (y_1 - y_3)^2} \quad (\text{II-101})$$

$$c = \sqrt{(x_1 - x_2)^2 + (y_1 - y_2)^2} \quad (\text{II-102})$$

$$S_1 = c \cos A \quad (\text{II-103})$$

$$S_2 = b - S_1 \quad (\text{II-104})$$

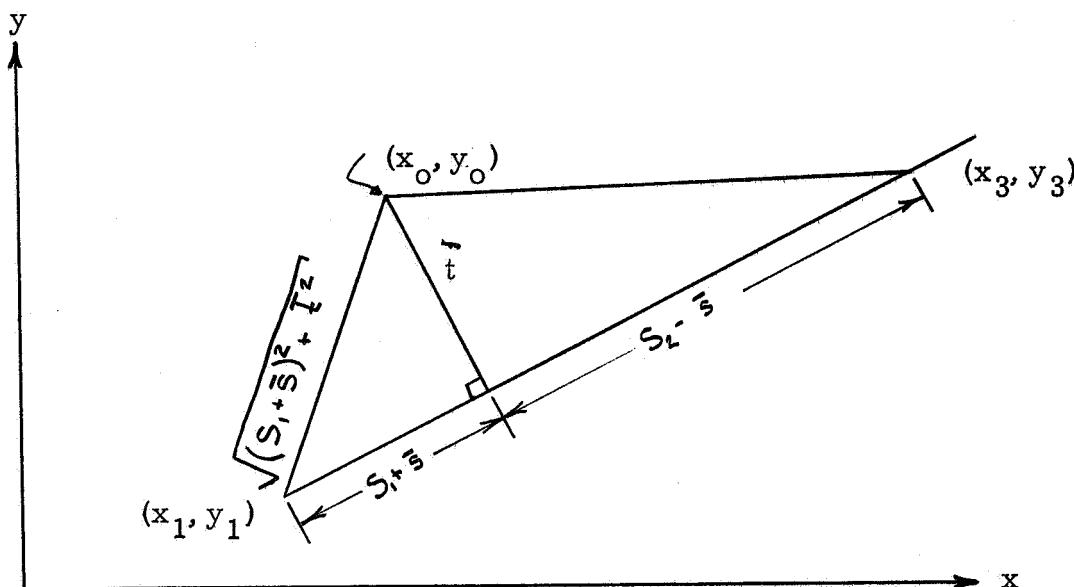
$$T = c \sin A \quad (\text{II-105})$$

But:  $\sin^2 A + \cos^2 A = 1 \quad (\text{II-106})$

$$\sin A = \sqrt{1 - \cos^2 A} \quad (\text{II-107})$$

$$T = c \sqrt{1 - \cos^2 A} \quad (\text{II-108})$$

With Equations (II-100) through (II-108) we can immediately find  $\bar{s}$  and  $\bar{t}$  from Equations (II-98) and (II-97). We may now convert these to the (x, y) coordinate system.



$$\sqrt{(S_1 + \bar{s})^2 + \bar{t}^2} = \sqrt{(x_1 - x_0)^2 + (y_1 - y_0)^2} = \sqrt{D} \quad (\text{II-109})$$

$$\sqrt{(S_2 - \bar{s})^2 + \bar{t}^2} = \sqrt{(x_3 - x_0)^2 + (y_3 - y_0)^2} = \sqrt{G} \quad (\text{II-110})$$

$$x_1^2 - 2x_0 x_1 + x_0^2 + y_1^2 - 2y_0 y_1 + y_0^2 = D \quad (\text{II-111})$$

$$x_3^2 - 2x_0 x_3 + x_0^2 + y_3^2 - 2y_0 y_3 + y_0^2 = G \quad (\text{II-112})$$

$$x_0 = \frac{D - G - x_1^2 + x_3^2 - y_1^2 + y_3^2 + 2y_0 [y_1 - y_3]}{-2 [x_1 - x_3]} \quad (\text{II-113})$$

This can be written as:

$$x_0 = E + F y_0 \quad (\text{II-114})$$

and we may substitute this into Equation (II-111) to obtain:

$$x_1^2 - 2x_1 [E + Fy_0] + [E + Fy_0]^2 + y_1^2 - 2y_0 y_1 + y_0^2 = D \quad (\text{II-115})$$

$$\begin{aligned} & [F^2 + 1] y_0^2 + [-2x_1 F + 2E F - 2y_1] y_0 + \\ & + x_1^2 - 2x_1 E + E^2 + y_1^2 - D = 0 \end{aligned} \quad (\text{II-116})$$

which can be written:

$$a y_0^2 + b y_0 + C = 0 \quad (\text{II-117})$$

with the solution:

$$y_0 = \frac{-b \pm \sqrt{b^2 - 4ac}}{2a} \quad (\text{II-118})$$

To determine which solution is to be used we first note that  $y_0$  must be between  $y_1$  and  $y_3$ ,  $y_1$  and  $y_2$ , or  $y_2$  and  $y_3$ ; and  $x_0$  must be between  $x_1$  and  $x_3$ ,  $x_1$  and  $x_2$ , or  $x_2$  and  $x_3$ . It is not likely both the  $y_0$ 's will satisfy this requirement and we will not investigate this further unless a problem is found to exist in the actual calculations.

There can be a problem if  $x_1 = x_3$  because of a zero in the denominator of Equation (II-113). In this case we write:

$$y_1^2 - y_3^2 - 2y_0 [y_1 - y_3] = D - G \quad (\text{II-119})$$

$$\frac{y_1^2 - y_3^2 - D + G}{2 y_1 - y_3} = y_0 \quad (\text{II-120})$$

$$x_0^2 + (-2x_1) x_0 + (x_1^2 + y_1^2 - 2y_0 y_1 + y_0^2 - D) = 0 \quad (\text{II-121})$$



and:

$$x_0 = \frac{-bt \pm \sqrt{bt^2 - 4 * at * ct}}{2 * at} \quad (\text{II-122})$$

With the "center" of the triangle determined to be  $(x_0, y_0)$ , we may now determine the new center of the segment. This will be located at:

$$x_s = x_n A_n + x_0 A_t \quad (\text{II-123})$$

$$y_s = y_n A_n + y_0 A_t \quad (\text{II-124})$$

where: s refers to the edge segment center

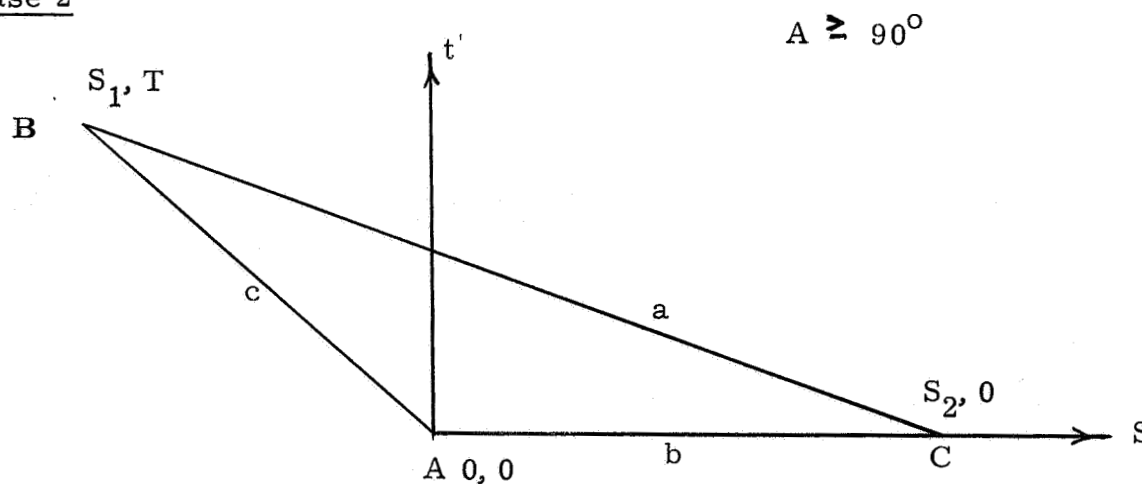
n refers to the original node

$A_t$  = triangle area, positive if the triangle is to be added,  
negative if removed from the original node.

If a trapezoid is to be added or removed, the location of the center is a simple addition of two triangles and will not be discussed here.

Two other "types" of triangles can result during the calculation process, which will be referred to as Cases 2 and 3. Similar investigations result in the following:

#### Case 2



$$s_1 = c (-\cos A) \quad (\text{II-125})$$

$$s_2 = b \quad (\text{II-126})$$

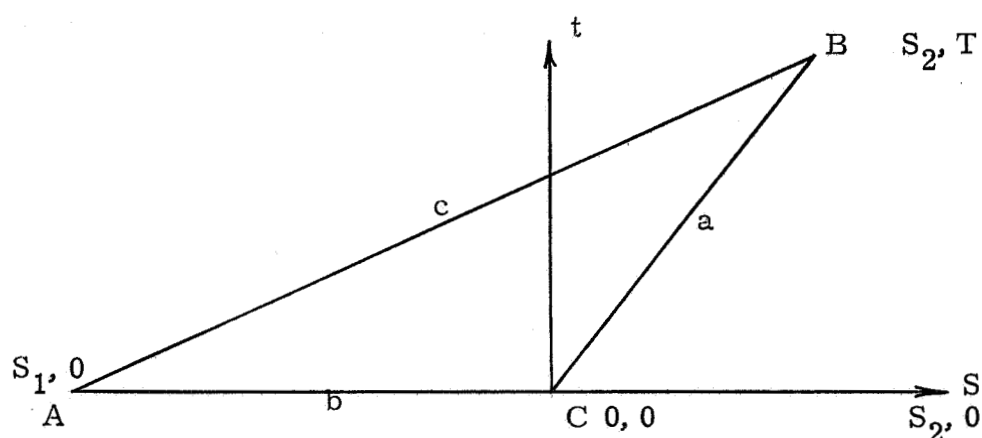
$$T = c \sqrt{1 - \cos^2 A} \quad (\text{II-127})$$

$$\bar{s} = \frac{s_1 s_2 - s_1^2}{3(s_1 + s_2)} \quad (\text{II-128})$$

$$\bar{t} = \frac{T s_1}{3(s_1 + s_2)} \quad (\text{II-129})$$

Case 3

$$C \geq 90^\circ$$



$$s_1 = b \quad (\text{II-130})$$

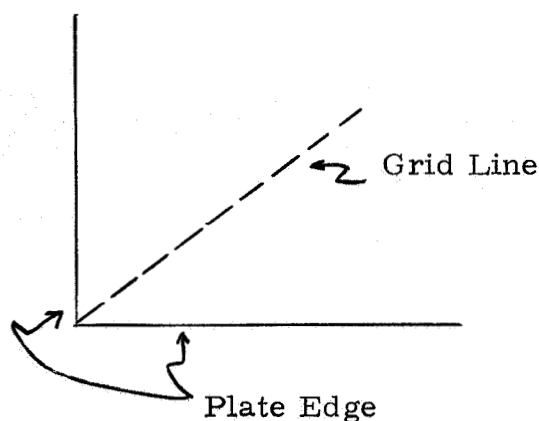
$$s_2 = a (-\cos C) \quad (\text{II-131})$$

$$T = a \sqrt{1 - \cos^2 C} \quad (\text{II-132})$$

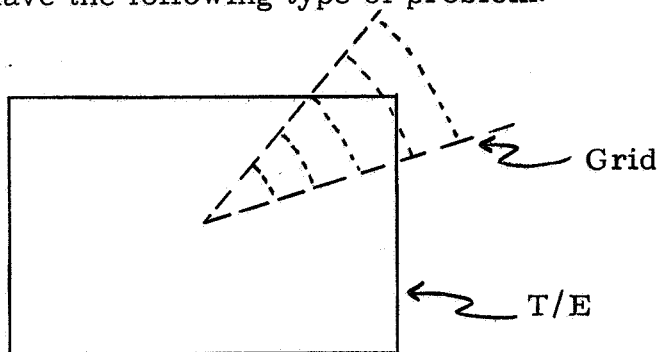
Plate corners will introduce no problem in mesh construction because of the angular construction technique. In all cases along the outside edges, the node construction is of the type:

$$\bar{s} = \frac{s_2^2 - s_2 s_1}{3(s_2 + s_1)} \quad (\text{II-133})$$

$$\bar{t} = \frac{t}{3} \frac{s_2}{s_1 + s_2} \quad (\text{II-134})$$

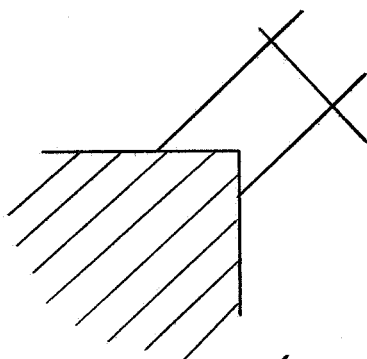


for all four corners. However, this is not the case for the thermoelement edge, and here we can have the following type of problem:

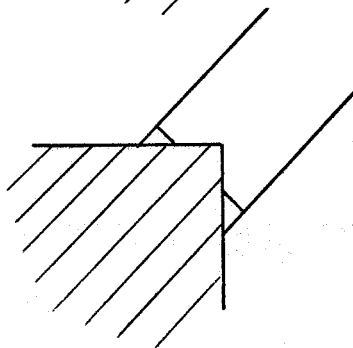


which must be considered. The following cases can result:

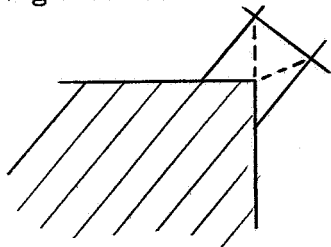
(1)



(2)



Case two will be treated by merely adding the triangular segments to the adjoining segment (the increased  $r$  position) and adjusting the segment center location accordingly (as previously discussed). Case one can be considered by the following construction:



which gives three triangles and which may be immediately considered by the techniques already discussed.

Unfortunately, the ability to compute the various node characteristics is not worthwhile if the computer cannot be instructed to select the type of node.

To determine the type of node under consideration, we first compare point A on the trapezoid to the thermoelement edge at the same angular location and the corresponding plate edge. An index value,  $n_A$ , is assigned according to the following:

Location	Index, $n_A$
In plate	1
In T/E	2
Outside plate	3

Values on an edge are treated as though they were in the plate. Similar assignments are then made for points B, C and D. We may then compute a "case index",  $MZU$ , according to the equation:

$$MZU = n_D + 3 (n_C - 1) + 9 (n_B - 1) + 27 (n_A - 1) \quad (\text{II-135})$$

There results the values shown in Table II-2. The item is the case to be treated, and these are defined in Table II-3. The item numbers also correspond to the source program statement numbers in subroutine MESHA. The branching is performed by using  $MZU$  as the index for a computed go to statement (see source statement number 24).

TABLE II-2. CORNER TREATMENT CHARACTERISTICS

$n_A$	$n_B$	$n_C$	$n_D$	MZU	Item	$n_A$	$n_B$	$n_C$	$n_D$	MZU	Item
1	1	1	1	1	36	2	2	2	3	42	56
1	1	1	2	2	37	2	2	3	1	43	57
1	1	1	3	3	38	2	2	3	2	44	58
1	1	2	1	4	37	2	2	3	3	45	59
1	1	2	2	5	37	2	3	1	1	46	37
1	1	2	3	6	37	2	3	1	2	47	37
1	1	3	1	7	39	2	3	1	3	48	37
1	1	3	2	8	37	2	3	2	1	49	37
1	1	3	3	9	40	2	3	2	2	50	37
1	2	1	1	10	41	2	3	2	3	51	37
1	2	1	2	11	37	2	3	3	1	52	70
1	2	1	3	12	42	2	3	3	2	53	60
1	2	2	1	13	68	2	3	3	3	54	61
1	2	2	2	14	37	3	1	1	1	55	37
1	2	2	3	15	69	3	1	1	2	56	37
1	2	3	1	16	37	3	1	1	3	57	62
1	2	3	2	17	37	3	1	2	1	58	37
1	2	3	3	18	43	3	1	2	2	59	37
1	3	1	1	19	37	3	1	2	3	60	37
1	3	1	2	20	37	3	1	3	1	61	37
1	3	1	3	21	37	3	1	3	2	62	37
1	3	2	1	22	37	3	1	3	3	63	63
1	3	2	2	23	37	3	2	1	1	64	37
1	3	2	3	24	37	3	2	1	2	65	37
1	3	3	1	25	44	3	2	1	3	66	64
1	3	3	2	26	37	3	2	2	1	67	37
1	3	3	3	27	45	3	2	2	2	68	37
2	1	1	1	28	46	3	2	2	3	69	65
2	1	1	2	29	47	3	2	3	1	70	37
2	1	1	3	30	37	3	2	3	2	71	37
2	1	2	1	31	37	3	2	3	3	72	66
2	1	2	2	32	37	3	3	1	1	73	37
2	1	2	3	33	37	3	3	1	2	74	37
2	1	3	1	34	48	3	3	1	3	75	37
2	1	3	2	35	49	3	3	2	1	76	37
2	1	3	3	36	50	3	3	2	2	77	37
2	2	1	1	37	51	3	3	2	3	78	37
2	2	1	2	38	52	3	3	3	1	79	37
2	2	1	3	39	53	3	3	3	2	80	37
2	2	2	1	40	54	3	3	3	3	81	67
2	2	2	2	41	55						

TABLE II-3. ITEM DEFINITIONS

Item (MESHA Source Program Statement Number)	Case
36	8
37	impossible mesh specification
38	4'
39	5'
40	1'
41	3
42	3 - 4"
43	3 - 1"
44	6'
45	2'
46	2
47	6
48	2 - 5"
49	6 - 5"
50	2 - 1"
51	1
52	5
53	1 - 4"
54	4
55	all T/E
56	4 - 4"
57	1 - 5"
58	5 - 5"
59	1 - 1"
60	6 - 6"
61	2 - 2"
62	7'
63	3'
64	3 - 7"
65	7 - 7"
66	3 - 3"
67	all outside
68	7
69	7 - 4"
70	2 - 6"

### D. Steady State Calculations

Equation (II-16) may be rewritten in the following form in the steady state:

$$\begin{aligned} & \frac{k_{n+1} + k_n}{2 \Delta Z} [T_{n+1} - T_n] - \frac{k_n + k_{n-1}}{2 \Delta Z} [T_n - T_{n-1}] + \\ & + \frac{\rho_n I^2 \Delta Z}{A^2} - \frac{T_n I}{2A} [\alpha_{n+1} - \alpha_{n-1}] - \frac{F P}{A} \\ & [T_n^4 - \overline{T_s^4}] = 0 \end{aligned} \quad (\text{II-135})$$

This can be rewritten as:

$$\begin{aligned} & \frac{k_{n+1} + k_n}{2} [T_{n+1} - T_n] - \frac{k_n + k_{n-1}}{2} [T_n - T_{n-1}] + \frac{\rho_n I^2 \Delta Z^2}{A^2} - \\ & - \frac{T_n I \Delta Z}{2A} [\alpha_{n+1} - \alpha_{n-1}] - \frac{F P}{A} [T_n^4 - \overline{T_s^4}] = 0 \end{aligned} \quad (\text{II-136})$$

or:

$$\begin{aligned} T_n = & \left[ \frac{\rho_n I^2 \Delta Z}{A^2} - \frac{T_n I \Delta Z}{2A} (\alpha_{n+1} - \alpha_{n-1}) - \frac{F P}{A} \right. \\ & (T_n^4 - \overline{T_s^4}) + \frac{(k_n + k_{n+1})}{2} T_{n+1} + \frac{(k_n + k_{n-1})}{2} T_{n-1} \left. \right] \left[ \right. \\ & \left. \frac{k_n + k_{n+1}}{2} + \frac{k_n + k_{n-1}}{2} \right]^{-1} \end{aligned} \quad (\text{II-137})$$

Equation (II-137) forms the basis for the steady state calculation for the thermoelement.

The absorber and the radiator may be considered by rewriting Equation (II-36) for the steady state situation:

$$\frac{Q_{i,j}}{t_p} - \frac{\sigma F_{i,j}}{t_p r_j \Delta \Theta \Delta r} [T_{i,j}^4 - \overline{T'}^4] + S_{i,j} - \sigma \epsilon_{o,i,j} T_{i,j}^4 +$$

$$\begin{aligned}
& + \frac{1}{r_j \Delta \Theta \Delta r^2} \left\{ \Delta \Theta k_{i,j} \left[ r_j - \frac{\Delta r}{2} \right] [T_{i,j-1} - T_{i,j}] + \right. \\
& + \Delta \Theta k_{i,j+1} \left[ r_j + \frac{\Delta r}{2} \right] [T_{i,j+1} - T_{i,j}] + \\
& + k_{i,j} \frac{\Delta r^2}{r_j \Delta \Theta} [T_{i+1,j} - T_{i,j}] + \frac{k_{i,j} \Delta r^2}{r_j \Delta \Theta} \\
& \left. [T_{i-1,j} - T_{i,j}] \right\} = 0 \tag{II-138}
\end{aligned}$$

where the thermal conductivity behavior has been approximated. Now consider the following:

$$k_{i,j} \frac{\left[ r_j - \frac{\Delta r}{2} \right] \Delta \Theta t_p}{\Delta r} = Y_{r_{i,j}} = \text{radial conductance} \tag{II-139}$$

$$k_{i,j+1} \frac{\left[ r_j + \frac{\Delta r}{2} \right] \Delta \Theta t_p}{\Delta r} = Y_{r_{i,j+1}} = \text{radial conductance} \tag{II-140}$$

$$k_{i,j} \frac{\Delta r t_p}{r_j \Delta \Theta} = Y_{\Theta_{i,j}} = \text{angular conductance} \tag{II-141}$$

$$r_j \Delta \Theta \Delta r = A_{i,j} = \text{surface area} \tag{II-142}$$

Use of these equations allows Equation (II-138) to be rewritten as:

$$\begin{aligned}
& Q_{i,j} = \frac{\sigma F_{i,j}}{A_{i,j}} [T_{i,j}^4 - T^4] + S_{i,j} t_p - \sigma \epsilon_{o_{i,j}} t_p T_{i,j}^4 + \\
& + \frac{1}{A_{i,j}} \left\{ Y_{r_{i,j}} [T_{i,j-1} - T_{i,j}] + Y_{r_{i,j+1}} (T_{i,j+1} - T_{i,j}) + \right.
\end{aligned}$$



$$+ Y_{\Theta_{i+1,j}} \left[ T_{i+1,j} - T_{i,j} \right] + Y_{\Theta_{i,j}} \left[ T_{i-1,j} - T_{i,j} \right] \} = 0 \quad (\text{II-143})$$

or:

$$\begin{aligned} T_{i,j} = & A_{i,j} \left[ Q_{i,j} - \frac{\sigma F_{i,j}}{A_{i,j}} \left[ T_{i,j}^4 - \overline{T}^4 \right] + S_{i,j} t_p - \right. \\ & - \sigma \epsilon_{o i,j} t_p T_{i,j}^4 + \frac{1}{A_{i,j}} \left\{ Y_{r i,j} T_{i,j-1} + Y_{r i,j+1} T_{i,j+1} + \right. \\ & \left. \left. + Y_{\Theta_{i+1,j}} T_{i+1,j} + Y_{\Theta_{i,j}} T_{i-1,j} \right\} \right] \left[ Y_{r i,j} + Y_{r i,j+1} + \right. \\ & \left. \left. + Y_{\Theta_{i+1,j}} + Y_{\Theta_{i,j}} \right]^{-1} \end{aligned} \quad (\text{II-144})$$

Equation (II-144) represents the form utilized for the steady state calculations of the absorber or the radiator temperatures. Although this equation has been derived by considering the difference equation which describes the plate behavior, it is virtually identical to the one obtained by the computer program from the automatically generated mesh arrangement. The only basic differences pertain to the determination of the conductance, which in the computer program, is obtained with a variable increment size in the radial and angular directions.

Investigation of Equations (II-137) and (II-144) show that the temperature which appears on the left side of the equation also appears on the right side to the fourth power. This is an unstable configuration if the term contributes to a significant degree to the solution. Normally, the contribution in Equation (II-137) is small and the effect is damped so that instability does not occur except in extreme circumstances. This is not the case when Equation (II-144) is applied to the radiator, although a problem probably will not occur in the absorber. The radiator rejects virtually all of the heat associated with the system to the surroundings via this temperature term to the fourth power. This represents an extremely unstable form. The instability will normally take the appearance of calculating a temperature on the left side which is too low, followed by the recalculation using the new value on the right side which gives a temperature on the left that is too high, followed again by one which

is too low, with divergence occurring very rapidly. Damping of this effect with this alternating type behavior is relatively easy to obtain by incorporating a fractional portion of the new value as computed by Equation (II-144) into the value which appears on the right side. This approach is utilized in the computer program as convergence to the correct answers is obtained.

It is not desirable to use strong damping during the entire calculation because this slows convergence to the correct answer. However, criteria still must be applied with this type of an equation so that stability is assured if it is not damped. The calculation approach utilized herein was to assume an allowed temperature change per iteration, and damp or extrapolate the results accordingly. The application of this process to the solutions may be obtained by considering:

$$T_{\text{new}} = T_{\text{old}} + f(T_{\text{old}}) + C T_{\text{old}}^4 \quad (\text{II-145})$$

where:

$T_{\text{new}}$	=	new value of temperature
$T_{\text{old}}$	=	previous value of temperature
$f$	=	a linear function of $T_{\text{old}}$ which need not be defined here
$C$	=	a constant

which is the type of equation that must be handled. Since the temperature change per iteration is:

$$\Delta T = T_{\text{new}} - T_{\text{old}} \quad (\text{II-146})$$

Equation (II-145) becomes:

$$\Delta T = f(T_{\text{old}}) + C T_{\text{old}}^4 \quad (\text{II-147})$$

Now consider that the calculation is to be forced so that a change of  $\Delta T'$  is to occur per iteration, and that a number of equations of the type of (II-147) are applied to different nodes with some other node temperatures appearing on the right side. Then each of the  $\Delta T_i$  (for the  $i$ th node) may be computed. If  $\Delta T_{\text{max}}$  is the largest of the  $\Delta T_i$ , then all of the  $\Delta T_i$  are recalculated according to:

$$\Delta T_{i_{\text{new}}} = \Delta T_{i_{\text{old}}} \frac{\Delta T'}{\Delta T_{\text{max}}} \quad (\text{II-148})$$

Immediately, the new temperature for the next iteration may be obtained from:

$$T_{\text{new}_i} = T_{\text{old}_i} + \Delta T_{i_{\text{new}}} \quad (\text{II-149})$$

This type of iteration will never converge to steady state answers because it automatically extrapolates so that the greatest temperature change is equal to  $\Delta T'$ . Consequently, as the correct value is approached the forcing technique represented by Equations (II-148) and (II-149) is removed from the calculation procedure and a damping factor, as previously discussed, is applied. The point at which this occurs in the computer program is determined from the user's estimate of how many iterations will be required to reach steady state. This means that for most problems the user's estimate of the number of iterations will represent the minimum number of iterations the code will perform. It will virtually always require more iterations than estimated before it will automatically terminate the calculation procedure. If the termination occurs shortly after the estimated number of iterations has been reached, then the estimated value for future problems of that type should be reduced.

### III. COMPUTER PROGRAM USE INSTRUCTIONS

#### A. Data Input

##### 1. Introduction

Input for this program is flexible and for practical purposes need not follow any particular format. The location of the data on the cards, the number of items per card, the field length, etc., are entirely at the discretion of the user. Columns 1-72 may be used for input information. Other restrictions are that individual numbers must be separated by at least one blank, and one number may not be continued from one card to another. Extra blanks can be included as desired, since they will be ignored by the code. In general, however, the more compact the input the faster will be the data processing.

There is no designation between fixed point and floating point variables in the input. Hence, the following may all be used to read a "one" into the machine:

1  
1.  
1.0  
1.0E+0  
10.E-1  
1.+0  
10.-1  
1.000000  
+1  
+10.-1  
.1E+1  
0.10E+1  
+0.1E+1  
0.1+1  
etc.

The only restriction for inputting numbers, aside from number separation by blanks, is that numbers greater than approximately 32,000 must be specified in an exponential form.

This computer program is written in such a manner that it may be readily expanded. Therefore, the input sections of the program have been programmed so that more information can be provided than is presently utilized. The input description is written to be compatible with the existing

input routines, and where information that is provided in the input is not utilized, comments are made to that effect. This approach has been used so that as additions are made to the program, the use instructions may be kept up to date by merely deleting the necessary comments.

The input data are divided into major groups, which are in turn divided into small groups or individual items. Specification of the group (for example \*3 for group 3) and a subgroup number defines the data to follow. Property tables may be read in any order, and the length of the table need not be given. The code will recognize the end of tables from the input. Groups and subgroups may be read in any desired order. When the end of input data is signaled, the code will perform a logical check to be sure that all of the necessary data have been read and that no conflicts have been specified. The data tables will be placed in the proper order and calculations begun.

Following problems may be investigated very easily when parametric investigations are to be performed. In these cases it is only necessary to read those data that are to be changed.

The panel may be defined by reading the overall dimensions of each of the constituents in terms of an (x, y) coordinate system oriented at one corner of the panel, and by reading thermoelement length. Specification of the number of divisions desired for computational purposes along each major dimension is sufficient for setting up a mesh. The code utilizes this information to generate a mesh approximately conforming to the input specifications.

## 2. Control Cards

Control cards are designated with a \* in column one and a number in column two. The remainder of the card may be used for input information if desired. The following designations have been established:

<u>Control Card</u>	<u>Meaning</u>
*1	Descriptive Information
*2	Dimensions
*3	Thermoelement Properties
*4	Other Properties
*5	Heating Information
*6	Specifications
*7	Input complete - Partial input for another problem follows
*8	Input complete, complete input for another problem follows
*9	Input complete, no input follows

With the exception of \*7 - \*9 cards, the information may be read in any desired order. The \*7 - \*9 cards are used to signal the end of input for a problem and are therefore located last. The \*1 - \*6 cards may be read in any order and, if desired, partial information in one group may be read. This makes possible such input options as reading some \*2 information, then some \*5 information, then some more \*2, etc. Further, if only one item is to be changed between problems, this may be done by reading only that one item. Other information previously read into the machine will be saved unchanged with the exception of \*1 information.

It is never necessary to specify how much information is to be read before reading. Termination of a group of information will occur if a "Z" is inserted following the data. This can appear anywhere on a card, and can be followed on the same card by other numerical information. The first number following the Z must be a control number (as discussed under the \* headings). A Z may not be used if the information to be read consists of a fixed number of words.

### 3. \*1, Descriptive Information

Information read by the \*1 card is printed as output at the beginning of the problem for which it is read and is then destroyed. The first card of this group must contain \*1 in columns 1 and 2. Columns 3-72 may be used for any title information that is desired. Following cards do not need \*1 in columns 1 and 2 although no harm will be done if this is included. In either case, only columns 3-72 will be printed as output. As many cards as desired may be read and printed in this manner.

It is not necessary to reread data already in the machine since this information is utilized automatically. (See \*7 and \*8 for further discussion of this point).

### 4. \*2, Dimensions

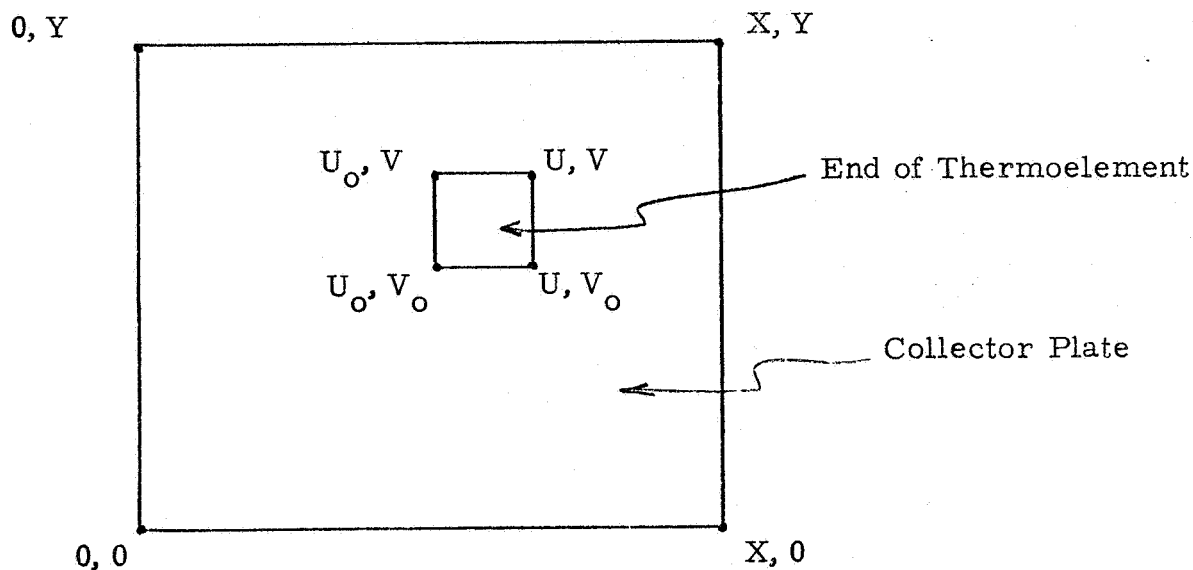
The first number read following \*2 is a control number, which will be referred to as C throughout the remainder of this report. The value for C is then followed by the required information as specified in this section and summarized in Table 1. Another C may follow the presented information or a new control card may be used. The C's may be read in any desired order and as few C's as desired may be used, subject to the restriction that the dimensions must be fully described or the information must have been previously read.

TABLE III-1.\* \*2 CONTROL NUMBERS (C)

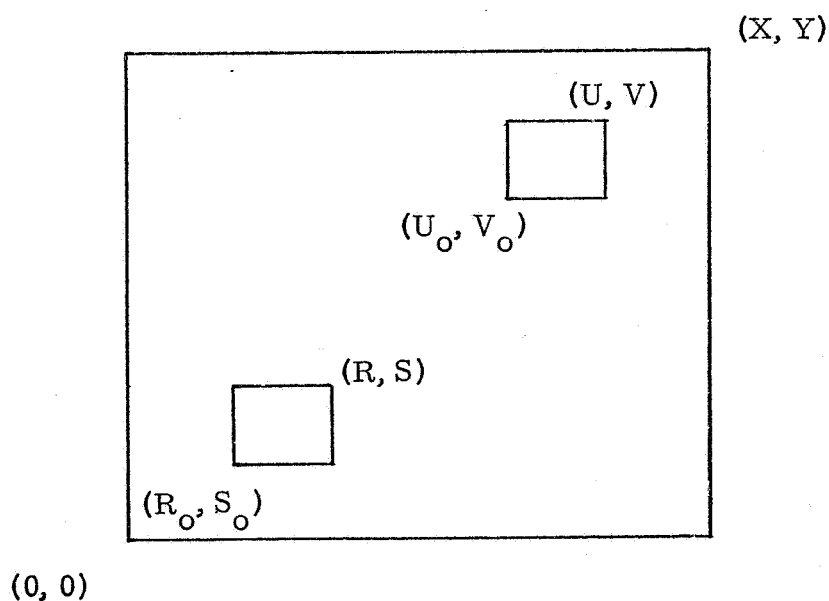
<u>C Value</u>	<u>Data Required</u>
1	Collector thickness
2	Collector dimensions X, Y and Z
3	Thermoelement dimensions Uo, Vo, U and V
4	Thermoelement dimensions Uo, Vo, U, V, Ro, So, R, S
5	Thermoelement length for non-segmented thermoelements
6	Number of segments followed by lengths for segmented elements
7	Thermoelement material designations
8	Radiator and edge dimensions
9	Hot shoe thickness and material designations
10	Cold shoe thicknesses and material designations
11	Mesh information
12	Boundary information

\* The type 2 information used by the program in the present version consists of X and Y. Z must be read, but is not used for calculations. The options offered under the type 4 and type 6 input are not available, and C values corresponding to these should not be used. The type 8 information pertains to radiator edge thickness and edge width as well as radiator thickness. Only the radiator thickness is used in the calculations, although all 3 values must be provided for proper functioning of the input. The type 9 and type 10 information is used only to provide contact resistance data in the present version.

Two types of collector plates can be considered:

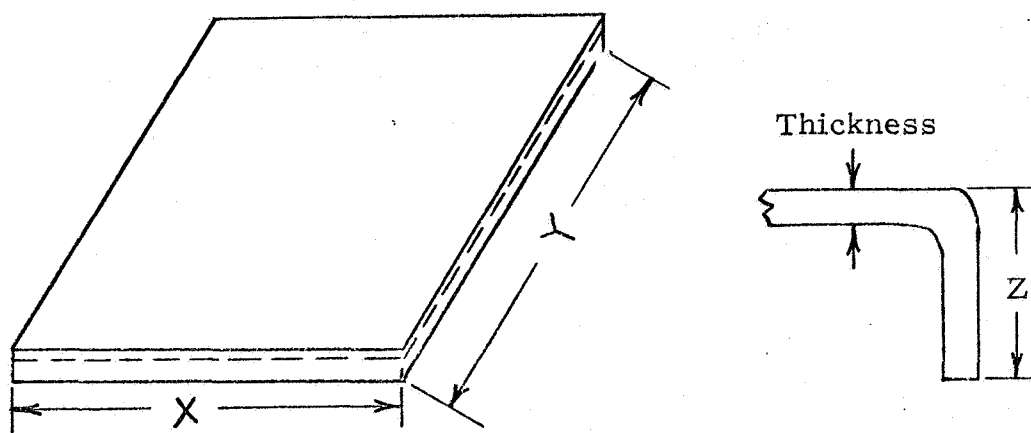


and a similar arrangement with two attached thermoelements rather than one. In the latter case the thermoelement corners will be specified by  $R$  and  $S$  as well as  $U$  and  $V$ . The meaning of  $R$  is the same as  $U$ , that of  $S$  the same as  $V$ , except we now describe two thermoelements:





Only one piece of information will be read with a C=1 type input, the absorber (or hot side fin) thickness. Any dimensions may be used as long as a consistent set of data are given for all of the input. Type C=2 information reads the collector dimensions X, Y and Z (in that order) as shown on the following sketch:



Note that Z is not the thickness, but the total outside dimension of the plate edge that has been bent over.

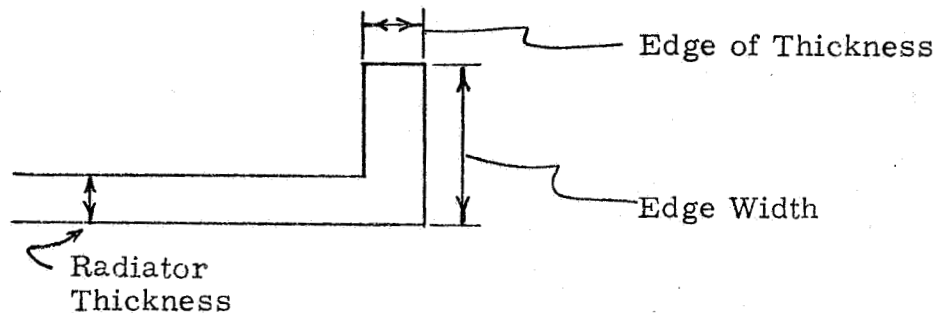
Type C=3 and 4 information refers to the thermoelement dimensions as previously discussed. This information will be read in the order presented under the headings in Table III-1. If for some reason both 3 and 4 type information were read, an error would result and this case would be skipped. Further, if type 3 information had been read for a previous problem, and type 4 information read for a present problem, then sufficient additional information would be required that the problem were fully described or the case would be refused. This will be discussed further at the end of this subsection.

Type C=5 information requires the thermoelement length for non-segmented thermoelements. If C=3 information has been (or is to be) read, one length is required; if C=4, two lengths, the first for the UV T/E, the second the RS T/E. The end of this information must be signaled by a "Z" since the code "doesn't know" how many numbers to read.

Type C=6 information is used if segmented thermoelements are to be used. The first number read is the number of segments in the UV system (10 or less are allowed) and this is followed by the segment lengths. If information for an RS system is required, this is read next in the same manner. The end of this information must be signaled by a "Z". The segments should be read beginning at the hot end.

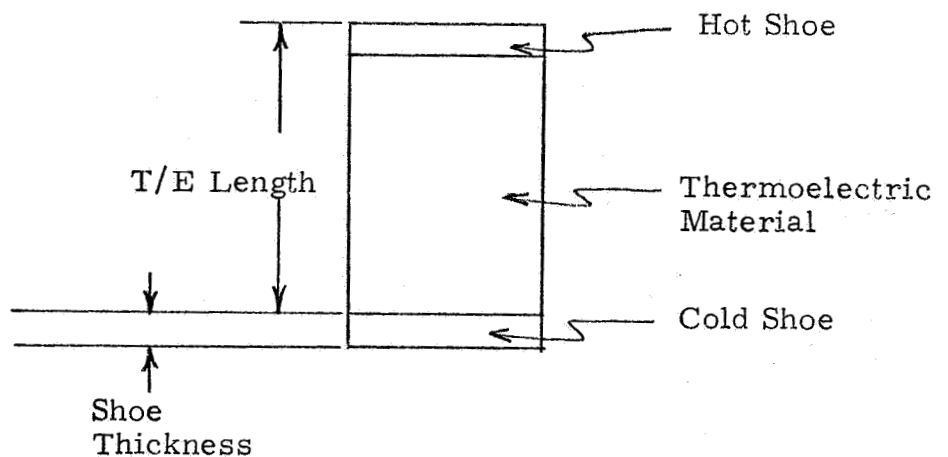
Type C=7 are thermoelement material designations read in the same order as for type 6. These material designations are numbers referring to properties read using \*3 input and, therefore, specify the thermoelement material. A "Z" must terminate this information.

Type C=8 input is three numbers, the radiator thickness, the edge width, and the edge thickness.



These are read in the presented order.

Each thermoelement or thermoelement segment is considered to have the following form:



C=9 is used to read in the hot shoe thickness followed by the hot shoe material designation. If multiple thermoelements or segments are used, the readings are repeated in the order read in C=7. All thicknesses are read first, and then all material numbers. If R-S as well as U-V thermoelements are used, the thicknesses for the U-V T/E's are read, then for the R-S T/E's. The material designations are then read in the same order. C=10 does the same thing for the cold shoes. Note that zero thickness shoes are allowed, which is the same as no shoe. A material still must be specified, however. The end of C=9 and of C=10 information must be specified by using a "Z".

Mesh information is given through the use of C=11. The number of mesh points are read for each of the dimensions in the following order:

- (1) Each collector side starting at (0, 0) and proceeding in a counter-clockwise direction.
- (2) Each collector side starting at (Uo, Vo) and proceeding as before.
- (3) Repeat Item 2 (above) for R S if used, otherwise this is ignored.
- (4) The number of mesh points in the thermoelements reading first the UV T/E and then the R S T/E. If segmented T/E's are used, begin at the hot end and work toward the cold end reading the number of mesh points per segment for the UV T/E. Repeat for the R/S T/E if one is used.

These data are used as guides in setting up the mesh for calculation purposes. They will seldom be used exactly as specified. The total number of mesh points must be less than 1000 (the collector and radiator have the same number of points). The end of this information must be indicated by a "Z".

The C=12 information pertains to whether the edges are exposed or "see" other flat plate packages and hence are insulated. Ones should be read if the edges are exposed, zeroes if insulated. The edge should be traced starting at point (0, 0) and going in a counterclockwise direction.

A typical set of information under \*2 could be read using the following cards\*:

```
*2 1 .002 2 .5394 .5394 .028 3 .2452 .2942 .2942 5 .0984Z7 1 Z 8 .0025
.007 .028 .005 9 .0055 2 Z 10 .0055 2Z 11 4 4 4 4 4 4 4 6 Z
12 0 0 0 0
```

Although the entire \*2 information is shown in one place, if desired it could have been scattered throughout the input. Further, the values of C can be taken in any desired order, rather than the order taken here. Note that blanks need not be left between Z and a number, but this can be done. In general, the only restriction on the order or quantity of data is that each C group must be complete. If partial input is required and the remainder of the information was read for a previous problem, only the C groups that are to be changed need be read. However, if the change in the C group changes other requirements, then these must be changed as well. An example of this would be a change from one T/E to two. Then new dimensional, material, mesh information, etc. would all be required.

#### 5. \*3, Thermoelectric Material Properties

Allowance is made for up to 6 (inclusive) tables of thermoelectric properties. Control numbers, C, are used to indicate the table number (1, 2, 3, ..., 6) and are followed by absolute temperature, seebeck coefficient, thermal

---

\* The sample problem is based on information given in MELPAR and the dimensions are given in inches. Any consistent set of dimensions may be used.

conductivity, electrical resistivity, heat capacity, density, and relative emissivity, in the presented order. To read data, first read a control number, then a temperature, then the properties at that temperature, then another temperature, more properties, etc. The end of a table must be indicated by a "Z". This can be followed by another control number if desired, in which case another table can be read. The maximum total number of temperatures that can be read is 200; the minimum number is 3. Linear interpolation is used in the tables.

The temperatures can be read in any desired order, and it is not necessary to number the tables consecutively. If data for one table are to be changed, only that one table need be reread, the others will be unaffected. The same table number may not be read twice during the same reading, but a table number used for a previous problem can be reused for a present problem. In this case the previous table is eliminated.

#### 6. \*4, Other Properties

As in the \*3 tables, control numbers are utilized to indicate the table (or material) number. The following designations are used:

C	Meaning
7 - 12	Shoe Information
13 - 14	Absorber Information
15 - 16	Radiator (Backup Plate) Information

Information read by C=7 - 12 is a temperature (absolute) and values for thermal conductivity, heat capacity, density, electrical resistivity, and contact resistance (in that order) corresponding to that temperature. "Z's" must be used to indicate the end of each table.

The C=13 - 16 information is identical to C=7 - 12, except a value for relative absorptivity and two values for relative emissivity are read following the electrical value and no contact resistance is read. The relative absorptivity and the first relative emissivity will be applied to the outside (exposed) surface, the second relative emissivity to the internal surface.

Other comments made in the \*3 input data section are applicable in \*4 data input.

#### 7. \*5, Heating Information

This information pertains to the environment as "seen" by the thermoelectric power supply. The following designations for control numbers are used:

<u>C</u>	<u>Meaning</u>
1	Solar constant (heat flux)
2	Planet Reflectivity Constant (albedo)
3	Stefan Boltzmann Constant
4	Time

<u>C</u>	<u>Meaning</u>
5	Mean Planet Temperature as "seen" by the plate
6	Planet view factor for radiant heat transfer from the planet to the absorber
7	Planet view factor for radiant heat transfer from the planet to the radiator
8	Planet view factor for the planet albedo to the absorber
9	Planet view factor for the planet albedo to the radiator
10	Solar angle (Radians)
11	Function Calculation

The first three C's are single-valued constants, and as such need be read only once. The end of these numbers need not be signaled by using a "Z". The other C's are used to designate groups, and the end of each group of numbers must be signaled by a "Z".

If only one value of time (C=4 information) is read, and that time is zero, then a steady state problem will be assumed. If nonzero values are read, then at least three values must be read, and they must be read in increasing order. The lowest value may be zero. Up to 100 (inclusive) values may be read.

The C=5 - 10 information corresponds to the values of time (C=4). If a constant is desired, only one value need be read. Otherwise, the same number of values as used for time (C=4) must be read. The meaning of these numbers can be obtained by considering the following brief discussion.

The solar heat flux absorbed by a flat surface is given by:

$$\begin{aligned}
 q &= S \cos(\delta) \quad 0 \leq \delta < \frac{\pi}{2} \\
 &= 0 \quad \frac{\pi}{2} \leq \delta \leq \pi
 \end{aligned}
 \tag{III-1}$$

where:

$q$  = heat flux  
 $S$  = solar constant  
 $\delta$  = angle measure from a normal to the surface

In this analysis,  $0 \leq \delta \leq \frac{\pi}{2}$  will be taken to mean that solar heat is impinging on the collector;  $\frac{\pi}{2} < \delta = \pi$  will mean it is impinging on the radiator, and  $\delta = \frac{\pi}{2}$  means that no solar heat is being absorbed. The heat flux absorbed by the thermoelectric system from a planet is given by:

$$q = \sigma F_1 T^4 \epsilon \quad (\text{III-2})$$

where:

$\sigma$  = Stefan Boltzmann Constant

$F_1$  = Radiation view factor

$T$  = average planet temperature as seen by the thermoelectric system

The albedo heat flux is given by:

$$q = \text{SRF}_2 \alpha \quad (\text{III-3})$$

where:

$R$  = Reflectivity Constant (albedo)

Much of this information can be calculated if orbital parameters are defined, and provision is made to do this if desired by using the C=11 input. In this case, subroutine HEAT (TIME, T, FIC, FIR, F2C, F2R, DELTA) will be called.\*

#### 8. \*6, Other Data

Information read using the \*6 designation has the following meanings:

<u>C</u>	<u>Meaning</u>
1	Load resistance
2	Symmetry designation
3	Transient problem investigation time
4	Minimum allowable time increment
5	Maximum allowable number of iterations
6	Maximum allowable temperature change per iteration
7	Steady state convergence factor
8	Frequency of long print-out in transient calculations

---

\* The arguments are the C=4 - 10 variables, and each is a 100 number array. These data should be placed in the array in the same manner as though they were read, except the use of "Z's" is not permitted. Zeros must be placed in all portions of the array not specifically used. The same number of values as times must be used. This subroutine is not presently available, and a dummy subroutine of the same name has been included in the deck.

<u>C</u>	<u>Meaning</u>
9	Frequency of short print-out in transient calculations
10	Mesh specification: If zero or negative, the mesh calculations are to be performed; if equal to one, the mesh calculations are to be skipped.
11	Node map generation designation: If zero, no map is generated in the print-out; if one, a map is generated.
12	Transient-steady state designation: If one, a steady state problem; if zero, a transient problem
13	Convergence criterion for steady state calculations
14	Frequency of long print-out in terms of number of iterations
15	Frequency of short print-out in terms of number of iterations
16	Estimated number of iterations required for convergence in steady state calculations
17	Collector material number
18	Radiator material number
19	Not assigned
20	System initial temperature

Each of these items is discussed in more detail, where required, in the following paragraphs.

The load resistance, specified by a control number equal to one, is that resistance "seen" by the portion of the solar flat plate being simulated by the computer program.

A control number of two refers to a symmetry designation. Proper use of this term will result in drastic decreases in computation time and a saving of close to a factor of eight can be obtained in many cases. The symmetry designation number is the number of nodes in a repeating portion of the solar flat plate. To see exactly what is meant by this statement, consider Figure III-1. This figure illustrates a completely symmetrical solar flat plate with the thermoelement located in the center and with six nodes specified on each of the four sides. Each of these "rays" is shown in the figure. Specification of a symmetry number of 12 with this configuration would cause the computer program to calculate temperatures beginning at line AO and continuing counterclockwise to line DO. No calculations would be performed for temperatures from line DO counterclockwise to line AO since they would be a mirror image of the previously calculated temperature. Any time a detailed print-out was required, the temperatures from the calculated portion would be reproduced in the uncalculated portion and then all nodal points would be represented in the output.

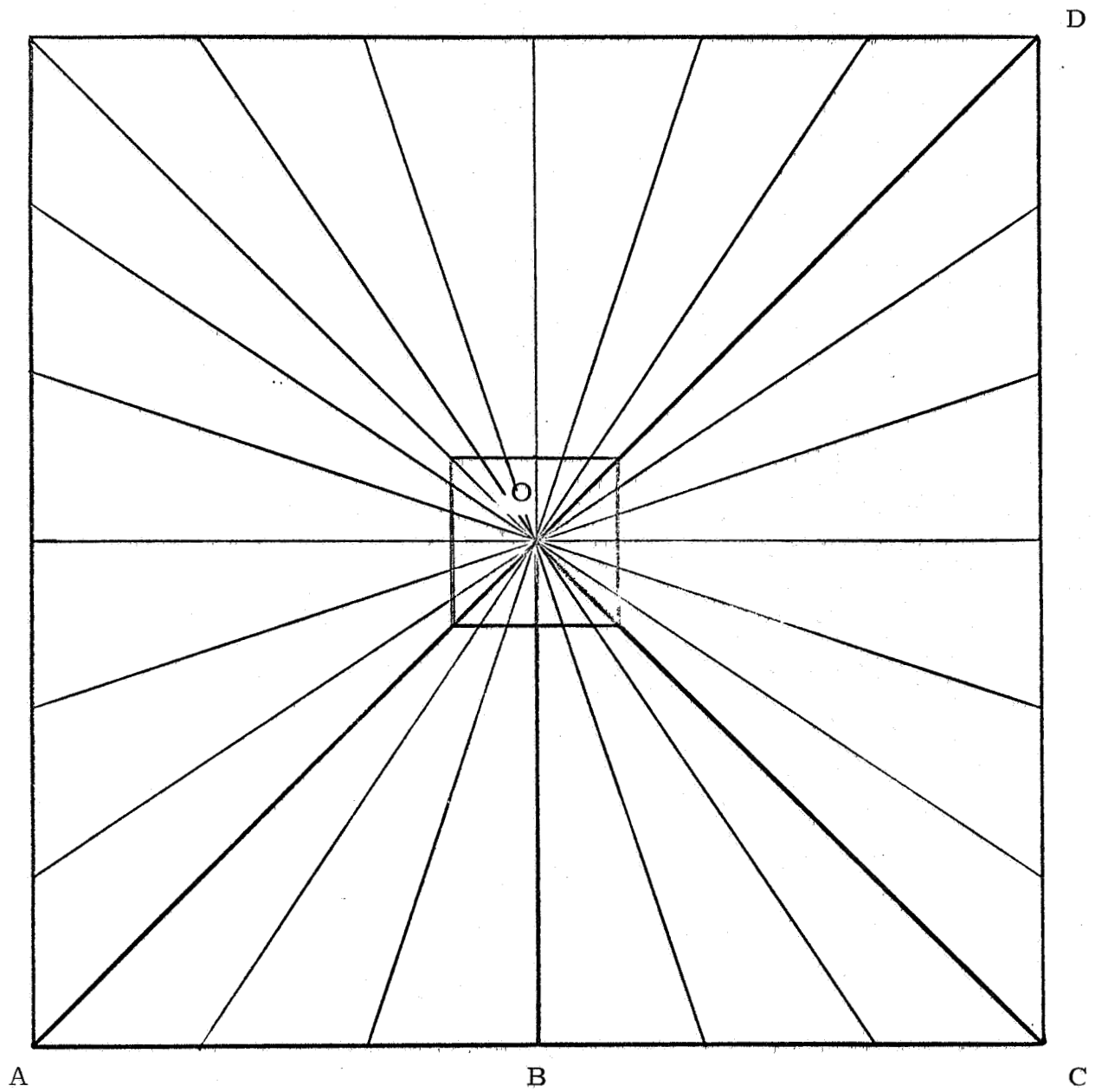


Figure III-1. Illustration of Symmetry Designation Number



Specification of a symmetry number of six would cause calculations to be performed beginning at line AO and progressing in a counterclockwise direction to line CO. No other calculations would be performed, and a detailed print-out would be obtained in a manner similar to that discussed for a symmetry number of 12. Specification of a symmetry number of three would cause calculations to begin at line AO and progress in a counterclockwise direction to line BO. This latter specification would result in a factor of eight less calculations than if no symmetry number were defined or if a symmetry number of 24 were read. The advantage of taking into account the symmetry of the solar flat plate immediately becomes apparent.

Note that a requirement for a symmetrical designation is that the lines of symmetry correspond to nodal separation lines as set up by the computer program. For example, if the number of nodes between points A and C were specified as five rather than six, and similar designations were made for the other three sides, a symmetry specification of five or ten could be made. In this case, a factor of four or two, respectively, decrease in the number of calculations would be obtained. A factor of eight decrease in the number of calculations would be impossible with this designation because integer numbers must be specified, and a specification of 2.5 would be required to take full advantage of the symmetry of the figure, an impossible arrangement.

Care should be taken in the selection of the symmetry number since if a designation is given which is inconsistent with the figure generated by the computer program, the symmetry specification will be neglected and calculations will be performed for all nodes in the solar flat plate. This obviously could cause a considerable increase in running time over that expected with a correct symmetry designation.

A control number of three specifies the maximum time which is to be simulated for investigation of a transient calculation. The problem is terminated when this simulated time has been reached by the computer program. This particular number is not used if a steady state type of calculation has been specified.

A control number of four can be used to represent the minimum time step that is allowable for a transient calculation. The problem will be rejected if a lower time step is required for calculation purposes. A zero is allowable as an input for this number if desired, and a zero will be used if no number is read.

A control number of five defines the maximum number of iterations which will be permitted during a run. If this number is exceeded, the problem will be terminated.

The maximum temperature change per iteration that will be allowed is specified with a control number of six. The calculations will be performed for both the transient and the steady state utilizing this value. In the transient calculation, the time increment will be selected so that the maximum change in temperature is approximately the input value. In the steady state, this value will be the allowable change per iteration, and an extrapolation factor will be selected so that this value is attained (see also the input with a control number of 16).

Input with a control number of seven is the convergence factor utilized for a steady state calculation. A value between 0.5 and 0.8 is recommended (see also the input under a control number of 16).

A control number of eight is used to specify the time separation between detailed temperature print-outs. The temperatures which are outputted will be presented as a function of node number, and ordinarily a node map will be required to interpret the results. This number is ignored for steady state calculations.

The time separation between concise print-outs is specified with a control number of nine. A concise print-out consists of the time, time increment, iteration number, electrical power output, electrical current, the error in the heat balance, the overall conversion efficiency based upon the heat received by the absorber and the electrical output, the hot junction temperature, the cold junction temperature, the average radiator temperature, and the average absorber temperature. A concise print-out is obtained automatically whenever a detailed print-out is output. A long and short print-out is given at the termination of each problem. Specifications with a control number of nine are ignored if steady state calculations are being performed.

A control number of ten is a flag which defines whether the mesh is to be set up or not. The mesh must be calculated if any input has been used in a problem which can change the volume of a node, the separation of one node from another, or the node connection arrangements. If a mesh calculation is specified by reading a zero or negative number with the control number of ten, the temperature specified as an initial condition (\*6 20) is used to set the temperature of all of the nodes. If the mesh is not to be calculated, which occurs with a specification of one, then the node temperatures which were previously used represent the starting point of the calculations.

Changes in the following input designations do not require the generation of a new mesh: \*1, \*2 5, \*2 7, \*2 9, \*2 10, \*3, \*4, \*5, \*6 1, \*6 2, \*6 3, \*6 4, \*6 5, \*6 6, \*6 7, \*6 8, \*6 9, \*6 12, \*6 13, \*6 14, \*6 15, \*6 16, \*6 17, \*6 18, and \*7. All other input specifications require that a new mesh be generated.

A control number of 11 specifies whether a print-out of the node map is desired or not. If a value of zero is read, no map will be detailed. If a value of one is input, then a complete map will be generated. This map will consist of a scale sketch generated by the computer using the output printer which illustrates the outline of the absorber, and each of the node centers will be located within this sketch to within roughly one part in one hundred. Coordinate numbers will be generated along the sides of this map so that each of the nodes may be referred to by two coordinate numbers. A table will be generated which lists the node numbers corresponding to each of the node centers in terms of the reference numbers generated on the map. The node representations for the radiator will not be generated in a map since the map would be identical; however, the complete representation of these in tabular form will be generated.

The data read using a control number of 13 will be utilized by the computer program to determine when convergence has been achieved in steady state calculations. This number should be the temperature change per iteration that it is desired to attain during steady state calculations. When a maximum temperature change during an iteration has resulted which is less than this number, the problem will be terminated. (Note that in setting up the print specification requirements, final answers are always given upon problem termination.) The number input using a control number of 13 is ignored if a transient calculation is specified.

A control number of 14 causes detailed print-out of results to occur for both the steady state and transient calculations. The value read is the number of iterations between each detailed print-out. If detailed print-outs as a function of iterations are not desired, a large number should be used.

The information obtained by the program using a control number of 15 is identical to that previously presented for 14, except in this case the concise print-out is to be described.

A control number of 16 causes an estimate of the number of steps required for convergence in the steady state to be read. This estimate is used by the computer program during the calculation process. For all calculations performed prior to reaching the number of iteration steps given in this estimate, a change in temperature per iteration as specified using a control number of six is forced. After this estimate has been reached, the temperature change is multiplied by the factor input using a control number of seven in computing temperature changes per iteration. This means that the steady state calculation procedure cannot terminate until after the number of iterations input under the control number 16 has been exceeded. However, the convergence rate will be forced prior to this number being reached. The effectiveness of this "forcing" decreases with the number of iterations performed.

A control number of 17 refers to the material specified for the collector, and the one under a control number of 18 refers to the radiator material. These values are the table numbers which were input using the \*4 input.

The initial temperature of the entire generator should be input using a control number of 20. Absolute values of temperature must be used, but other than this restriction, any units desired can be selected, provided consistency is used in all input.

#### 9. \*7, \*8, \*9 Information

These control cards signal the end of input for a particular problem. If \*7 is used, calculations will be performed for the data read into the machine. When the required computations have been completed, control will be transferred to read further input information for a new problem. The input for the new problem need contain only those items that have been changed from the previous calculation. All of the previously read information is preserved unless replaced by new input.

The use of a \*8 control card will cause behavior similar to the \*7 card, except the input for the new problem must be complete. None of the previously read information will be available for a new problem.

The \*9 card signals that no input data follows. The required calculations will be performed, and then control will be transferred back to the computer systems tape and the run will be terminated.

## B. PROGRAM DESCRIPTION

### 1. Introduction

This computer program is written for an IBM 7094 which has approximately 32,000 words of storage. Roughly 20,000 words are provided at all times within the core for storage of variables so that the lengthy input-output of data required for calculations is not necessary except to obtain initial information or to provide answers. However, the computer program requires much more memory than the remaining 12,000 words. For this reason, it has been broken into a number of links. The main control of the computation process and those subroutines which require extensive use are always left in the core of the machine. The other subroutines are brought into the core as necessary. The arrangement of subroutines is such that a subroutine need be brought into the core only once for each problem that is to be investigated.

The names of the subroutines and the link to which they are assigned are shown in Table III-2. The overlay construction is shown in Figure III-2.

### 2. Main Program

The main program serves basically as a calling sequence and controller for all subroutines. It first causes all storage locations used by this program to be cleared. It then causes the input data routines to be selected, selects the routines which process the data and store it in the proper locations, sets up the routines which automatically generate the mesh, and then calls in the basic calculation routine. When calculations have been completed on one problem, it either selects input for a new problem or causes the machine to be returned to the system control.

### 3. Subroutine LENGTH

This is a very short subroutine which merely computes the straight line distance between two points when the x and y coordinates are given.

### 4. Subroutine INTER

This subroutine is used to compute the (x,y) coordinates of the point of intersection of two straight lines. The straight lines are described by giving the (x,y) coordinates of two points on each of the lines.

TABLE III-2. COMPUTER PROGRAM OVERLAY  
CONSTRUCTION

<u>Subroutine Name</u>	<u>Link Number</u>
MAIN	0
POINT	1
LENGTH	1
INTER	1
AREA	1
CHECK	2
PRO-3	3
PRO-5	4
HEAT	4
PRO-4	5
PRO-6	6
PRO-2	6
NUMBER	7
READIT	7
UNPAK	7
PACK	7
SQUASH	7
MESH	8
MESHA	9
MESHB	10
MESHC	10
FPMAP	11
PROP	12
CALCS	13
CALCT	13

MAIN (0)

POINT (1)	MESH A (8)									
LENGTH (1)										
INTER (1)										
AREA (1)										
FLOG (2)										
FXPF (2)	MESH (7)									
FXP3 (2)										
CHECK (2)										
FPMAP (10)										
	MESH E (9) MESH C (9)									
	PROP (11)									

CALCS (12) CALCT (13)

Figure III-2. Overlay Construction

## 5. Subroutine POINT

The basic calculations are set up using a cylindrical coordinate system. However, it is necessary to relate this to the (x,y) coordinate system which is utilized in providing input to the program. This subroutine takes the radius and angular information being utilized in the program and converts to Cartesian coordinates.

## 6. Subroutine AREA

Each of the nodes used for calculation purposes is considered as being located at the centroid of its assigned area. This latter value is computed by considering smaller areas of triangular cross section. Subroutine AREA computes the centroid locations in Cartesian coordinates and the surface area from the Cartesian coordinates which describe the three corners of the triangle.

## 7. Subroutine HEAT

As explained in the input description section, subroutine HEAT is not presently available. However, a dummy subroutine is presently provided. If it is desired in the future to provide automatic calculation of the surroundings as "seen" by the solar flat plate power supply, a suitable subroutine may be added at this point. The actual location of this subroutine in the link structure is relatively unimportant since it will only be called once for each problem. The data it provides is utilized in exactly the same manner as though input data were provided. The calculation process is based upon a linear interpolation of this information.

## 8. Subroutine NUMBER

This subroutine generates integer or decimal numbers from the components as read by the machine using subroutine READIT. An integer number furnished to this subroutine will consist of only two portions, a sign and a significant figure. Subroutine NUMBER multiplies these two portions, and then stores the result in storage allocations corresponding to the type 2, 3, 4, 5, or 6 input.

A decimal or floating point number consists of several portions. The exponent consists of a sign and a significant figure portion, and the significant figure portions consist of a sign, the integer (that portion of the number to the left of the decimal point), and the significant figures which occur to the right of the decimal point. Subroutine NUMBER compiles these various pieces to form the completed number and then stores them in the proper location as previously described for an integer number.

## 9. Subroutine READIT

All input to this program is by A-mode conversion. This enables the program to read any recognized FORTRAN character. Subroutine READIT causes these data to be read and then processes the information.

The initial character on an input card is first checked to see if it has an asterisk, and if an asterisk is found, the character in column 2 is shifted to the right-most position in its storage location so that the type of

control card may be determined. If an illegal character is found in column 2, the entire card is printed out with a comment to the effect that an illegal control card was found, and the program is removed from the computer. If the first card of data read is found not to be a control card, it is assumed that the control card is missing and the problem will be refused. If a legitimate control card is found, it is analyzed to see if this is information pertaining to input data, or if the information defines the end of a problem. If an end of problem is found, control is transferred back to the MAIN PROGRAM for the beginning of the processing of the information which has been read and stored in the machine. If the control card indicates that data are to be processed, then the required action is taken.

If the card read is a comment card (\*1), then the first two columns of the card are considered to be blank, and the remainder of the card is printed as output. If the card is determined to contain numerical information, then that information is processed. Each of the allowed characters - BLANK, -, +, ., E, Z, 0, or 1 through 9 - is checked and appropriate action is taken.

Construction of a number first begins with a sign indication, or if no sign indication is found, with an integer portion. If a BLANK is encountered during the compilation of the integer, the end of the number is assumed, and subroutine NUMBER is called. Any other allowed character will either be a single digit integer, a decimal point, or exponent type information. An integer is treated by simply shifting the previous integer to the left and adding the new one. A decimal point merely sets a flag so that further integers are stored in a different location and are treated as the significant figure portion to the right of the decimal point. Any of several characters may indicate an exponent, and if this is found, a flag is set. Any further information obtained after the flag is set, but before a BLANK is found, is incorporated into the exponential portion of the number prior to calling subroutine NUMBER which processes the information and stores it in the right location.

If at any time an illegal character or an unallowed number is found, or a return is received from subroutine NUMBER with an error designation, then a check is made to determine if any further problems remain. If no problems remain, the program is automatically removed from the computer. If further problems remain, then a flag is set and subroutine READIT transfers control back to the MAIN PROGRAM. The MAIN PROGRAM is caused to clear the portions of the machine allotted for data storage, set a flag, and reenter subroutine READIT. This institutes a search process of the remaining data until a complete new problem is found. At this point, analysis will proceed.

#### 10. Subroutine UNPAK

One of the basic ground rules under which this computer program was written was that the input-output of information from the computer core would be limited to input data and output answers. The initial processing of input information required the storage of a large number of one or two digit integer numbers. This type of information requires only a few bits in a word, and the remainder of the word is not used. Consequently, when a shortage of core storage was encountered, the integer numbers were compressed so that eight were stored in a single word location in the IBM 7094 memory. Subroutine PACK causes this compression of the integer numbers to take place. Subroutine UNPAK causes the inverse to occur.



These subroutines require a small MAP routine (the equivalent of the FORTRAN IV machine language). The two entry points of this machine language portion are titled SQUASH and SPREAD. Their functions are self-explanatory.

#### 11. Subroutine CHECK

There is no way of determining the difference between a minus zero and a plus zero using FORTRAN IV. Consequently, a small MAP subroutine was written which provides a solution to the problem.

#### 12. Subroutine PRO-2

This subroutine is called after all of the information has been processed by READIT. This particular routine processes all of the information that has been input using the \*2 control. The subroutine causes all of the data to be processed and stored in the necessary memory allocations which have been assigned for each of the variables. All of the input data are cross-checked to be certain that sufficient data have been provided and that contradictory or inconsistent data have been excluded. The first of several checks on the number of mesh points specified is also made in this subroutine. If an error is found, a flag is set and control is returned to the MAIN PROGRAM for processing of the remaining information to determine if another problem remains which can be attempted.

#### 13. Subroutine PRO-3

The function of this subroutine is quite similar to PRO-2 except that the type of information to be stored is different. This subroutine first causes all of the thermoelectric information that has been previously stored in the computer memory to be moved to a different location for temporary storage. The new information is then processed and all of the property data are stored in the designated locations. It is possible at this point that the data could have been input to the computer in a nonascending temperature order, and consequently the data are immediately reprocessed so that all information is in the order of increasing temperature. After this has been done, checks are made for specifications of duplicate table numbers on the input data. If none are found, then the information which was saved from previous problems is merged with the new information so that a complete set of data are available. If, during this merge process, any duplicate tables are encountered, the old tables are eliminated.

Once all of the tabular information has been compiled and sorted, a check is made of the information which will be required during the calculation process. If it is found that information is going to be requested which is unavailable, a comment will be made to this effect and an error return procedure will be instigated. Otherwise, notations are set up so that the information can be used during the calculation process.

#### 14. Subroutine PRO-4

This calculation process is virtually identical to that previously discussed in subroutine PRO-3. The only differences are slight perturbations in the material which is to be stored and in the references which are set up so that the data can be later utilized.

15. Subroutine PRO-5

This subroutine is relatively straightforward and short, and serves to merely store the input data in the proper locations. It also contains the calling sequence for subroutine HEAT.

16. Subroutine PRO-6

This subroutine processes the remaining data which are necessary for the utilization of the computer program. In each case, each piece of data information is preceded by a control number which is then utilized in subroutine PRO-6 for storage information purposes.

17. Subroutine MESH

This subroutine contains the beginning of the automatic mesh generation procedure. The first items which are computed are the angles interconnecting the center of the thermoelement with its corners, and the center of the thermoelement with the corners of the attached plate. The number of points specified for the edges of the plate and thermoelement are next investigated to determine the more stringent requirements. When this has been obtained, the previously calculated angles are divided up into small segments. The segments are next smoothed by an iterative process so that the accuracy of the calculations will be improved.

Once these calculations have been done, it is possible to compute the radii corresponding to each of the angular segments at the intersection of the thermoelement and the plate, and at the plate edge. These are stored at the proper locations and then the angular segments are integrated to provide the angle between a line connecting the center of the thermoelement with the Cartesian coordinate point (0,0), and the angular segment. When this calculation has been completed, and an average angular segment size has been determined for calculation purposes, control is transferred back to the MAIN PROGRAM which then immediately calls subroutine MESHA.

18. Subroutine MESHA

This subroutine continues the calculations which were begun in subroutine MESH. It first causes the stored locations assigned for the areas, the centroids, and the inside and back length to be cleared. It then selects the first node and assigns a trapezoid to the node. The Cartesian coordinates of each of the trapezoid corners are assigned and each of the corners is investigated to see whether it lies within the plate, within the portion of the plate assigned to the thermoelement, or outside of the plate. An index is set up as each of these determinations is made. This index is next used with a computed "GO TO" statement so that the proper breakdown of the trapezoid into triangles for calculation of area and centroids is obtained. (See Section II-C.)

During this calculation process, several nodes will be obtained which are totally within the area assigned to the thermoelement, and others will be obtained which are totally outside of the plate. Suitable indices are set up so that these nodes are not referred to or incorporated into the calculation process. Further, several logical calculations are made to assure that the memory allocation assigned for the storage of node information is not exceeded.

When the calculations in this subroutine have been satisfactorily completed, transfer is made back to the MAIN PROGRAM which then calls in subroutine MESHB.

19. Subroutine MESHB

A number of very small nodes can be readily obtained during the previously described calculation process. These nodes, which can result from portions of the trapezoid either being within the thermoelement or outside of the plate, will contribute nothing to the calculation, and indeed could considerably slow the calculations. Consequently, they are combined in this subroutine with adjoining nodes in such a manner that accuracy is for practical purposes not compromised. This joining process is made by comparing the size of a node with its nearest neighbor. If a factor of four difference in the size is obtained, and certain other criteria are also satisfied, then the node is combined with its neighbor. This combination process consists of the addition of the areas, and the weighted relocation of the node centroid in the Cartesian coordinate system. Information must also be incorporated for the logical calculation of the cross sectional area common to one node and its neighbors.

When this information has been compiled, transfer is made back to the MAIN PROGRAM which then calls in subroutine MESHC.

20. Subroutine MESHC

This subroutine computes the area-to-length ratios necessary for the future calculation of conductances between the various nodes. It also sets up the initial mesh temperatures and finishes setting up the boundary information which is required.

21. Subroutine FPMAP

This subroutine generates a scaled map of the mesh arrangement which has been generated by the computer program. The map is caused to be printed on output paper to a precision of approximately one part in a hundred. Tabular information which will later enable the user to relate temperatures to specific positions is also presented.

22. Subroutine PROP

This subroutine provides linear interpolation of all input data which is either a function of temperature or a function of time. It is utilized during the calculation process so that the physical properties of the absorber, radiator, and thermoelement are treated as temperature dependent variables. Transient characteristics as generated by subroutine HEAT or as described in the input are also processed in this subroutine.

23. Subroutine CALCS

This subroutine constitutes the main steady state calculation process of the program and is the portion of the program where most of the steady state calculation time is expended. It is broken into three portions: mesh property evaluation, steady state calculation, and print-out of results.

The properties of the absorber, radiator, and thermoelements are determined by calling subroutine PROP. These properties are utilized to compute conductances between each node and its interconnected nodes, the heat capacity of the node, and the surroundings information (such as solar input, albedo, planet input, and heat rejection from the exposed surfaces). When these data have been computed, a number of other characteristics which will be necessary are computed and stored (such as temperature to the fourth power).

The calculation is begun by first considering the hot junction node. The conductance for this node to all of the surrounding nodes is obtained by integrating with respect to angle through  $360^\circ$  (or less if a symmetry specification has been incorporated). The conductance to the thermoelement is next computed, and the various effects such as Peltier cooling and contact resistance heating are included. The calculation of a new temperature immediately follows.

The calculation of the cold end node temperature is virtually identical. Once this has been computed, the thermoelement calculations are made on a node-by-node basis, and then the absorber calculations are made. The same calculations are utilized for the absorber as for the radiator, and the radiator is computed by merely resetting some indices and using the same equations.

Once new temperatures have been obtained, they are multiplied by the extrapolation factor and checks are made to determine whether further calculations must be performed.

The generation of print-out information is relatively straightforward and will not be discussed in detail.

#### 24. Subroutine CALCT

This subroutine performs the transient calculations for the program, and most of the calculation time for this type of problem will be spent here. The routine is very similar to subroutine CALCS, discussed in the previous subsection, and the only basic difference is the addition of heat capacity and time terms to the steady state calculation. The calculation process is virtually identical and need not be discussed further.

#### IV. CALCULATIONS PERFORMED USING THE PROGRAM

##### A. Sample Problem

The input data for a sample program and the resulting computer output are presented in the following pages. Figure IV-1 shows sample input data for the calculation of the characteristics of a repeating section of the present solar flat plate configuration. This specific problem did not contain a request for a map of the array. The array which would have been generated is shown in Figure IV-2 which was obtained from a previous computer run. Figure IV-3 gives map coordinates, x - y coordinates, and the node number. Also shown are the summary results when convergence was obtained. For this steady state problem, the time and time increment are meaningless. The iteration number indicates that it took 2456 iterations for the specified convergence to be obtained. The power output from the repeating section was 0.00566 watts, the current was 0.373 amps, and the fractional error in the answer obtained by comparing with an overall heat balance was 0.00259. The overall efficiency based upon the heat absorbed by the panel is 2.63 percent. The hot junction temperature computed was 922°R, and the cold junction temperature 636°R. These compare to an average absorber temperature of 937°R and an average radiator temperature of 624°R. These average temperatures were obtained by a weighting of temperature to the fourth power so that the true heat rejection characteristics of the surfaces are obtained. The computer printout which gives the detailed temperature distribution is also shown in Figure IV-3. The location of each of the presented temperatures may be obtained by referring to the previously generated node numbers.

Similar results are obtained if the node specification is changed to three or four nodes on a side. The node map for a three node specification is shown in Figure IV-4 and Table IV-1, that for a four node specification in Figure IV-5 and Table IV-2.

##### B. Comparison with Previous Calculations

Fuschillo\* has presented an analysis of the Solar Flat Plate Generator using manual calculation techniques. Since this is the most complete analysis which has been published, it will be used as a basis for comparison.

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\*Fuschillo, N., R. Gibson, F.K. Eggleston and J. Epstein, "Flat Plate Solar Thermoelectric Generator for Near-Earth Orbits," Advanced Energy Conversion, Vol. 6, pp. 103-125, 1966.

SS BASE CASE

IV-2

\*2 1.002 2.5394 .5394 .028 3.2452 .2452 .2942 .2942 5.098427 1 2  
8.0025 .028 .005 9.0055 7 7 10 .0055 72 11 2 2 2 2 2 2 2 6

7 12 0 0 0 0

\*3 1 200 1.06-4 1.56E-2 .996E-3 35.5 .277 .1

500. 1.06E-4 1.56E-2 .996E-3 35.5 .277 .1

1000 1.06-4 1.56E-2 .996E-3 35.5 .277 .1 7

\*4 7 200 100 20 20 .1 0

500 100 20 20 .1 0

1000 100 20 20 .1 0 7

13 200 3.22 229. .1 .1 .82 .08 .03

500 3.22 229. .1 .1 .82 .08 .03

1000 3.22 229. .1 .1 .82 .08 .037

15 200 3.22 229. .1 .1 .14 .95 .03

500 3.22 229. .1 .1 .14 .95 .03

1000 3.22 229. .1 .1 .14 .95 .03 7

\*5 1.903 2.0 3.00349E-9 4 0 7 5 0 7 6 .094 7 7 .094 7 10 0 7 8

.155 7 9 .155 7

\*6 1 4.07-2 2 1 3 150 4 1-5 5 1+5 6 1.7 .7 8 5.9 .5 10 0 11 0

12 1 13 1-3 14 500 15 50 16 2000 17 13 18 15 19 0 20 700

\*9

Figure IV-1. Sample Input

[illegible]

Figure IV-2. Node Map

Figure IV-3. Computed Results

MAP		X-Y		NODE	
COORDINATES	COORDINATES	NUMBER	NUMBER		
28 4	0.1325E-00 0.5073E-00	18			
85 6	0.4142E-00 0.4906E-00	16			
10 15	0.4876E-01 0.4142E-00	20			
43 16	0.2103E-00 0.4131E-00	17			0.
68 16	0.3291E-00 0.4131E-00	15			1.000000
104 16	0.5073E-00 0.4069E-00	14			2456.0
26 26	0.1263E-00 0.3291E-00	19			0.00566
85 26	0.4131E-00 0.3291E-00	13			0.37278
26 40	0.1263E-00 0.2103E-00	21			0.002588
85 40	0.4131E-00 0.2103E-00	11			2.625324
7 49	0.3210E-01 0.1325E-00	22			TEMPERATURES
43 50	0.2103E-00 0.1263E-00	7			HOT JUNCTION
68 50	0.3291E-00 0.1263E-00	9			COLD JUNCTION
101 50	0.4906E-00 0.1252E-00	12			RADIATOR (AVG)
26 59	0.1252E-00 0.4876E-01	8			ABSORBER (AVG)
83 61	0.4069E-00 0.3210E-01	10			

## THERMOEQUIVALENT MESH INFORMATION

NODE		X-Y-Z	
NUMBER	COORDINATES		
1	0.26970E-00 0.26970E-00 0.82000E-02		
2	0.26970E-00 0.26970E-00 0.24600E-01		
3	0.26970E-00 0.26970E-00 0.41000E-01		
4	0.26970E-00 0.26970E-00 0.57400E-01		
5	0.26970E-00 0.26970E-00 0.73800E-01		
6	0.26970E-00 0.26970E-00 0.90200E-01		

## TEMPERATURES

NODE										
NUMBER		0	1	2	3	4	5	6	7	8
0		660.25	708.95	757.13	804.84	852.12	899.02	936.80	937.47	936.80
10		937.47	936.80	937.47	937.47	936.80	937.47	936.80	937.47	936.80
20		937.47	936.80	937.47	922.31	624.50	624.50	623.99	623.99	624.50
30		624.50	623.99	624.50	623.99	624.50	624.50	623.99	623.99	624.50
40		635.62	0.	0.	0.	0.	0.	0.	0.	0.



[illegible]

Figure IV-4. Three Node Map

[illegible]

Figure IV-5. Four Node Map

MAP			Node			MAP			Node		
COORDINATES	X-Y	COORDINATES	NUMBER	COORDINATES	X-Y	COORDINATES	X-Y	COORDINATES	NUMBER	COORDINATES	X-Y
56	5	0.2697E-00	0.5037E 00	46	89	33	0.4329E-00	0.2697E-00	30		
26	7	0.1247E-00	0.4888E-00	51	103	33	0.5037E 00	0.2697E-00	31		
85	7	0.4147E-00	0.4887E-00	41	48	35	0.2322E-00	0.2484E-00	62		
56	13	0.2697E-00	0.4329E-00	45	63	35	0.3071E-00	0.2483E-00	22		
11	15	0.5067E-01	0.4147E-00	56	43	37	0.2088E-00	0.2345E-00	63		
100	15	0.4887E-00	0.4147E-00	36	51	37	0.2483E-00	0.2323E-00	7		
39	16	0.1881E-00	0.4111E-00	50	60	37	0.2910E-00	0.2322E-00	17		
12	16	0.3513E-00	0.4111E-00	40	68	37	0.3306E-00	0.2345E-00	23		
56	20	0.2697E-00	0.3768E-00	44	55	38	0.2697E-00	0.2280E-00	12		
45	22	0.2161E-00	0.3625E-00	49	37	39	0.1769E-00	0.2161E-00	64		
66	22	0.3233E-00	0.3625E-00	39	74	39	0.3625E-00	0.2161E-00	24		
27	23	0.1283E-00	0.3513E-00	55	48	40	0.2345E-00	0.2088E-00	8		
84	23	0.4111E-00	0.3513E-00	35	63	40	0.3049E-00	0.2088E-00	18		
56	24	0.2697E-00	0.3400E-00	43	55	41	0.2697E-00	0.1994E-00	13		
48	25	0.2345E-00	0.3306E-00	48	27	42	0.1283E-00	0.1881E-00	65		
63	25	0.3049E-00	0.3306E-00	38	84	42	0.4111E-00	0.1881E-00	25		
37	26	0.1769E-00	0.3233E-00	54	45	44	0.2161E-00	0.1769E-00	9		
74	26	0.3625E-00	0.3233E-00	34	66	44	0.3233E-00	0.1769E-00	19		
43	28	0.2088E-00	0.3049E-00	53	55	45	0.2697E-00	0.1626E-00	14		
51	28	0.2484E-00	0.3072E-00	47	11	50	0.5067E-01	0.1247E-00	66		
56	28	0.2697E-00	0.3114E-00	42	39	50	0.1881E-00	0.1283E-00	10		
60	28	0.2911E-00	0.3071E-00	37	72	50	0.3513E-00	0.1283E-00	20		
68	28	0.3306E-00	0.3049E-00	33	100	50	0.4887E-00	0.1247E-00	26		
48	30	0.2323E-00	0.2911E-00	52	55	52	0.2697E-00	0.1065E-00	15		
63	30	0.3072E-00	0.2910E-00	32	26	59	0.1247E-00	0.5067E-01	11		
8	33	0.3566E-01	0.2697E-00	61	85	59	0.4147E-00	0.5066E-01	21		
22	33	0.1065E-00	0.2697E-00	60	55	61	0.2697E-00	0.3566E-01	16		
34	33	0.1626E-00	0.2697E-00	59							
41	33	0.1994E-00	0.2697E-00	58							
47	33	0.2280E-00	0.2697E-00	57							
64	33	0.3114E-00	0.2697E-00	27							
70	35	0.3400E-00	0.2697E-00	28							
77	33	0.3768E-00	0.2697E-00	29							
THERMOELEMENT MESH INFORMATION											
NODE X-Y-Z											
NUMBER COORDINATES											
1 0.26970E-00 0.26970E-00 0.82000E-02											
2 0.26970E-00 0.26970E-00 0.24600E-01											
3 0.26970E-00 0.26970E-00 0.41000E-01											
4 0.26970E-00 0.26970E-00 0.57400E-01											
5 0.26970E-00 0.26970E-00 0.73800E-01											
6 0.26970E-00 0.26970E-00 0.90200E-01											

Table IV-1. Three Node Specification

MAP COORDINATES	X-Y COORDINATES	Node NUMBER	MAP COORDINATES	X-Y COORDINATES	Node NUMBER
44	0.2122E-00	83	57	0.2768E-00	70
67	0.3272E-00	76	98	0.4773E-00	54
20	0.9640E-01	90	25	0.1206E-00	102
91	0.4430E-00	69	46	0.2229E-00	92
47	0.2284E-00	82	51	0.2485E-00	84
64	0.3110E-00	75	60	0.2906E-00	63
8	0.3434E-01	97	65	0.3165E-00	57
32	0.1521E-00	89	86	0.4188E-00	53
79	0.3873E-00	68	34	0.1627E-00	101
103	0.5051E 00	62	49	0.2374E-00	91
49	0.2401E-00	81	62	0.3017E-00	56
62	0.2993E-00	74	77	0.3767E-00	52
38	0.1853E-00	88	40	0.1929E-00	100
73	0.3541E-00	67	44	0.2145E-00	99
20	0.9372E-01	96	67	0.3249E-00	50
91	0.4457E-00	61	71	0.3465E-00	51
51	0.2484E-00	80	48	0.2335E-00	98
60	0.2910E-00	73	63	0.3059E-00	49
43	0.2091E-00	87	40	0.1929E-00	107
68	0.3303E-00	66	44	0.2145E-00	106
30	0.1433E-00	95	48	0.2336E-00	105
52	0.2544E-00	79	63	0.3059E-00	42
59	0.2850E-00	72	67	0.3249E-00	43
81	0.3961E-00	60	71	0.3465E-00	44
37	0.1790E-00	94	34	0.1627E-00	108
47	0.2262E-00	86	49	0.2377E-00	112
64	0.3132E-00	65	62	0.3020E-00	35
74	0.3604E-00	59	77	0.3767E-00	45
4	0.1535E-01	104	25	0.1206E-00	109
53	0.2587E-00	78	46	0.2229E-00	113
58	0.2807E-00	71	60	0.2909E-00	28
107	0.5241E 00	55	65	0.3165E-00	36
42	0.2046E-00	93	86	0.4188E-00	46
49	0.2384E-00	85	51	0.2488E-00	7
62	0.3010E-00	64	54	0.2626E-00	14
69	0.3349E-00	58	57	0.2769E-00	21
13	0.6211E-01	103			
54	0.2626E-00	77			

Table IV-2. Four Node Specification

map coordinates		x-y coordinates		node number
13	38	0.6211E-01	0.2284E-00	110
42	38	0.2045E-00	0.2262E-00	114
49	38	0.2384E-00	0.2229E-00	8
62	38	0.3010E-00	0.2229E-00	29
69	38	0.3348E-00	0.2262E-00	37
98	38	0.4773E-00	0.2284E-00	47
53	39	0.2587E-00	0.2145E-00	15
58	39	0.2807E-00	0.2145E-00	22
4	40	0.1535E-01	0.2122E-00	111
37	40	0.1790E-00	0.2091E-00	115
47	40	0.2262E-00	0.2046E-00	9
64	40	0.3132E-00	0.2046E-00	30
74	40	0.3604E-00	0.2091E-00	38
107	40	0.5240E-00	0.2122E-00	48
52	42	0.2544E-00	0.1929E-00	16
59	42	0.2850E-00	0.1929E-00	23
30	43	0.1433E-00	0.1853E-00	116
43	43	0.2091E-00	0.1790E-00	10
68	43	0.3303E-00	0.1790E-00	31
81	43	0.3961E-00	0.1853E-00	39
51	45	0.2484E-00	0.1627E-00	17
60	45	0.2910E-00	0.1627E-00	24
20	47	0.9372E-01	0.1521E-00	117
91	47	0.4457E-00	0.1521E-00	40
38	48	0.1853E-00	0.1433E-00	11
73	48	0.3541E-00	0.1433E-00	32
49	50	0.2401E-00	0.1206E-00	18
62	50	0.2993E-00	0.1206E-00	25
8	53	0.3434E-01	0.9641E-01	118
103	53	0.5051E-00	0.9641E-01	41
32	54	0.1521E-00	0.9372E-01	12
79	54	0.3873E-00	0.9372E-01	33
47	57	0.2284E-00	0.6211E-01	19
64	57	0.3110E-00	0.6211E-01	26
20	61	0.9641E-01	0.3434E-01	13
91	61	0.4430E-00	0.3434E-01	34
44	63	0.2122E-00	0.1535E-01	20
67	63	0.3272E-00	0.1535E-01	27

## THEMOELEMENT MESH INFORMATION

NODE NUMBER	X-Y-Z COORDINATES		
1	0.26970E-00	0.26970E-00	0.82000E-02
2	0.26970E-00	0.26970E-00	0.24600E-01
3	0.26970E-00	0.26970E-00	0.41000E-01
4	0.26970E-00	0.26970E-00	0.57400E-01
5	0.26970E-00	0.26970E-00	0.73800E-01
6	0.26970E-00	0.26970E-00	0.90200E-01

Table IV-2 Continued. Four Node Specification

An examination of the data presented by Fuschillo shows that thermal conductivity data are not presented for the thermoelements analyzed. Further, use of the thermoelectric properties presented in his Figures 8 and 9 is not always in agreement with the mean property values utilized for the calculations. Consequently, to provide the same basis between the calculations, the data have been obtained by "back calculations" from his results.

A design point is selected by Fuschillo on page 112 for the manufactured solar flat plate panels. They are stated to have the following characteristics:

Hot Junction Temperature ( $T_h$ )	520°K
Absorber Relative Absorbtivity ( $\alpha_a$ )	0.82
Absorber Relative Emissivity ( $\epsilon_a$ )	0.08
Cold Junction Temperature ( $T_c$ )	350°K
Radiator Relative Emissivity ( $\epsilon_r$ )	0.95
Average Figure of Merit ( $\bar{Z}$ )	$1.3 \times 10^{-3} \text{ } ^\circ\text{K}^{-1}$
$ZT_h$	0.67
Power Output	3.3 watts/ft <sup>2</sup>
Plate Area (A) to Length ( $\ell$ ) Ratio	30 cm
Open Circuit Voltage ( $V_o = S \Delta T$ , S = Seebeck Coefficient)	$64 \times 10^{-3}$ Volts
Internal Plate Relative Emissivity	0.03

Examination of these data shows that Fuschillo's Figure 7, line D is applicable. This yields a value of  $\bar{k}A/\ell = 0.31$  where  $\bar{k}$  is the mean thermoelement thermal conductivity. This immediately allows the thermal conductivity of the thermoelement to be calculated. Since  $\bar{Z}$  and  $\bar{k}$  are now known, and by definition:

$$\bar{Z} = \frac{\bar{S}^2}{\bar{\rho} \bar{k}}$$

where  $\bar{S}$  = Seebeck voltage;  $\bar{\rho}$  = electrical resistivity

We find:

$$\frac{\bar{S}^2}{\bar{\rho}} = \bar{k} \bar{Z} = 1.440 \times 10^{-5} \frac{\text{watts}}{\text{cm}^2 \text{ } ^\circ\text{K}^2}$$

From Fuschillo, page 112, for these temperature conditions:

$$V_o = S \Delta T = 64 \times 10^{-3} \text{ v.}$$

where  $V_o$  = open circuit voltage;  $\Delta T = T_h - T_c$

$$S = \frac{64 \times 10^{-3}}{520 - 350} = 3.82 \times 10^{-4} \text{ v/}^\circ\text{K}$$

Now:

$$\bar{S} = \frac{S}{2} = \frac{3.82 \times 10^{-4}}{2} = 1.91 \times 10^{-4} \text{ v/}^\circ\text{K}$$

$$\bar{\rho} = \frac{(1.91)^2(10^{-8})}{(1.44)(10^{-5})} = 2.53 \times 10^{-3} \text{ } \Omega \text{ cm}$$

The length of the thermoelements is 0.25 cm. An area to length ratio of thirty (30) yields a corresponding plate side dimension of 2.74 cm. Since the repeating section analyzed by the code is one fourth of the area, or one half of the side dimension, the repeating section analyzed is 1.37 cm which is very close to the value presented in Fuschillo's Figure 11 for the thermoelements which have been constructed and which are analyzed herein.

Fuschillo's analysis, as shown in his Figure 1, is based upon a load resistance equal to the internal thermoelement resistance. Since the tested thermoelement is 0.25 cm long and has a cross sectional area of  $(0.1245)^2 \text{ cm}^2$ , the resistance is:

$$R = \frac{(0.25)(2.53)(10^{-3})}{(0.1245)^2} = 4.07 \times 10^{-2} \text{ } \Omega$$

These data have been utilized in a computer run with the code modified to hold the hot and cold junction temperatures at  $520^\circ\text{K}$  and  $350^\circ\text{K}$ , respectively. The thermoelectric behavior predicted by Fuschillo was identical to that predicted by the code under these conditions.

The full calculational capabilities of the code were used with the same input data. These results were presented as the sample problem in Section IV-A. A comparison of the calculated temperatures and the results from Fuschillo's Figure 2 is shown in Table IV-3. It is interesting that the computer results for the average collector temperature and Fuschillo's calculations are virtually identical. The computer predicts an average radiator temperature that is  $3^\circ\text{K}$  lower, a hot junction temperature that is  $8^\circ\text{K}$  lower, and a cold junction temperature that is  $3^\circ\text{K}$  higher. The effect is essentially due to temperature gradient in the radiator and collector, which is to be expected. Since the average collector temperature is by definition that temperature which represents the overall collector from a radiant heat rejection standpoint, the agreement between the manual and computer calculations is to be expected. The temperature decrease to  $512^\circ\text{K}$  at the hot junction and the increase to  $353^\circ\text{K}$  at the cold junction results in a lower

temperature differential across the thermoelement. This in turn causes a six to seven percent decrease in the power output. The three degree difference in the average radiator temperature is probably due to the cross sectional area employed. A difference between the actual number used by Fuschillo and that utilized herein of one percent in the linear side dimension would account for this difference. It would also account for the slightly higher average collector temperature.

Table IV-3 Comparison of Computed Temperature Data

<u>Item</u>	<u>Fuschillo</u>	<u>Code</u>
Hot Junction Temperature, °K	520	512.39
Cold Junction Temperature, °K	350	353.12
Average Radiator Temperature, °K	350	346.83
Average Collector Temperature, °K	520	520.61

#### C. Transient Behavior

A transient investigation was conducted to estimate the time required for the solar flat plate to reach its steady state value and also to investigate how rapidly the power dropped when the solar input was removed. The heat up curve shown in Figure IV-6 indicates that the power output is to within 90 percent of the steady state value in 100 seconds and is approaching the steady state value at 200 seconds. The cool down curve indicates that the power output drops extremely rapidly when the solar flux is removed and is approximately one half of the steady state value within 15 seconds.

#### D. Parametric Investigation With Constant Properties

Variation of the design parameters is of particular interest to this program because of its extensive calculational capability. The data presented in this section are based upon the bismuth telluride data which have been utilized for the previously presented calculations.

The effective changes in the collector and radiator thickness are indicated in Figure IV-7. The present configuration is considered to have a collector thickness of 0.002 in. and a radiator thickness including the effect of the stiffening rib of 0.0025 in. This operating point is indicated in the Figure. The power output shown is for one repeating section of the plate. The efficiency is based upon the electrical power produced at a useful load compared to the energy intercepted by the plate. As would be expected, the effect of increasing collector thickness or radiator thickness



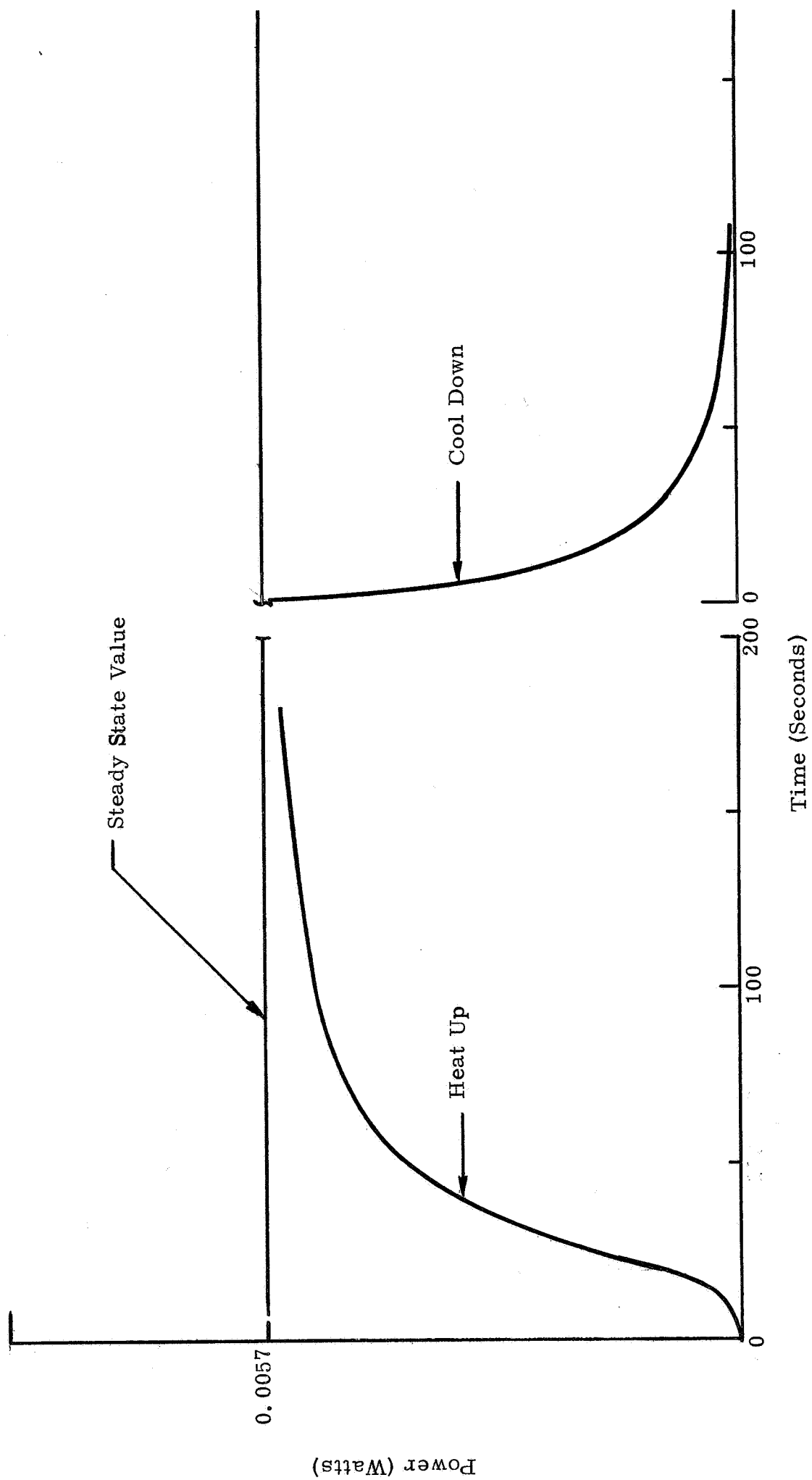


Figure IV-6. Time Transient Investigation

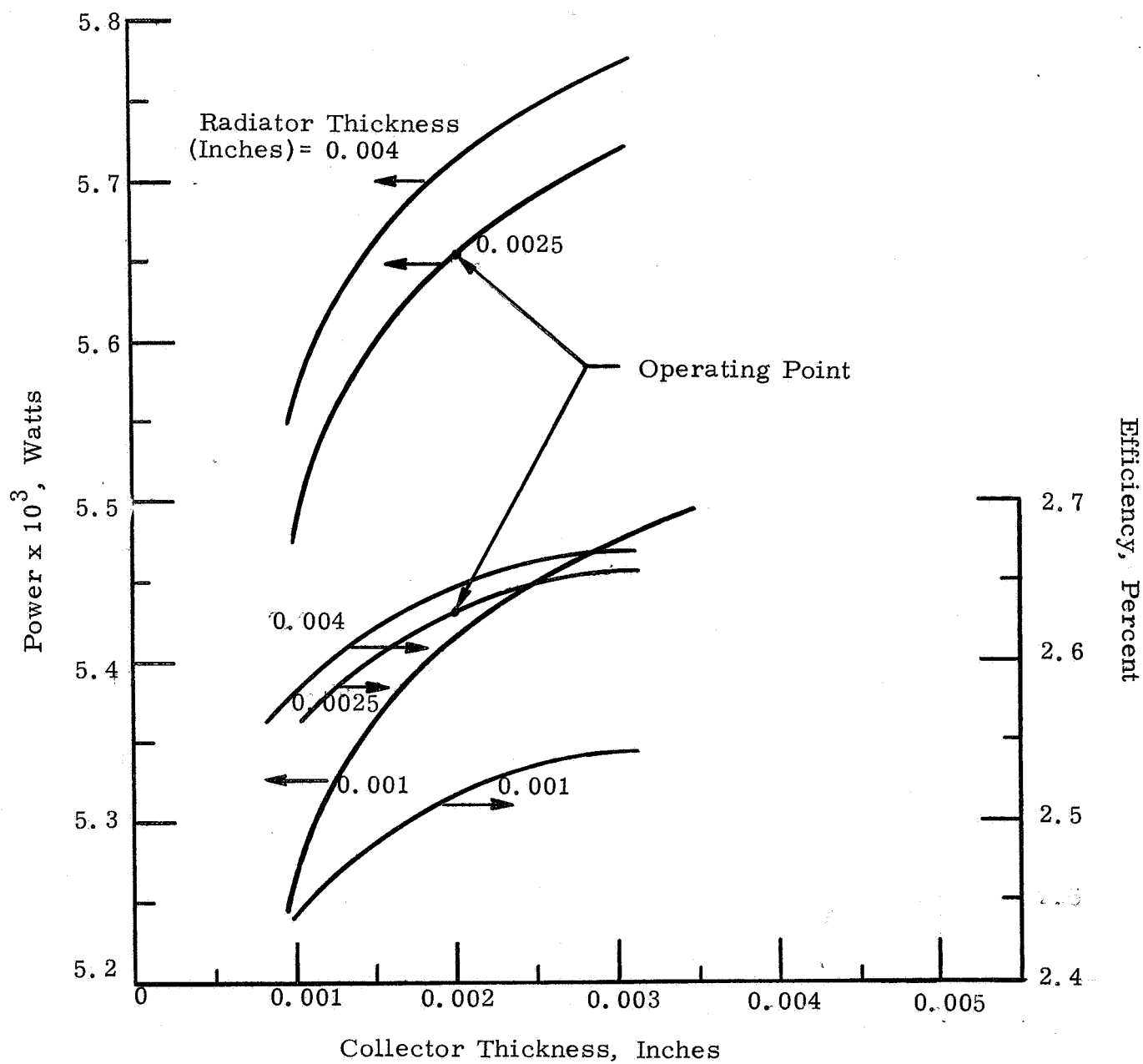


Figure IV-7. Effect of Changes in Plate Thickness

is to increase the power output as well as the efficiency. The effect is relatively flat with respect to thickness for small changes (the scales in Figure IV-7 may be somewhat misleading). However, a change in thickness of both collector and radiator from 0.001 in. to 0.004 in. results in a ten percent increase in the power output, which indicates that larger changes are not negligible.

The effect of changing the thermoelement length with all other parameters constant is indicated in Figure IV-8. This figure immediately shows that the overall efficiency of the configuration will increase as the thermoelement length is increased. This particular effect could not be observed utilizing the approach taken by Fuschillo because of his assumption that the internal load resistances were equal. It is known that the best efficiency for conversion of heat to electricity with thermoelements occurs when the load resistance is approximately twenty to forty percent greater than the internal resistance of the thermoelements. This occurs because of the secondary effect of Peltier cooling which is increased by increasing current. Figure IV-8 indicates that the present thermoelements are not an optimum configuration with respect to the thermoelement area to length ratio since the efficiency and power are increasing as the thermoelement length increased. Further, the load resistance utilized for this combination is also not an optimum. This would indicate that further analysis should be employed to generate a more optimum configuration.

The effect of changing the load resistance is shown in Figure IV-9. The present operating point occurs with a load resistance of 0.0407 ohms. This figure indicates that the power output and efficiency will increase with increasing load resistance with a maximum in the vicinity of 0.05 ohms. This value is in the range of the optimum ratio of external to internal load which would normally be expected. The effect of contact resistance or other internal resistances is also indicated in Figure IV-9. This indicates the need for minimizing such resistances, and also strongly emphasizes the need for including them in calculations if realistic answers are to be obtained. It perhaps should be emphasized that all previous answers presented in this analysis have assumed a zero contact resistance and have additionally assumed no wiring resistance.

#### E. Parametric Investigations Conducted With Variable Bismuth-Telluride Properties

Bismuth-telluride properties should not be represented as constants for parametric studies, particularly when the temperature can change drastically, because several of the variables are immediately being ignored. Typical bismuth-telluride data are shown in Figures IV-10 through IV-12. The thermal conductivity information principally is based upon estimates. These data have been utilized in several additional parametric studies.

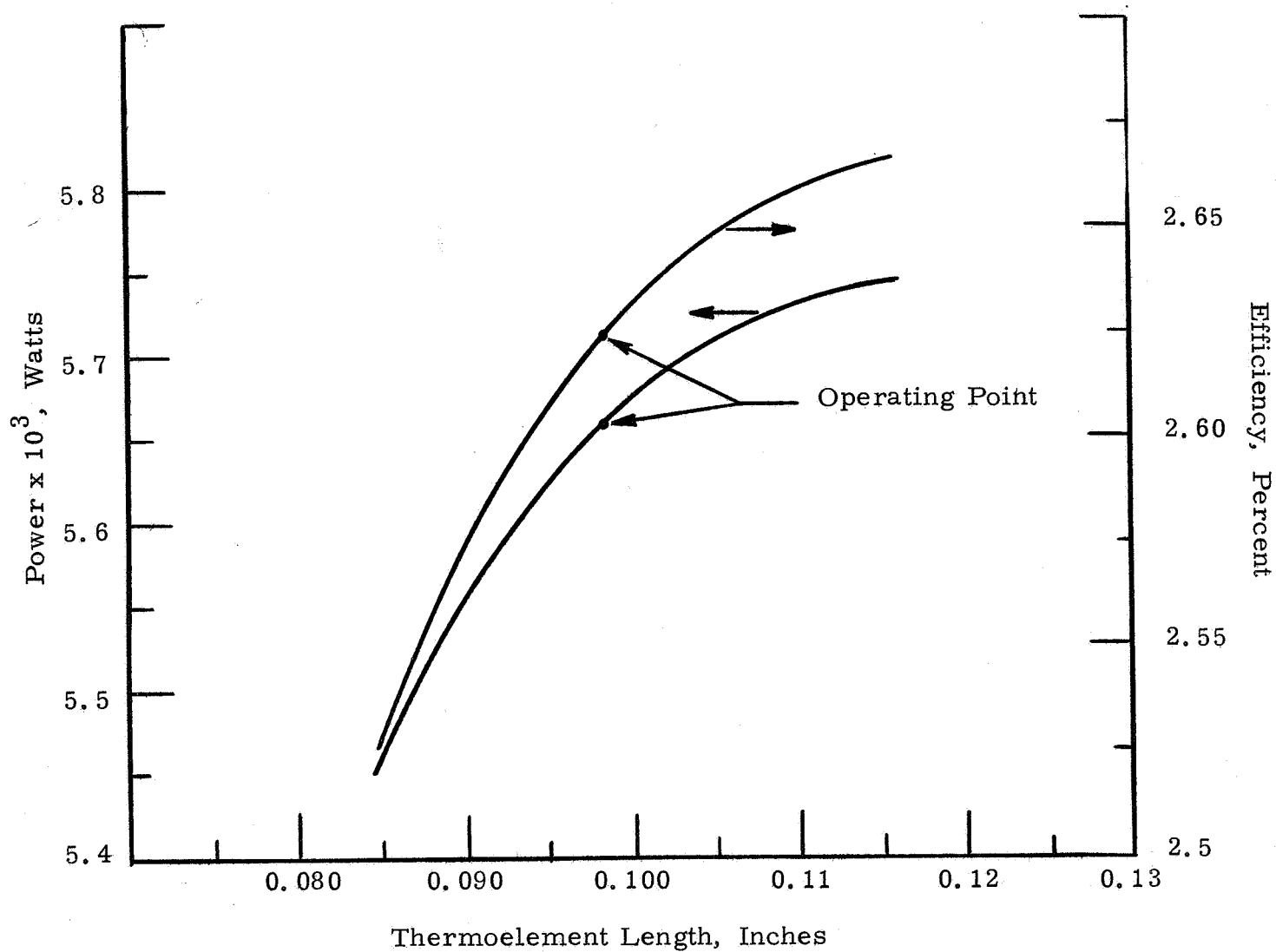


Figure IV-8. The Effect of Changing Thermoelement Length  
With a Load Resistance of 0.0407 Ohms

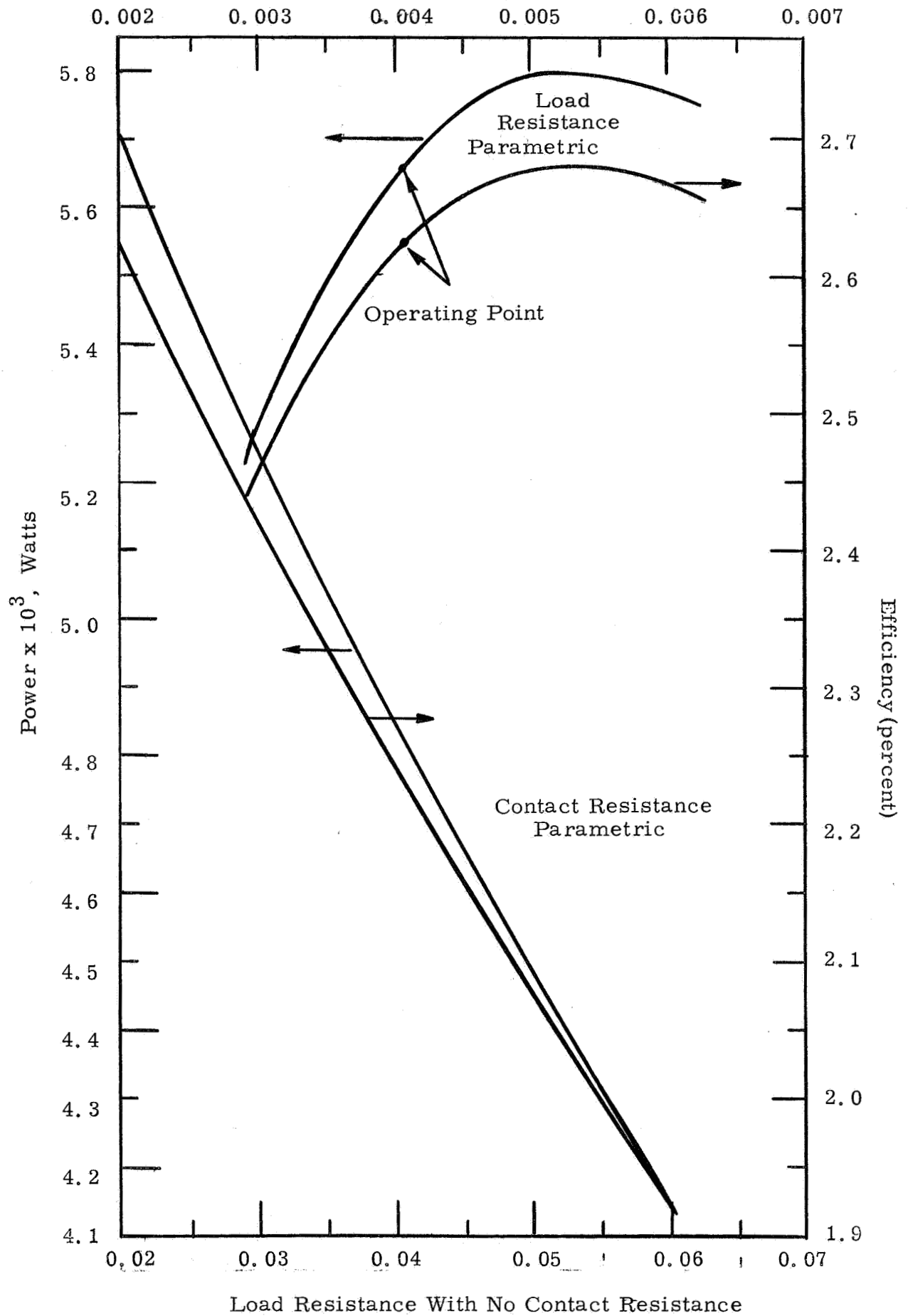


Figure IV-9. Effect of Load and Contact Resistance

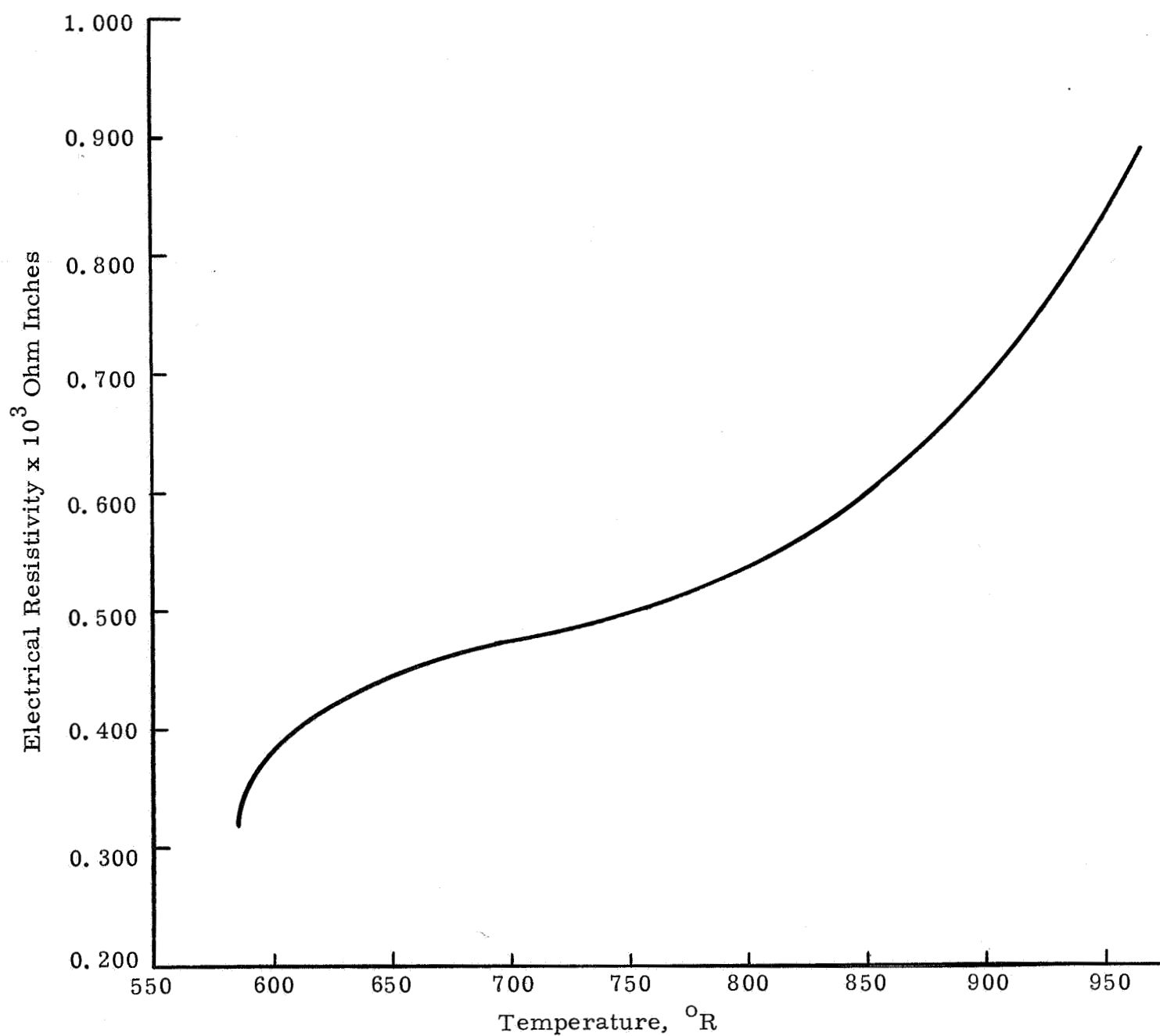


Figure IV-10. BiTe Electrical Resistivity

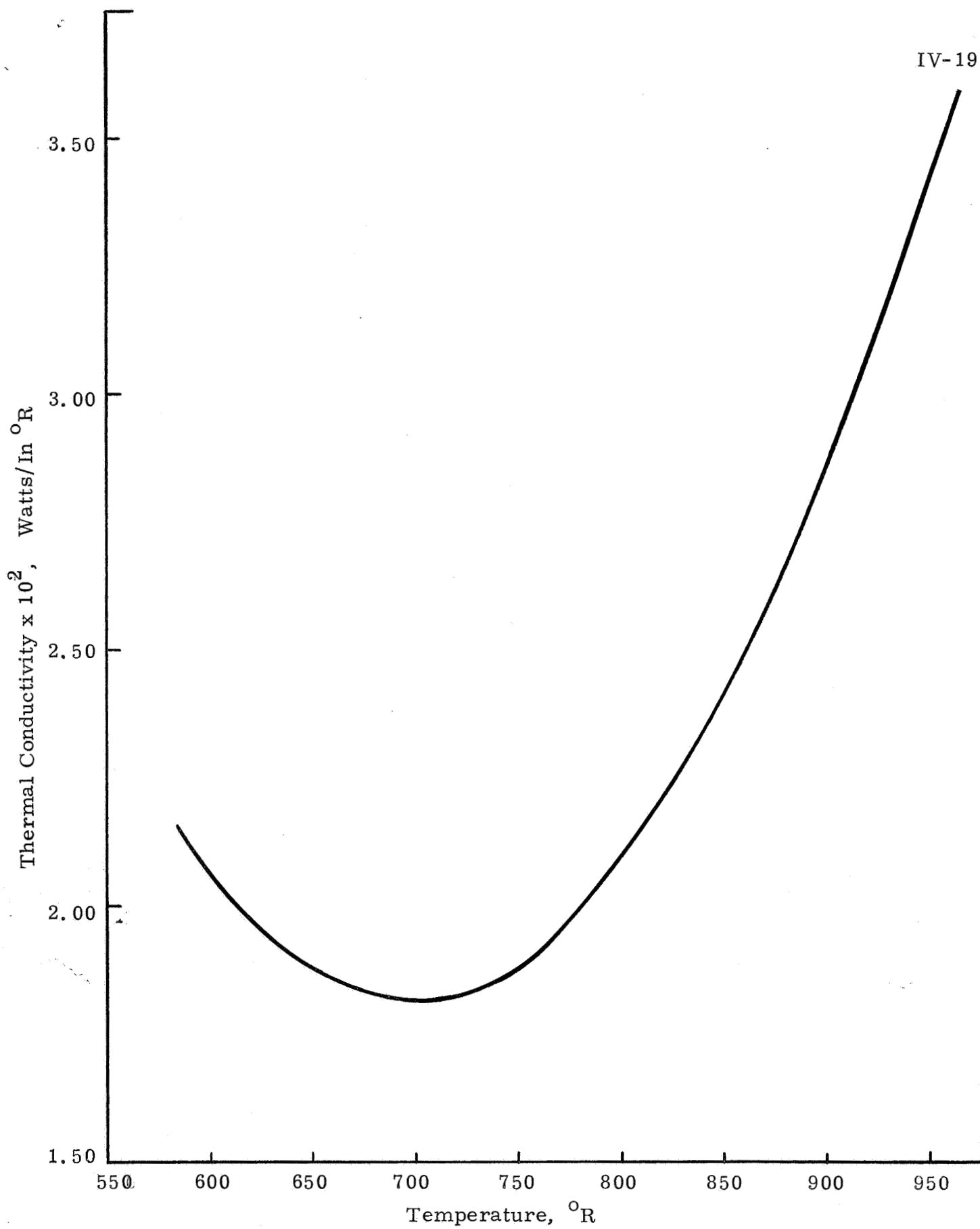


Figure IV-11. BiTe Thermal Conductivity

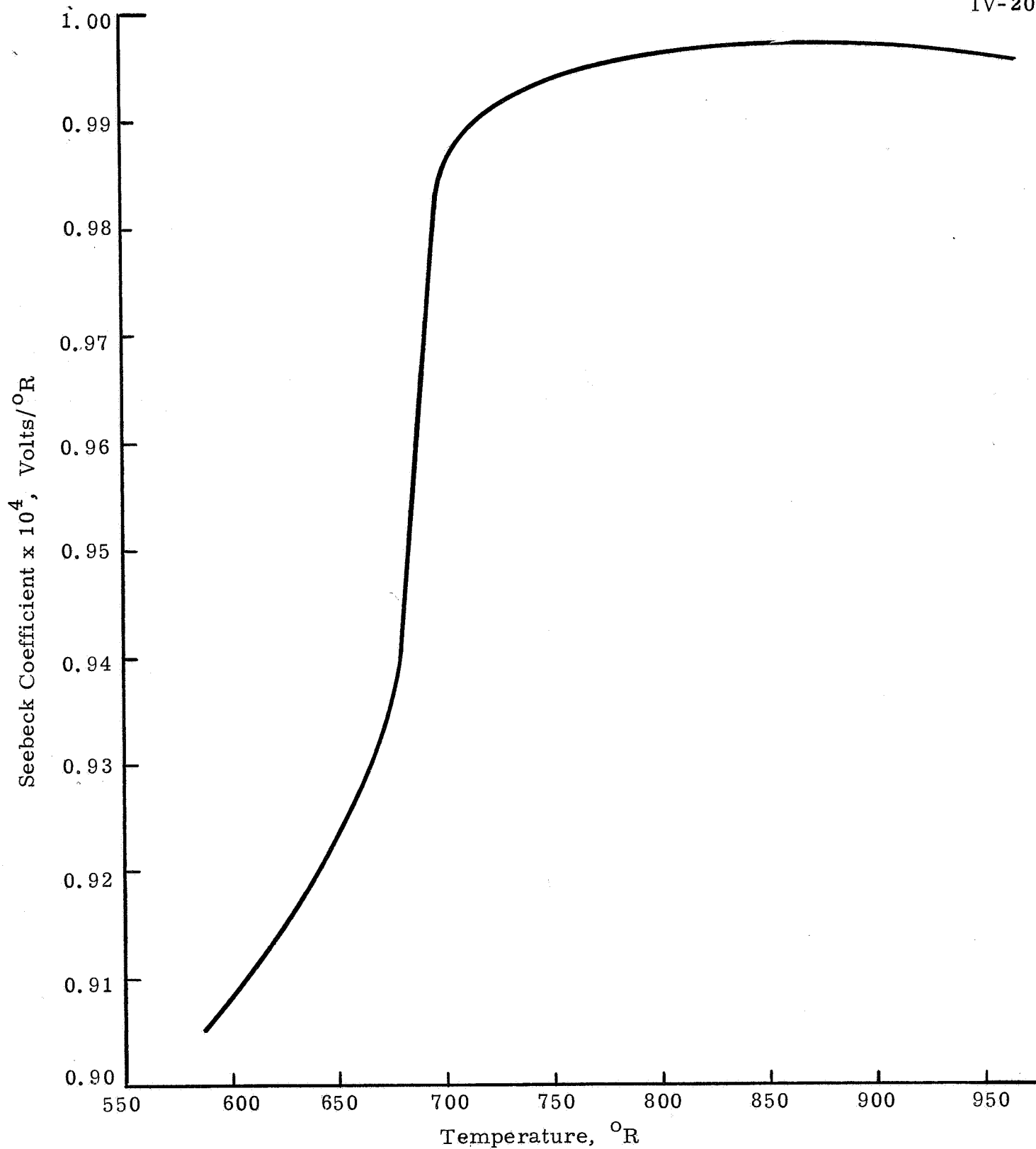


Figure IV-12. BiTe Seebeck Coefficient



The effect of variations in the collector and radiator thickness is shown in Figure IV-13. The reason for the lower power output of these runs as compared to the previously presented information is due to the data used, a mismatch in the load resistance, and the configuration being further removed from optimum than was previously the case. This effect is further indicated in Figure IV-14 which indicates the effect of changing thermoelement length. The steepness of the power output curve with respect to thermoelement length indicates the distance that the indicated design point is removed from the optimum point.

The effect of changing load resistance and contact resistance with these data is indicated in Figure IV-15. The conclusions previously mentioned in regard to contact and wiring resistance remain unchanged. However, here it is seen that the power output, and hence the efficiency, is decreasing with increasing load resistance. This is taking place because the resistivity of the bismuth-telluride used for these calculations was lower than the previous values, and the optimum load resistance is on the opposite side of the operating point.

The effect of changing the plate area while holding other independent values constant is shown in Figure IV-16. As would be expected, the power input increases with increasing surface area because this allows the collector to run hotter while the radiator runs cooler. Of particular interest, however, is the effect of changing the area on power density. Here, it is seen that a maximum is obtained as the area is properly matched to the thermoelement characteristics. This immediately indicates the importance of considering the variation of plate area on the power density. It is further expected that the effect of plate thickness has a rather large effect on the optimum area because of the temperature gradient which would occur.

These parametric studies show that to realize the best performance it is necessary to optimize with respect to thermoelement length, thermoelement cross sectional area, load resistance, and plate area. Further, the effect of plate thickness will have a perturbing effect on the conclusions. In general, the thicker the plate, the more attractive will be the performance, but the heavier will be the resulting solar panel.

#### F. Solar Flat Plate Using Silicon-Germanium Thermoelements

Application of the solar flat plate to produce electrical power for missions close to the sun is particularly attractive if high temperature thermoelements are utilized. Bifano\* has reported results for several investigations at varying distances from the sun. Two of his investigations have been selected for comparison purposes, at 0.388 and 0.25AU from the sun. Typical characteristics are presented in Table IV-4. Since he did not present silicon germanium data, data for this material was obtained from

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\* Bifano, William J., "Analysis of Solar Thermoelectric Flat-Plate Generators Operating in the Range of 1.0 to 0.1 Astronomical Unit", E-3478, Lewis Research Center, NASA, Cleveland, Ohio.

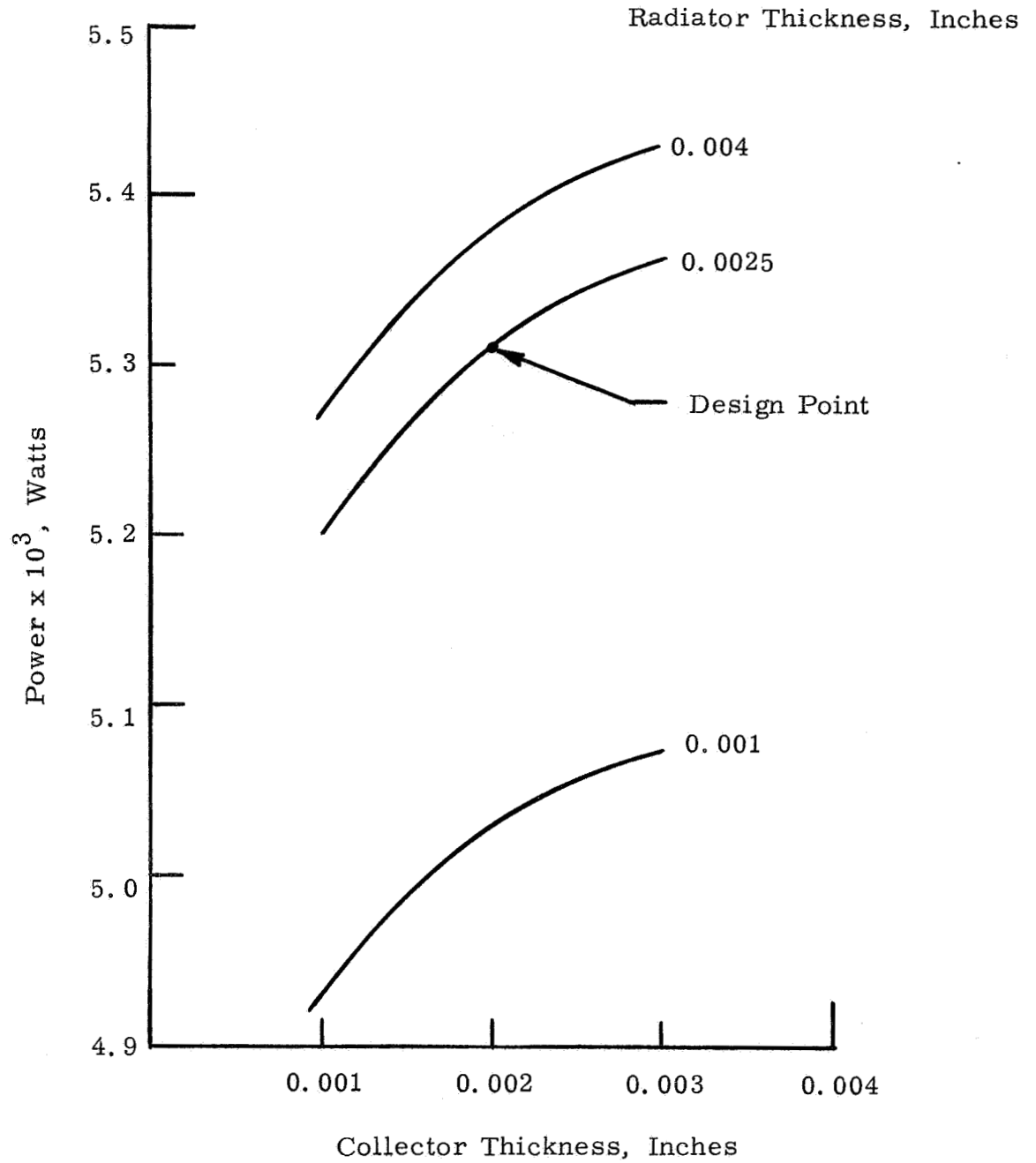


Figure IV-13. Thickness Parametric With Variable Bi-Te Properties

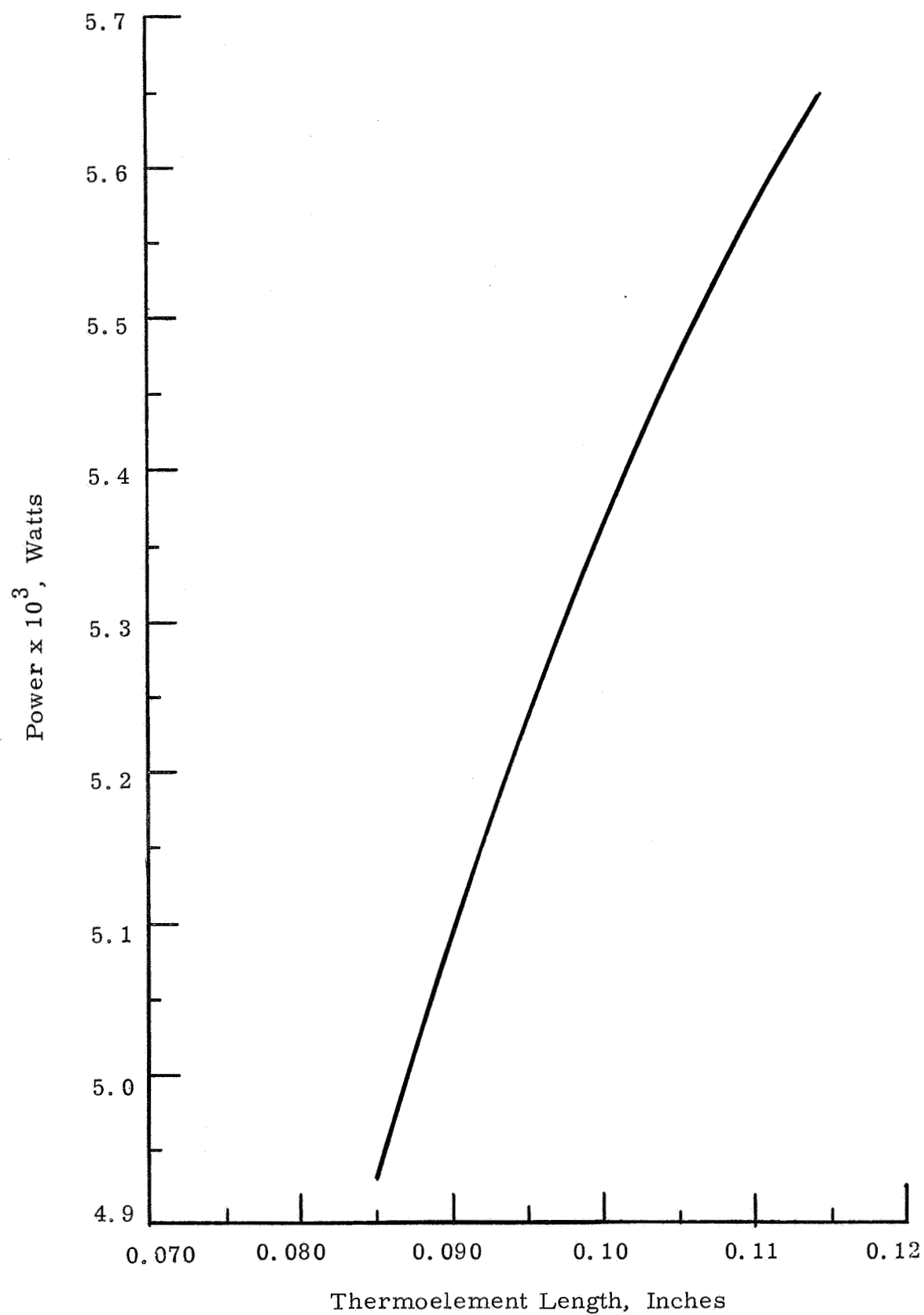


Figure IV-14. Thermoelement Length Parametric With Bi-Te Variable Properties

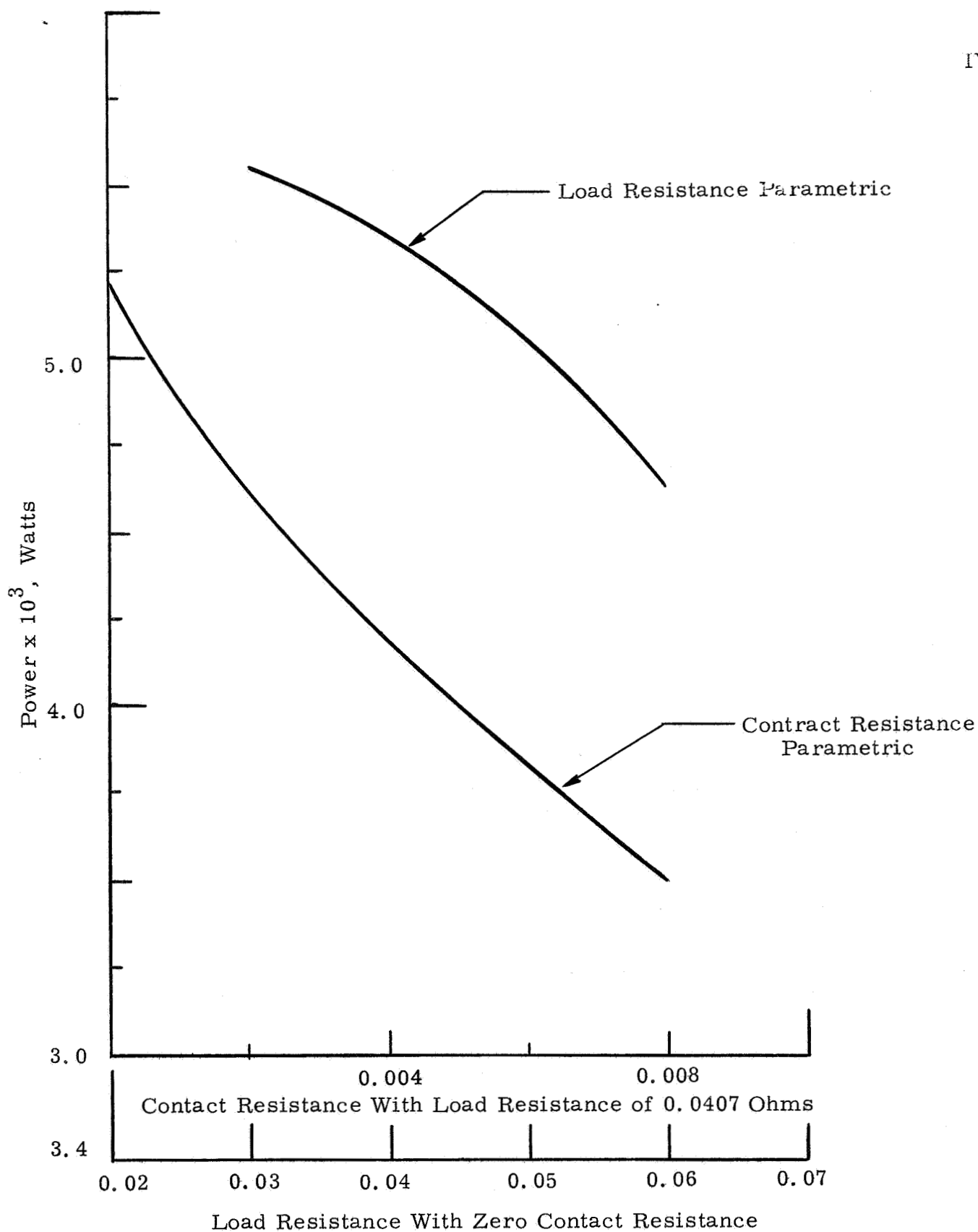


Figure IV-15. Bi-Te Variable Properties Resistance Parametric

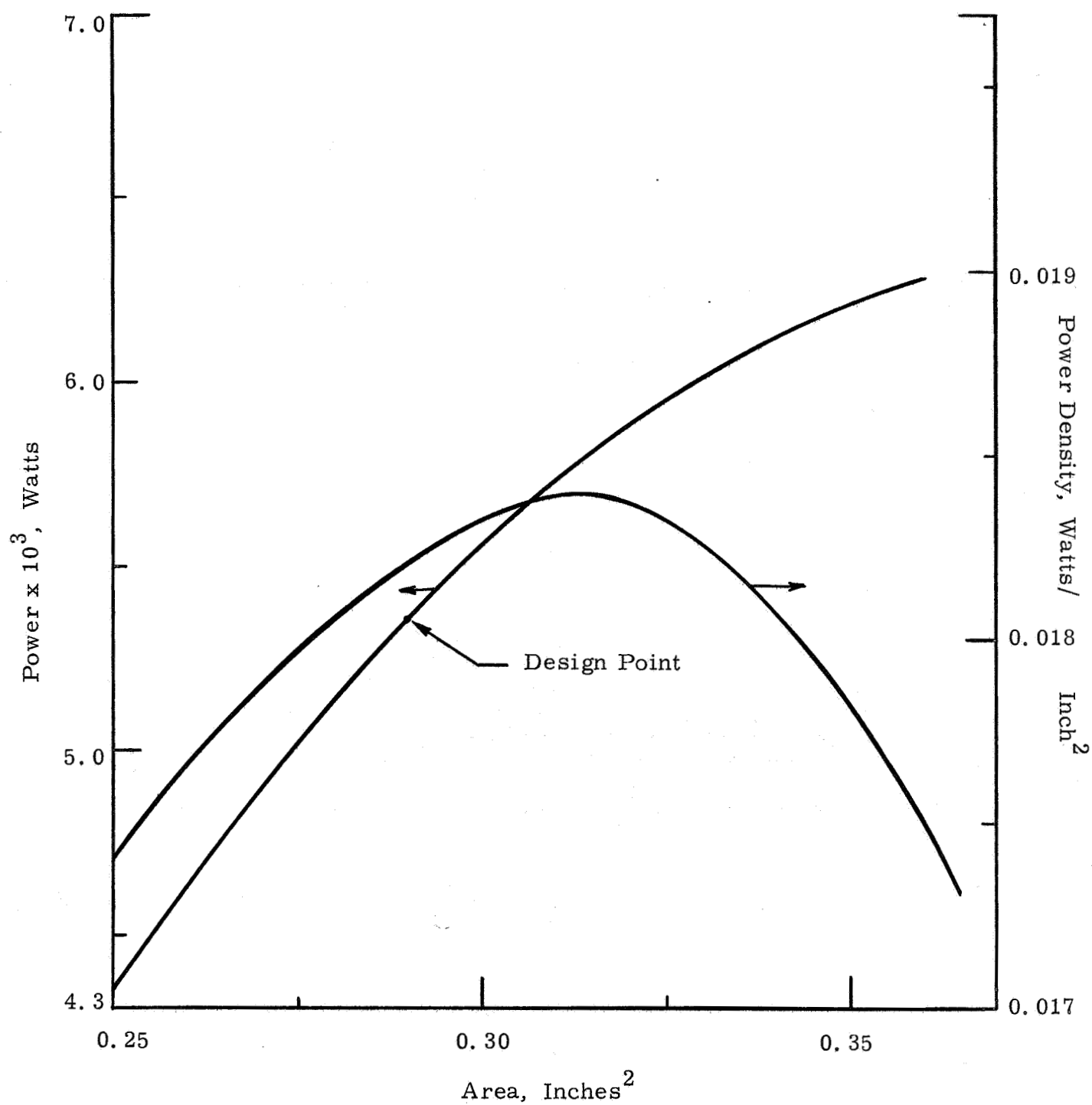


Figure IV-16. Plate Area Parametric Bi-Te Variable Properties

RCA for purposes of the calculation. These data are presented in Figures IV-17 through IV-19 (inclusive). The collector and radiator surface characteristics with the exception of the internal surface are shown in Figure IV-20.

TABLE IV-4. SOLAR FLAT PLATE CHARACTERISTICS FOR  
HIGH TEMPERATURE APPLICATIONS

Case I: Distance from Sun = 0.388AU

Collector Material	Molybdenum
Radiator Material	Molybdenum
Thermoelement Material	Silicon-Germanium
Collector Thickness	0.005 in
Radiator Thickness	0.005 in
Thermoelement Length	0.24 in
Collector or Radiator Side Dimension	0.804 in
Thermoelement Side Dimension (equivalent)	0.088 in
Interplate Relative Emissivity (assumed)	0.03
Solar Flux	6w/in <sup>2</sup>

\* Case II: Solar Distance = 0.25AU

Plate Side Dimension	0.574 in
Thermoelement Side Dimension (equivalent)	0.088 in
Solar Flux	14.4w/in <sup>2</sup>

\* NOTE: All other items identical to Case I values

A comparison of the characteristics computed by Bifano and those obtained by the computer program is shown in Table IV-5 for the Case I calculations.

TABLE IV-5. CALCULATED RESULTS FOR  
SOLAR FLAT PLATE AT 0.388AU FROM THE SUN

<u>Item</u>	<u>Bifano</u>	<u>Computer Program</u>
Hot Junction Temperature (°R)	1660	1544.5
Cold Junction Temperature (°R)	1000	969.40
Collector Temperature (°R)	1660	1581.9

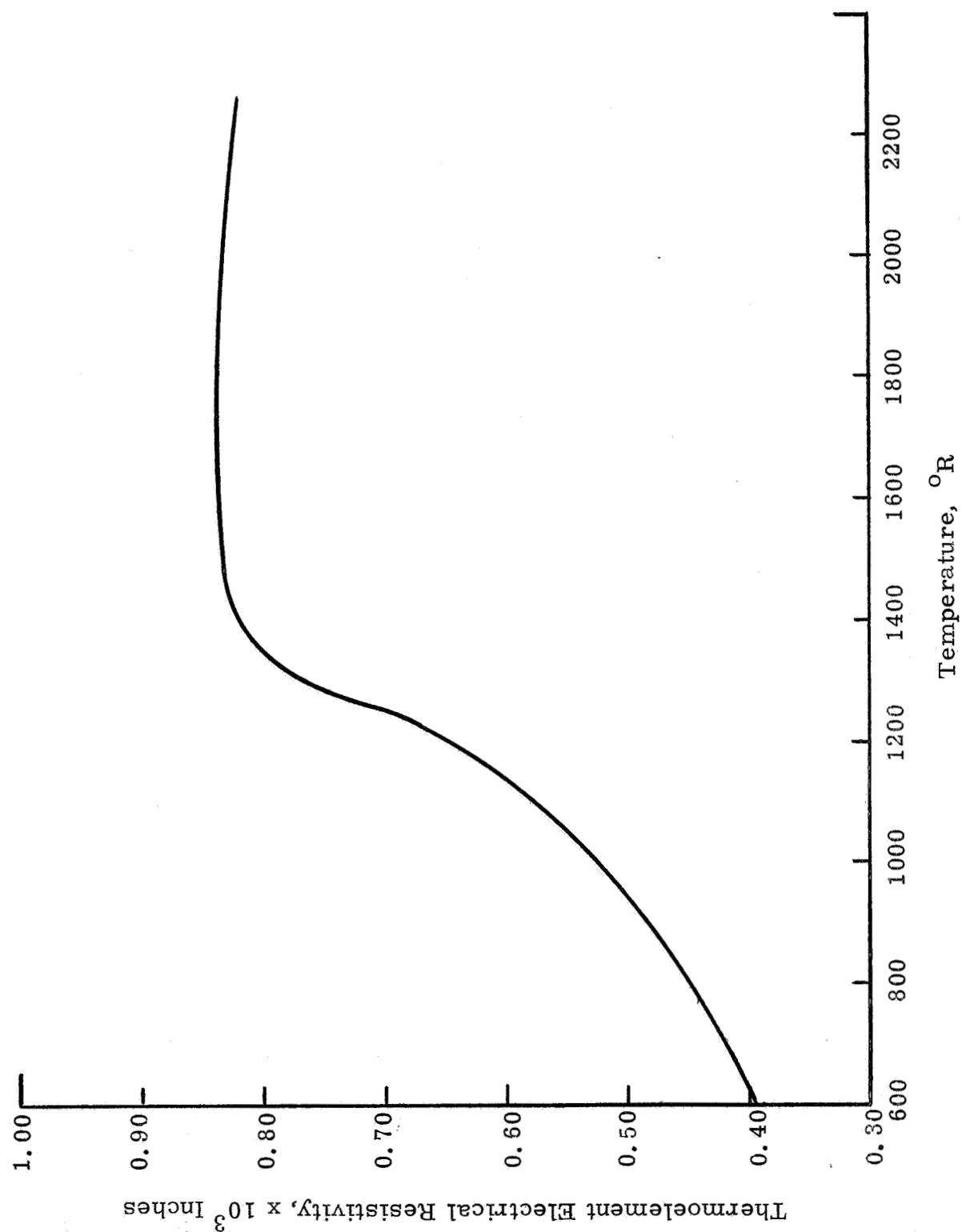


Figure IV-17. SiGe Electrical Resistivity

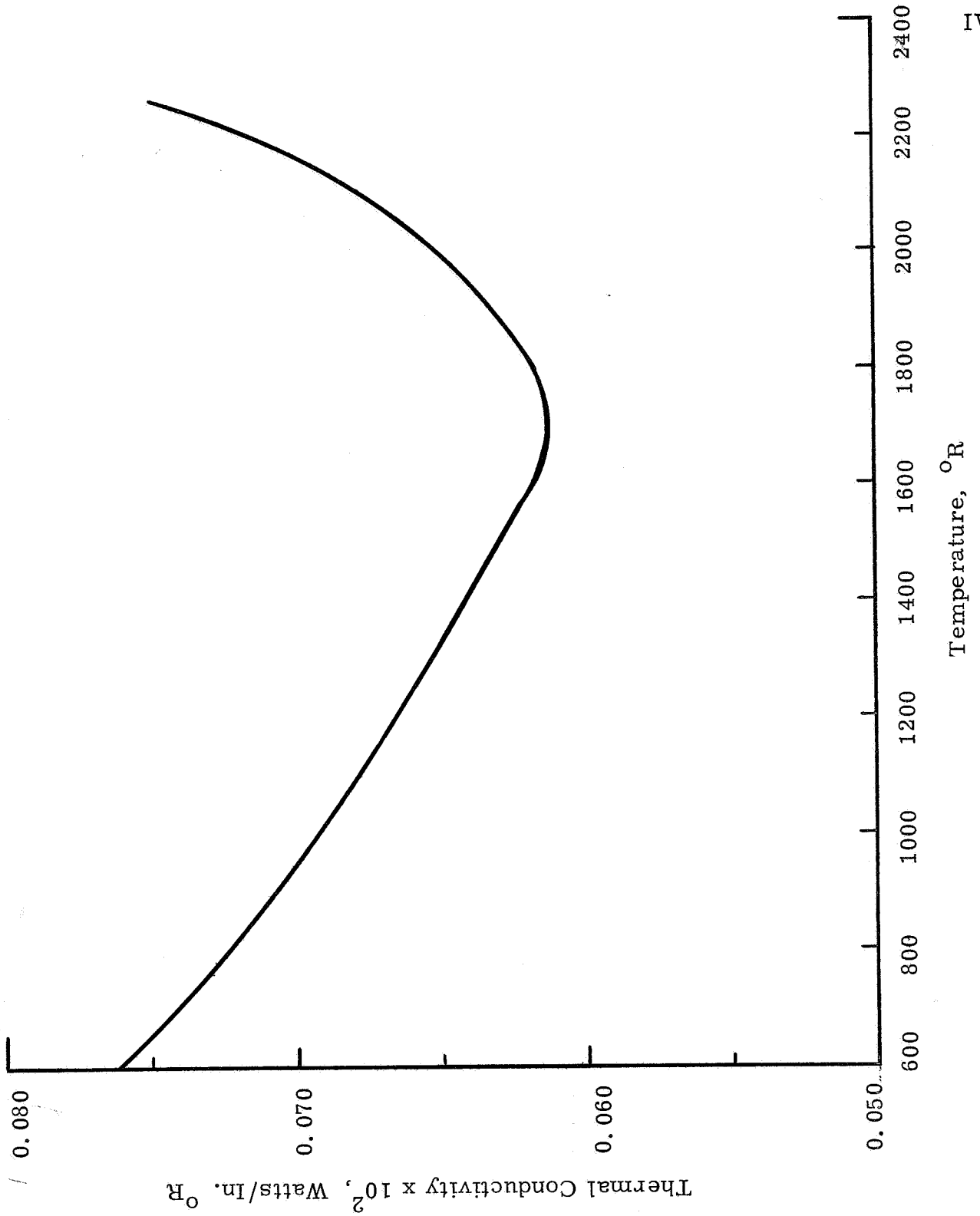


Figure IV-18, SiGe Thermal Conductivity



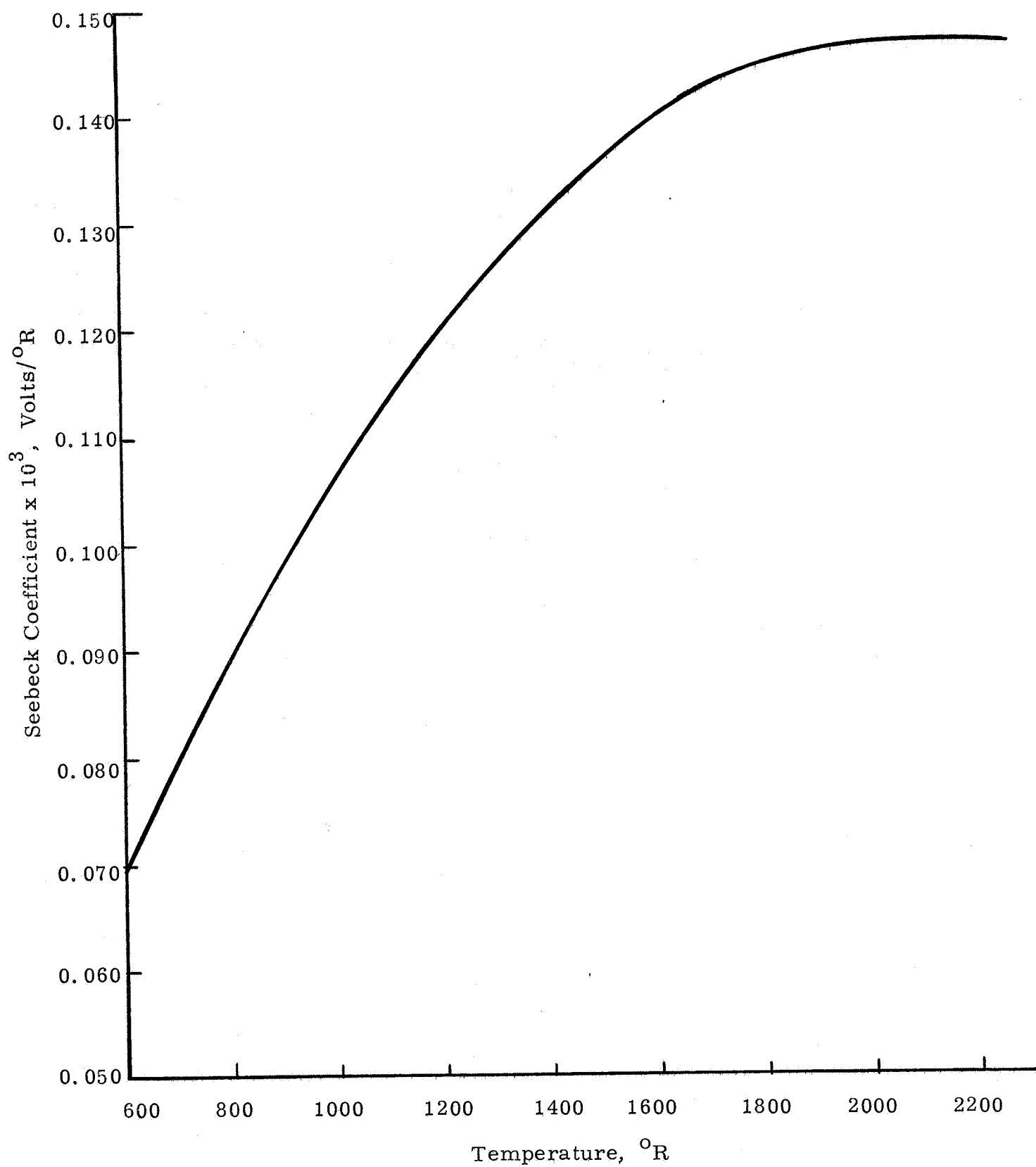


Figure IV-19. SiGe Seebeck Coefficient

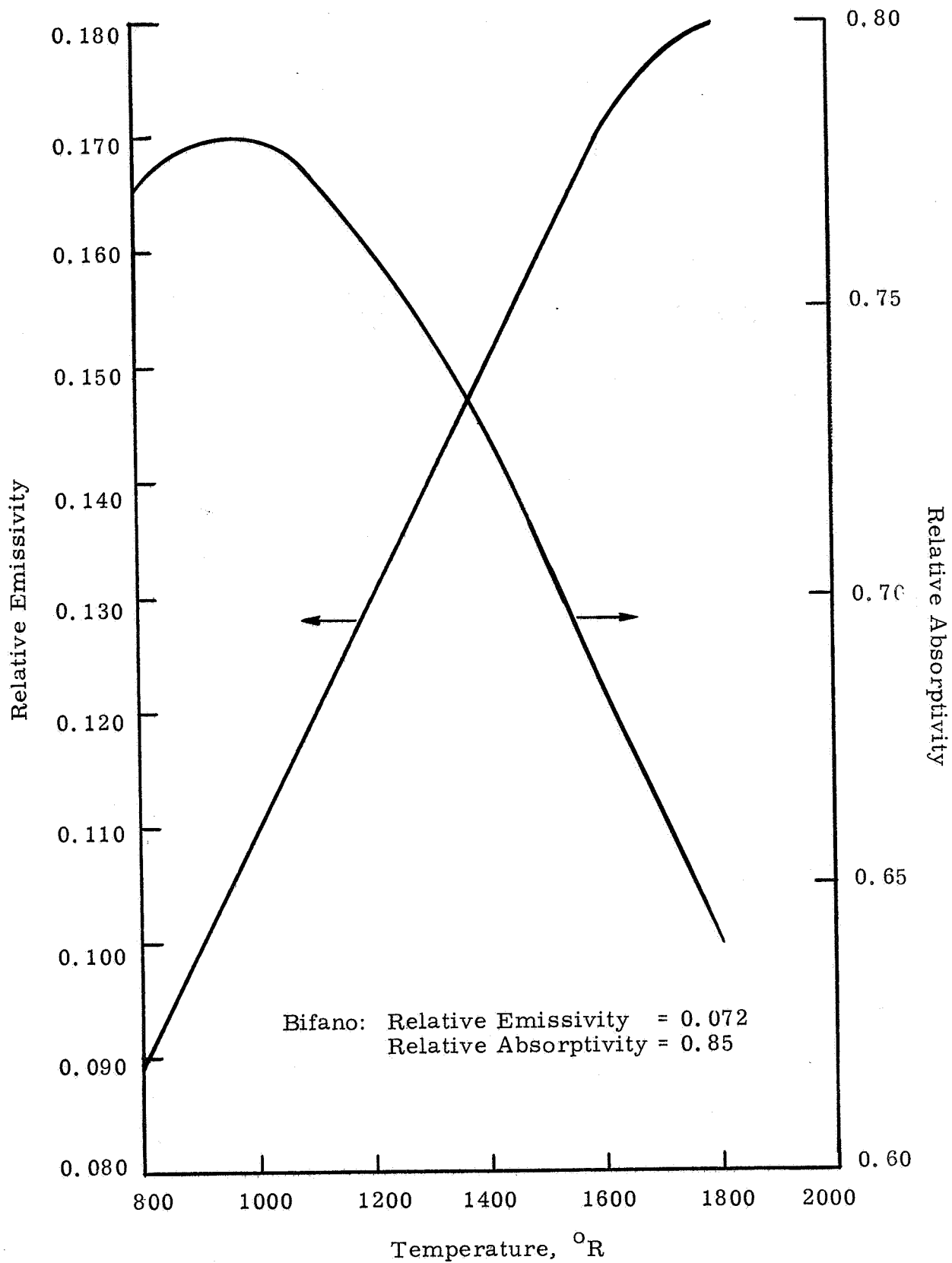


Figure IV-20. Absorber Surface Characteristics

TABLE IV-5. SOLAR FLAT PLATE CHARACTERISTICS FOR  
HIGH TEMPERATURE APPLICATIONS (CONTINUED)

<u>Item</u>	<u>Bifano</u>	<u>Computer Program</u>
Radiator Temperature ( $^{\circ}\text{R}$ )	1000	935.9
Power Density Based Upon Plate Area (watts/ft <sup>2</sup> )	17	11.9

The difference between the calculated power output is due to the decreased temperature difference across the thermoelement which was predicted by the computer program. The reason for the sizeable drop in the collector temperature predicted by the computer program as compared to Bifano's calculations is due to the relative absorptivity of 0.85 which he used compared to the lower values used in the code. A secondary drop in the power output occurs because of the temperature difference in the collector and radiator from the center to the edges. Both of these effects act to decrease the temperature difference across the thermoelement which in turn decreases the output power.

No attempt was made during this investigation to optimize the thermoelement shape with respect to the area to length ratio or the thermoelement dimensions with respect to the plate area. However, the load resistance was varied to assure that the maximum power was being generated. The data presented in Table IV-5 were based upon the best load resistance.

The Case II calculations are summarized in Table IV-6. Again, most of the reason for the difference between the computer predicted power output and that shown by Bifano is due to the lower temperature difference across the thermoelement. This principally results from the lower collector temperature predicted by the computer, but secondarily results from the temperature gradient across the plate which Bifano's calculation could not investigate. Again, the computer results are based upon an optimized power output with respect to variation of the load resistance.

TABLE IV-6. CALCULATED RESULTS FOR SOLAR FLAT PLATE  
AT 0.25AU FROM THE SUN

<u>Item</u>	<u>Bifano</u>	<u>Computer Program</u>
Hot Junction Temperature ( $^{\circ}\text{R}$ )	2060	1909.5
Cold Junction Temperature ( $^{\circ}\text{R}$ )	1245	1208.3
Average Collector Temperature ( $^{\circ}\text{R}$ )	2060	1947.7
Average Radiator Temperature ( $^{\circ}\text{R}$ )	1245	1174.1
Power Density Based on Plate Area (watts/sq. ft.)	49.5	38.1

## V. CONCLUSIONS

An accurate mathematical model of the solar flat plate has been obtained and a computer program written for the IBM-7094 has been utilized to obtain solutions for the mathematical model. The predicted characteristics agree closely with the experimental data which are available. Good correlation between the computer calculations and the previously available manual calculations have been obtained when suitable restrictions were placed upon the model utilized by the computer so that it corresponded to that upon which the manual calculations were based. Full use of the computer code capabilities shows a slight decrease in performance from that predicted for the existing configurations with manual calculations.

A number of calculations are possible with this computer program that previously could not be made. Transient behavior of the solar flat plate which results from movement from shade to sun or vice versa conditions may be predicted. The effect of variations in the absorber and radiator thickness can be determined so that true optimization with respect to weight can be considered.

Computer program input is very simplified and a comprehensive logical analysis of the input data is built into the code. Input data are read independent of format or the manner in which the data are written. Comprehensive parametric studies may be easily obtained because only those data which are changed from a previous run need be read by the computer program. Sufficient storage capability for tabular information has been incorporated into the computer program so that physical property data for several materials of interest may be stored and referred to by the computer as desired.

Although the program in its present form is quite generalized and applicable to most situations, its application to segmented or multiple thermoelements or the automatic calculation of orbital characteristics is not at present possible. These applications have been anticipated in developing the program and sufficient flexibility has been incorporated into the coding techniques so that the modifications may be made in a straightforward manner, if desired, at a later date.

APPENDIX A

DISCUSSION OF THE SELECTION  
OF A  
COORDINATE SYSTEM

## APPENDIX A

### Discussion of the Selection of a Coordinate System

Two basic approaches may be utilized in the solution of a heat conduction problem using numerical techniques. In the first, a differential equation represents the starting point, and the differential equation is expressed as a difference equation. This is solved by using suitable calculation techniques. In the second, the geometry which is to be analyzed is broken up into a number of small volumes, and each volume is considered to be represented by the properties of a point assigned within the volume. Normally, the points are called nodes and are considered as being interconnected to other nodes by conductances. The transient behavior of the volumes is represented by the "capacitance" of the volume, which may be obtained immediately from the heat capacity, the volume and the density. Representation in this technique essentially analyzes the equivalent electrical circuit to the heat conduction problem. Either approach should yield identical results if the analysis is applied properly.

The differential equation applicable to the solar flat plate absorber or radiator is second order with respect to  $x$  and  $y$  in the Cartesian coordinate system. This means that if it is reduced to a difference equation, the temperatures in the steady state representation for a central node and four surrounding nodes will appear if a Taylor series expansion of the second derivative is used and higher order terms are neglected. It is common practice to solve equations of this type using equal spacing between all node centers. Since the absorber or radiator surface area is large compared to the cross sectional area of a thermoelement, a node size that was uniform everywhere would be determined by the minimum node size acceptable along a line of intersection between the thermoelement and the absorber or radiator, because the heat flux is highest at this line. Such an approach would result in a very large number of nodes being required for the analysis of the heat conduction. Although there is nothing wrong with this insofar as accuracy is concerned, the computation time increases very rapidly as the number of nodes is increased because the number of iterations required to attain suitable convergence becomes very large.

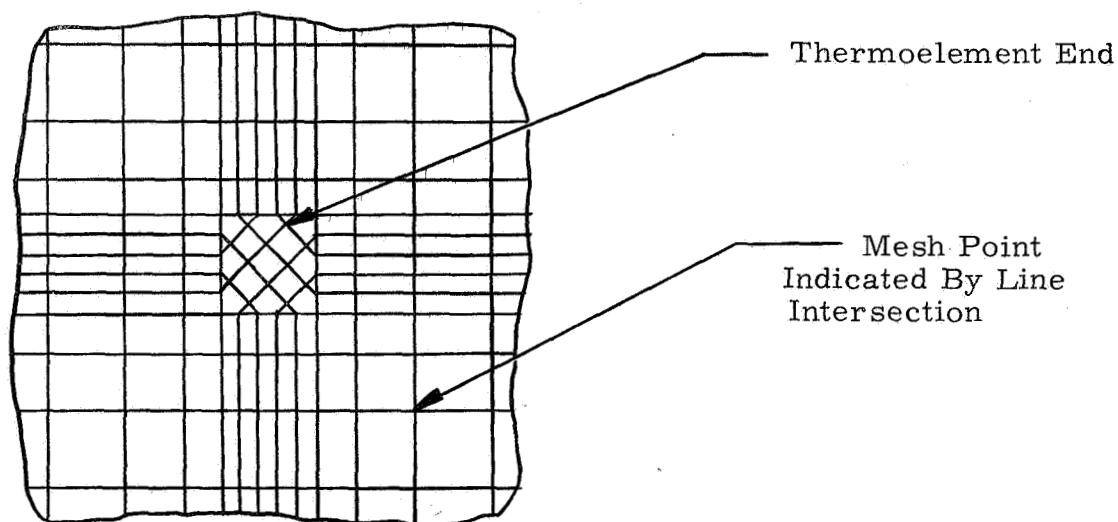
All of the heat absorbed at the hot end of the thermoelement must be conducted to that position from the absorber plate. Since the thermoelement end is small, the assumption was made that the thermoelement temperature at the hot junction could be represented by a single value. It is therefore reasonable to assume that the entire portion of the absorber formed by an extension of the thermoelement peripheral surfaces perpendicular to the plane of the absorber is also a uniform temperature region. This means that all of the heat collected by the absorber, with the small exception of that which is radiated directly to the small uniform region, must be conducted through this "thermoelement peripheral area". The heat fluxes in this region will be quite high, as will the temperature gradients. If an imaginary path is selected which is perpendicular to the periphery of the thermoelement and lies within the absorber, and one investigates temperature gradient while traveling along this path away from the thermoelement, it will be found that the gradient lessens and approaches zero at the edge of the absorber. Since

the gradients are smaller further away from the thermoelement, larger sections of the absorber could be selected to be represented by a single node and an acceptable accuracy would still be attained. This immediately suggests that a numerical technique is desired wherein the size of the section assigned to a node is increased with increasing distance from the thermoelement. This will considerably reduce the required computation time with little or no loss in accuracy. An approach of this type is perhaps most easily understood by considering the electrical analog to heat transfer which was previously mentioned.

In utilizing this approach, it is vitally important to represent each node in a suitably designed configuration. It must be remembered that even though it may appear that a somewhat arbitrary approach is being taken, it is still necessary to mathematically represent the behavior. A haphazard sort of arrangement of interconnecting the nodes will not reduce to the necessary differential equation when the node size is allowed to approach zero. If the differential equation is not realized, then the required numerical solution may also not be realized.

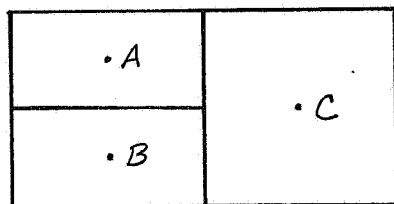
We therefore restrict the discussion for the present time to the selection of rectangular nodes which fit within the rectangular geometry which describes the absorber and connected thermoelement.

If we first consider that a large number of nodes must intersect the thermoelement-absorber intersection, and we wish to reduce the number of nodes as the distance from the thermoelement is increased, an arrangement of the following type can occur:

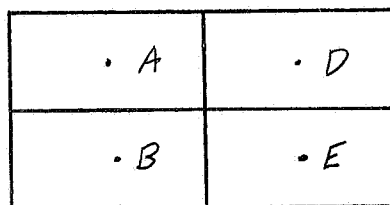


Although this reduces the number of points required for calculation purposes, we notice there are still a large number of extra points not required for accuracy. The ability to eliminate these is also important.

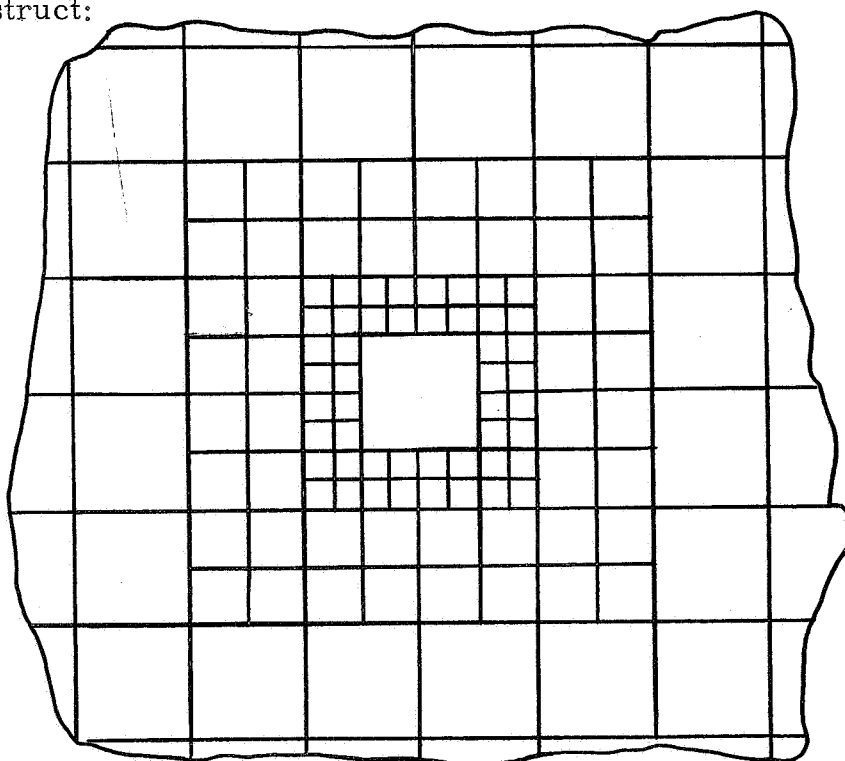
We now consider the following mesh point arrangement and surrounding area:



If the points A, B and C are properly interconnected so that this mesh is the equivalent of:



then we can construct:

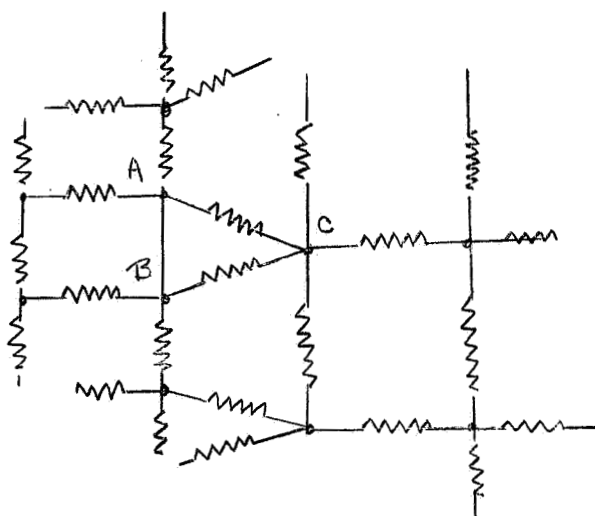




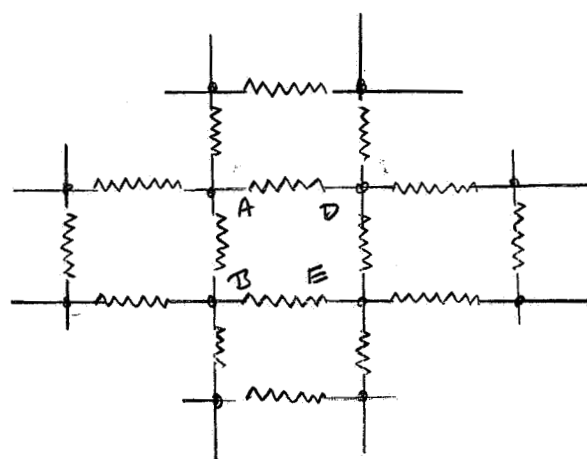
Here we see that the number of points is high near the thermoelement where the heat fluxes are high, and becomes lower as one moves away from the high heat flux area. The accuracy is maintained, and the number of mesh points reduced further.

To introduce "no" error in the system means that the conductances (or resistances) of each of the grids must give identical behavior of the mesh. Further, the capacitances associated with each of the points must also give the "same" overall mesh behavior. In our case, the grid is further complicated by the addition and loss of heat over the surface, which must also be included. Immediately, it is seen that the connecting of the mesh points is not just an arbitrary selection, but must be done carefully with a complete understanding of the heat conduction problem.

For this system to work properly, the electrical circuits shown in the following sketch must be identical:



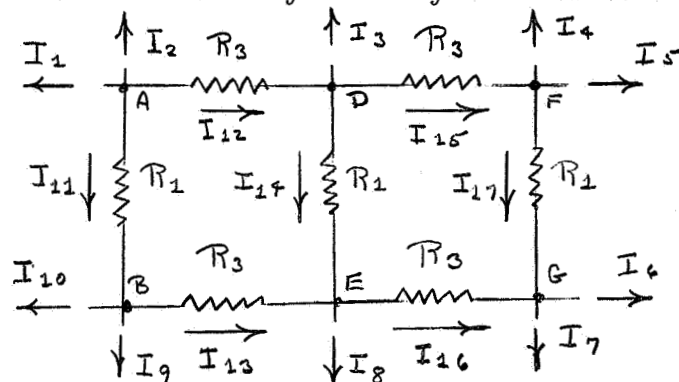
Reduction of Points Mesh

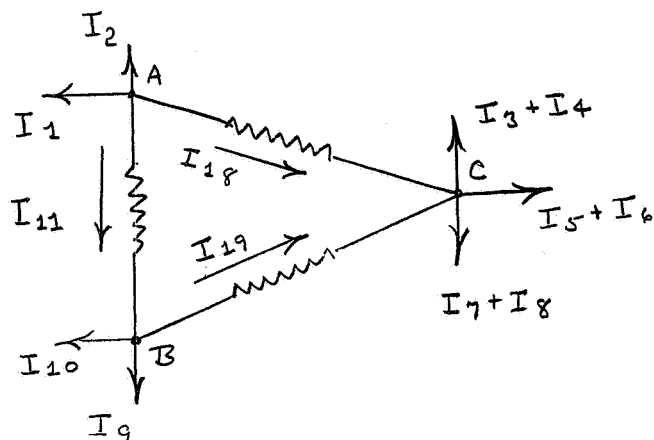


"Exact" Mesh

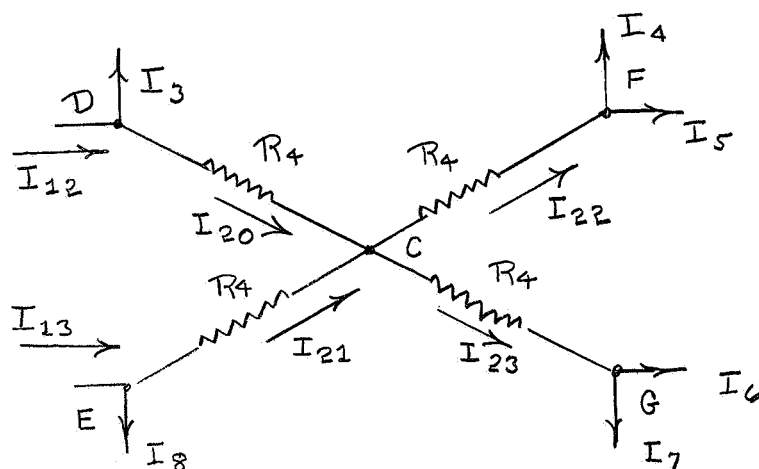
This requires careful calculation of all of the individual connectors, capacitances, etc., a problem generally ignored.

These circuits may be analyzed considering the following configurations:

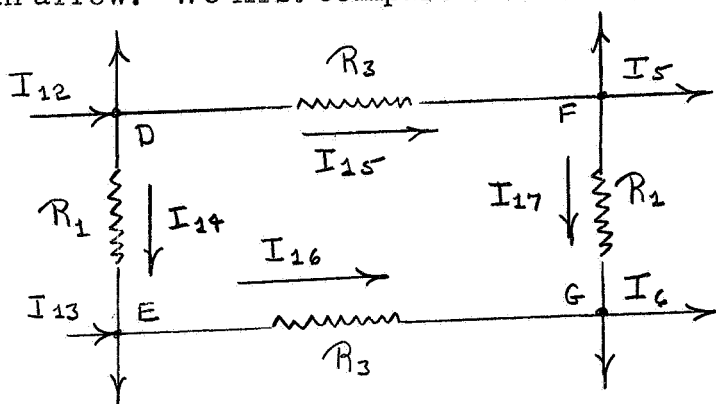
Circuit 1

Circuit 2

With the additional stipulation that the point C be chosen such that:

Circuit 3

Where  $I$  = current,  $R$  = resistance, the first direction of which is indicated by an arrow. We first compare circuit 3 to:

Circuit 4

Immediately we see that:

$$V_D - I_{15} R_3 = V_F \quad (A-1)$$

$$V_D - I_{20} R_4 - I_{22} R_4 = V_F \quad (A-2)$$

Subtracting:

$$-I_{15} R_3 + [I_{20} + I_{22}] R_4 = 0 \quad (A-3)$$

Similarly:

$$I_{14} R_1 + [I_{21} - I_{20}] R_4 = 0 \quad (A-4)$$

$$I_{16} R_3 - [I_{23} + I_{21}] R_4 = 0 \quad (A-5)$$

$$-I_{17} R_1 + [-I_{22} + I_{23}] R_4 = 0 \quad (A-6)$$

Further:

$$I_{20} = I_{14} + I_{15} \quad (A-7)$$

$$I_{22} = I_{15} - I_{17} \quad (A-8)$$

$$I_{21} = -I_{14} + I_{16} \quad (A-9)$$

$$I_{23} = I_{16} + I_{17} \quad (A-10)$$

Using this, Equations (A-3) through (A-6) become:

$$-I_{15} R_3 + [I_{14} + I_{15} + I_{15} - I_{17}] R_4 = 0 \quad (A-11)$$

$$I_{14} R_1 + [-I_{14} + I_{16} - I_{14} - I_{15}] R_4 = 0 \quad (A-12)$$

$$I_{16} R_3 - [I_{16} + I_{17} - I_{14} + I_{16}] R_4 = 0 \quad (A-13)$$

$$-I_{17} R_1 + \left[ -I_{15} + I_{17} + I_{16} + I_{17} \right] R_4 = 0 \quad (A-14)$$

which can be rewritten:

$$I_{15} \left[ 2R_4 - R_3 \right] + I_{14} R_4 - I_{17} R_4 = 0 \quad (A-15)$$

$$I_{14} \left[ -2R_4 + R_1 \right] - I_{15} R_4 + I_{16} R_4 = 0 \quad (A-16)$$

$$I_{16} \left[ -2R_4 + R_3 \right] + I_{14} R_4 - I_{17} R_4 = 0 \quad (A-17)$$

$$I_{17} \left[ 2R_4 - R_1 \right] - I_{15} R_4 + I_{16} R_4 = 0 \quad (A-18)$$

Now subtract Equation (A-17) from (A-15):

$$\left[ I_{15} + I_{16} \right] \left[ 2R_4 - R_3 \right] = 0 \quad (A-19)$$

Immediately:

$$R_4 = \frac{R_3}{2} \quad (A-20)$$

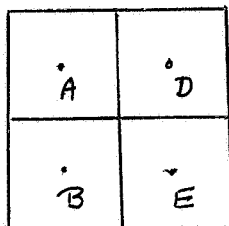
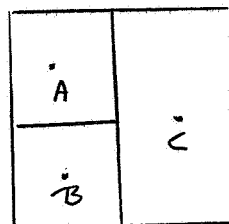
Finally, subtract Equation (A-18) from (A-16):

$$\left[ I_{14} + I_{17} \right] \left[ -2R_4 + R_1 \right] = 0 \quad (A-21)$$

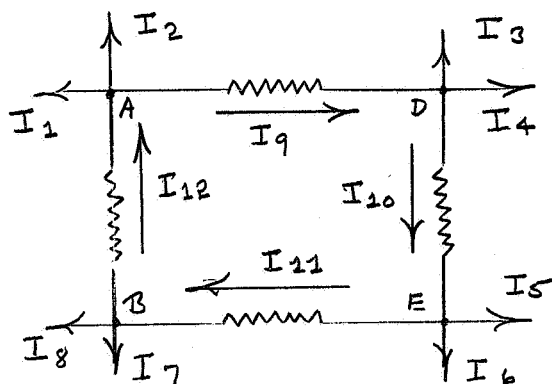
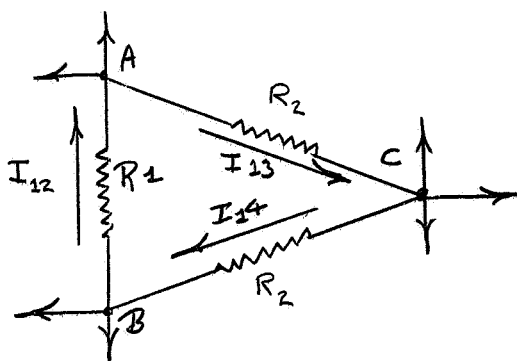
$$R_4 = \frac{R_1}{2} \quad (A-22)$$

But Equations (A-20) and (A-22) contradict unless  $R_1 = R_3$ , which immediately indicates that circuits 3 and 4 cannot be made equivalent except for square arrays, a prohibitive requirement since a non-square rectangular mesh is necessary for the general case. It is concluded that a mesh of the type indicated in configuration 2 cannot be converted to configuration 1 with the connecting point (C) in the center of the block (the circuit 3 requirement).

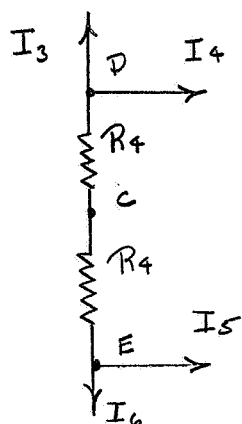
We therefore reject the requirement that we expand in both directions at once, and instead consider the change from configuration 2 to 3:

Configuration 2Configuration 3

which have the equivalent circuits:

Circuit 5Circuit 6

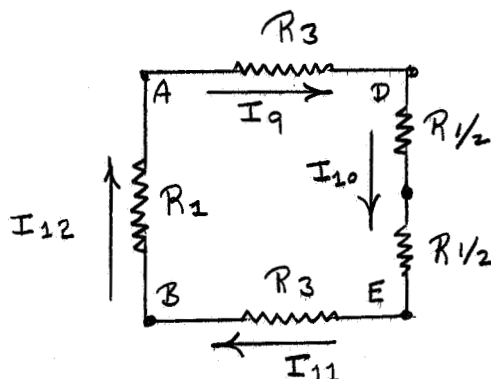
Here we require that:

Circuit 7

since C is at the midpoint of D and E, and further that:

$$R_4 = \frac{R_1}{2} \quad (\text{A-23})$$

which fixes the position of C as in the midpoint of rectangle C. We immediately see that circuits 5 and 7 can be combined to:



Circuit 8

and this gives behavior identical to circuit 5. We now require that circuits 6 and 8 give identical behavior insofar as points A and B are concerned. Hence:

$$V_A - I_{13} R_2 = V_C \quad (\text{A-24})$$

$$V_C - I_{14} R_2 = V_B \quad (\text{A-25})$$

$$I_{13} = I_9 \quad (\text{A-26})$$

$$I_{14} = I_{11} \quad (\text{A-27})$$

$$V_A - I_9 R_3 = V_D \quad (\text{A-28})$$

$$V_D - I_{10} \frac{R_1}{2} = V_C \quad (\text{A-29})$$

$$V_C - I_{10} \frac{R_1}{2} = V_E \quad (\text{A-30})$$

A-11

$$V_E - I_{11} R_3 = V_B \quad (A-31)$$

Now subtract Equation (A-30) from (A-29):

$$V_D - \cancel{I_{10} \frac{R_1}{2}} - V_C + \cancel{I_{10} \frac{R_1}{2}} = V_C - V_E \quad (A-32)$$

$$2V_C = V_D + V_E \quad (A-33)$$

$$V_C = \frac{V_D + V_E}{2} \quad (A-34)$$

(which would be expected).

Now substitute Equations (A-28) and (A-31) in (A-34):

$$V_C = \frac{V_A - I_9 R_3 + V_B + I_{11} R_3}{2} \quad (A-35)$$

Now we use Equations (A-24) through (A-27) with (A-35) to obtain:

$$2V_C = V_C + I_9 R_2 - I_9 R_3 + V_C - I_{11} R_2 + I_{11} R_3 \quad (A-36)$$

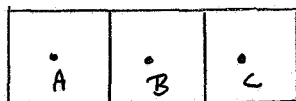
$$I_9 [R_2 - R_3] - I_{11} [R_2 - R_3] = 0 \quad (A-37)$$

$$[I_9 - I_{11}] [R_2 - R_3] = 0 \quad (A-38)$$

which for the most general behavior requires simply that:

$$R_2 = R_3 \quad (A-39)$$

This enables us to expand the mesh in one direction (say, for example, the y direction). We now must derive a technique that will enable expansion in the other direction (the x direction). Hence we require that the following configurations be equivalent:

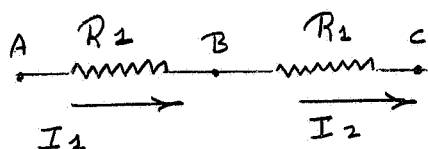


Configuration 4

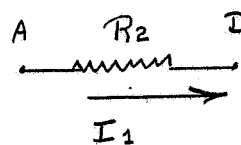


Configuration 5

The equivalent circuits are:



Circuit 9



Circuit 10

Since we are doubling size, and D is to be placed at the midpoint, there results:

$$R_2 = \frac{3R_1}{2} \quad (\text{A-40})$$

as the requirement that positions point D. Further, we see that:

$$V_A - I_1 R_1 = V_B \quad (\text{A-41})$$

$$V_A - I_1 R_2 = V_D \quad (\text{A-42})$$

which combine to form:

$$V_A - I_1 \frac{3R_1}{2} = V_D \quad (\text{A-43})$$

$$\frac{2V_A}{3} - I_1 R_1 = \frac{2V_D}{3} \quad (\text{A-44})$$

$$\frac{V_A}{3} = V_B - \frac{2V_D}{3} \quad (\text{A-45})$$

$$V_A = 3V_B - 2V_D \quad (\text{A-46})$$



$$V_D = \frac{3V_B - V_A}{2} \quad (A-47)$$

which is reasonable weighting and replacing circuit 9 by 10 is therefore reasonable. Similar changes may be made if other than doubling of the mesh size is desired.

This academic type discussion indicates that the node interconnection problem is more complicated than generally realized. It further indicates that if the rectangular type of coordinate system is to be utilized, it is going to be difficult to obtain suitable expansion factors in going from one node size to another so that suitably sized nodes can be fitted to the different requirements of the plate edge as compared to the intersection of the plate with the thermoelement.

It is not absolutely necessary that the electrical analogy hold identically, as implied in the foregoing analysis. If suitable resistances are selected which come close to representing the overall behavior, suitable answers will still be attained. Such a relaxation of the rigorousness of the approach is commonly utilized, but the authors suspect that the assumptions involved are not fully understood.

One of the basic requirements imposed upon the analysis was that the resulting computer program should be "easy to use". The authors concluded early in the study of mesh arrangements that the only information necessary for general prescription of the solar flat plate was the coordinates of the corners of the plate and of the thermoelement along with specifications in regard to the number of mesh points which should be used. A mesh generation technique which was easier to set up in a logical manner than the rectangular approach just discussed was therefore desired. As a result, the rectangular system was rejected in favor of the cylindrical coordinate system.

The cylindrical coordinate system with the origin centered at the thermoelement fits the requirements for an expanding mesh very nicely. The width of "rays" close to the thermoelement is small, and increases with increasing distance from the thermoelement. Selection of increment sizes such that an approximately "square" node results allows the automatic generation of the desired nodal arrangement.

A complication with the cylindrical geometry approach that was not evident in the rectangular geometry approach is the "non-fit" of the geometry to the thermoelement-plate intersection and to the outside edges of the plate. The logic involved in these sections was easier to set up than was the logic involved in generating an expanding generalized node system based upon rectangular geometry. Cylindrical geometry was therefore selected as the basis for the calculations.

APPENDIX B

PROGRAM LISTING

\$IBFTC MAIN LIST,DECK,DD,XR7

~~C FLAT PLATE T/E GENERATOR ANALYSIS~~

~~C C. ANDERSON AND W. LYON, HITTMAN ASSOCIATES, BALTIMORE, MD., 1966.~~

COMMON Z

DIMENSION Z(20000),MA2(12),JS(6),MS(12),ITAB(10),ITABA(30),ICA(200  
2),IGS(200)  
EQUIVALENCE (Z(1),IN),(Z(2),IOUT),(Z(3763),MS(1)),(Z(614),MA2(1)),  
2(Z(827),JS(1)),(Z(13),LINE),(Z(12),IPR),(Z(2443),NP3),(Z(640),  
3NOLU),(Z(663),NP4),(Z(2433),ITAB(1)),(Z(15940),ITABA(1)),(Z(15740)  
4,ICA(1)),(Z(2233),IGS(1)),(Z(3746),JZZ),(Z(11),LERR),(Z(3),IQUIT),  
5(Z(6316),A(1)),(Z(9016),CENX(1)),(Z(9916),CENY(1)),  
6(Z(8116),PLEIN(1)),(Z(10816),NODE(1)),(Z(7216),PLEBAK(1)),  
7(Z(12816),CONDB(1)),(Z(11916),CONDI(1)),(Z(13716),T(1)),  
8(Z(7216),XK(1)),(Z(8116),CAP(1)),(Z(14616),EPSC1(1)),  
9(Z(15066),EPSC2(1)),(Z(15066),EPSR1(1)),(Z(15516),EPSR2(1))

EQUIVALENCE (Z(9016),ALPH(1)),(Z(9916),T4(1)),(Z(16416),TERMF(1)  
1),(Z(10816),TERMO(1)),(Z(4409),TMAP),(Z(5200),LB,L22),  
3(Z(5213),L2),(Z(5214),ITOT,IX),  
4(Z(16416),NOD(1)),(Z(14616),XPRNT(1)),(Z(15516),YPRNT(1)),  
5(Z(4408),XINPUT),(Z(4),K),(Z(4410),ISS)

~~NAMLIST/NAM1/Z~~

C SET VARIABLES FOR PROBLEM USING ALL NEW DATA

MAI

103 DO 260 I=1,20000

260 Z(I)=0.

C SET TAPE DESIGNATIONS, IN IS INPUT, IOUT OUTPUT

MAI

IN=5

MAI

IOUT=6

MAI

C SET VARIABLES FOR PROBLEM USING PARTIAL NEW DATA

MAI

239 DO 3 I=2,6

3 JS(I)=0

MAI

C READ INFORMATION

9 CALL READIT(JZZ)

~~IF(IQUIT)103,210,210~~

C PROCESS NUMBERS FOR CORRECT LOGIC AND PLACE IN CORRECT LOCATIONS.

C JS IS \* INFORMATION, CONTAINS 0 IF NOTHING WAS READ.

210 IF(JS(3)+JS(4))100,4,26

26 CALL PACK

GO TO 4

100 WRITE(IOUT,101)

101 FORMAT(32HOMACHINE ERROR - I GOOFED - MAIN,

11 CALL EXIT

4 DO 1011 I=2,6

GO TO (100,12,14,13,15,16),II

12 IF(JS(2))100,10,6

6 CALL PRO2

18 IF(LERR)100,10,11

17 JZZ=1

IF(K-8)9,103,11

13 IF(JS(5))100,10,29

29 CALL PRO5

GO TO 18

14 IF(JS(6))100,10,30

30 CALL PRO6

GO TO 18

15 IF(JS(3))100,10,31

31 I=JS(4)

JS(4)=0

CALL UNPAK

C SAVE LOWER MEMORY

DO 35 J=1,1580

35 Z(J+10249)=Z(J+3758)

CALL PRO3

C RESTORE LOWER MEMORY

DO 36 J=1,1580

36 Z(J+3758)=Z(J+10249)

JS(4)=1

GO TO 18

16 IF(JS(4))100,10,37

37 I=JS(3)

JS(3)=0

CALL UNPAK

CALL PRO4

JS(3)=1

GO TO 18

10 CONTINUE

IF(XINPUT)310,10,309

310 CALL MESH

IF(LERR)100,251,17

251 CALL MESH(A,CENX,CENY,PLEBAK,PLEIN,NODE,IX,L2)

IF(LERR)100,252,17

252 CALL MESH(B,NODE,A,CENX,CENY,PLEBAK,PLEIN,IX,L2)

CALL MESH(C,CONDB,NODE,CONDI,PLEBAK,CENX,CENY,A,T,PLEIN,IX,L2)

309 IF(TMAP)201,202,201

201 CALL FPMAP(CENX,CENY,NOD,XPRNT,YPRNT,IX,L2)

202 IF(ISS)305,305,304

304 CALL CALCS(XK,CAP,EPSC1,EPSC2,EPSR1,EPSR2,ALPH,T,A,T4,

1CONDI,CONDB,TERMF,TERMO,IX,L2,L22)

GO TO 306

305 CALL CALCT(XK,CAP,EPSC1,EPSC2,EPSC3,EPSC4,ALPH,T,A,T4,

1CONDI,CONDB,TERMF,TERMO,IX,L2,L2)

306 CONTINUE

IF(IQUIT)100,238,11

238 IF(K-6)239,103,11

END

<del>SIBFTC</del>	<del>READIT</del>	<del>LIST,DECK,DD,XR7</del>	
		SUBROUTINE READIT(JZZ)	READIT
		COMMON Z	READIT
		DIMENSION Z(16000),ISS(72),IS(72)	READIT
		<del>EQUIVALENCE (Z(1),IN), (Z(2),IOUT), (Z(3),IQUIT), (Z(4),K), (Z(5),NUM)</del>	<del>READIT</del>
		<del>2, (Z(6),IGN), (Z(7),DEC), (Z(8),NUMB), (Z(9),ZEXP), (Z(10),DECP), (Z(11)</del>	<del>READIT</del>
		<del>3,LERR), (Z(13),LINE), (Z(9900),ISS(1)), (Z(9800),IS(1)), (Z(832),MACH)</del>	<del>READIT</del>
		<del>4, (Z(3747),KB), (Z(7900),IG), (Z(7901),ITE), (Z(7902),TEMP)</del>	<del>READIT</del>
		<del>INTEGER A,AST,ZZ,BLANK,DOT,PLUS,E</del>	<del>READIT</del>
		<del>DATA A,AST,ZZ,BLANK,DOT/1HA,1H*,1HZ,1H ,1H./,PLUS/1H+/,E/1HE/</del>	<del>READIT</del>
		<del>CHECK FOR NORMAL ENTRY,321 IS YES</del>	<del>READIT</del>
		LERR=0	NUMBER
		<del>IF(JZZ)100,321,32</del>	<del>READIT</del>
		321 K=0	READIT
		<del>18 READ (IN,1) (IS(I),I=1,72)</del>	<del>READIT</del>
		1 FORMAT(72A1)	READIT
		<del>WRITE (IOUT,500) (IS(I),I=1,72)</del>	
		500 FORMAT (1H0,72A1)	
		IA=1	READIT
		CHECK FOR *	READIT
		<del>IF (IS(1)-AST)6,3,6</del>	<del>READIT</del>
		CONTROL CARD, DETERMINE TYPE	READIT
		<del>3 IS(2)=IS(2)/1073741824</del>	
		IF (IS(2))9,9,10	READIT
		<del>9 IS(2)=IS(2)*1073741824</del>	
		WRITE(IOUT,500)(IS(I),I=1,72)	
		<del>WRITE(IOUT,12)</del>	<del>READIT</del>
		12 FORMAT(21H011LEGAL CONTROL CARD)	READIT
		<del>322 CALL EXIT</del>	<del>READIT</del>
		10 IF(9-IS(2))9,13,14	READIT
		<del>13 IQUIT=1</del>	<del>READIT</del>
		RETURN	READIT
		<del>14 IF(7-IS(2))16,16,17</del>	<del>READIT</del>
		100 WRITE(IOUT,101)MACH	READIT
		<del>101 FORMAT(42HOMACHINE ERROR IN READIT, STATEMENT NUMBER 14)</del>	<del>READIT</del>
		CALL EXIT	READIT
		<del>16 IQUIT=0</del>	<del>READIT</del>
		RETURN	READIT
		<del>17 K=IS(2)</del>	<del>READIT</del>
		IA=3	READIT
		<del>6 IF(K-1)102,19,20</del>	<del>READIT</del>
		102 WRITE(IOUT,103)	READIT
		<del>103 FORMAT(21HOMISSING CONTROL CARD)</del>	<del>READIT</del>
		CALL EXIT	READIT
		<del>COMMENT CARD WAS READ</del>	<del>READIT</del>
		19 IS(1)=BLANK	READIT
		IS(2)=BLANK	READIT
		WRITE(IOUT,500)(IS(I),I=1,72)	
		LINE=LINE+1	READIT
		GO TO 18	READIT
		<del>CALCULATE NUMERICAL DATA</del>	<del>READIT</del>
		20 NUM=0	READIT
		IGN=0	READIT
		DEC=0.	READIT
		NUMB=0	READIT
		KB=0	READIT
		ZEXP=0.	READIT
		DECP=0.	READIT

JZZ=0

LINE=0

READIT

IF(K-8)410,418,410

410 IF(K-9)400,401,402

READIT

402 MACH=32

READIT

GO TO 100

READIT

418 IQUIT=-1

RETURN

401 CALL EXIT

READIT

C SEARCH FOR NEW PROBLEM

READIT

400 READ(IN,405)IS(1),IS(2)

READIT

IF(IS(1)-AST)400,406,400

READIT

405 FORMAT(2A1)

READIT

406 IS(2)=IS(2)/1073741824

IF(IS(2)-8)408,418,408

408 IF(IS(2)-9)400,401,400

READIT

END

READIT

\$JOB	1202C003 427 N0CJA N0001B1270	
\$EXECUTE	IBJOB	
\$IBJOB	MAP, FIOCS	
\$IBFTC	NUMBER LIST, DECK, DD, XR7	
	SUBROUTINE NUMBER(KB)	NUMBER
C		NUMBER
	COMMON Z	NUMBER
	DIMENSION Z(16000), I2(200), N2(200), P2(200), I3(1600), N3(1600), P3(	NUMBER
	2(1600), I4(1830), N4(1830), P4(1830), N5(730), P5(730), I5(730), N6(600),	NUMBER
	3P6(600), I6(600)	NUMBER
	EQUIVALENCE (Z(2), IOUT), (Z(4), K), (Z(5), NUM), (Z(6), IGN), (Z(7), DEC),	NUMBER
	2(Z(8), NUMB), (Z(9), ZEXP), (Z(10), DECP), (Z(11), LERR), (Z(13), LINE), (Z(	NUMBER
	33575), I2(1)), (Z(3375), N2(1), P2(1)), (Z(828), J2), (Z(829), J3),	NUMBER
	4(Z(830), J4), (Z(831), J5), (Z(832), J6), (Z(5379), I3(1)), (Z(3779), N3(	NUMBER
	51)), (Z(3779), P3(1)), (Z(10000), I4(1)), (Z(11830), P4(1)), (Z(11830),	NUMBER
	6N4(1)), (Z(6991), N5(1)), (Z(6991), P5(1)), (Z(7721), I5(1)), (Z(8451),	NUMBER
	7N6(1)), (Z(8451), P6(1)), (Z(9051), I6(1)), (Z(9651), I), (Z(9652), TEMP),	NUMBER
	8(Z(9653), TEM), (Z(9654), TEMA)	NUMBER
C	SET ERROR RETURN FOR NO ERROR	NUMBER
	LERR=0	
	K=K	NUMBER
C	CHECK FOR NUMBER OR Z	NUMBER
	7 IF(KB)100,3,6	NUMBER
	6 I=-1	NUMBER
	GO TO 5	NUMBER
C	DETERMINE TYPE OF NUMBER	NUMBER
	3 IF(DEC)100,1,2	NUMBER
	100 WRITE(IOUT,101)	NUMBER
	101 FORMAT(24HOMACHINE ERROR IN NUMBER)	NUMBER
	LERR=1	NUMBER
	RETURN	
	1 IF(IGN)2,4,2	NUMBER
C	FIXED POINT NUMBER, SET SIGN	NUMBER
	4 NUMB=NUM*NUMB	NUMBER
	I=0	NUMBER
	GO TO 5	NUMBER
C	FLOATING POINT NUMBER, COMPILE PIECES	NUMBER
	2 TEMP=NUMB	NUMBER
	TEM=NUM	NUMBER
	TEMA=IGN	NUMBER
	PNUMB=(TEMP+DECP)*TEM*10.** (TEMA*ZEXP)	NUMBER
	I=1	NUMBER
C	BRANCH TO TYPE OF INPUT AND STORE	NUMBER
	5 GO TO (100,12,13,14,15,16),K	NUMBER
	12 J2=J2+1	NUMBER
	18 IF(201-J2)102,102,17	NUMBER
	102 WRITE(IOUT,103)K	NUMBER
	103 FORMAT(2H0*I1,45HTABLE OVERFLOW - YOU JUST KAUNT USE THIS MUCH)	NUMBER
	LERR=1	
	19 RETURN	NUMBER
	17 I2(J2)=I	NUMBER
	IF(I)19,20,21	NUMBER
	20 N2(J2)=NUMB	NUMBER
	GO TO 19	NUMBER
	21 P2(J2)=PNUMB	NUMBER
	GO TO 19	NUMBER
	13 J3=J3+1	NUMBER
	22 IF(1410-J3)102,102,23	NUMBER

45 IF(DEC)46,47,147	
46 MACH=45	READIT
GO TO 100	READIT
47 NUMB=NUMB*10	READIT
GO TO 21	READIT
147 DEC=DEC*10	
GO TO 21	
CHECK NEXT VALUE FOR BLANK	READIT
44 IF(72-1)49,62,61	READIT
49 MACH=44	READIT
GO TO 100	READIT
144 IF(72-1)49,145,146	
145 ZEXP=NUMB1	
GO TO 201	
146 ZEXP=NUMB1	
IF(IS(I+1)-BLANK)163,200,163	
163 IF(IS(I+1))189,64,64	
189 IF(IS(I+1)-ZZ)29,92,29	
62 ZEXP=ZEXP*10.	READIT
GO TO 201	READIT
61 IF(IS(I+1)-BLANK)63,200,63	READIT
63 IF(IS(I+1))189,62,64	
64 TEMP=IS(I+1)/1073741824	
ZEXP=10.*ZEXP+TEMP	READIT
GO TO 200	READIT
CHECK FOR Z	READIT
89 IF(IS(I)-ZZ)29,92,29	
92 IF(NUM)93,90,93	
93 CALL NUMBER(KB)	
90 KB=1	READIT
GO TO 34	READIT
CHECK FOR PLUS	READIT
42 IF(IS(I)-PLUS)72,73,72	READIT
C PLUS OR E FOUND - SET EXPONENT	READIT
73 IGN=1	READIT
GO TO 21	READIT
CHECK FOR E, IT WAS NOT PLUS	READIT
72 IF(IS(I)-E)74,73,74	READIT
CHECK FOR DECIMAL POINT, IT WAS NOT E	READIT
74 IF(IS(I)-DOT)76,75,76	READIT
C DECIMAL - SET VALUE	READIT
75 DEC=1.	READIT
GO TO 21	READIT
CHECK FOR NUMBER, NOT DECIMAL	READIT
76 IF(IS(I)-10737418240)77,29,89	
CHECK FOR TYPE OF NUMBER AND SET ACCORDINGLY	READIT
77 IF(IGN)144,177,144	
177 IF(DEC)78,79,80	
78 MACH=77	READIT
GO TO 100	READIT
79 NUMB=10*NUMB+NUMB1	
GO TO 21	READIT
80 DEC=DEC*10.	READIT
TEMP=NUMB1	
DECP=DECP+TEMP/DEC	READIT
21 CONTINUE	READIT
IF(NUM)201,303,201	
201 CALL NUMBER(KB)	READIT
303 IF(LERR)18,18,32	
32 WRITE(1OUT,420)	READIT
420 FORMAT(1H1)	READIT



DO 21 1=1A,72	READIT
ISS(I)=IS(I)	READIT
NUMB1=IS(I)/1073741824	
CHECK FOR PROCESSING A NUMBER	READIT
22 IF(NUM)25,24,25	READIT
CHECK FOR BLANK - NOT A NUMBER	READIT
24 IF(IS(I)-BLANK)26,21,26	READIT
CHECK FOR MINUS - NOT BLANK, ITE=-1 IF -0, =0 IF +0, = +1 OTHERWISE	READIT
26 IG=IS(I)	
CALL CHECK(IG,ITE)	READIT
IF(ITE)91,21,28	
CONTAINS MINUS, SET SIGN	READIT
91 IF(IS(I)-ZZ)29,92,30	
30 NUM=-1	READIT
GO TO 21	READIT
CHECK FOR + OR -	READIT
28 IF(IS(I))89,728,628	READIT
728 MACH=28	READIT
GO TO 100	READIT
CHECK FOR LEGALITY OF POSITIVE IS, FIRST DECIMAL POINT	READIT
628 IF(IS(I)-DOT)36,37,29	READIT
29 IG=I	READIT
DO 327 IH=IG,72	READIT
327 ISS(IH)=IS(IH)	READIT
WRITE(IOUT,500)(ISS(IH),IH=1,72)	
WRITE(IOUT,31)	READIT
31 FORMAT(50H0YOU GOOFED - ILLEGAL CHARACTER ON PRECEEDING CARD)	READIT
CHECK FOR NEW PROBLEM	READIT
GO TO 32	READIT
37 DEC=1.	READIT
NUM=1	READIT
GO TO 21	READIT
CHECK NEXT NUMBER	READIT
36 IF(NUMB1-10)33,29,38	
38 IF(IS(I)-PLUS)29,21,29	READIT
CONTROL SET AND STORAGE OF LEGITIMATE NUMBER	READIT
33 NUM=1	READIT
NUMB=NUMB1	
GO TO 21	READIT
CHECK FOR BLANK IN NUMBER BEING PROCESSED	READIT
25 IF(IS(I)-BLANK)89,34,35	READIT
C BLANK FOUND - END OF NUMBER	READIT
200 CONTINUE	READIT
34 CALL NUMBER(KB)	READIT
301 NUM=0	READIT
IGN=0	READIT
DEC=0.	READIT
NUMB=0	READIT
KB=0	READIT
ZEXP=0.	READIT
DECP=0.	READIT
GO TO 21	READIT
CHECK FOR MINUS, IT WAS NOT BLANK	READIT
35 CALL CHECK(IS(I),ITE)	READIT
IF(ITE)40,41,42	READIT
C MINUS FOUND - SET AS EXPONENT	READIT
40 IGN=-1	READIT
GO TO 21	READIT
CHECK FOR EXPONENT SINCE IS CONTAINS ZERO	READIT
41 IF(IGN)44,45,44	READIT
CHECK FOR DECIMAL - IT WAS NOT EXPONENT	READIT

23	I3(J3)=I	NUMBER
	IF(I)19,24,25	NUMBER
24	N3(J3)=NUMB	NUMBER
	GO TO 19	NUMBER
25	P3(J3)=PNUMB	NUMBER
	GO TO 19	NUMBER
14	J4=J4+1	NUMBER
26	IF(731-J4)102,102,27	NUMBER
27	I4(J4)=I	NUMBER
	IF(I)19,28,29	NUMBER
28	N4(J4)=NUMB	NUMBER
	GO TO 19	NUMBER
29	P4(J4)=PNUMB	NUMBER
	GO TO 19	NUMBER
15	J5=J5+1	NUMBER
30	IF(731-J5)102,102,31	NUMBER
31	I5(J5)=I	NUMBER
	IF(I)19,32,33	NUMBER
32	N5(J5)=NUMB	NUMBER
	GO TO 19	NUMBER
33	P5(J5)=PNUMB	NUMBER
	GO TO 19	NUMBER
16	J6=J6+1	NUMBER
34	IF(601-J6)102,102,35	NUMBER
35	I6(J6)=I	NUMBER
	IF(I)19,36,37	NUMBER
36	N6(J6)=NUMB	NUMBER
	GO TO 19	NUMBER
37	P6(J6)=PNUMB	NUMBER
	GO TO 19	NUMBER
	END	NUMBER

\$IBFTC FPMAP LIST,DECK,REF,DD,XR7

SUBROUTINE FPMAP(CENX,CENY,NOD,XPRNT,YPRNT,IX,L2)

COMMON Z

DIMENSION Z(20000),IB(100),IE(100),XITEM(2),CENX(IX,L2),

1CENY(IX,L2),ZA(200),ZTE(100),XPRNT(IX,L2),YPRNT(IX,L2),

2PRNT(1:13),NOD(IX,L2)

EQUIVALENCE (Z(6116),IB(1)),(Z(6216),IE(1)),(Z(5200),LB),

1(ZA(166),XITEM(1)),

2(Z(627),ZA(1)),(Z(2),IOUT),(Z(11),LERR),

3(Z(17316),ZTE(1)),

4(Z(5425),UC),(Z(5426),VC),

5(Z(5431),ITEMP),(Z(5427),RS),(Z(5418),THE),(Z(5417),IJ),

6(ZA( 2 ),Y),

7(Z(5415),II),(Z(5429),XS),(Z(5430),YS),

8(Z(5414),DY),(ZA(1),X),(Z(5413),DX)

EQUIVALENCE (Z(5716),PRNT(1)),(Z(5410),B),

1(Z(5409),D),(Z(5408),C),(Z(5407),G),(Z(5406),LINE),

2(Z(5405),LU),(Z(5404),LL),(Z(5403),I),(Z(5402),IC),(Z(5401),IA),

3(Z(5400),J),(Z(5428),NT),(Z(5432),NODA),(Z(5211),DELX),

4(ZA(4),GO),(ZA(5),VO),(ZA(6),U),(ZA(7),V)

DATA LANK,JPR,MINU,MINI/1H,1HX,1H-,1HI/

INTEGER PRNT,XPRNT,YPRNT,XPRN,YPRN

12 FORMAT(1H1)

10 FORMAT(1H-,12A1,I4)

11 FORMAT( 32H1COLLECTOR AND RADIATOR NODE MAP/112H0 1234567891111111

111112222222222333333333344444444445555555556666666666777777777788

28888888899999999991111111111/11X,9(10H0123456789),11H000000000001/1

301X,11H01234567890)

13 FORMAT(1H0,5X,3HMAP,12X,3HX-Y,18X, 14X,4HNODE)

235 FORMAT(12HCOORDINATES,5X,11HCOORDINATES,12X, 6H  
1NUMBER)

30 FORMAT(3X,4HNODE,13X,5HX-Y-Z)

234 FORMAT(8H NUMBER,10X,11HCOORDINATES)

32 FORMAT(16,3E12.5)

18 FORMAT(2I6,2E12.4,I7)

RSO=0.

RS=0.

SSO=0.

SS=0.

XITEM(2)=0.

IMAX=XITEM(1)

ZTE(1)=DELX/2.

DO 801 I=2,IMAX

ZTE(I)=ZTE(I-1)+DELX

801 CONTINUE

NODA=XITEM(1)+XITEM(2)

NT = 1+NODA

DO 1020 J=1,LB

IA=IB(J)

IC=IE(J)

DO 1021 I=IA,IC

NOD(I,J)=NT

1021 NT=NT+1

1020 CONTINUE

NT=NT+1

LL=LB+1

LU=2\*LB

DO 1022 J=LL,LU

IA= IB(J)

IC= IE(J)

DO 1023 I=IA,IC

```

      .NOD(I,J)=NT
1023 NT=NT+1
1022 CONTINUE
      111 WRITE(IOUT,11)
          LINE=5
          G=ABS((VO-V)/10.)
          C=(VO+V)/2.
          D=(UO+U)/2.
          B=ATAN(C/D)*57.296
          DO 24 I=1,111
              PRNT(I+1)=MINU
24 CONTINUE
      26 PRNT(1)=MINU
          PRNT(113)=0
          WRITE(IOUT,10)(PRNT(II),II=1,113)
          PRNT(113)=1
      27 PRNT(1)=MINI
          PRNT(112)=MINI
          DX=X/110.
          DY=1.71*DX
          YS=Y+DY
          IJ=1
112 YS=YS-DY
          XS=-DX
          II=1
103 XS=XS+DX
          J=1
102 I=IB(J)
101 IF(ABS(YS-CENY(I,J))-DY)1201,201,200
1201 IF(YS-GE-CENY(I,J)) GO TO 201
200 PRNT(II+1)= LANK
      GO TO 1
201 IF(ABS(XS-CENX(I,J))-DX)1202,202,200
1202 IF(XS-LE-CENX(I,J)) GO TO 202
      GO TO 200
202 PRNT(11+1)=JPR
      XPRNT(I,J)=II
      YPRNT(I,J)=IJ
      GO TO 3
1 IF(I-IE(J))104,2,2
104 I=I+1
      GO TO 101
2 IF(J-LB)105,3,3
105 J=J+1
      GO TO 102
3 IF(II-110)106,107,107
106 II=II+1
      GO TO 103
107 WRITE(IOUT,10)(PRNT(II),II=1,113)
      PRNT(113)=PRNT(113)+1
      IF(YS)4,4,108
108 IF(LINE-55)110,109,109
110 LINE=LINE+1
113 IJ=IJ+1
      GO TO 112
109 WRITE(IOUT,11)
      LINE=5
      GO TO 113
4 I=1
      DO 339 I=1,112
339 PRNT(I)=MINU

```

PRNT(113)=0

WRITE(IOUT,10)-(PRNT(I),I=1,113)

41 IJJ=IJ

IF(LINE-30)124,123,123

123 WRITE(IOUT,12)

LINE=0

124 WRITE(IOUT,13)

WRITE(IOUT,235)

LINE=LINE+3

IJ=1

122 II=1

121 J=1

120 I=IB(J)

119 IF(XPRNT(I,J)-II)16,116,16

116 IF(YPRNT(I,J)-IJ)16,117,16

117 IF(ABS(CENX(I,J)-D)-1.0E-4)118,118,156

156 THE=ATAN((CENY(I,J)-C)/(CENX(I,J)-D))\*57.296-B

IF(CENX(I,J)-D)321,321,322

321 IF(CENY(I,J)-C)328,330,324

330 IF(CENX(I,J)-D)324,324,131

324 THE=THE+360.

GO TO 323

328 IF(THE)324,323,323

322 IF(CENY(I,J)-C)340,330,340

340 THE=THE+180.

323 GO TO 315

118 THE=ATAN(D/C)\*57.296

130 IF(CENY(I,J)-C)315,131,131

131 THE=180.+THE

315 CONTINUE

15 THEP=THE/57.296

IF(ABS(CENY(I,J)-C)-G)150,150,151

150 RS=(CENX(I,J)-D)/COS(THEP+B)

GO TO 17

151 RS=(CENY(I,J)-C)/SIN(THEP+B)

RS=ABS(RS)

17 CONTINUE

IF(LINE-55)153,152,152

152 WRITE(IOUT,12)

WRITE(IOUT,13)

WRITE(IOUT,235)

LINE=2

153 WRITE(IOUT,18)II,IJ,CENX(I,J),CENY(I,J),

NOD(I,J)

LINE=LINE+1

GO TO 158

16 IF(IE(J)-I)155,155,154

154 I=I+1

GO TO 119

155 IF(LB-J)158,158,157

157 J=J+1

GO TO 120

158 IF(110-II)159,159,160

160 II=II+1

GO TO 121

159 IF(IJJ-IJ)23,23,161

161 IJ=IJ+1

GO TO 122

23 IF(LINE-30)162,163,163

163 LINE=0

WRITE(IOUT,12)

162 WRITE(IOUT,31)

```

WRITE(IOUT,30)
WRITE(IOUT,234)
LINE=LINE+4
I=1
ITEMP=XITEM(1)
UC=(UO+U)/2.
VC=(VO+V)/2.
RC=(RSO+RS)/2.
SC=(SSO+SS)/2.
133 IF(LINE-55)34,165,165
165 WRITE(IOUT,12)
WRITE(IOUT,30)
WRITE(IOUT,234)
LINE=2
34 WRITE(IOUT,32)I,UC,VC,ZTE(I)
IF(ITEMP-I)167,167,166
166 I=I+1
GO TO 133
167 IF(XITEM(2))39,39,36
36 ITEMP=XITEM(1)+XITEM(2)
I=XITEM(1)+1.
135 IF(LINE-55)37,168,168
168 WRITE(IOUT,12)
WRITE(IOUT,30)
WRITE(IOUT,234)
LINE=2
37 WRITE(IOUT,32)I,RC,SC,ZTE(I)
IF(ITEMP-I)39,39,169
169 I=I+1
GO TO 135
31 FORMAT( 31H0 THERMOELEMENT MESH INFORMATION)
39 RETURN
END

```

SIBFTC PRO2 LIST,DECK,DD,XR7  
SUBROUTINE PRO2

COMMON Z

EQUIVALENCE (Z(2),IOUT),(Z(11),LERR),(Z(3575),I2(1)),(Z(3375),N2(1)),  
(Z(614),M1),(Z(615),M2),(Z(616),M3),(Z(617),M4),  
3(Z(618),M5),(Z(619),M6),(Z(620),M7),(Z(621),M8),(Z(622),M9),(Z(623),  
4),M10),(Z(624),M11),(Z(625),M12),(Z(626),CTH),(Z(627),ZA(1)),(Z(82  
58),J2),(Z(3375),P2(1)),(Z(614),MA2(1)),(Z(3763),MS(1)),(Z(3762),I  
6,(Z(3761),N), (Z(3760),NA),(Z(3759),NZ)

DIMENSION Z(16000)

DIMENSION I2(200),N2(200),P2(200),ZA(200),MS(12),MA2(12)

NAMELIST /NAM1/CTH/NAM2/ZA

M6=0

DO 170 I=1,12

MS(I)=MA2(I)

170 MA2(I)=0

LERR=0

I=1

17 IF(I2(I))13,14,31

13 WRITE(IOUT,15)

15 FORMAT(27H0ILLEGAL CONTROL IN #2 DATA)

30 LERR=1

25 RETURN

14 IF(N2(I)-1)13,1,16

16 IF(I2-N2(I))13,12,18

18 N=N2(I)

34 GO TO (100,2,3,4,5,6,7,8,9,10,11,12),N

100 WRITE(IOUT,101)

101 FORMAT (32H0MACHINE ERROR - SUBROUTINE PRO2)

GO TO 30

1 I=I+1

36 IF(I2(I)-1)19,20,21

19 IF(I-J2)22,26,27

22 I=I+1

24 IF(I2(I))19,14,31

20 CTH=N2(I)

M1=1

GO TO 19

21 CTH=P2(I)

M1=1

GO TO 19

175 IF(M1)100,27,28

27 WRITE(IOUT,29)

29 FORMAT(43H0INSUFFICIENT #2 DATA - I CANNOT DO PROBLEM)

GO TO 30

31 IF(P2(I)-1.)13,1,32

32 IF(I2.-P2(I))13,12,33

33 N=P2(I)

TEMP=N

35 IF(P2(I)-TEMP)13,34,13

28 IF(N2)100,27,37

37 IF(M3)100,39,38

39 IF(M4)100,27,40

38 IF(M4)100,40,41

41 WRITE(IOUT,42)

42 FORMAT(41H0TOO MUCH #2 DATA - IT CONTRADICTS - GOOF)

GO TO 30

40	IF(M5)100,43,44	PRO2
43	IF(M6)100,27,45	PRO2
44	IF(M6)100,45,41	PRO2
45	IF(M7)100,27,46	PRO2
46	IF(M8)100,27,47	PRO2
47	IF(M9)100,27,48	PRO2
48	IF(M10)100,27,49	PRO2
49	IF(M11)100,27,79	PRO2
79	IF(M12)100,27,200	
200	IF(M6)80,201,80	
201	M6=M60	
80	IF(M7-M6)67,81,67	
81	IF(M9-M6)67,82,67	
82	IF(M10-M6)67,83,67	
83	IF(M3)100,116,115	
115	IF(8+M6-M11)67,117,67	PRO2
116	IF(12+M6-M11)67,117,67	PRO2
117	IF(M11-1000)118,119,119	PRO2
119	WRITE(IOUT,120)	PRO2
120	FORMAT(31H000 MANY MESH POINTS SPECIFIED)	PRO2
	GO TO 30	PRO2
118	CONTINUE	
	RETURN	
2	NA=1	PRO2
	NZ=3	PRO2
	M2=1	PRO2
59	DO 53 II=NA,NZ	PRO2
	I=I+1	PRO2
57	IF(I2(I))54,55,56	PRO2
55	ZA(I1)=N2(I)	PRO2
	GO TO 53	PRO2
56	ZA(I1)=P2(I)	PRO2
	GO TO 53	PRO2
54	WRITE(IOUT,58)	PRO2
58	FORMAT(20H0IMPROPER Z LOCATION)	PRO2
	GO TO 30	PRO2
53	CONTINUE	PRO2
	GO TO 19	PRO2
3	NA=4	PRO2
	NZ=7	PRO2
	M3=1	PRO2
	M60=1	
	GO TO 59	PRO2
4	NA=4	PRO2
	NZ=11	PRO2
	M4=1	PRO2
	M60=2	
	GO TO 59	PRO2
5	NA=12	PRO2
	IF(I2(I+2))60,61,61	
60	NZ=12	PRO2
	M5=1	PRO2
	GO TO 59	PRO2
61	NZ=13	PRO2
	M5=2	PRO2
	GO TO 59	PRO2
6	NA=14	PRO2
	DO 62 II=1,40	PRO2
	JA=I+II+1	PRO2
	IF(I2(JA))63,62,62	PRO2
63	NZ=NA+II-1	PRO2



```

M6=(NZ-13)/2
GO TO 59
62 CONTINUE
M6=20
GO TO 59
7 NA=54
DO 64 II=1,20
JA=I+II+1
IF(I2(JA))65,64,64
65 NZ=NA+II-1
M7=NZ-53
GO TO 59
64 CONTINUE
M7=20
GO TO 59
67 WRITE(IOUT,68)
68 FORMAT(31H0INCONSISTENT *2,DATA - TCH TCH)
GO TO 30
8 M8=1
NA=75
NZ=77
GO TO 59
9 NA=78
DO 69 II=1,40
JA=I+II+1
IF(I2(JA))70,69,69
70 NZ=NA+II-1
M9=(NZ-77)/2
GO TO 59
69 CONTINUE
M9=20
GO TO 59
10 NA=118
DO 72 II=1,40
JA=I+II+1
IF(I2(JA))73,72,72
73 NZ=NA+II-1
M10=(NZ-117)/2
GO TO 59
72 CONTINUE
M10=20
GO TO 59
11 NA=158
DO 90 II=1,32
JA=I+II+1
IF(I2(JA))91,90,90
91 NZ=NA+II-1
M11=NZ-157
GO TO 59
90 CONTINUE
M11=32
GO TO 59
12 NA=190
NZ=193
M12=1
GO TO 59
26 IF(M1)100,171,172
171 M1=MS(1)
172 IF(M2)100,173,174
173 M2=MS(2)
174 IF(M3)100,176,179

```

176 IF(M4)100,178,179

178 M3=MS(3)

M4=MS(4)

179 DO 180 I=5,12

182 IF(MA2(I))100,181,180

181 MA2(I)=MS(I)

180 CONTINUE

GO TO 175

END

PRO2

PRO2

PRO2

PRO2

PRO2

PRO2

PRO2

PRO2

PRO2

SORIGIN

ALPHA

<del>\$IBFTC</del>	PRO3	LIST,DECK,DD,XR7	
		SUBROUTINE PRO3	PRO3
		COMMON Z	
		DIMENSION Z(20000),TES(200),ALPHAS(200),THERMS(200),RESIS(200),	PRO3
		2CAPS(200),DENS(200),EPSIS(200),ICS(200),IZC(10),ITAB(10),IZD(10),	PRO3
		3I3(1600),P3(1600),N3(1600),IC(200),TE(200),ALPHA(200),THERM(200),	PRO3
		4RESIST(200),CAP(200),DEN(200),EPSI(200),IZA(200)	PRO3
		DIMENSION ZA(200),IXB(20),IXE(20),MATTE(15)	
		EQUIVALENCE (Z(12),IPR),(Z(832),MACH),(Z(11),LERR),(Z(833),TES(1))	PRO3
		2,(Z(1033),ALPHAS(1)),(Z(1233),THERMS(1)),(Z(1433),RESIS(1)),(Z(	PRO3
		31633),CAPS(1)),(Z(1833),DENS(1)),(Z(2033),EPSIS(1)),(Z(2233),ICS(	PRO3
		41)),(Z(2433),ITAB(1)),(Z(2443),NP3),(Z(8589),IZC(1)),(Z(8579),	PRO3
		5IZD(1)),(Z(8379),IC(1)),(Z(8179),TE(1)),(Z(7979),ALPHA(1)),(Z(7779	PRO3
		6),THERM(1)),(Z(7579),RESIST(1)),(Z(7379),CAP(1)),(Z(7179),DEN(1)),	PRO3
		7(Z(6979),EPSI(1)),(Z(5379),I3(1)),(Z(3779),P3(1)),(Z(3779),N3(1))	
		8,(Z(3778),I),(Z(3777),N),(Z(3776),TEMP),(Z(3775),IP),(Z(829 ),J3),	PRO3
		9(Z(13661),IZA(1)),(Z(2),IOUT)	
		EQUIVALENCE (Z(17416),IXB(1)),(Z(17436),IXE(1)),(Z(627),ZA(1)),	
		1(ZA(54),MATTE(1))	
		NAMLIST/NAM1/TES/NAM2/ALPHAS/NAM3/THERMS/NAM4/RESIS/NAM5/CAPS/	
		1NAM6/DENS/NAM7/EPSIS/NAM8/ICS	
		LERR=0	PRO3
		IPRO=IPR	PRO3
		IPR=1	PRO3
		1 IF(NP3)2,205,4	
		2 MACH=1	PRO3
		100 WRITE(IOUT,101) MACH	
		101 FORMAT(32HOMACHINE ERROR, PRO3, STATEMENT 14)	PRO3
		GO TO 8	
		4 DO 5 I=1,IPRO	PRO3
		Z(I+9799)=TES(I)	PRO3
		Z(I+9599)=ALPHAS(I)	PRO3
		Z(I+9399)=THERMS(I)	PRO3
		Z(I+9199)=RESIS(I)	PRO3
		Z(I+8999)=CAPS(I)	PRO3
		Z(I+8799)=DENS(I)	PRO3
		Z(I+8599)=EPSIS(I)	PRO3
		5 IZA(I)=ICS(I)	PRO3
		205 DO 94 I=1,6	
		IZC(I)=ITAB(I)	PRO3
		IZD(I)=0	PRO3
		94 ITAB(I)=0	PRO3
		3 I=1	PRO3
		21 IF(I3(I))6,15,10	PRO3
		6 WRITE(IOUT,7)	PRO3
		7 FORMAT(20H0ILLEGAL CONTROL, #3)	
		8 LERR=1	
		9 RETURN	PRO3
		10 IF(P3(I)-1.)6,11,12	PRO3
		12 IF(P3(I)- 6.)13,11,6	
		13 N=P3(I)-1.01	PRO3
		TEMP=N	PRO3
		IF(ABS(P3(I)-TEMP)-.01)110,110,6	
		11 N=P3(I)	PRO3
		110 IC(IPR)=N	PRO3
		GO TO 14	PRO3
		15 IF(N3(I)-1)6,16,17	PRO3
		17 IF(N3(I)- 6)16,16,6	

16	IC(IPR)=N3(I)	PRO3
14	IP=1	PRO3
	I=I+1	PRO3
61	IF(I-J3)19,70,60	PRO3
19	IF(I3(I))20,22,31	PRO3
20	I=I+1	PRO3
	IC(IPR-1)=-1	
	GO TO 21	PRO3
60	MACH=61	PRO3
	GO TO 100	PRO3
22	P3(I)=N3(I)	PRO3
24	GO TO (31,32,33,34,35,36,37),IP	PRO3
31	TE(IPR)=P3(I)	PRO3
26	IP=IP+1	PRO3
	I=I+1	PRO3
	IF(I3(I))23,22,24	PRO3
23	WRITE(IOUT,25)	PRO3
25	FORMAT(12HOBAD *3 DATA)	PRO3
	GO TO 8	PRO3
32	ALPHA(IPR)=P3(I)	PRO3
	GO TO 26	PRO3
33	THERM(IPR)=P3(I)	PRO3
	GO TO 26	PRO3
34	RESIST(IPR)=P3(I)	PRO3
	GO TO 26	PRO3
35	CAP(IPR)=P3(I)	PRO3
	GO TO 26	PRO3
36	DEN(IPR)=P3(I)	PRO3
	CAP(IPR)=CAP(IPR)*DEN(IPR)	
	GO TO 26	PRO3
37	EPSI(IPR)=P3(I)	PRO3
	IPR=IPR+1	PRO3
	IC(IPR)=0	PRO3
	GO TO 14	PRO3
70	IC(IPR-1)=-1	
	IPR=IPR-1	
	IH=2	PRO3
	IM=1	PRO3
	IG=1	PRO3
	TMIN=TE(I)	PRO3
83	DO 42 I=IH,IPR	PRO3
43	IF(IC(I))42,42,45	
45	IJ=I-1	PRO3
	GO TO 51	
42	CONTINUE	PRO3
	IJ=IPR	PRO3
51	DO 40 I=IH,IJ	PRO3
	IF(TMIN-TE(I))40,23,41	
41	TMIN=TE(I)	PRO3
	IM=I	PRO3
40	CONTINUE	PRO3
46	IF(IM-IG)47,50,52	PRO3
47	MACH=46	PRO3
	GO TO 100	PRO3
50	IG=IG+1	PRO3
183	IF(IG-IJ)80,81,182	
80	IH=IG+1	PRO3
	TMIN=TE(IG)	PRO3
	IM=IG	PRO3
	GO TO 51	PRO3
182	MACH=183	

GO TO 100	PRO3
52 TEMP=TE(IG)	PRO3
TE(IG)=TE(IM)	PRO3
TE(IM)=TEMP	PRO3
TEMP=ALPHA(IG)	PRO3
ALPHA(IG)=ALPHA(IM)	PRO3
ALPHA(IM)=TEMP	PRO3
TEMP=THERM(IG)	PRO3
THERM(IG)=THERM(IM)	PRO3
THERM(IM)=TEMP	PRO3
TEMP=RESIST(IG)	PRO3
RESIST(IG)=RESIST(IM)	PRO3
RESIST(IM)=TEMP	PRO3
TEMP=CAP(IG)	PRO3
CAP(IG)=CAP(IM)	PRO3
CAP(IM)=CAP(IG)	PRO3
TEMP=DEN(IG)	PRO3
DEN(IG)=DEN(IM)	PRO3
DEN(IM)=TEMP	PRO3
TEMP=EPSI(IG)	PRO3
EPSI(IG)=EPSI(IM)	PRO3
EPSI(IM)=TEMP	PRO3
GO TO 50	PRO3
81 IF(IJ-IPR)82,85,84	PRO3
82 IG=IG+1	
IH=IG+1	
IM=IG	
TMIN=TE(IM)	PRO3
GO TO 83	PRO3
84 MACH=81	PRO3
GO TO 100	PRO3
85 CONTINUE	
C ALL INFO IN TABLES AND ARRANGED IN INCREASING TEMPERATURE ORDER	PRO3
C NOW PICK OUT TABLE NUMBERS	PRO3
DO 87 I=1,IPR	PRO3
IF(IC(I))87,87,88	PRO3
88 IM=IC(I)	PRO3
89 IF(IZD(IM))90,91,92	PRO3
90 MACH=89	PRO3
GO TO 100	PRO3
91 IZD(IM)=I	PRO3
GO TO 87	PRO3
92 WRITE(1OUT,93)	PRO3
93 FORMAT(27H0DUPLICATE TABLE NUMBER, *3)	
GO TO 23	PRO3
87 CONTINUE	PRO3
C MERGE TABLES TO COMPLETED FORM	PRO3
IA=1	PRO3
DO 95 I=1,6	
IH=1	PRO3
IF(IZC(I))96,97,98	PRO3
96 MACH=95	PRO3
GO TO 100	PRO3
97 IF(IZD(I))96,95,99	PRO3
99 IG=IZD(I)	PRO3
ITAB(I)=IA	PRO3
104 TES(IA)=TE(IG)	PRO3
ALPHAS(IA)=ALPHA(IG)	PRO3
THERMS(IA)=THERM(IG)	PRO3
RESIS(IA)=RESIST(IG)	PRO3
CAPS(IA)=CAP(IG)	PRO3

DENS(IA)=DEN(IG)	PRO3
EPSIS(IA)=EPSI(IG)	PRO3
ICS(IA)=IC(IG)	PRO3
GO TO (102,103,107),IH	PRO3
102 IH=2	PRO3
IA=IA+1	PRO3
IG=IG+1	PRO3
GO TO 104	PRO3
103 IA=IA+1	PRO3
IG=IG+1	PRO3
IF(IC(IG))105,104,107	PRO3
105 IH=3	PRO3
GO TO 104	PRO3
98 IF(IZD(I))96,108,99	PRO3
108 IG=IZC(I)	PRO3
ITAB(I)=IA	PRO3
114 TES(IA)=Z(IG+9799)	PRO3
ALPHAS(IA)=Z(IG+9599)	PRO3
THERMS(IA)=Z(IG+9399)	PRO3
RESIS(IA)=Z(IG+9199)	PRO3
CAPS(IA)=Z(IG+8999)	PRO3
DENS(IA)=Z(IG+8799)	PRO3
EPSIS(IA)=Z(IG+8599)	PRO3
ICS(IA)=IZA(IG)	PRO3
GO TO (112,113,107),IH	PRO3
112 IH=2	PRO3
IA=IA+1	PRO3
IG=IG+1	PRO3
GO TO 114	PRO3
113 IA=IA+1	PRO3
IG=IG+1	PRO3
IF(IC(IG))115,114,107	PRO3
115 IH=3	PRO3
GO TO 114	PRO3
107 IA=IA+1	PRO3
95 CONTINUE	PRO3
IA=IA-1	PRO3
IF(200-IA)117,118,118	PRO3
117 WRITE(IOUT,119) IA	
119 FORMAT(I4,17HTABLE ENTRIES, *3)	
GO TO 23	PRO3
118 IPR=IA	PRO3
NP3=1	PRO3
IM=1	
NSG=1	
1001 IX=1	
1006 XXX=ICS(IX)	
NN=53+IM	
IF(ABS(XXX-ZA(NN))-1.E-5)1003,1003,1002	
1002 IF(IX-200)1004,1005,1005	
1004 IX=IX+1	
GO TO 1006	
1005 WRITE(IOUT,1007)	
1007 FORMAT(21H0 TE MATERIAL MISSING)	
GO TO 8	
1003 NONIE=ICS(IX)	
IXB(NONIE)=IX	
1008 IF(ICS(IX))1010,2081,2081	
2081 IF(IX-200)1009,1010,1010	
1009 IX=IX+1	
GO TO 1008	

1010 IXE(NONIE)=IX

NSG=NSG-1

IM=IM+1

IF(NSG)1001,1011,1001

1011 CONTINUE

RETURN

END

PRO3

\$IBFTC PRO4 LIST,DECK,REF,DD,XR7

SUBROUTINE PRO4

COMMON Z

DIMENSION Z(20000),CONTRA(200),EMISIA(200),TEA(200),ALPHA(200),  
2THERMA(200),RESIA(200),CAPA(200),DENA(200),EMISOA(200),IZA(200),  
3ICA(200),IZC(30),ITABA(30),IZD(30),I4(1830),P4(1830),N4(1830),  
4TE(200),THERM(200),CAP(200),DEN(200),RESIST(200),ALPHA(200),EMISO  
5(200),EMISI(200),CONTR(200),IC(200)

DIMENSION IXB(20),IXE(20),ZA(200)

EQUIVALENCE (Z(832),MACH),(Z(11),LERR),(Z(14),CONTRA(1)),(Z(214),  
2EMISIA(1)),(Z(414),TEA(1)),(Z(2446),ALPHA(1)),(Z(2646),THERMA(1))  
3,(Z(2846),RESIA(1)),(Z(3046),CAPA(1)),(DENA(1 ),Z(3246)),(Z(3446  
4),EMISOA(1)),(Z(5000 ),ICA(1)),(Z(8169),IZA(1)),(Z(8139),IZC(1)),  
5(Z(8109),IZD(1)),(Z(15940),ITABA(1)),(Z(10000),I4(1)),(Z(11830),  
6P4(1)),(Z(11830),N4(1)),(Z(8108),I),(Z(8107),N),(Z(8106),TEMP),  
7(Z(8105),IPRO),(Z(662),IPR4),(Z(663),NP4),(Z(8104),IP),(Z(8103),  
8NTEMP),(Z(830 ),J4),(Z(7900),THERM(1)),(Z(7700),CAP(1)),(Z(7500),  
9DEN(1)),(Z(7300),RESIST(1)),(Z(7100),ALPHA(1))

EQUIVALENCE (Z(6900),EMISO(1)),(Z(6700),EMISI(1)),(Z(6500),CONTR  
2(1)),(Z(8102),IH),(Z(8101),IM),(Z(8100),IG),(Z(6499),TMIN),(Z(6299  
3),TE(1)),(Z(6298),IJ),(Z(6297),IA),(Z(5997),IC(1)),(Z(2),IOUT)

EQUIVALENCE (Z(4415),MATC),(Z(4416),MATR),(Z(17416),IXB(1)),  
1(Z(17436),IXE(1)),(Z(627),ZA(1)),(ZA(79),MH),(ZA(119),MC)

NAMelist/NAM1/ICA/NAM2/TEA/NAM3/THERMA/NAM4/CAPA/NAM5/DENA/  
1NAM6/RESIA/NAM7/CONTRA/NAM8/ALPHA/NAM9/EMISIA/NAM10/EMISOA

NAMelist/NAM11/IXB/NAM12/IXE

LERR=0

IPRO=IPR4

IPR4=1

1 IF(NP4)2,205,4

2 MACH=1

100 WRITE(IOUT,101) MACH

101 FORMAT(32HOMACHINE ERROR, PRO4, STATEMENT 14)

GO TO 8

4 DO 5 I=1,IPRO

Z(I+9799)=TEA(I)

Z(I+9599)=ALPHA(I)

Z(I+9399)=THERMA(I)

Z(I+9199)=RESIA(I)

Z(I+8999)=CAPA(I)

Z(I+8799)=DENA(I)

Z(I+8599)=EMISOA(I)

Z(I+8399)=EMISIA(I)

Z(I+8199)=CONTRA(I)

5 IZA(I)=ICA(I)

205 DO 94 I=1,30

IZC(I)=ITABA(I)

IZD(I)=0

94 ITABA(I)=0

3 I=1

21 IF(I4(I))6,15,10

6 WRITE(IOUT,7)

7 FORMAT(20H0ILLEGAL CONTROL, \*4)

8 LERR=1

9 RETURN

10 IF(P4(I)-1.)6,11,12

12 IF(P4(I)-30.)13,11,6

13 N=P4(I) +.01

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110	IC (IPR4)=N	
	GO TO 14	PRO4
15	IF(N4(I)-1)6,16,17	PRO4
17	IF(N4(I)-30)16,16,6	PRO4
16	IC (IPR4)=N4(I)	
	NTEMP=N4(I)	PRO4
14	IP=1	PRO4
	I=I+1	PRO4
61	IF(I-J4)19,70,60	PRO4
19	IF(I4(I))20,22,31	PRO4
20	I=I+1	PRO4
	IC(IPR4-1)=-1	
	GO TO 21	PRO4
60	MACH=61	PRO4
	GO TO 100	PRO4
402	GO TO (31,32,33,34,35,36,37,38),IP	PRO4
31	TE(IPR4)=P4(I)	PRO4
26	IP=IP+1	PRO4
	I=I+1	PRO4
	IF(I4(I))23,22,24	PRO4
23	WRITE(IOUT,25)	PRO4
25	FORMAT(12H0BAD *4 DATA)	
	GO TO 8	PRO4
32	THERM(IPR4)=P4(I)	PRO4
	GO TO 26	PRO4
33	CAP(IPR4)=P4(I)	PRO4
	GO TO 26	PRO4
34	DEN(IPR4)=P4(I)	PRO4
	CAP(IPR4)=CAP(IPR4)*DEN(IPR4)	
	GO TO 26	PRO4
35	RESIST(IPR4)=P4(I)	PRO4
	GO TO 26	
403	CONTR(IPR4)=ALPHA(IPR4)	PRO4
400	IPR4=IPR4+1	
	IC (IPR4)=0	
	GO TO 14	PRO4
22	P4(I)=N4(I)	PRO4
24	IF(NTEMP-21)402,403,403	PRO4
36	ALPHA(IPR4)=P4(I)	PRO4
	IF(NTEMP-10)403,403,26	PRO4
37	EMISO(IPR4)=P4(I)	PRO4
	GO TO 26	PRO4
38	EMISI(IPR4)=P4(I)	
	GO TO 400	PRO4
70	IC(IPR4-1)=-1	
	IPR4=IPR4-1	
	IH=2	PRO4
	IM=1	PRO4
	IG=1	PRO4
	TMIN=TE(1)	PRO4
83	DO 42 I=IH,IPR4	PRO4
43	IF(IC (I))42,42,45	
45	IJ=I-1	PRO4
	GO TO 51	
42	CONTINUE	PRO4
	IJ=IPR4	PRO4
51	DO 40 I=IH,IJ	PRO4
	IF(TMIN-TE(I))40,23,41	
41	TMIN=TE(I)	PRO4
	IM=I	PRO4
40	CONTINUE	PRO4

46	IF(IM-IG)47,50,52	PRO4
47	MACH=46	PRO4
	GO TO 100	PRO4
50	IG=IG+1	PRO4
183	IF(IG-IJ)80,81,182	
80	IH=IG+1	PRO4
	TMIN=TE(IG)	PRO4
	IM=IG	PRO4
	GO TO 51	PRO4
182	MACH=183	
	GO TO 100	PRO4
52	TEMP=TE(IG)	PRO4
	TE(IG)=TE(IM)	PRO4
	TE(IM)=TEMP	PRO4
	TEMP=THERM(IG)	PRO4
	THERM(IG)=THERM(IM)	PRO4
	THERM(IM)=TEMP	PRO4
	TEMP=CAP(IG)	PRO4
	CAP(IG)=CAP(IM)	PRO4
	CAP(IM)=TEMP	PRO4
	TEMP=DEN(IG)	PRO4
	DEN(IG)=DEN(IM)	PRO4
	DEN(IM)=TEMP	PRO4
	TEMP=RESIST(IG)	PRO4
	RESIST(IG)=RESIST(IM)	PRO4
	RESIST(IM)=TEMP	PRO4
	TEMP=ALPHA(IG)	PRO4
	ALPHA(IG)=ALPHA(IM)	PRO4
	ALPHA(IM)=TEMP	PRO4
	TEMP=EMISO(IG)	PRO4
	EMISO(IG)=EMISO(IM)	PRO4
	EMISO(IM)=TEMP	PRO4
	TEMP=EMISI(IG)	PRO4
	EMISI(IG)=EMISI(IM)	PRO4
	EMISI(IM)=TEMP	PRO4
	TEMP=CONTR(IG)	PRO4
	CONTR(IG)=CONTR(IM)	PRO4
	CONTR(IM)=TEMP	PRO4
	GO TO 50	PRO4
81	IF(IJ-IPR4)82,85,84	PRO4
82	IG=IG+1	
	IH=IG+1	
	IM=IG	
	TMIN=TE(IM)	PRO4
	GO TO 83	PRO4
84	MACH=81	PRO4
	GO TO 100	PRO4
85	CONTINUE	
	DO 87 I=1,IPR4	PRO4
	IF(IC(I))87,87,88	PRO4
88	IM=IC(I)	PRO4
89	IF(IZD(IM))90,91,92	PRO4
90	MACH=89	PRO4
	GO TO 100	PRO4
91	IZD(IM)=I	PRO4
	GO TO 87	PRO4
92	WRITE(IOUT,93)	PRO4
93	FORMAT(26H0DUPLICATE *4 TABLE NUMBER)	
	GO TO 23	PRO4
87	CONTINUE	PRO4
	IA=1	PRO4

	D0 95 I=1,30	PRO4
	IH=1	PRO4
	IF(IZC(I))96,97,98	PRO4
96	MACH=95	PRO4
	GO TO 100	PRO4
97	IF(IZD(I))96,95,99	PRO4
99	IG=IZD(I)	PRO4
	ITABA(I)=IA	PRO4
104	TEA(IA)=TE(IG)	PRO4
	ALPHAA(IA)=ALPHA(IG)	PRO4
	RESIA(IA)=RESIST(IG)	PRO4
	THERMA(IA)=THERM(IG)	PRO4
	CAPA(IA)=CAP(IG)	PRO4
	DENA(IA)=DEN(IG)	PRO4
	EMISOA(IA)=EMISO(IG)	PRO4
	EMISIA(IA)=EMISI(IG)	PRO4
	ICA(IA)=IC(IG)	PRO4
	CONTRA(IA)=CONTR(IG)	PRO4
	GO TO (102,103,107),IH	PRO4
102	IH=2	PRO4
	IA=IA+1	PRO4
	IG=IG+1	PRO4
	GO TO 104	PRO4
103	IA=IA+1	PRO4
	IG=IG+1	PRO4
	IF(IC(IG))105,104,107	PRO4
105	IH=3	PRO4
	GO TO 104	PRO4
98	IF(IZD(I))96,108,99	PRO4
108	IG=IZC(I)	PRO4
	ITABA(I)=IA	PRO4
114	TEA(IA)=Z(IG+9799)	PRO4
	ALPHAA(IA)=Z(IG+9599)	PRO4
	THERMA(IA)=Z(IG+9399)	PRO4
	RESIA(IA)=Z(IG+9199)	PRO4
	DENA(IA)=Z(IG+8799)	PRO4
	EMISOA(IA)=Z(IG+8599)	PRO4
	EMISIA(IA)=Z(IG+8399)	PRO4
	CONTRA(IA)=Z(IG+8199)	PRO4
	CAPA(IA)=Z(IG+8999)	PRO4
	ICA(IA)=IZA(IG)	PRO4
	GO TO (112,113,107),IH	PRO4
112	IH=2	PRO4
	IA=IA+1	PRO4
	IG=IG+1	PRO4
	GO TO 114	PRO4
113	IA=IA+1	PRO4
	IG=IG+1	PRO4
	IF(IC(IG))115,114,107	PRO4
115	IH=3	PRO4
	GO TO 114	PRO4
107	IA=IA+1	PRO4
95	CONTINUE	PRO4
	IA=IA-1	PRO4
	IF(200-IA)117,118,118	PRO4
117	WRITE(ROUT,119) IA	
119	FORMAT(I4,17HTABLE ENTRIES, *4)	
	GO TO 23	PRO4
118	IPR4=IA	PRO4
	NP4=1	PRO4
	IX=1	PRO4

```

1004 XX=MATC
      XXX=ICA(IX)
      IF(XXX-XX      )1001,1002,1001
1001 IF(IX-200)1003,1005,1005
1003 IX=IX+1
      GO TO 1004
1005 WRITE(IOUT,1006)
1006 FORMAT( 28H0 COLLECTOR MATERIAL MISSING)
1472 LERR=1
      RETURN
1002 IXB(MATC)=IX
1010 IF(ICA(IX))1008,1007,1007
1007 IF(IX-200)1009,1008,1008
1009 IX=IX+1
      GO TO 1010
1008 IXE(MATC)=IX
      IX=1
1014 XX=MATR
      XXX=ICA(IX)
      IF(XXX-XX      )1011,1012,1011
1011 IF(IX-200)1013,1015,1015
1013 IX=IX+1
      GO TO 1014
1015 WRITE(IOUT,1016)
1016 FORMAT( 28H0 RADIATOR MATERIAL MISSING )
      LERR=1
      RETURN
1012 IXB(MATR)=IX
1020 IF(ICA(IX))1018,1017,1017
1017 IF(IX-200)1019,1018,1018
1019 IX=IX+1
      GO TO 1020
1018 IXE(MATR)=IX
      IX=1
1034 XXX=ICA(IX)
      IF(ABS(XXX-ZA(79))-.1.E-5)1032,1032,1031
1031 IF(IX-200)1033,1035,1035
1033 IX=IX+1
      GO TO 1034
1035 WRITE(IOUT,1036)
1036 FORMAT( 27H0 HOT SHOE MATERIAL MISSING)
      GO TO 1472
1032 NONIE=ICA(IX)
      IXB(NONIE)=IX
1040 IF(ICA(IX))1038,1037,1037
1037 IF(IX-200)1039,1038,1038
1039 IX=IX+1
      GO TO 1040
1038 IXE(NONIE)=IX
      IX=1
1054 XXX=ICA(IX)
      IF(ABS(XXX-ZA(119))-.1.E-5)1052,1052,1051
1051 IF(IX-200)1053,1055,1055
1053 IX=IX+1
      GO TO 1054
1055 WRITE(IOUT,1056)
1056 FORMAT(28H0 COLD SHOE MATERIAL MISSING)
      GO TO 1472
1052 NONIE=ICA(IX)
      IXB(NONIE)=IX
1060 IF(ICA(IX))1058,1057,1057

```

1057 IF(IX-200)1059,1058,1058

1059 IX=IX+1

GO TO 1060

1058 IXE(NONIE)=IX

1303 CONTINUE

RETURN

END

PRO4

\$IBFTC-PRO5- LIST,DECK,DD,XR7  
SUBROUTINE PRO5

COMMON Z

DIMENSION Z(16000),I5(730),N5(730),P5(730),TIME(100),TSUR(10),  
2T(100),F1C(100),F1R(100),F2C(100),F2R(100),DELTA(100),TSUR(10)

DIMENSION IXB(20),IXE(20)

EQUIVALENCE (Z(3656),TIME(1)),(Z(8459),ISUR(1)),(Z(3756),T(1)),

2(Z(3856),F1C(1)),(Z(3956),F1R(1)),(Z(4056),F2C(1)),(Z(4156),F2R(1))

3),(Z(4256),DELTA(1)),(Z(6991),N5(1)),(Z(6991),P5(1)),(Z(7721),

4I5(1)),(Z(2),IOUT),(Z(11),LERR),(Z(8451),I),(Z(8452),N),(Z(8453),

5NA),(Z(8454),TEMP),(Z(4356),OLDSOL),(Z(4357),ALBEDO),(Z(4358),

6STEFAN),(Z(8455),TMIN),(Z(8456),J),(Z(4359),ITIME),(Z(8458),K),

7(Z(831),J5),(Z(4360),TSUR(1))

EQUIVALENCE (Z(17416),IXB(1)),(Z(17438),IXE(1))

NAMELIST /NAM1/OLDSOL/NAM2/ALBEDO/NAM3/STEFAN/NAM4/TIME/NAM5/ISUR

1/NAM6/T/NAM7/F1C/NAM8/F1R/NAM9/F2C/NAM10/F2R/NAM11/DELTA

I=1

LERR=0

IF(I5(1))1,3,4

1 WRITE(IOUT,2)

2 FORMAT(19H0ILLEGAL \*5 CONTROL)

22 LERR=1

15 RETURN

5 I=I+1

GO TO (11,12,13,34,34,34,34,34,34,34,21),N

3 N=N5(I)

GO TO 5

4 NA=P5(I)

TEMP=NA

IF(ABS(TEMP-P5(I))-1.E-7)6,6,1

6 N=NA

GO TO 5

11 IF(I5(1))7,8,9

7 WRITE(IOUT,10)

10 FORMAT(19H0YOU GOOFED \*5 DATA)

GO TO 22

8 OLDSOL=N5(I)

17 IF(J5-I)26,28,16

9 OLDSOL=P5(I)

GO TO 17

16 I=I+1

IF(I5(I))17,3,4

12 IF(I5(I))7,18,19

18 ALBEDO=N5(I)

GO TO 17

19 ALBEDO=P5(I)

GO TO 17

13 IF(I5(I))7,20,23

20 STEFAN=N5(I)

GO TO 17

23 STEFAN=P5(I)

GO TO 17

21 CALL HEAT(TIME,T,F1C,F1R,F2C,F2R,DELTA)

TMIN=TIME(1)

DO 24 J=2,100

IF(TMIN-TIME(J))24,7,25

25 IF(TIME(J))7,26,7

```

26 ITIME=J-1
   GO TO 27
24 CONTINUE
   ITIME=100
27 DO 29 J=2,10
29 TSUR(J)=ITIME
   GO TO 17
34 J=N-3
   ISUR(J)=1
45 K=ISUR(J)+100*(J-1)
   IF(I5(I))41,42,43
41 ISUR(J)=ISUR(J)-1
   IF(N-4)17,401,17
401 IXB(17)=1
   IXE(17)=ISUR(J)
   GO TO 17
42 TIME(K)=N5(I)
   GO TO 44
43 TIME(K)=P5(I)
44 I=I+1
   ISUR(J)=ISUR(J)+1
   GO TO 45
28 DO 50 I=1,7
   IF(-ISUR(I))51,51,50
51 WRITE(IOUT,52)
52 FORMAT(27H0PROGRAM NEEDS MORE *5 DATA)
   GO TO 22
50 CONTINUE
   IF(ISUR(1)-2)54,51,54
54 DO 58 I=2,7
   IF(ISUR(I)-1)51,58,56
56 IF(-ISUR(I)-ISUR(1))59,58,59
59 WRITE(IOUT,60)
60 FORMAT(21H0INCONSISTENT *5 DATA)
   GO TO 22
58 CONTINUE
   RETURN
   END

```

SIBFTC PRO6 LIST,DECK,REF,DD,XR7

SUBROUTINE PRO6

COMMON Z

DIMENSION Z(20000),I6(200),P6(200),N6(200)

EQUIVALENCE (Z(9051),I6(1)),(Z(8451),N6(1)),(Z(8451),P6(1)),

1(Z(832),J6),(Z(4400),RE),(Z(4401),SEGM),(Z(4402),TIMAX),

2(Z(4403),TMIN),(Z(4404),TMX),(Z(4405),TIMF),(Z(4406)

3,TIMPL),(Z(4407),TIMPS),(Z(4408),XINPUT),(Z(4409),TMAP),

4(Z(4410),ISS),(Z(4411),EPST),(Z(4412),NL),(Z(4413),NS),

5(Z(4414),TMACH),(Z(4415),MATC),(Z(4416),MATR),(Z(4417),

6NRSTRT),(Z(4418),TINIT),(Z(11),LERR),(Z(5200),LB),(Z(2),IOUT),

7(Z(4419),CFACT)

I=1

LERR=0

20 IF(I-J6)105,105,121

23 WRITE(IOUT,22)

22 FORMAT(11HOPRO6 ERROR)

LERR=1

RETURN

105 IF(I6(I))23,130,131

130 NP=N6(I)

GO TO 132

131 NP=P6(I)+1.E-5

132 I=I+1

GO TO (1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,120,121),NP

1 IF(I6(I))23,30,31

30 RE=N6(I)

GO TO 24

31 RE=P6(I)

24 I=I+1

GO TO 20

2 IF(I6(I))23,32,33

32 SEGM=N6(I)

GO TO 24

33 SEGM=P6(I)

GO TO 24

3 IF(I6(I))23,34,35

34 TIMAX=N6(I)

GO TO 24

35 TIMAX=P6(I)

GO TO 24

4 IF(I6(I))23,36,37

36 TMIN=N6(I)

GO TO 24

37 TMIN=P6(I)

GO TO 24

5 IF(I6(I))23,38,39

38 TMX=N6(I)

GO TO 24

39 TMX=P6(I)

GO TO 24

6 IF(I6(I))23,40,41

40 TIMF=N6(I)

GO TO 24

41 TIMF=P6(I)

GO TO 24

7 IF(I6(I))23,42,43

42 CFACT = N6(I)



```
GO TO 24
43 CFACT = P6(I)
GO TO 24
8 IF(I6(I))23,44,45
44 TIMPL = N6(I)
GO TO 24
45 TIMPL = P6(I)
GO TO 24
9 IF(I6(I))23,46,47
46 TIMPS = N6(I)
GO TO 24
47 TIMPS = P6(I)
GO TO 24
10 IF(I6(I))23,48,49
48 XINPUT = N6(I)
GO TO 24
49 XINPUT = P6(I)
GO TO 24
11 IF(I6(I))23,50,51
50 TMAP = N6(I)
GO TO 24
51 TMAP = P6(I)
GO TO 24
12 IF(I6(I))23,52,53
52 ISS = N6(I)
GO TO 91
53 ISS = P6(I) + .01
91 IF(ISS - 1)92,24,92
92 NP = NP + 4
I = I + 1
GO TO 20
13 IF(I6(I))23,54,55
54 EPST = N6(I)
GO TO 24
55 EPST = P6(I)
GO TO 24
14 IF(I6(I))23,56,57
56 NL = N6(I)
GO TO 24
57 NL = P6(I) + .01
GO TO 24
15 IF(I6(I))23,58,59
58 NS = N6(I)
GO TO 24
59 NS = P6(I) + .01
GO TO 24
16 IF(I6(I))23,60,61
60 TMACH = N6(I)
GO TO 24
61 TMACH = P6(I)
GO TO 24
17 IF(I6(I))23,62,63
62 MATC = N6(I)
GO TO 24
63 MATC = P6(I) + .01
GO TO 24
18 IF(I6(I))23,64,65
64 MATR = N6(I)
GO TO 24
65 MATR = P6(I) + .01
GO TO 24
```

19 IF(I6(I))23,66,67

66 NRSTRT =N6(I)

GO TO 94

67 NRSTRT = P6(I) +.01

94 IF(NRSTRT-1)24,95,24

95 NP=NP+2

I=I+1

GO TO 20

120 IF(I6(I))23,68,69

68 TINIT =N6(I)

GO TO 24

69 TINIT = P6(I)

GO TO 24

121 RETURN

END

SIBFTC HEAT LIST,DECK,DD,XR7

SUBROUTINE HEAT(XA,XB,XC,XD,XE,XF,XG)

COMMON Z

DIMENSION Z(16000)

EQUIVALENCE (Z(2),IOUT)

DIMENSION XA(100),XB(100),XC(100),XD(100),XE(100),XF(100),XG(100)

XA=XB+XC+XD+XE+XF+XG

WRITE(IOUT,1)

1 FORMAT(33H0SUBROUTINE HEAT IS NOT AVAILABLE)

CALL EXIT

RETURN

END

SIBFTC UNPAK LIST,DECK,DD,XR7

SUBROUTINE UNPAK

COMMON Z

DIMENSION Z(16000),KC(8),I3(1600),K3(220),K4(250),I4(1830),KS(250)

2,JS(10)

EQUIVALENCE (Z(5379),I3(1)),(Z(829),J3),(Z(6979),IX),(Z(6980),KA),

2(Z(6981),KB),(Z(6982),KC(1)),(Z(6990),KD),(Z(10250),K3(1)),(Z(830)

3,J4),(Z(10000),K4(1)),(Z(10000),I4(1)),(Z(7979),KS(1)),(Z(827),

4JS(1))

IF(JS(3))313,313,1

1 KB=J3/8

IX=0

DO 310 KA=1,KB

KD=K3(KA)

CALL SPREAD(KC(1),KD)

DO 311 I=1,8

IX=IX+1

311 I3(IX)=KC(I)-1

310 CONTINUE

KA=KB+1

314 IF(IX-J3)312,313,313

312 IX=IX+1

I3(IX)=K3(KA)

KA=KA+1

GO TO 314

313 IF(JS(4))413,413,2

2 KB=J4/8+8

IF(KB-250) 14, 14,315

315 KB=250

14 DO 316 I=1,KB

316 KS(I)=K4(I)

KB=J4/8

IX=0

DO 410 KA=1,KB

KD=KS(KA)

CALL SPREAD(KC(1),KD)

DO 411 I=1,8

IX=IX+1

411 I4(IX)=KC(I)-1

410 CONTINUE

KA=KB+1

414 IF(IX-J4)412,413,413

412 IX=IX+1

I4(IX)=KS(KA)

KA=KA+1

GO TO 414

413 RETURN

END

\$1BFTC PACK LIST,DECK,DD,XR7

SUBROUTINE PACK

COMMON Z

DIMENSION Z(16000),KC(8),K4(250),I4(1830),I3(1600),K3(220),JS(10)

EQUIVALENCE (Z(5379),I3(1)), (Z(829),J3),(Z(830),J4),

2Z(6979),IX),(Z(6980),KA),(Z(6981),KB),(Z(6982),KC(1)),(Z(6990),KD)

3,(Z(10000),K4(1)),(Z(10250),K3(1)),(Z(10000),I4(1)),(Z(827),JS(1))

IF(JS(4))303,303,1

1 KB=J4/8

IX=0

DO 300 KA=1,KB

DO 301 I=1,8

IX=IX+1

301 KC(I)=I4(IX)+1

CALL SQUASH(KC(1),KD)

300 K4(KA)=KD

KA=KB+1

304 IF(IX-J4)302,303,303

302 IX=IX+1

K4(KA)=I4(IX)

KA=KA+1

GO TO 304

303 IF(JS(3))308,308,2

2 KB=J3/8

IX=0

DO 305 KA=1,KB

DO 306 I=1,8

IX=IX+1

306 KC(I)=I3(IX)+1

CALL SQUASH(KC(1),KD)

305 K3(KA)=KD

KA=KB+1

309 IF(IX-J3)307,308,308

307 IX=IX+1

K3(KA)=I3(IX)

KA=KA+1

GO TO 309

308 RETURN

END

\$1BMAP CHECK M94,999

ENTRY CHECK

CHECK CLA\* 3,4

TMI CHK2

TZE CHK1

CLA =1

CHK1 STO\* 4,4

TRA 1,4

CHK2 CLS =1

STO\* 4,4

TRA 1,4

END

\$1BMAP SQUASH M94,999

ENTRY SQUASH

ENTRY SPREAD

SQUASH SXA SQU2,4

CLA 3,4

ADD =8

STA SQU1

PXA 0,0

	AXT	8,4
	ALS	2
SQU1	ORA	** ,4
	TIX	*-2,4,1
SQU2	AXT	** ,4
	STO*	4,4
	TRA	1,4
SPREAD	SXA	UNSQ2,4
	CLA	3,4
	ADD	=8
	STA	UNSQ1
	LDQ*	4,4
	LGL	20
	AXT	8,4
	PXA	0,0
	LGL	2
UNSQ1	STO	** ,4
	TIX	*-3,4,1
UNSQ2	AXT	** ,4
	TRA	1,4
	END	

SIBFTC MESH LIST,DECK,REF,DD,XR7

SUBROUTINE MESH

COMMON Z

DIMENSION Z(20000),PL(8),L(8),PHI(8),DELTH(8),DELTHE(100),CORA(8),  
2CORR(8),LLA(4),THETAX(100),ANGLE(100),KC(100),RIL(100),ROL(100),  
3RI(100),RO(100),ZA(200)

EQUIVALENCE (Z(5428),TEMPB),(Z(5500),L(1)),(Z(5470),PHI(1)),(Z(546  
22),DELTH(1)),(Z(11916),DELTHE(1)),(Z(5700),CORA(1)),(Z(5708),CORR(  
31)),(Z(5458),LLA(1)),(Z(5716),THETAX(1)),(Z(5816),ANGLE(1)),(Z(  
412016),KC(1)),(Z(5916),RIL(1)),(Z(6016),ROL(1)),(Z(11816),RI(1)),  
5(Z(11716),RO(1)),(Z(5454),TEMPA),(Z(627),ZA(1)),(ZA(158),PL(1)),  
6(Z(5453),TEMPC),(Z(5451),TEMPCD),(Z(5452),TEMPE),(Z(5420),TEMPCG),  
7(Z(5422),TEMP),(Z(5427),LA),(Z(5450),NCOUNT),

8(ZA(1),X),(ZA(2),Y),(ZA(4),UO),(ZA(5),VO),(ZA(6),U),(ZA(7),V),  
9(Z(2),IOUT),(Z(11),LERR),(Z(5419),LC),(Z(5418),LD),(Z(5417),PLARGE  
9)

EQUIVALENCE (Z(5416),SUM),(Z(5415),J),(Z(5414),I),(Z(5413),LINE),  
2(Z(5412),THETA),Z(5411),M),(Z(5410),K),(Z(5409),TEMG),(Z(5408),  
3TEMGA),(Z(5207),RMIN),(Z(5201),R),(Z(5406),THETM),(Z(5202),THETAV)  
4,(Z(5203),DELR),(Z(5405),KA),(Z(5404),KB),(Z(5403),TEMPH),(Z(5402  
5),TEMPJ),(Z(5401),TEMPC),(Z(5400),N),

7(Z(5200),LB,L22),(Z(5213),L2),(Z(5214),ITOT,IX),  
8(ZA(166),XIMX),(Z(5208),IMAXP),(ZA(12),XLENGT),(Z(5211),DELX)  
9,(Z(5221),IMAX)

NAMLIST/NAM1/UO,VO,U,V,X,Y,XIMX,XLENGT,PL/NAM2/PHI,DELTH,  
1DELTHE,PL,L/NAM3/PLARGE,LB,DELTHE,NCOUNT/NAM4/CORA,CORR,LLA/  
2 NAM5/R,RMIN,THETAV,DELR/NAM6/RIL,ROL,THETAX

LERR=0

DO 1 I=1,8

L(I)=PL(I)+.01

1 CONTINUE

COMPUTE ANGLES

TEMPA=(UO+U)\*(UO+U)

TEMPB=(VO+V)\*(VO+V)

TEMPC=(2.\*X-UO-U)\*\*2

TEMPCD=(2.\*Y-VO-V)\*\*2

PHI(1)=(4.\*X\*X-TEMPA-2.\*TEMPB-TEMPC)/(-2.\*SQRT((TEMPA+TEMPB)\*(TEMP  
2C+TEMPB)))

IF(ABS(PHI(1)-1.E-3))601,602,602

601 PHI(1)=1.57079

GO TO 603

602 PHI(1)=ATAN(SQRT(1.-PHI(1)\*PHI(1))/PHI(1))

603 IF(PHI(1))200,201,201

200 PHI(1)=3.14159+PHI(1)

201 CONTINUE

PHI(2)=(4.\*Y\*Y-2.\*TEMPC-TEMPB-TEMPCD)/(-2.\*SQRT((TEMPC+TEMPB)\*(TEMP  
2D+TEMPC)))

IF(ABS(PHI(2)-1.E-3))605,606,606

605 PHI(2)=1.57079

GO TO 607

606 PHI(2)=ATAN(SQRT(1.-PHI(2)\*PHI(2))/PHI(2))

607 IF(PHI(2))202,203,203

202 PHI(2)=3.14159+PHI(2)

203 CONTINUE

PHI(3)=(4.\*X\*X-2.\*TEMPCD-TEMPA-TEMPB)/(-2.\*SQRT((TEMPCD+TEMPB)\*(TEMP  
2A+TEMPCD)))

IF(ABS(PHI(3)-1.E-3))609,610,610

609 PHI(3)=1.57079

GO TO 611

610 PHI(3)=ATAN(SQRT(1.-PHI(3)\*PHI(3))/PHI(3))

611 IF(PHI(3))204,205,205

```

204 PHI(3)=3.14159+PHI(3)
205 CONTINUE
    PHI(4)=6.283184-PHI(1)-PHI(2)-PHI(3)
    TEMPE=(U-U0)/(V-V0)
    TEMPG=1./TEMPE
    PHI(5)=2.*ATAN(TEMPE)
    PHI(6)=2.*ATAN(TEMPG)
    PHI(7)=PHI(5)
    PHI(8)=PHI(6)
COMPUTE DELTA THETA
    DO 2 I=1,8
2 DELTH(I)=PHI(I)/PL(I)
    COMPARE DELTA THETAS AND RE-SET POINTS AND DELTA THETAS AS NECESSARY
    DO 3 I=1,4
    IF(DELTH(I)-DELTH(I+4))3,3,4
4 PL(I)=PL(I)*DELTH(I)/DELTH(I+4)
    L(I)=PL(I) +.01
    PL(I)=L(I)
6 DELTH(I)=PHI(I)/PL(I)
3 CONTINUE
CALCULATE WORKING DELTA THETAS
    LB=0
    DO 7 J=1,4
    LA=LB+1
    LB=L(J)+LB
    DO 8 I=LA,LB
8 DELTHE(I)=DELTH(J)
7 CONTINUE
    IF(LB-100)9,9,10
10 WRITE(10OUT,11)
11 FORMAT(32H0NUMBER OF POINTS TABLE EXCEEDED)
    LERR=1
    RETURN
9 NCOUNT=0
31 LC=1
    PLARGE=0.
    LD=0
    THETAX(1)=DELTHE(1)
    DELTHE(1)=(DELTHE(2)+DELTHE(LB))/4. +DELTHE(1)/2.
    SUM=DELTHE(1)
    J=1
18 LA=LC+1
    LC=L(J)+LD
    DO 13 I=LA,LC
    THETAX(I)=DELTHE(I)
    DELTHE(I)=(DELTHE(I+1)+DELTHE(I-1))/4. +DELTHE(I)/2.
13 SUM=SUM+DELTHE(I)
    IF(J-3)21,21,20
21 TEMP=PHI(J)/SUM
    LA=LD+1
    DO 16 I=LA,LC
    DELTHE(I)=TEMP*DELTHE(I)
    IF(PLARGE-ABS(THETAX(I)-DELTHE(I)))14,16,16
14 PLARGE=ABS(THETAX(I)-DELTHE(I))
16 CONTINUE
    IF(J-3)17,19,24
17 LD=LC
30 J=J+1
    SUM=0.
    GO TO 18
19 LD=LC-1

```

GO TO 30	MESH
20 LC=LC+1	MESH
TEMP=DELTHE(LC)	MESH
DELTHE(LC)=(DELTHE(1)+DELTHE(LC-1))/4.+DELTHE(LC)/2.	
IF(PLARGE-ABS(TEMP-DELTHE(LC)))22,23,23	MESH
22 PLARGE=ABS(TEMP-DELTHE(LC))	MESH
23 SUM=SUM+DELTHE(LC)	MESH
LD=LD+1	MESH
GO TO 21	MESH
24 IF(PLARGE-1.E-3)25,25,26	
26 IF(NCOUNT-250)27,27,25	MESH
27 NCOUNT=NCOUNT+1	MESH
GO TO 31	MESH
25 CONTINUE	MESH
CALCULATE ANGLE OF LOWER LEFT T/E CORNER	MESH
TEMP=(UO+U)/2.	MESH
TEMPA=(VO+V)/2.	MESH
THETA=ATAN(TEMP/TEMPA)-PHI(5)/2.	MESH
CALCULATE CORNER LOCATIONS	MESH
CORA(1)=0.	
CORA(2)=PHI(1)	
CORA(3)=PHI(1)+PHI(2)	
CORA(4)=PHI(3)+CORA(3)	
IF(THETA)41,42,42	MESH
41 CORA(5)= THETA+PHI(5)	MESH
CORA(6)=CORA(5)+PHI(6)	MESH
CORA(7)=CORA(6)+PHI(5)	MESH
CORA(8)=6.283184+THETA	MESH
GO TO 43	MESH
42 CORA(5)=THETA	MESH
CORA(6)=THETA+PHI(5)	MESH
CORA(7)=CORA(6)+PHI(6)	MESH
CORA(8)=CORA(7)+PHI(5)	MESH
43 CONTINUE	MESH
COMPUTE CORNER RADII	MESH
CORR(1)=SQRT(TEMP*TEMP+TEMPA*TEMPA)	MESH
CORR(2)=SQRT((X-TEMP)**2+TEMPA**2)	MESH
CORR(3)=SQRT((X-TEMP)**2+(Y-TEMPA)**2)	MESH
CORR(4)=SQRT(TEMP**2+(Y-TEMPA)**2)	MESH
CORR(5)=SQRT(-(U-UO)**2+(V-VO)**2)/4.	
CORR(6)=CORR(5)	MESH
CORR(7)=CORR(5)	MESH
CORR(8)=CORR(5)	MESH
CALCULATE SIDE NUMBERS	MESH
LLA(1)=L(1)	MESH
LLA(2)=L(2)+LLA(1)	MESH
LLA(3)=L(3)+LLA(2)	MESH
LLA(4)=L(4)+LLA(3)	MESH
CALCULATE MINIMUM RADIUS	MESH
TEMG=(U-UO)/2.	MESH
TEMG=(V-VO)/2.	MESH
IF(TEMG-TEMG) 32,32,33	MESH
32 RMIN=TEMG	MESH
GO TO 34	MESH
33 RMIN=TEMG	MESH
34 R=RMIN	MESH
COMPUTE AVERAGE THETAS	MESH
SUM=0.	MESH
DO 35 I=1,LC	MESH
36 SUM=SUM+DELTHE(I)	MESH
35 CONTINUE	MESH



TEMPB=LC	MESH
THETAV=SUM/TEMPB	MESH
CHOOSE DELTA R	MESH
38 DELR=R*THETAV	
CALCULATE INCREMENT NUMBER	MESH
I=1	MESH
C SET UP SUBROUTINE	MESH
CALL POINT(UO,U,TEMPC,TEMPO,0.)	MESH
CALL POINT(VO,V,TEMPC,TEMPO,-1.)	MESH
CALCULATE RADII AT BACK SIDE OF INCREMENTS ALONG THE EDGE (RL) AND	
C ALONG PLATE EDGE (RL) ALSO COMPUTE THETA AT THE SAME POINTS	
TEMPC=ATAN(TEMPA/TEMP)	MESH
TEMPO=COS(TEMPC)	MESH
TEMPE=SIN(TEMPC)	MESH
THETAX(1)=0.	MESH
N=1	MESH
TEMPO=VO	MESH
TEMPE=U	MESH
TEMPO=0.	MESH
TEMPO=X	MESH
LA=LLA(1)	MESH
LB=1	MESH
49 DO 45 K=LB,LA	MESH
TEMPI=-(-SIN(THETAX(K))*TEMPO+COS(THETAX(K))*TEMPE)	MESH
RIL(K)=(TEMPO-TEMP)/TEMPI	MESH
ROL(K)=(TEMPE-TEMP)/TEMPI	MESH
45 THETAX(K+1)=THETAX(K)+ DELTHE(K)	MESH
N=N+1	MESH
LB=LA+1	MESH
LA=LLA(N)	MESH
DO 46 K=LB,LA	MESH
TEMPI=-COS(THETAX(K))*TEMPO+SIN(THETAX(K))*TEMPE	MESH
RIL(K)=(TEMPE-TEMP)/TEMPI	MESH
ROL(K)=(TEMPO-TEMP)/TEMPI	MESH
46 THETAX(K+1)=THETAX(K)+ DELTHE(K)	MESH
IF(N-4) 48,47,47	MESH
48 N=N+1	MESH
LB=LA+1	MESH
LA=LLA(N)	MESH
TEMPO=V	MESH
TEMPE=UO	MESH
TEMPO=Y	MESH
TEMPO=0.	MESH
GO TO 49	MESH
47 M=5	
LB=LA	
DO 70 K=1,LB	
IF(M-9) 502,501,501	
501 CORAA=CORA(5)+6.283184	
IF(THETAX(K)-CORAA) 71,73,72	
502 IF(THETAX(K)-CORA(M)) 71,73,72	
71 ANGLE(K)=0.	
GO TO 70	
72 IF(K-1) 74,74,575	
575 IF(M-9) 75,576,576	
576 ANGLE(K-1)=CORAA	
GO TO 577	
75 ANGLE(K-1)=CORA(M)	
577 M=M+1	
GO TO 70	
73 ANGLE(K)=0.	

M=M+1

70 CONTINUE

IF (M-8) 74, 77, 78 .

77 ANGLE (LB) = CORA (M)

78 CONTINUE

IMAX=X-IMX

IMAXP=IMAX+1

DELX=XLENGT/XIMX

L2=2\*LB

ITOT=900/L2

RETURN

74 WRITE (IOUT, 76)

76 FORMAT ( 17H0 BLOW UP IN MESH )

LERR=1

RETURN

END

MESH

MESH

SIBFTC MESHA LIST,DECK,REF,DD,XR7

SUBROUTINE MESHA(A,CENX,CENY,PLEBAK,PLEIN,NODE,IX,L2)

COMMON Z

DIMENSION Z(20000),ANGLE(100),RIL(100),ROL(100),ZA(200),THETAX(100)

1)A(IX,L2),CENX(IX,L2),CENY(IX,L2),PLEBAK(IX,L2),PLEIN(IX,L2),

2 IB(100),IE(100),NODE(IX,L2),CORR(8),XPNT(100),YPNT(100)

3,XZ(4),YZ(4),AY(6),AX(6),MA(10),CORA(8)

EQUIVALENCE (Z(5201),R),(Z(5203),DELR),(Z(5816),ANGLE(1)),

1(Z(5200),LB),(Z(5916),RIL(1)),(Z(6016),ROL(1)),(Z(2),IOUT),

2(Z(11),LERR),(Z(627),ZA(1)),(ZA(4),UO),(ZA(5),VO),(ZA(6),U),

3(ZA(7),V),(Z(5716),THETAX(1)),(Z(5202),THETAV),

5 (Z(6116),IB(1)),(Z(6216),IE(1)),

6 (Z(5708),CORR(1)),(Z(11716),XPNT(1))

EQUIVALENCE (Z(11816),YPNT(1)),(Z(5204),RP),(Z(5205),KRET),

1(Z(5700),CORA(1)),(Z(5400),NGOA),(Z(5486),YE),(Z(5492),XE),

2(Z(5487),YF),(Z(5405),ITE),(Z(5406),NGOB),(Z(5493),XF),

3(Z(5407),IY4),(Z(5408),IY3),(Z(5409),IY2),(Z(5410),IY1),

4(Z(5411),IX4),(Z(5412),IX3),(Z(5413),IX2),(Z(5414),IX1),

5(Z(5415),NGO),(Z(5416),AA),(Z(5417),CENYA),(Z(5418),CENXA),

6(Z(5419),MA3),(Z(5420),MA2),(Z(5421),MA1),(Z(5422),JX),

7(Z(5423),THE3),(Z(5424),THE2),(Z(5425),THE1),(Z(5426),RO3),

8(Z(5427),RI3),(Z(5428),RO2),(Z(5429),RI2),(Z(5430),RO1)

EQUIVALENCE (Z(5431),RI1),(Z(5432),MA(1)),(Z(5442),NAA),

1(Z(5443),MG),(Z(5445),MDU),(Z(5446),MCU),

2(Z(5447),MBU),(Z(5448),MAU),(Z(5449),PV),(Z(5450),PU),

3(Z(5451),YFP),(Z(5452),XFP),(Z(5453),YEP),(Z(5454),XEP),

4(Z(5455),YDP),(Z(5456),XDP),(Z(5457),YCP),(Z(5458),XCP),

5(Z(5459),YBP),(Z(5460),XBP),(Z(5461),YAP),(Z(5462),XAP),

6(Z(5463),KB),(Z(5464),K),(Z(5485),YD),(Z(5491),XD),(Z(5484),YC),

7(Z(5490),XC),(Z(5483),YB),(Z(5489),XB),(Z(5482),YA),

8(Z(5488),XA),(Z(5473),J),(Z(5474),YZ(1)),(Z(5478),XZ(1)),

9(Z(5482),AY(1)),(Z(5488),AX(1)),(Z(5444),NG)

NAMELIST/NAM7/IB,IE,XPNT,YPNT

REAL LENGTH

LB2=2\*LB

DO 2901 J=1,LB2

IB(J)=0

IE(J)=0

2901 CONTINUE

I=2

GO TO 705

1703 DELR=RP\*THETAV

R=RP

705 RP=R+DELR

J=1

701 K=J

IF(I-15)704,704,1026

704 A(I,J)=.0

CENX(I,J)=.0

CENY(I,J)=.0

PLEBAK(I,J)=.0

PLEIN(I,J)=.0

GO TO 102

104 CALL POINT(R,THETAX(K),XA,YA,1.)

CALL POINT(R,THETAX(KB),XB,YB,1.)

CALL POINT(RP,THETAX(KB),XC,YC,1.)

CALL POINT(RP,THETAX(K),XD,YD,1.)

GO TO 109

```

102 IF(K-LB)3,4,4
4 KB=1
GO TO 104
3 KB=K+1
GO TO 104
109 CONTINUE
IF(ANGLE(K))300,300,301
300 KRET=0
ANGLE(K)=.0
GO TO 105
301 IF(R-CORR(5))101,4083,4083
4083 ANGLE(K)=.0
GO TO 300
101 XAP=XA
KRET=1
XBP=XB
XCP=XC
XDP=XD
YAP=YA
YBP=YB
YCP=YC
YDP=YD
4082 CALL POINT(R,ANGLE(K),XEP,YEP,1.)
PU=(U0+U)/2.
PV=(V0+V)/2.
XZ(1)=XEP
YZ(1)=YEP
XZ(2)=PU
YZ(2)=PV
XZ(3)=XCP
YZ(3)=YCP
XZ(4)=XDP
YZ(4)=YDP
CALL INTER(XZ,YZ,XFP,YFP)
4081 RI1=RIL(K)
RO1=ROL(K)
RI2=RIL(KB)
RO2=ROL(KB)
RI3=CORR(5)
DO 4001 JX=1,4
JXX=JX
IF(CORA(JX)-ANGLE(K))4001,4002,4001
4001 CONTINUE
4002 RO3=CORR(JXX)
THE1=THETAX(K)
THE2=THETAX(KB)
THE3=ANGLE(K)
CC1=.1*THETAV
4092 IF(THE3-THE1-CC1)4083,4083,4093
4093 IF(THE2-THE3-CC1)4083,4083,4091
4091 XB=XEP
YB=YEP
XC=XFP
YC=YFP
RIL(KB)=RI3
ROL(KB)=RO3
THETAX(KB)=THE3
105 CONTINUE
C WHERE IS A, 1 IN PLATE, 2 IN T/E, 3 OUTSIDE PLATE (OR NO PLATE)
IF(R-RIL(K))6,7,9
6 MAU=2

```

```

      GO TO 8
7 MAU=1
      GO TO 8
9 IF(R-ROL(K))7,7,10
10 MAU=3
C WHERE IS B
8 IF(R-RIL(KB))11,12,13
11 MBU=2
      GO TO 14
12 MBU=1
      GO TO 14
13 IF(R-ROL(KB))12,12,15
15 MBU=3
C WHERE IS D
14 IF(RP-RIL(K))16,17,19
16 MDU=2
      GO TO 18
17 MDU=1
      GO TO 18
19 IF(RP-ROL(K))17,17,20
20 MDU=3
C WHERE IS C
18 IF(RP-RIL(KB))21,22,23
21 MCU=2
      GO TO 24
22 MCU=1
      GO TO 24
23 IF(RP-ROL(KB))22,22,25
25 MCU=3
24 MZU=MDU+3*(MCU-1)+9*(MBU-1)+27*(MAU-1)
      GO TO (36,37,38,37,37,37,39,37,40,41,37,42,68,37,69,37,37,43,37,37,
2,37,37,37,37,44,37,45,46,47,37,37,37,37,48,49,50,51,52,53,54,55,
356,57,58,59,37,37,37,37,37,37,70,60,61,37,37,62,37,37,37,37,37,
463,37,37,64,37,37,65,37,37,66,37,37,37,37,37,37,37,37,67),MZU
37 WRITE(IOUT,4069) MZU
4069 FORMAT(63HODAD BRANCH IN MESH CALC., SUBROUTINE MESHA, STATEMENT 2
24, MZU=I3)
      LERR=1
      RETURN
497 NG=1
      MG=100
504 IF((NAA.LT.2*MG)GO TO 500
      IF(NAA.LT.3*MG)GO TO 501
      IF(NAA.LT.4*MG)GO TO 502
      IF(NAA.LT.5*MG)GO TO 505
      IF(NAA.LT.6*MG)GO TO 506
      MA(NG)=6
      GO TO 503
505 MA(NG)=4
      GO TO 503
506 MA(NG)=5
      GO TO 503
502 MA(NG)=3
      GO TO 503
501 MA(NG)=2
      GO TO 503
500 MA(NG)=1
503 NAA=NAA-MA(NG)*MG
      MG=MG/10
      NG=NG+1
      IF(NG.LE.3) GO TO 504

```

```

MA1=MA(1)
MA2=MA(2)
MA3=MA(3)
CALL AREA(AX(MA1),AX(MA2),AX(MA3),AY(MA1),AY(MA2),AY(MA3),AA,CENXA
2,CENYA)
A(I,J)=A(I,J)+AA
CENX(I,J)=CENX(I,J) + CENXA*AA
CENY(I,J)=CENY(I,J) + CENYA*AA
GO TO NGO,(510,511,512,513,600,515,517)

```

#### CASE 8

```

36 IX1=1
IX2=4
IX3=1
IX4=2
NODE(I,J)=5
ASSIGN 901 TO NGOB
36 ASSIGN 510 TO NGO
NAA=123
GO TO 497
510 ASSIGN 511 TO NGO
NAA=134
GO TO 497
511 IY1=IX1
IY2=IX2
IY3=IX3
IY4=IX4
ZZX =LENGTH(AX(IX1),AY(IY1),AX(IX2),AY(IY2))
PLEBAK(I,J)=ZZX+PLEBAK(I,J)
ZZX = LENGTH(AX(IX3),AY(IY3),AX(IX4),AY(IY4))
PLEIN(I,J)=ZZX+PLEIN(I,J)
GO TO 600

```

#### CASE 1

```

51 IX1=1
IX2=4
IX3=1
IX4=2
NODE(I,J)=1
ASSIGN 901 TO NGOB
ITE=1
950 IF(ITE)1002,1001,1002
1001 CALL POINT (ROL(K),THETAX(K),XA,YA,1.)
CALL POINT (ROL(KB),THETAX(KB),XB,YB,1.)
GO TO 836
1002 CALL POINT (RIL(K),THETAX(K),XA,YA,1.)
CALL POINT (RIL(KB),THETAX(KB),XB,YB,1.)
GO TO 836

```

#### CASE 2

```

46 IX1=6
IX2=4
IX3=6
IX4=5
NODE(I,J)=1
ASSIGN 901 TO NGOB
ITE=1
846 ASSIGN 512 TO NGO
NAA=235
951 IF(ITE)1003,1004,1003
1004 CALL POINT (ROL(K),THETAX(K),XF,YF,1.)
CALL POINT (ROL(KB),THETAX(KB),XZ(1),YZ(1),1.)
GO TO 516
1003 CALL POINT (RIL(K),THETAX(K),XF,YF,1.)

```

CALL POINT (RIL(KB),THETAX(KB),XZ(1),YZ(1),1.)

516 XZ(2)=XF

YZ(2)=YF

XZ(3)=XA

YZ(3)=YA

XZ(4)=XB

YZ(4)=YB

618 CALL INTER(XZ,YZ,XE,YE)

GO TO 497

512 ASSIGN 513 TO NGO

NAA=356

GO TO 497

513 ASSIGN 511 TO NGO

NAA=346

GO TO 497

CASE 3

41 IX1=1

IX2=4

IX3=5

IX4=6

NODE(I,J)=1

ASSIGN 901 TO NGOB

ITE=1

841 ASSIGN 515 TO NGO

NAA=156

952 IF(ITE)1005,1006,1005

1006 CALL POINT(ROL(KB),THETAX(KB),XF,YF,1.)

CALL POINT(ROL(K),THETAX(K),XZ(1),YZ(1),1.)

GO TO 516

1005 CALL POINT(RIL(KB),THETAX(KB),XF,YF,1.)

CALL POINT(RIL(K),THETAX(K),XZ(1),YZ(1),1.)

GO TO 516

515 ASSIGN 517 TO NGO

NAA=136

GO TO 497

517 ASSIGN 511 TO NGO

NAA=134

GO TO 497

CASE 4

54 IX1=4

IX2=6

IX3=5

IX4=6

NODE(I,J)=1

ASSIGN 901 TO NGOB

ITE=1

854 ASSIGN 511 TO NGO

NAA=456

953 IF(ITE)1007,1008,1007

1008 CALL POINT(ROL(K),THETAX(K),XF,YF,1.)

CALL POINT(ROL(KB),THETAX(KB),XZ(1),YZ(1),1.)

GO TO 1009

1007 CALL POINT(RIL(K),THETAX(K),XF,YF,1.)

CALL POINT(RIL(KB),THETAX(KB),XZ(1),YZ(1),1.)

1009 XZ(2)=XF

YZ(2)=YF

XZ(3)=XC

YZ(3)=YC

XZ(4)=XD

YZ(4)=YD

GO TO 618

# CASE 5

```

52 IX1=1
   IX2=1
   IX3=5
   IX4=6
   NODE(I,J)=1
   ASSIGN 901 TO NGOB
   ITE=1
852 ASSIGN 511 TO NGO
   NAA=356
   IF(ITE)1011,1010,1011
1010 CALL POINT(ROL(KB),THETAX(KB),XE,YE,1.)
      CALL POINT(ROL(K),THETAX(K),XZ(1),YZ(1),1.)
      GO TO 1012
1011 CALL POINT(RIL(KB),THETAX(KB),XE,YE,1.)
      CALL POINT(RIL(K),THETAX(K),XZ(1),YZ(1),1.)
1012 XZ(2)=XE
      YZ(2)=YE
      XZ(3)=XC
      YZ(3)=YC
      XZ(4)=XD
      YZ(4)=YD
      CALL INTER(XZ,YZ,XF,YF)
      GO TO 97

```

# CASE 6

```

47 IX1=1
   IX2=4
   IX3=1
   IX4=2
   NODE(I,J)=1
   ASSIGN 901 TO NGOB
   ITE=1
847 ASSIGN 520 TO NGOA
955 IF(ITE)523,1014,523
1014 CALL POINT(ROL(K),THETAX(K),XZ(1),YZ(1),1.)
      CALL POINT(ROL(KB),THETAX(KB),XZ(2),YZ(2),1.)
      GO TO 1013
523 CALL POINT(RIL(K),THETAX(K),XZ(1),YZ(1),1.)
      CALL POINT(RIL(KB),THETAX(KB),XZ(2),YZ(2),1.)
1013 XZ(3)=XA
      YZ(3)=YA
      XZ(4)=XB
      YZ(4)=YB
      CALL INTER(XZ,YZ,XE,YE)
      XZ(3)=XC
      YZ(3)=YC
      XZ(4)=XD
      YZ(4)=YD
      CALL INTER(XZ,YZ,XF,YF)
      GO TO NGOA,(520,522)
520 XA=XE
      YA=YE
      XD=XF
      YD=YF
      GO TO 836

```

# CASE 7

```

68 IX1=1
   IX2=4
   IX3=1
   IX4=2
   NODE(I,J)=1

```



ASSIGN 901 TO NGOB

ITE=1

868 ASSIGN 522 TO NGOA

GO TO 955

522 XB=XE

YB=YF

XC=XF

YC=YF

GO TO 836

CASE 1P

40 IX1=1

IX2=4

IX3=1

IX4=2

NODE(I,J)=3

ASSIGN 901 TO NGOB

840 CALL POINT(ROL(K),THETAX(K),XD,YD,1.)

CALL POINT(ROL(KB),THETAX(KB),XC,YC,1.)

GO TO 836

CASE 2P

45 IX1=1

IX2=6

IX3=1

IX4=5

NODE(I,J)=3

ASSIGN 901 TO NGOB

ITE=0

845 ASSIGN 511 TO NGO

NAA=156

GO TO 951

CASE 3P

63 IX1=1

IX2=1

IX3=2

IX4=5

NODE(I,J)=3

ASSIGN 901 TO NGOB

ITE=0

863 ASSIGN 511 TO NGO

NAA=256

GO TO 952

CASE 4P

38 IX1=1

IX2=6

IX3=1

IX4=2

NODE(I,J)=3

ASSIGN 902 TO NGOB

ASSIGN 1836 TO NGOC

ITE=0

GO TO 854

CASE 5P

39 IX1=1

IX2=4

IX3=1

IX4=2

NODE(I,J)=3

ASSIGN 902 TO NGOB

ASSIGN 1836 TO NGOC

ITE=0

GO TO 852

CASE 6P

44 IX1=1  
IX2=4  
IX3=1  
IX4=2  
NODE(I,J)=3  
ASSIGN 901 TO NGOB  
ITE =0  
GO TO 868

CASE 7P

62 IX1=1  
IX2=4  
IX3=1  
IX4=2  
NODE(I,J)=3  
ASSIGN 901 TO NGOB  
ITE=0  
GO TO 847

CASE 3-4PP

42 IX1=4  
IX2=6  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ITE=0  
ASSIGN 970 TO NGOC  
GO TO 854

CASE 3-1PP

43 IX1=4  
IX2=6  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ASSIGN 970 TO NGOC  
ITE=0  
GO TO 950

CASE 2-5PP

48 IX1=1  
IX2=1  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ASSIGN 46 TO NGOC  
ITE =0  
GO TO 852

CASE 6-5PP

49 IX1=1  
IX2=1  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ITE=0  
ASSIGN 47 TO NGOC  
GO TO 852

CASE 2-1PP

50 IX1=4  
IX2=6

IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ASSIGN 972 TO NGOC  
ITE=0  
GO TO 950

CASE 1-4PP

53 IX1=6  
IX2=4  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ASSIGN 973 TO NGOC  
ITE=0  
GO TO 854

CASE ALL T/E

55 NODE(I,J)=-1  
ASSIGN 901 TO NGOB  
GO TO 600

CASE 4-4PP

56 IX1=4  
IX2=6  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ITE=0  
ASSIGN 974 TO NGOC  
GO TO 854

CASE 1-5PP

57 IX1=1  
IX2=1  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ITE=0  
ASSIGN 51 TO NGOC  
GO TO 852

CASE 5-5PP

58 IX1=1  
IX2=1  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ASSIGN 52 TO NGOC  
ITE=0  
GO TO 852

CASE 1-1PP

59 IX1=4  
IX2=6  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ASSIGN 973 TO NGOC  
ITE=0  
GO TO 950

CASE 6-6PP

60 IX1=1  
IX2=1  
IX3=5  
IX4=5  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ASSIGN 971 TO NGOC  
ITE=0  
GO TO 847

CASE 2-2PP

61 IX1=4  
IX2=6  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ASSIGN 972 TO NGOC  
ITE=0  
GO TO 846

CASE 3-7PP

64 IX1=1  
IX2=1  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ASSIGN 41 TO NGOC  
ITE=0  
GO TO 868

CASE 7-7PP

65 IX1=1  
IX2=1  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ASSIGN 68 TO NGOC  
ITE=0  
GO TO 868

CASE 3-3PP

66 IX1=1  
IX2=1  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB  
ASSIGN 41 TO NGOC  
ITE=0  
GO TO 841

CASE ALL OUTSIDE PLATE

67 NODE(I,J)=0  
ASSIGN 901 TO NGOB  
GO TO 600

CASE 7-4PP

69 IX1=4  
IX2=6  
IX3=1  
IX4=1  
NODE(I,J)=4  
ASSIGN 902 TO NGOB

```

ASSIGN 974 TO NGOC
ITE=0
GO TO 854
CASE 2-6PP
70 IX1=1
IX2=1
IX3=1
IX4=1
NODE(I,J)=4
ASSIGN 902 TO NGOB
ASSIGN 46 TO NGOC
ITE=0
GO TO 847
600 GO TO NGOB,(901,902)
901 IF(KRET)106,108,106
106 KRET=0
XA=XEP
YA=YEP
XD=XFP
YD=YFP
XB=XBP
YB=YBP
XC=XCP
YC=YCP
ROL(K)=RO3
RIL(K)=RI3
ROL(KB)=RO2
RIL(KB)=RI2
THETAX(K)=THE3
THETAX(KB)=THE2
PBAK=PLEBAK(I,J)
NIJP=NODE(I,J)
GO TO 105
108 IF(ANGLE(K)) 107,107,1015
1015 PLEBAK(I,J)=PBAK
XA=XAP
YA=YAP
XD=XDP
YD=YDP
RIL(K)=RI1
ROL(K)=RO1
THETAX(K)=THE1
IF(NIJP-4)5024,5025,5024
5024 IF(NODE(I,J)-4)5026,5025,5026
5025 NODE(I,J)=4
GO TO 5027
5026 IF(NIJP+1)5029,5028,5029
5028 IF(NODE(I,J)+1)5029,5031,5029
5031 NODE(I,J)=-1
GO TO 5027
5029 NODE(I,J)=1
5027 CONTINUE
GO TO 107
902 A(I,J)=-A(I,J)
CENX(I,J)=-CENX(I,J)
CENY(I,J)=-CENY(I,J)
ASSIGN 901 TO NGOB
GO TO NGOC,(836,970,971,46,972,973,974,51,41,68,1974,1836)
970 PLEBAK(I,J)=-PLEBAK(I,J)
GO TO 41
971 PLEIN(I,J)=-PLEIN(I,J)

```

```

GO TO 47
972 PLEBAK(I,J)=-PLEBAK(I,J)
GO TO 46
973 PLEBAK(I,J)=-PLEBAK(I,J)
GO TO 51
974 PLEBAK(I,J)=-PLEBAK(I,J)
GO TO 68
1974 PLEBAK(I,J)=-PLEBAK(I,J)
GO TO 54
1836 IX1=1
IX2=1
IX3=1
IX4=1
GO TO 836
107 CENX(I,J)=CENX(I,J)/A(I,J)
CENY(I,J)=CENY(I,J)/A(I,J)
IF(IB(J))1016,1017,1016
1017 IF(NODE(I,J)-1)1019,1018,1018
1018 IB(J)=1
1016 IF(NODE(I,J))1021,1020,1021
1020 IF(IE(J))1021,1022,1021
1022 IE(J)=I-1
GO TO 1021
1019 IF(NODE(I,J))1021,1023,1021
1023 IB(J)=-1
1021 IF(I-IB(J))1701,2023,1701
2023 CALL POINT (RIL(K),THETAX(K),XZ(1),YZ(1),1.)
CALL POINT (RIL(KB),THETAX(KB),XZ(2),YZ(2),1.)
XZ(3)=CENX(I,J)
YZ(3)=CENY(I,J)
XZ(4)=(UO+U)/2.
YZ(4)=(VO+V)/2.
CALL INTER(XZ,YZ,XPNT(J),YPNT(J))
1701 IF(J-LB)1024,1025,1025
1024 J=J+1
GO TO 701
1025 DO 1026 J=1, LB
IF(IE(J))1027,1027,1026
1026 CONTINUE
RETURN
1027 I=I+1
IF(I-IX)1703,1029,1029
1029 WRITE(IOUT,1030)
1030 FORMAT(22H0 TOO MANY MESH POINTS)
LERR=1
RETURN
END

```

```

SIBFTC MESHB LIST,DECK,REF,DD,XR7
SUBROUTINE MESHB(NODE,A,CENX,CENY,PLEBAK,PLEIN,IX,L2)
COMMON Z
DIMENSION Z(20000),IB(100),NODE(IX,L2),A(IX,L2),CENX(IX,L2),
2CENY(IX,L2),PLEBAK(IX,L2),PLEIN(IX,L2),IE(100)
EQUIVALENCE (Z(5200),LB),(Z(2),IOUT),(Z(11),LERR),
2(Z(6116),IB(1)),
4 (Z(6216),IE(1)),(Z(5400),I),(Z(5401),JP),
5(Z(5402),AMIN),(Z(5403),RATIO),(Z(5404),J),(Z(5405),IFLAG)
IFLAG=0
AMIN=.45E-4
12 J=1
101 K=J
2 I=IB(J)
103 IF(NODE(I+1,J)-1)30,30,130
130 RATIO= A(I+1,J)/A(I,J)
IF(RATIO-2.)126,126,26
126 IF(A(I,J)-AMIN)26,26,403
403 IF(NODE(I,J)-1)404,405,404
404 IF(NODE(I,J)-4)3,405,3
405 IF(NODE(I+1,J)-1)406,26,406
406 IF(NODE(I+1,J)-4)3,26,3
26 IF(I-IB(J)-3)25,3
25 CENX(I+1,J)=(A(I+1,J)*CENX(I+1,J)+A(I,J)*CENX(I,J))/(A(I+1,J)+A(I,
1J))
CENY(I+1,J)=(A(I+1,J)*CENY(I+1,J)+A(I,J)*CENY(I,J))/(A(I+1,J)+A(I,
1J))
A(I+1,J)=A(I+1,J)+A(I,J)
PLEBAK(I+1,J)=PLEBAK(I+1,J)+PLEBAK(I,J)
IF(NODE(I+1,J)-1)200,201,200
200 PLEIN(I+1,J)=PLEIN(I,J)
GO TO 202
201 PLEIN(I+1,J)=PLEIN(I+1,J)+PLEIN(I,J)
202 NODE(I+1,J)=NODE(I,J)
NODE(I,J)=-1
IB(J)=IB(J)+1
A(I,J)=.0
CENX(I,J)=.0
CENY(I,J)=.0
PLEBAK(I,J)=.0
PLEIN(I,J)=.0
GO TO 3
30 IF(NODE(I-1,J))31,131,131
131 RATIO= A(I-1,J)/A(I,J)
IF(RATIO-2.)135,135,36
135 IF(A(I,J)-AMIN)35,3,3
31 IF(J-LB)204,203,204
203 JP=1
GO TO 205
204 JP=J+1
205 IF(NODE(I,JP)-3)3,3,206
206 IF(A(I,J)-AMIN)45,3,3
36 IF(I-IE(J)-3)35,3,3
35 CENX(I-1,J)=(A(I,J)*CENX(I,J)+A(I-1,J)*CENX(I-1,J))/(A(I,J)+A(I-1,
1J))
CENY(I-1,J)=(A(I,J)*CENY(I,J)+A(I-1,J)*CENY(I-1,J))/(A(I,J)+A(I-1,
1J))
A(I-1,J)=A(I-1,J)+A(I,J)

```

```

PLEBAK(I-1,J)=PLEBAK(I,J)+PLEBAK(I-1,J)
NODE(I-1,J)=NODE(I,J)
NODE(I,J)=0
IE(J)=IE(J)-1
A(I,J)=.0
CENX(I,J)=.0
CENY(I,J)=.0
PLEBAK(I,J)=.0
PLEIN(I,J)=.0
GO TO 3
45 CENX(I,J)=(A(I,J)*CENX(I,J)+A(I,JP)*CENX(I,JP))/(A(I,J)+A(I,JP))
CENY(I,J)=(A(I,J)*CENY(I,J)+A(I,JP)*CENY(I,JP))/(A(I,J)+A(I,JP))
PLEIN(I,J)=PLEIN(I,J)+PLEIN(I,JP)
A(I,J)=A(I,J)+A(I,JP)
LB=LB-1
DO 305 J=JP,LB
I=IB(J)
304 A(I,J)=A(I,J+1)
CENX(I,J)=CENX(I,J+1)
CENY(I,J)=CENY(I,J+1)
PLEBAK(I,J)=PLEBAK(I,J+1)
PLEIN(I,J)=PLEIN(I,J+1)
IF(I-IE(J))303,305,305
303 I=I+1
GO TO 304
305 CONTINUE
3 IF(I-IE(J))306,1,1
306 I=I+1
GO TO 103
1 IF(J-LB)308,307,307
308 J=J+1
GO TO 101
307 IF(IFLAG-1)310,309,310
310 IFLAG=1
GO TO 12
309 RETURN
END

```



\$IBFTC MESHCLIST,DECK,REF,DD,XR7

SUBROUTINE MESHCL(CONDB,NODE,CONDI,PLEBAK,CENX,CENY,A,T,PLEIN,IX,L2

1)

COMMON Z

DIMENSION Z(20000),CONDB(IX,L2),NODE(IX,L2),IB(100),IE(100),

1CONDI(IX,L2),PLEBAK(IX,L2),XPNT(100),CENX(IX,L2),CENY(IX,L2),

2A(IX,L2),T(IX,L2),TTE(100),PLEIN(IX,L2),YPNT(100),ZA(200)

EQUIVALENCE (Z(2),IOUT),(Z(5200),LB),(Z(11),LERR),

2 (Z(6216),IE(1)),(Z(6116),IB(1)),

3 (Z(11716),XPNT(1)),

4(Z(11816),YPNT(1)),

5(Z(4418),TIN),(Z(5206),TC),(Z(5207),TR),(Z(5208),IMAXP),

6(Z(6016),TTE(1)),(Z(627),ZA(1)),(Z(626),TP),(Z(5400),JX),

7(Z(5401),YZ),(Z(5402),XZ), (Z(5404),X1),

8 (Z(5406),JXP),(Z(5407),JPP),(Z(5408),JPX),

9(Z(5409),JP),(Z(5410),J),(Z(5411),I),(ZA(75),TPR)

REAL LENGTH

DO 302 J=1,L2

DO 301 I=1,IX

CONDB(I,J)=.0

CONDI(I,J)=.0

301 CONTINUE

302 CONTINUE

DO 100 J=1,LB

JX=J+LB

1 I=IB(J)

IF(J-1)102,101,102

101 JP=LB

JPX=2\*LB

GO TO 2

102 JP=J-1

JPX=JP+LB

2 CONTINUE

11 IF(J-LB)104,103,104

103 JPP=1

JXP=LB+1

GO TO 105

104 JPP=J+1

JXP=JPP+LB

105 IF(NODE(I,JPP))106,106,107

106 CONDB(I,JPP)=.0

CONDB(I,JXP)=CONDB(I,JPP)

107 IF(NODE(I,JP))108,108,7

108 CONDB(I,J)=.0

CONDB(I,JX)=CONDB(I,J)

9 IF(NODE(I+1,J))10,109,10

109 CONDI(I+1,J)=.0

CONDI(I+1,JX)=.0

GO TO 10

7 X2=CENX(I,JP)

Y2=CENY(I,JP)

X1=CENX(I,J)

Y1=CENY(I,J)

XLENT=LENGTH(X1,Y1,X2,Y2)

CONDB(I,J)=(-TP\*PLEBAK(I,J))/XLENT

CONDB(I,JX)=CONDB(I,J)\*TPR/TP

GO TO 9

10 IF(I-IB(J))25,4,25

```

4 X1=XPNT(J)
Y1=YPNT(J)
GO TO 5
25 X1=CENX(I-1,J)
Y1=CENY(I-1,J)
5 X2=CENX(I,J)
Y2=CENY(I,J)
XLENTH=LENGTH(X1,Y1,X2,Y2)
CONDI(I,J)=(TP*PLEIN(I,J))/XLENTH
CONDI(I,JX)=CONDI(I,J)*TPR/TP
6 A(I,JX)=A(I,J)
T(I,J)=TIN
T(I,JX)=TIN
16 IF(I-IE(J))112,111,111
112 I=I+1
GO TO 2
111 IB(JX)=IB(J)
IE(JX)=IE(J)
100 CONTINUE
TC=TIN
TR=TIN
DO 200 I=2,IMAXP
200 TTE(I)=TIN
RETURN
END

```

```

$IBFTC AREA LIST,DECK,REF,DD,XR7
SUBROUTINE AREA (XA,XB,XC,YA,YB,YC,A,CA,CB)
DIMENSION XZ(4),YZ(4)
DIMENSION Z(20000),XP(3),C(4)
EQUIVALENCE(Z(2),IOUT)
EQUIVALENCE (Z(5300),M),(Z(5301),J),(Z(5302),C(1)),(Z(5306),XP(1))
2,(Z(5309),K),(Z(5310),I),(Z(5311),CT),(Z(5312),BT),(Z(5313),AT),
3(Z(5314),F),(Z(5315),E),(Z(5316),G),(Z(5317),D),(Z(5318),TBAR),
4(Z(5319),SBAR),(Z(5320),T),(Z(5321),S2),(Z(5322),S1),(Z(5323),COSA
5),(Z(5324),S),(Z(5325),ZA),(Z(5326),Y),(Z(5327),X)
COMMON Z
LOGICAL AA,AB,AC
REAL LENGTH
IOUT=6
COMPUTE LINE LENGTHS
ZA=LENGTH(XA,YA,XB,YB)
Y=LENGTH(XA,YA,XC,YC)
X=LENGTH(XB,YB,XC,YC)
S=(X+Y+ZA)/2.
COMPUTE AREA
A=SQRT(S*(S-X)*(S-Y)*(S-ZA))
COMPUTE CENTROID IF REQUIRED
GO TO 2
1 CONTINUE
101 FORMAT(1H,6E15.6)
102 FORMAT(1H,4I6)
201 CONTINUE
RETURN
2 COSA=(Y*Y+ZA*ZA-X*X)/(2.*Y*ZA)
COSC=(X*X+Y*Y-ZA*ZA)/(2.*X*Y)
IF(COSA)103,103,104
104 IF(COSC)106,106,105
103 S1=-ZA*COSA
S2=Y
T=ZA*SQRT(ABS(1.-COSA*COSA))
CF=S2/(S1+S2)
SBAR=((S2-S1)/3.)*CF
S1=.0
GO TO 108
106 S1=Y
S2=-X*COSC
T=X*SQRT(ABS(1.-COSC*COSC))
CF=S1/(S1+S2)
SBAR=((S2-S1)/3.)*CF
S2=.0
GO TO 108
105 S1=ZA*COSA
S2=Y-S1
T=ZA*SQRT(ABS(1.-COSA*COSA))
CF=1.
SBAR=((S2-S1)/3.)*CF
108 TBAR=(T/3.)*CF
D=(S1+SBAR)**2+TBAR**2
G=(S2-SBAR)**2+TBAR**2
IF(ABS(XA-XC)-1.E-5)110,110,109
110 IF(YA-YC)111,121,111
111 C(1)=(YA*YA-YC*YC-D+G)/(2.*(YA-YC))
C(2)=C(1)
BT=-2.*XA
CT=XA*XA+YA*YA-2.*YA*C(1)+C(1)*C(1)-D
AT=1.

```

```

CA=BT*BT-4.*AT*CT
IF(CA)3,112,112
112 CA=SQRT(ABS(CA))
C(3)=(CA-BT)/2.
C(4)=(-BT-CA)/2.
GO TO 21
109 E=(D-G-XA*XA+XC*XC-YA*YA+YC*YC)/(2.*(XC-XA))
F=(YA-YC)/(XC-XA)
AT=F*F+1.
BT=2.*(F*E-XA*F-YA)
CT=XA*XA-2.*XA*E+E*E+YA*YA-D
CA=BT*BT-4.*AT*CT
IF(CA)3,4,4
3 WRITE(IOUT,5)
5 FORMAT(44HOIMAGINARY NUMBER FAILURE IN SUBROUTINE AREA)
18 GO TO 1
4 CA=SQRT(ABS(CA))
CB=(CA-BT)/2./AT
CT=(-BT-CA)/2./AT
CHECK FOR BEHAVIOR OF VALUE - IT MUST BE IN RANGE OF POINTS
I=1
K=1
XP(1)=YA
XP(2)=YB
XP(3)=YC
C(1)=CB
C(2)=CT
14 CONTINUE
7 IF(C(I)-XP(1))8,9,10
8 IF(C(I)-XP(2))11,9,9
10 IF(C(I)-XP(2))9,9,12
11 IF(C(I)-XP(3))13,9,9
12 IF(C(I)-XP(3))9,9,13
13 GO TO (15,16,26,28),K
15 J=0
K=2
I=2
GO TO 14
16 IF(J)20,121,20
121 WRITE(IOUT,17)
17 FORMAT(41HOCENTROID TEST FAILURE IN SUBROUTINE AREA)
GO TO 18
9 GO TO (19,22,27,31),K
19 K=2
I=2
J=1
GO TO 14
22 IF(J)23,24,23
23 J=-1
GO TO 20
24 J=2
20 I=3
XP(1)=XA
XP(2)=XB
XP(3)=XC
K=3
C(3)=E+F*C(1)
C(4)=E+F*C(2)
GO TO 14
26 K=4
M=0

```

```

I=4
GO TO 14
27 K=4
I=4
M=1
GO TO 14
28 IF(M)29,121,29
31 IF(M)32,33,32
32 M=-1
GO TO 29
33 M=2
29 IF(M)62,121,63
62 IF(J)21,121,65
63 IF(J)65,121,64
64 IF(M-J)121,66,121
65 IF(J)66,121,35
66 CA=C(M+2)
CB=C(M)
GO TO 201
35 CA=C(J+2)
CB=C(J)
GO TO 201
21 XZ(1)=XB
YZ(1)=YB
XZ(2)=XC
YZ(2)=YC
XZ(3)=XA
YZ(3)=YA
XZ(4)=C(3)
YZ(4)=C(1)
CALL INTER(XZ,YZ,XPP,YP)
NN=1
CY=C(1)
CX=C(3)
151 FORMAT(1H ,4E15.6)
122 AA=XPP.GE.XB
AB=XPP.LE.XC
AC=AA.AND.AB
IF(AC) GO TO 126
XX=XPP-XB
XXXX=XPP-XC
124 IF((XPP.LE.XB).AND.(XPP.GE.XC)) GO TO 126
IF(ABS(XPP-XB)-1.E-6)205,205,125
205 IF(ABS(XPP-XC)-1.E-6)126,126,125
125 GO TO (140,141),NN
126 IF((YP.GE.YB).AND.(YP.LE.YC)) GO TO 128
127 IF((YP.LE.YB).AND.(YP.GE.YC)) GO TO 128
IF(ABS(YP-YB)-1.E-6)206,206,125
206 IF(ABS(YP-YC)-1.E-6)128,128,125
128 IF((CX.GE.XA).AND.(CX.LE.XPP)) GO TO 131
130 IF((CX.LE.XA).AND.(CX.GE.XPP)) GO TO 131
GO TO 125
131 IF((CY.GE.YA).AND.(CY.LE.YP)) GO TO 133
132 IF((CY.LE.YA).AND.(CY.GE.YP)) GO TO 133
GO TO 125
133 GO TO (142,143),NN
140 J=0
135 NN=2
XZ(4)=C(4)
YZ(4)=C(2)
CALL INTER(XZ,YZ,XPP,YP)

```

```
CX=C(4)
CY=C(2)
GO TO 122
141 IF(J)121,121,134
134 CA=C(3)
CB=C(1)
GO TO 201
142 J=1
GO TO 135
143 IF(J-1)136,121,136
136 CA=C(4)
CB=C(2)
GO TO 201
END
```

```

$IBFTC LENGTH LIST,DECK,DD,XR7
REAL FUNCTION LENGTH(XA,YA,XB,YB)
LENGTH=SQRT((-XB-XA)**2+(-YB-YA)**2)
RETURN
END

```

```

$IBFTC INTER LIST,REF,DD,XR7

```

```

SUBROUTINE INTER(X,Y,XA,YA)
DIMENSION X(4),Y(4),Z(20000)
COMMON Z
EQUIVALENCE (Z(2),IOUT)
IF(X(1)-X(2))10,11,10
11 IF(X(3)-X(4))10,4,10
10 IF(Y(1)-Y(2))1,2,1
2 IF(Y(3)-Y(4))3,4,3
4 WRITE(IOUT,5)
5 FORMAT(17H0FAILURE IN INTER)
CALL EXIT
3 YA=Y(1)
XA=(X(4)*Y(3)-X(3)*Y(4)-(X(4)-X(3))*Y(1))/(Y(3)-Y(4))
6 RETURN
1 IF(Y(3)-Y(4))7,8,7
8 YA=Y(3)
XA=(X(2)*Y(1)-X(1)*Y(2)-(X(2)-X(1))*Y(3))/(Y(1)-Y(2))
GO TO 6
7 YA=((X(4)*Y(3)-X(3)*Y(4))/(Y(3)-Y(4))-(X(2)*Y(1)-X(1)*Y(2))/(Y(1)-
2-Y(2)))/(X(4)-X(3))/(Y(3)-Y(4)-(X(2)-X(1))/(Y(1)-Y(2)))
XA=(X(2)*Y(1)-X(1)*Y(2)-(X(2)-X(1))*YA)/(Y(1)-Y(2))
GO TO 6
END

```

```

$IBFTC POINT LIST,DECK,DD,XR7

```

```

SUBROUTINE POINT(R,T,X,Y,B)
COMMON Z
DIMENSION Z(20000)
EQUIVALENCE (Z(5345),U),(Z(5344),V),(Z(5343),TEMP),(Z(5342),TEMPD) PO
2,(Z(5341),TEMPE),(Z(5340),TEMPE)
IF(B)3,1,2
1 U=(R+T)/2.
4 RETURN
3 V=(R+T)/2.
TEMP=ATAN(V/U)
TEMPD=COS(TEMP)
TEMPE=SIN(TEMP)
GO TO 4
2 TEMP=COS(T)
TEMPA=SIN(T)
X=-R*(TEMP*TEMPD-TEMPA*TEMPE)+U
Y=-R*(TEMPA*TEMPD+TEMP*TEMPE)+V
GO TO 4
END

```

```

SUBROUTINE PROP(T,V1,V2,V3,V4,V5,V6,MAY)
COMMON Z
DIMENSION Z(20000),I(17),IXB(20),IXE(20)
EQUIVALENCE (Z(17416),IXB(1)),(Z(17436),IXE(1))
DATA I(1),I(2),I(3),I(4),I(5),I(6),I(7),I(8),I(9),I(10),I(11),I(12)
1),I(13),I(14),I(15),I(16),I(17) / 2,2,2,2,2,2,2,2,2,2,2,2,2,2,2,2,
22/
IF(MAY-6)17,17,50
50 IF(MAY-12)37,37,51
51 IF(MAY-16)7,7,52
52 IF(MAY-17)53,27,53
53 WRITE(IOUT,54)
54 FORMAT(13H0 TABLE ERROR)
LERR=1
RETURN
7 IF(IXB(MAY)-IXE(MAY))407,400,400
400 IXX=IXB(MAY)+1
V1=Z(IXX+2644)
V2=Z(IXX+3044)
V3=Z(IXX+2444)
V4=Z(IXX+3444)
V5=Z(IXX+212)
RETURN
407 IXX=IXB(MAY)-1+ I(MAY)
1006 IF(IXX-IXE(MAY))2001,2001,2002
2002 IXX=IXX-1
GO TO 1006
2001 IF(IXX-IXB(MAY)-1)2003,6,6
2003 IXX=IXX+1
GO TO 2001
6 IF(T - Z(IXX+413 ))1003,2004,1
2004 IXX=IXX+1
2 A=0.0
GO TO 8
1 IXX=IXX+1
IF(IXX-IXE(MAY))1006,55,55
55 IXX=IXE(MAY)+1
GO TO 2
1003 IF(IXX-IXE(MAY))2011,2011,2012
2012 IXX=IXX-1
GO TO 1003
2011 IF(IXX-IXB(MAY)-1)2013,3,3
2013 IXX=IXX+1
GO TO 2011
3 IF(T - Z(IXX+413 ))4, 2,5
4 IXX=IXX-1
IF(IXX-IXB(MAY))57,57,1006
57 IXX=IXB(MAY)+1
GO TO 2
5 A=(T-Z(IXX+412 ))/(Z(IXX+413 )-Z(IXX+412 ))
8 V1=Z(IXX+2644) + A*(Z(IXX+2645)-Z(IXX+2644))
V2=Z(IXX+3044) + A*(Z(IXX+3045)-Z(IXX+3044))
V3=Z(IXX+2444) + A*(Z(IXX+2445)-Z(IXX+2444))
V4=Z(IXX+3444) + A*(Z(IXX+3445)-Z(IXX+3444))
V5=Z(IXX+212 ) + A*(Z(IXX+213 )-Z(IXX+212 ))
I(MAY)=IXX+1-IXB(MAY)
RETURN
17 IF(IXB(MAY)-IXE(MAY))417,401,401
401 IXX=IXB(MAY)+1
V1=Z(IXX+1031)
V2=Z(IXX+1231)

```



```

V3=Z(IXX+1431)
V4=Z(IXX+1631)
V5=Z(IXX+2031)
RETURN
417 IXX=IXB(MAY)-1 + I (MAY)
1016 IF(IXX-IXE(MAY))2014,2014,2015
2015 IXX=IXX-1
GO TO 1016
2014 IF(IXX-IXB(MAY)-1)2016,16,16
2016 IXX=IXX+1
GO TO 2014
16 IF(T - Z(IXX+832 ))1013,2017,11
2017 IXX=IXX+1
12 A=0.0
GO TO 18
11 IXX=IXX+1
IF(IXX-IXE(MAY))1016,155,155
155 IXX= IXE(MAY)+1
GO TO 12
1013 IF(IXX-IXE(MAY))2018,2018,2019
2019 IXX=IXX-1
GO TO 1013
2018 IF(IXX-IXB(MAY)-1)2020,13,13
2020 IXX=IXX+1
GO TO 2018
13 IF(T-Z(IXX+831 ))14, 12,15
14 IXX=IXX-1
IF(IXX-IXB(MAY))157,157,1016
157 IXX=IXB(MAY) +1
GO TO 12
15 A=(T-Z(IXX+831 ))/(Z(IXX+832 )-Z(IXX+831 ))
18 V1= Z(IXX+1031) +A*(Z(IXX+1032) -Z(IXX+1031))
V2= Z(IXX+1231) +A*(Z(IXX+1232) -Z(IXX+1231))
V3= Z(IXX+1431) +A*(Z(IXX+1432) -Z(IXX+1431))
V4= Z(IXX+1631) +A*(Z(IXX+1632) -Z(IXX+1631))
V5= Z(IXX+2031) +A*(Z(IXX+2032) -Z(IXX+2031))
I(MAY)=IXX+1-IXB(MAY)
RETURN
37 IF(IXB(MAY)-IXE(MAY))437,402,402
402 IXX=IXB(MAY)+1
V3=Z(IXX+12)
RETURN
437 IXX=IXB(MAY)-1+I(MAY)
1036 IF(IXX-IXE(MAY))2023,2023,2022
2022 IXX=IXX-1
GO TO 1036
2023 IF(IXX-IXB(MAY)-1)2024,36,36
2024 IXX=IXX+1
GO TO 2023
36 IF(T-Z(IXX+413))1033,2021,31
2021 IXX=IXX+1
32 A=.0
GO TO 38
31 IXX=IXX+1
IF(IXX-IXE(MAY))1036,355,355
355 IXX=IXE(MAY) +1
GO TO 32
1033 IF(IXX-IXE(MAY))2025,2025,2026
2026 IXX=IXX-1
GO TO 1033
2025 IF(IXX-IXB(MAY)-1)2027,33,33

```

```

2027 IXX=IXX+1
GO TO 2025
33 IF(T-Z(IXX+412))34, 32,35
34 IXX=IXX-1
IF(IXX-IXB(MAY))357,357,1036
357 IXX=IXB(MAY)+1
GO TO 32
35 A=(T-Z(IXX+412))/(Z(IXX+413)-Z(IXX+412))
38 V3=Z(IXX+12) +A*(Z(IXX+13)-Z(IXX+12))
I(MAY)=IXX+1-IXB(MAY)
RETURN
27 IF(IXB(MAY)-IXE(MAY))427,403,403
403 IXX=IXB(MAY)+1
V1=Z(IXX+3754)
V2=Z(IXX+3854)
V3=Z(IXX+3954)
V4=Z(IXX+4054)
V5=Z(IXX+4154)
V6=Z(IXX+4254)
RETURN
427 IXX=IXB(MAY)-1 + I(MAY)
1026 IF(IXX-IXE(MAY))2028,2028,2029
2029 IXX=IXX-1
GO TO 1026
2028 IF(IXX-IXB(MAY)-1)2030,26,26
2030 IXX=IXX+1
GO TO 2028
26 IF(T- Z(IXX+3655))1023,2031,21
2031 IXX=IXX+1
22 A=0.0
GO TO 28
21 IXX = IXX +1
IF(IXX-IXE(MAY))1026,255,255
255 IXX= IXE(MAY)+1
GO TO 22
1023 IF(IXX-IXE(MAY))2032,2032,2033
2033 IXX=IXX-1
GO TO 1023
2032 IF(IXX-IXB(MAY)-1)2034,23,23
2034 IXX=IXX+1
GO TO 2032
23 IF(T- Z(IXX+3654)) 24, 22,25
24 IXX = IXX-1
IF(IXX -IXB(MAY))257,257,1026
257 IXX = IXB(MAY) +1
GO TO 22
25 A=(T-Z(IXX+3654))/(Z(IXX+3655)-Z(IXX+3654))
28 V1=Z(IXX+3754) + A*(Z(IXX+3755)- Z(IXX+3754))
V2=Z(IXX+3854) + A*(Z(IXX+3855)- Z(IXX+3854))
V3=Z(IXX+3954) + A*(Z(IXX+3955)- Z(IXX+3954))
V4=Z(IXX+4054) + A*(Z(IXX+4055)- Z(IXX+4054))
V5=Z(IXX+4154) + A*(Z(IXX+4155)- Z(IXX+4154))
V6=Z(IXX+4254) + A*(Z(IXX+4255)- Z(IXX+4254))
I(MAY)=IXX+1-IXB(MAY)
RETURN

```

END

\$ORIGIN

ETA

\$IBFTC CALCS DECK, XR7

SUBROUTINE CALCS(XK,CAP,EPSC1,EPSC2,EPSR1,EPSR2,ALPH,T,A,T4,  
1CONDI,CONDB,TERMF,TERMO,IX,L2, L22)

COMMON Z

DIMENSION Z(20000),XK(IX,L2),XKTE( 100 ),CAP(IX,L2),CAPTE( 100 ),  
1ALPHA(100),RESIST(100),EPSC1(IX ,L22),EPSC2(IX ,L22),EPSR1(IX ,L22  
2),EPSR2(IX ,L22),ALPH(IX,L2),EPSI(100),T(IX,L2),TTE(100),A(IX,L2),  
3T4(IX,L2),CONDI(IX,L2),CONDB(IX,L2),TERMF(IX,L2),TERMO(IX,L2),  
4TP(100),IB(100),IE(100),ZA(200),TMPTR(1000),TPRE(1000) ,  
5WRIT(10,12) ,MATTE(15)

EQUIVALENCE (Z(2),IOUT),(Z(5200),LB),(Z(11),LERR),  
1(Z(11716),XKTE(1)),(Z(5916),EPSI(1)),(Z(6016),TTE(1)),  
2(Z(11816),CAPTE(1)),(Z(5716),ALPHA(1)),(Z(5816),RESIST(1)),  
3(Z(17316),TP(1)) , (Z(4414),TMACH) , (Z(3246),WRIT(1)) ,  
4(ZA(54),XMAT ) , (Z(4415),MATC), (Z(4416),MATR)

EQUIVALENCE (Z (6116),IB(1)),(Z (6216),IE(1)),  
1(Z(627),ZA(1)),(ZA(4),UO),(ZA(5),VO),(ZA(6),U),(ZA(7),V),  
2(Z (4356),SOLC),(Z (4357),ALBE),(Z (4358),SIGMA),  
3(Z (4400),RE),(Z(5208),IMAXP),(Z ( 626),TPLC),  
4(Z(5467),TPLAN),(Z(5466),F1C),(Z(5465),F1R),(Z(5464),F2C),  
5(Z(5463),F2R),(Z(5468),DELTA),(Z(5206),TC),(Z(5207),TR),  
6(Z(5462),ALPHAC),(Z(5461),EPSC),(Z(5469),TCP),(Z(5460),CAPC),  
7(Z(5459),DELTMP),(Z(5470),TRP),(Z(5458),F3),(Z(5457),F1),  
8(Z(5456),F2),(Z(4411),EPST),(Z(4402),TIMAX),(Z(4412), NL),  
9(Z(4413), NS),(Z(4419),CFACT),(Z(5455),NIT),(Z(5454),NITMAX)  
EQUIVALENCE (Z(5452),TAVC4),(Z(5451),EPSAR),  
1(Z(5450),TAVR4),(Z(5449),EPSAC),(Z(5448),RI ),(Z(5447),E),  
2(Z(5446),VOLT),(Z(5445),CURRNT),(Z(5444),POWR),(Z(4410), ISS),  
3(Z(5442),T14),(Z(5441),TCP4),(Z(5440),CCR),(Z(5439),RCR),  
4(Z(5438),TIME),(Z(5437),TIML),(Z(5436),TIMS),(Z(4406),TIMPL),  
5(Z(4407),TIMPS),(Z(4405),TIMF),(Z(5480),TERM3),  
6(Z(5435),V1),(Z(5434),V2),(Z(5433),V3),(Z(5432),V4),  
7(Z(5431),V5),(Z(5430),V6),(Z(5429),MAY),(Z(5428),J),  
8(Z(5427),TXX),(Z(5426),I),(Z(5425),JL),(Z(5424),JU) ,  
9(ZA(75),TPLR)

EQUIVALENCE (Z(5423),TERM2),(Z(5422),TERM6),(Z(5421),TERMR),  
1(Z(5420),EA),(Z(5419),ATOT),(Z(5418),AXT),(Z(5417),JJ),  
2(Z(5416),II),(Z(5478),LP), (Z(5414),TERM4),  
3(Z(5413),TERM5),(Z(5412),TERM7),(Z(5411),Q),(Z(5410),TERM1),  
4(Z(5409),LBJ),(Z(5408),IC),(Z(5407),JF),(Z(5473),TEPL),  
5(Z(5476),EPSP),(Z(5406),LONG),(Z(5474),NLINE),(Z(14616),TMPTR(1))  
6,(Z(5405),LL),(Z(5404),LU),(Z(5403),IA),  
7(Z(5472),LMAX),(Z(5471),TPR),(Z(5401),JZ),(Z(5400),JY),  
8 (Z(4403),TMIN),(Z(4404),TMX),(Z(4401),SEGM),  
9(Z(5211),DELX),(ZA(166),XIMAX),(Z(18000),TPRE(1))

DO 7900 IN=1,10

DO 7900 JN=1,12

7900 WRIT(IN,JN)=0.

QO2=0.

NITMAX=TMX

TMULT=TIMF

TDEC=TMULT/100.

SLB=LB

LBNEW=SLB/SEGM+.01

JJZ=1

DO 7501 I=1,4

IF(LBNEW-JJZ)7501,7502,7501

7501 JJZ=2\*JJZ

```

      LBNEW=1
7502 WJZ=LBNEW
      NOSEG=LB/LBNEW
      IF(NOSEG-LB)7503,7504,7504
7503 IL=IB(1)
      IU=IE(1)
      DO 7505 I=IL,IU
      CONDB(I,1)=0.
7505 CONDB(I,LB+1)=0.
      IL=IB(NOSEG+1)
      J=LB+NOSEG+1
      IU=IE(J)
      DO 7506 I=IL,IU
      CONDB(I,NOSEG+1)=0.
7506 CONDB(I,J)=0.
7504 CONTINUE
      TIMF=0.
      JFLAG=0
      NLINE=0
      JCNT=TMACH
      IMAX=XIMAX
      KX=0
      NIT=0
      DELT=1.0
25  TERM3=SOLC*ALBE
      TIMS = TIMPS
      TIML = TIMPL
      DELTM=10.
607  NIT=NIT+1
      TIME=0.
      IF(NIT-100)7010,7011,7011
7010 TIMF=TIMF+TDEC
7011 JTE=XMAT
      JCOL=MATC
      JRAD=MATR
      JCCON=ZA(79)+1.E-4
      JRCON=ZA(119)+1.E-4
      CALL PROP(TC,V1,CAPC,ALPHAC,EPSC,EPS2C,V6,JCOL)
      CALL PROP(TR,V1,CAPR,ALPHAR,EPSR,EPS2R,V6,JRAD)
      DO 600 J=1,NOSEG
        I=IB(J)
603  TXX = T(I,J)
      CALL PROP ( TXX,V1,V2,V3,V4,V5,V6,JCOL)
      XK(I,J) =V1
      CAP(I,J)=V2
      ALPH(I,J)=V3
      EPSC1(I,J)=V4
      EPSC2(I,J)=V5
      IF(I-IE(J))601,600,600
601  I =I+1
      GO TO 603
600  CONTINUE
      JL= LB+1
      JU=NOSEG+LB
      DO 602 J=JL,JU
        I=IB(J)
605  TXX = T(I,J)
      CALL PROP(TXX,V1,V2,V3,V4,V5,V6,JRAD)
      XK(I,J) =V1
      CAP(I,J)=V2
      ALPH(I,J)=V3

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PRNT

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        EPSR1(I,J)=V4
        EPSR2(I,J)=V5
        IF(I-IE(J))604,602,602
604    I=I+1
        GO TO 605
602    CONTINUE
        DO 606 I=2,IMAXP
        TXX=TTE(I)
        CALL PROP(TXX,V1,V2,V3,V4,V5,V6,JTE)
        ALPHA(I)=V1
        XKTE(I)=V2
        RESIST(I)=V3
        CAPTE(I)=V4
        EPSI(I)=V5
606    CONTINUE
        MAY=17
        CALL PROP(TIME,TPLAN,F1C,F1R,F2C,F2R,DELTA,MAY)
        DELTA=DELTA/57.296
        CALL PROP(TC,V1,V2,CCR,V4,V5,V6,JCCON)
        CALL PROP(TR,V1,V2,RCR,V4,V5,V6,JRCON)
        TERM2=SIGMA*TPLAN*TPLAN*TPLAN*TPLAN
        IF(DELTA-1.5708)810,810,811
810    TERMC=SOLC*COS(DELTA)
        TERMR=.0
        GOTO 812
811    TERMR=SOLC*COS(DELTA)
        TERMC=.0
812    CONTINUE
21    EA=.0
        EPSAR=.0
        TAVR4=.0
        ATOT=.0
        AXT=.0
        JJ=LB+1
11    II=IB(JJ)
12    ATOT=ATOT+A(II,JJ)
        T4(II,JJ)=T(II,JJ)*T(II,JJ)*T(II,JJ)*T(II,JJ)
        AXT=AXT+A(II,JJ)*T4(II,JJ)
        EA=EA+A(II,JJ)*EPSR2(II,JJ)
        IF(II-IE(JJ))101,100,100
101    II=II+1
        GO TO 12
100    LP=LB+NOSEG
        IF(JJ-LP)103,102,102
103    JJ=JJ+1
        GO TO 11
102    TAVR4=AXT/ATOT
        EPSAR=EA/ATOT
        TAVR=SQRT(TAVR4)
        TAVR=SQRT(TAVR)
20    EA=.0
        TAVC4=.0
        ATOT=.0
        AXT=.0
        EPSAC=.0
        JJ=1
13    II=IB(JJ)
14    ATOT=ATOT+A(II,JJ)
        T4(II,JJ)=T(II,JJ)*T(II,JJ)*T(II,JJ)*T(II,JJ)
        AXT=AXT+A(II,JJ)*T4(II,JJ)
        EA=EA+A(II,JJ)*EPSR2(II,JJ)

```

```

      IF(II-IE(JJ))108,109,109
108  II=II+1
      GO TO 14
109  IF(JJ-NOSEG)110,111,111
110  JJ=JJ+1
      GO TO 13
111  TAVC4=AXT/ATOT
      EPSAC=EA/ATOT
      TAVC=SQRT(TAVC4)
      TAVC=SQRT(TAVC)
      E=(TTE(2)-TR)*      ALPHA(2)
      DO 380 I=3,IMAXP
380  E=E+(TTE(I)-TTE(I-1))*0.5*(ALPHA(I)+ALPHA(I-1))
      E=E+(TC-TTE(IMAXP))*      ALPHA(IMAXP)
      RI=0.
      DO 381 I=2,IMAXP
381  RI=RI+DELX*RESIST(I)
      RI=1./((U-UO)*(V-VO))*(RI)+RCR+CCR
      CURRNT=E/(RI+RE)
      VOLT=CURRNT*RE
      POWR=VOLT*CURRNT
      GO TO 802
7042 DELTM=.9*TIME/DTMAX
      TC=TPRE(1)+DELT*(TC-TPRE(1))
      TR=TPRE(2)+DELT*(TR-TPRE(2))
      LI=2
      DO 5019 I=2,IMAXP
      LI=LI+1
5019 TTE(I)=TPRE(LI)+DELT*(TTE(I)-TPRE(LI))
      LL=1
      LU=NOSEG
      DO 5020 J=  LL,LU
      IL=IB(J)
      IU=IE(J)
      DO 5021 I=IL,IU
      LI=LI+1
5021 T(I,J)=TPRE(LI)+DELT*(T(I,J)-TPRE(LI))
5020 CONTINUE
      LL=LB+1
      LU=LB+NOSEG
      DO 5022 J=LL,LU
      IL=IB(J)
      IU=IE(J)
      DO 5023 I=IL,IU
      LI=LI+1
5023 T(I,J)=TPRE(LI)+DELT*(T(I,J)-TPRE(LI))
5022 CONTINUE
      GO TO 2074
366  IF (NIT - NITMAX)607,367,367
367  CONTINUE
      JFLAG=1
      LONG=1
2001 QIC=0.
      QRC=0.
      QIR=0.
      QRR=0.
      QOC=0.
      QOR=0.
      QINC=TERMC*ALPHAC*(U-UO)*(V-VO)+TERM3*F2C*ALPHAC*(U-UO)*(V-VO)
      I+TERM2*F1C*(U-UO)*(V-VO)*EPSC
      QRAC=SIGMA*EPSC*TC**4*(U-UO)*(V-VO)

```

```

      QIN=QINC
7901 QOUT=QRAC
7902 QINR=TERMR*ALPHAR*(U-UO)*(V-VO)+TERM3*F2R*ALPHAR*(U-UO)
      1*(V-VO)+TERM2*F1R*(U-UO)*(V-VO)*EPSR
      QRAR=SIGMA*EPSR*TR**4*(U-UO)*(V-VO)
      QIN=QIN+QINR
7904 QOUT=QOUT+QRAR
      DO 7909 J=1,NOSEG
      IL=IB(J)
      IU=IE(J)
      DO 7910 I=IL,IU
      QIC=QIC+TERMC*A(I,J)*ALPH(I,J)+TERM3*A(I,J)*F2C*ALPH(I,J)
      1+TERM2*A(I,J)*F1C*EPSC1(I,J)
      QX=SIGMA*A(I,J)*EPSC1(I,J)*T(I,J)**4
7912 QRC=QRC+QX
      QOC=QOC+QX
7910 CONTINUE
7909 CONTINUE
      QIC=QIC*WJZ
      QRC=QRC*WJZ
      QOC=QOC*WJZ
      QIN=QIN+QIC
      QINC=QINC+QIC
      QRAC=QRAC+QRC
      QOUT=QOUT+QOC
      LL=LB+1
      LU=LB+NOSEG
      DO 7913 J=LL,LU
      IL=IB(J)
      IU=IE(J)
      DO 7914 I=IL,IU
      QIR=QIR+TERMR*A(I,J)*ALPH(I,J)+ALPH(I,J)*TERM3*A(I,J)*F2R
      1+TERM2*A(I,J)*F1R*EPSR1(I,J)
      QY=SIGMA*A(I,J)*EPSR1(I,J)*T(I,J)**4
7916 QOR=QOR+QY
      QRR=QRR+QY
7914 CONTINUE
7913 CONTINUE
      QOR=QOR*WJZ
      QRR=QRR*WJZ
      QIR=QIR*WJZ
      QIN=QIN+QIR
      QINR=QINR+QIR
      QRAR=QRAR+QRR
      QOUT=QOUT+QOR
      QAVG=.5*(QIN+QOUT+POWR)
      ERROR=(QIN-(QOUT+POWR))/QAVG
      EFF=100.*(POWR/QINC)
      KX=KX+1
8300 WRIT(KX,1)=TIME
      WRIT(KX,2)=DELT
      WRIT(KX,3)=NIT
      WRIT(KX,4)=POWR
      WRIT(KX,5)=CURRNT
      WRIT(KX,6)=ERROR
      WRIT(KX,7)=EFF
      WRIT(KX,8)=TC
      WRIT(KX,9)=TR
      WRIT(KX,10)=TAVR
      WRIT(KX,11)=TAVC
      IF(KX-10)8318,8301,8301

```

```

8318 IF(JFLAG)8302,8302,8301
8301 WRITE(IOUT,8303)(WRIT(KX,1),KX=1,10)
8303 FORMAT(1H0,1X,4HTIME,12X,10F11.4)
      WRITE(IOUT,8305)(WRIT(KX,2),KX=1,10)
8305 FORMAT(1H ,1X,14HTIME INCREMENT,2X,10F11.6)
      WRITE(IOUT,8306)(WRIT(KX,3),KX=1,10)
8306 FORMAT(1H ,1X,13HITERATION NO.,3X,10F11.1)
      WRITE(IOUT,8307)(WRIT(KX,4),KX=1,10)
8307 FORMAT(1H ,1X,5HPOWER,11X,10F11.5)
      WRITE(IOUT,8308)(WRIT(KX,5),KX=1,10)
8308 FORMAT(1H ,1X,7HCURRENT,9X,10F11.5)
      WRITE(IOUT,8309)(WRIT(KX,6),KX=1,10)
8309 FORMAT(1H ,1X,5HERROR,11X,10F11.6)
      WRITE(IOUT,8310)(WRIT(KX,7),KX=1,10)
8310 FORMAT(1H ,1X,10HEFFICIENCY,6X,10F11.6)
      WRITE(IOUT,8312)
      WRITE(IOUT,8311)(WRIT(KX,8),KX=1,10)
8312 FORMAT(1H ,1X,12HTEMPERATURES)
8311 FORMAT(1H ,2X,12HHOT JUNCTION,3X,10F11.4)
      WRITE(IOUT,8313)(WRIT(KX,9),KX=1,10)
8313 FORMAT(1H ,2X,13HCOLD JUNCTION,2X,10F11.4)
      WRITE(IOUT,8314)(WRIT(KX,10),KX=1,10)
8314 FORMAT(1H ,2X,14HRADIATOR (AVG),1X,10F11.4)
      WRITE(IOUT,8315)(WRIT(KX,11),KX=1,10)
8315 FORMAT(1H ,2X,14HABSORBER (AVG),1X,10F11.4)
      KX=0
8302 IF(LONG)2019,2019,2008
2008 WRITE(IOUT,2009)
2009 FORMAT(1H1,52X,12HTEMPERATURES//)
      WRITE(IOUT,2011)
2011 FORMAT(3X,4HNODE)
      WRITE(IOUT,2012)
2012 FORMAT(3X,6HNUMBER,6X,1H0,9X,1H1,9X,1H2,9X,1H3,9X,1H4,9X,1H5,9X,1H
16,9X,1H7,9X,1H8,9X,1H9)
      L=1
      DO 2050 I=2,IMAXP
        TMPTR(L)=TTE(I)
2050 L=L+1
        IF(LBNEW-1)7600,7600,7601
7601 JJZ=LBNEW/2
        IF(JJZ-3)7602,7602,7603
7603 JJZ=3
7602 JJK=1
        LLR=NOSEG
        NONE=0
7608 LLS=NOSEG+1+NONE
        DO 7604 JJY=1,JJZ
          LLS=(LLS-1-NONE)*2+1+NONE
          DO 7605 J=JJK,LLR
            LL=IB(J)
            LU=IE(J)
            DO 7606 I=LL,LU
              LXY=LLS-J +NONE
7606 T(I,LXY )=T(I,J)
7605 CONTINUE
7604 LLR=(LLR-NONE)*2+NONE
        IF(JJK-1)7607,7607,7600
7607 LLR=LB+NOSEG
        JJK=LB+1
        NONE=LB
        GO TO 7608

```



```

7600 DO 2051 J=1, LB
      IA=IB(J)
      IC=IE(J)
      DO 2052 I=IA, IC
        TMPTR(L)=T(I,J)
2052 L=L+1
2051 CONTINUE
      TMPTR(L)=TC
      L=L+1
      LL=LB+1
      LU=2*LB
      DO 2053 J=LL, LU
        IA=IB(J)
        IC=IE(J)
        DO 2054 I=IA, IC
          TMPTR(L)=T(I,J)
2054 L=L+1
2053 CONTINUE
      TMPTR(L)=TR
      LMAX=L
      WRITE(IOUT,2014) (TMPTR(L), L=1, 9)
2014 FORMAT(5X, 2H 0, 13X, 9F10.2)
      L=10
2016 WRITE(IOUT,2015) L, TMPTR(L), TMPTR(L+1), TMPTR(L+2), TMPTR(L+3), TMPTR(
1L+4), TMPTR(L+5), TMPTR(L+6), TMPTR(L+7), TMPTR(L+8), TMPTR(L+9)
2015 FORMAT(17, 3X, 10F10.2)
      L=L+10
      IF(L-LMAX) 2016, 2016, 2019
2019 CONTINUE
      IF(JFLAG) 7018, 7018, 7019
7019 RETURN
7018 GO TO 366
802 CONTINUE
      TERM4 = 2.* XKTE(IMAXP)*(U-U0)*(V-V0)/DELX
      TERM5 = .0
      TERM7 = .0
      DO 300 J=1, NOSEG
        I=IB(J)
        TERM5 = XK(I,J)*CONDI(I,J)*T(I,J) + TERM5
300  TERM7 = XK(I,J)*CONDI(I,J) + TERM7
        TERM5=TERM5*WJZ
        TERM7=TERM7*WJZ
        Q = CCR*CURRNT**2-ALPHA(IMAXP)*CURRNT*TC+TERMC*ALPHAC*(U-U0)*(V-
1VO)+TERM3*F2C*ALPHAC*(U-U0)*(V-V0)+TERM2*F1C*(U-U0)*(V-V0)*EPSC
        2-SIGMA*EPSC*TC**4*(U-U0)*(V-V0) +Q02
        TCP=TC
6016 TPRED(1)=TCP
6015 CONTINUE
        TC=(Q+ TERM4*TTE(IMAXP)+ TERM5)/(TERM4 + TERM7)
        TC = TCP + CFACT *(TC -TCP)
        DTMAX=ABS(TC-TCP)
        TERM4 = 2.*XKTE(2)*(U-U0)*(V-V0)/DELX
        TERM5 = .0
        TERM7 = .0
        Q=RCR*CURRNT**2 +ALPHA(2) *CURRNT*TR +TERMR*ALPHAR*(U-U0)*(V-V0)
1 +TERM3*F2R*ALPHAR*(U-U0)*(V-V0)+TERM2*F1R*(U-U0)*(V-V0)*EPSR
        2 -SIGMA*TR**4*(U-U0)*(V-V0) *EPSR +Q02
        LL=LB +1
        LU=LB+NOSEG
        DO 310 J=LL, LU
          I=IB(J)

```

```

        TERM5=XK(I,J)*CONDI(I,J)*T(I,J)      +TERM5
310  TERM7=XK(I,J)*CONDI(I,J)      +TERM7
        TERM5=TERM5*WJZ
        TERM7=TERM7*WJZ
        TRP=TR
6018  TPRED(2)=TRP
6017  CONTINUE
        TR=(Q+TERM4*TTE(2)+TERM5)/(TERM4+TERM7)
        TR=TRP+CFACT*(TR-TRP)
        DTMAXP=ABS(TR-TRP)
        IF(DTMAXP-DTMAX)311,311,312
312  DTMAX=DTMAXP
311  CONTINUE
        TCP4=.5*(TAVC4+TAVR4)
        LI=2
        Q02=0.
        DO 320 I=2,IMAXP
            XKB=.5*(XKTE(I)+XKTE(I-1))
            XKT=.5*(XKTE(I)+XKTE(I+1))
            LI=LI+1
            TPR=TTE(I)
6020  TPRED(LI)=TPR
            IF(I-2)321,321,322
321  TTE(I-1)=TRP
            XKB=2.*XKTE(I)
            ALPHA(I-1)=ALPHA(2)
322  IF(I-IMAXP)324,323,323
323  TTE(I+1)=TCP
            XKT=2.*XKTE(I)
            ALPHA(I+1)=ALPHA(IMAXP)
324  TI4=TTE(I)*TTE(I)*TTE(I)*TTE(I)
            Q01=-SIGMA*EPSI(I)*2.*(U-UO+V-VO)*DELX*(TI4-TCP4)
            Q=+CURRNT**2*RESIST(I)*DELX/((U-UO)*(V-VO))
            TTE(I)=(1./((XKB+XKT))*(Q*DELX/((U-UO)*(V-VO))+
1TPRED(LI-1)*XKB+TTE(I+1)*XKT)
            TTE(I)=TPR+CFACT*(TTE(I)-TPR)
            DTMAXP=ABS(TTE(I)-TPR)
            IF(DTMAXP-DTMAX)320,320,326
326  DTMAX=DTMAXP
320  Q02=Q02-Q01
            Q02=Q02/2.
            LIP=LI
            DO 7001 J=1,NOSEG
                IL=IB(J)
                IU=IE(J)
                DO 7002 I=IL,IU
                    LI=LI+1
7002  TPRED(LI)=T(I,J)
7001  CONTINUE
                JL=LB+1
                JU=NOSEG+LB
                DO 7003 J=JL,JU
                    IL=IB(J)
                    IU=IE(J)
                    DO 7004 I=IL,IU
                        LI=LI+1
7004  TPRED(LI)=T(I,J)
7003  CONTINUE
                LIP=LIP
                LBJ=NOSEG

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```

      IC=1
4071 IF(IC-1)331,330,331
      331 J=LB +1
      GO TO 332
      330 J=1
      332 IF(J-LBJ)333,334,333
      334 IF(IC-1)335,336,335
      333 JF=J+1
      GO TO 337
      336 JF=1
      GO TO 337
      335 JF=LB +1
      337 IF(J-1)339,338,339
      339 IF(J-LB -1)340,338,340
      338 JB=LBJ
      GO TO 341
      340 JB=J-1
      341 IL=IB(J)
      IU=IE(J)
      DO 1031 I=IL,IU
      IF(IC-1)342,343,342
      342 F3=1./((1./EPSR2(I,J) +1./EPSAC -1.)
      F1= F1R
      F2= F2R
      TEPL=TPRE(2)
      TERM1 = TERMR
      EPSP=EPSR1(I,J)
      TAV4 = TAVC4
      GO TO 344
      343 F3=1./((1./EPSC2(I,J) +1./EPSAR -1.)
      F1 =F1C
      F2 =F2C
      TEPL=TPRE(1)
      TERM1 =TERMC
      TAV4 =TAVR4
      EPSP=EPSC1(I,J)
      344 Q=TERM1*A(I,J)*ALPH(I,J) +TERM2*A(I,J)*F1*EPSP      +TERM3*A(I,J) *
      1F2*ALPH(I,J) -SIGMA *A(I,J)*EPSP      *T4(I,J)-SIGMA*A(I,J)*F3
      2*(T4(I,J)-TAV4)
      LI=LI+1
      LIP=LI-(IE(J-1)-IB(J)+1 )
      IF(LBJ-LB)6031,6031,6032
      6031 LIPP=LIP+LB-1+IE(1)-IB(LB)
      LLL=2
      LLU=LB-1
      DO 6033 JN=LLL,LLU
      6033 LIPP=LIPP+IE(JN)-IB(JN)
      GO TO 6035
      6032 LXY=2*LB
      LZZ=LB+1
      LIPP=LIP+LB-1+IE(LZZ)-IB(LXY)
      LLL=LB+2
      LLU=2*LB-1
      DO 6034 JN=LLL,LLU
      6034 LIPP=LIPP+IE(JN)-IB(JN)
      6035 CONTINUE
      IF(I-IB(J))345,346,345
      346 TERMO(I-1,J)= - CONDI(I,J)*TEPL
      GO TO 5071
      345 TERMO(I-1,J)=-CONDI(I,J)*TPRE(LI-1)
      5071 IF(J-1)347,348,347

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347 IF(J-LB -1)349,348,349
348 TERMF(I,JB) = -CONDB(I,J)*TPRE(LIPP)
    TP(I) = T(I,J)
    GO TO 5070
349 TERMF(I,JB)=-CONDB(I,J)*TPRE(LIP)
5070 TERMO(I,J)= -CONDI(I+1,J)*T(I+1,J)
    IF(J-LBJ)350,351,350
351 TERMF(I,J) = -CONDB(I,JF)*TP(I)
    GO TO 2070
350 TERMF(I,J) = -CONDB(I,JF)*T(I,JF)
2070 TPR = T(I,J)
6022 TPRE(LI)=TPR
    COND=CONDI(I+1,J)+CONDB(I,JF)
    I+CONDI(I,J)+CONDB(I,J)
    T(I,J)=Q/(XK(I,J)*COND)+(-TERMO(I,J)-TERMF(I,J)
    I-TERMO(I-1,J)-TERMF(I,JB ))/COND
    T(I,J)=TPR + CFACT*(T(I,J)-TPR)
    DTMAXP=ABS(T(I,J)-TPR)
    IF(DTMAXP-DTMAX)1031,1031,353
353 DTMAX = DTMAXP
1031 CONTINUE
1032 IF(J-LBJ)1033,352,352
1033 J=J+1
    GO TO 332
352 IF(IC-1)2061,355,2061
355 IC=0
    LBJ=NOSEG+LB
    GO TO 4071
2061 IF(DTMAX-EPST)367,367,6011
6011 IF(NIT-JCNT)7014,7014,7013
7013 CFACT=0.7
    TIME=.0
    GO TO 7015
7014 CFACT=CFACT/DTMAX*TIME*.9
7015 IF(DTMAX-TIME)2074,2074,7042
2074 JZ=MOD(NIT,NL )
    IF(JZ)2076,2075,2076
2075 LONG=1
    GO TO 2001
2076 JY=MOD(NIT,NS )
    IF(JY)366,2078,366
2078 LONG=0
    GO TO 2001
    END

```

SIBFTC CALCT DECK,XR7

SUBROUTINE CALCT(XK,CAP,EPSC1,EPSC2,EPSR1,EPSR2,ALPH,T,A,T4,  
1CONDI,CONDB,TERMF,TERMO,IX,L2, L22)

COMMON Z

DIMENSION Z(20000),XK(IX,L2),XKTE( 100 ),CAP(IX,L2),CAPTE( 100 ),  
1ALPHA(100),RESIST(100),EPSC1(IX ,L22),EPSC2(IX ,L22),EPSR1(IX ,L22  
2),EPSR2(IX ,L22),ALPH(IX,L2),EPSI(100),T(IX,L2),TTE(100),A(IX,L2),  
3T4(IX,L2),CONDI(IX,L2),CONDB(IX,L2),TERMF(IX,L2),TERMO(IX,L2),  
4TP(100),IB(100),IE(100),ZA(200),TMPTR(1000),TPRE(1000) ,  
5WRIT(10,12) ,MATTE(15)

EQUIVALENCE (Z(2),IOUT),(Z(5200),LB),(Z(11),LERR),

1(Z(11716),XKTE(1)),(Z(5916),EPSI(1)),(Z(6016),TTE(1)),  
2(Z(11816),CAPTE(1)),(Z(5716),ALPHA(1)),(Z(5816),RESIST(1)),  
3(Z(17316),TP(1)) ,(Z(4414),TMACH) ,(Z(3246),WRIT(1)) ,  
4(ZA(54),XMAT ) ,(Z(4415),MATC),(Z(4416),MATR)

EQUIVALENCE (Z (6116),IB(1)),(Z (6216),IE(1)),

1(Z(627),ZA(1)),(ZA(4),UO),(ZA(5),VO),(ZA(6),U),(ZA(7),V),  
2(Z (4356),SOLC),(Z (4357),ALBE),(Z (4358),SIGMA),  
3(Z (4400),RE),(Z(5208),IMAXP),(Z ( 626),TPLC),  
4(Z(5467),TPLAN),(Z(5466),F1C),(Z(5465),F1R),(Z(5464),F2C),  
5(Z(5463),F2R),(Z(5468),DELTA),(Z(5206),TC),(Z(5207),TR),  
6(Z(5462),ALPHAC),(Z(5461),EPSC),(Z(5469),TCP),(Z(5460),CAPC),  
7(Z(5459),DELTMP),(Z(5470),TRP),(Z(5458),F3),(Z(5457),F1),  
8(Z(5456),F2),(Z(4411),EPST),(Z(4402),TIMAX),(Z(4412), NL),  
9(Z(4413), NS),(Z(4419),CFACT),(Z(5455),NIT),(Z(5454),NITMAX)

EQUIVALENCE (Z(5452),TAVC4),(Z(5451),EPSAR),

1(Z(5450),TAVR4),(Z(5449),EPSAC),(Z(5448),RI ),(Z(5447),E),  
2(Z(5446),VOLT),(Z(5445),CURRNT),(Z(5444),POWR),(Z(4410), ISS),  
3(Z(5442),TI4),(Z(5441),TCP4),(Z(5440),CCR),(Z(5439),RCR),  
4(Z(5438),TIME),(Z(5437),TIML),(Z(5436),TIMS),(Z(4406),TIMPL),  
5(Z(4407),TIMPS),(Z(4405),TIMF),(Z(5480),TERM3),  
6(Z(5435),V1),(Z(5434),V2),(Z(5433),V3),(Z(5432),V4),  
7(Z(5431),V5),(Z(5430),V6),(Z(5429),MAY),(Z(5428),J),  
8(Z(5427),TXX),(Z(5426),I),(Z(5425),JL),(Z(5424),JU) ,  
9(ZA(75),TPLR)

EQUIVALENCE (Z(5423),TERM2),(Z(5422),TERM6),(Z(5421),TERMR),

1(Z(5420),EA),(Z(5419),ATOT),(Z(5418),AXT),(Z(5417),JJ),  
2(Z(5416),II),(Z(5478),LP), (Z(5414),TERM4),  
3(Z(5413),TERM5),(Z(5412),TERM7),(Z(5411),Q),(Z(5410),TERM1),  
4(Z(5409),LBJ),(Z(5408),IC),(Z(5407),JF),(Z(5473),TEPL),  
5(Z(5476),EPSP),(Z(5406),LONG),(Z(5474),NLINE),(Z(14616),TMPTR(1))  
6,(Z(5405),LL),(Z(5404),LU),(Z(5403),IA),  
7(Z(5472),LMAX),(Z(5471),TPR),(Z(5401),JZ),(Z(5400),JY),  
8 (Z(4403),TMIN),(Z(4404),TMX),(Z(4401),SEGM),  
9(Z(5211),DELX),(ZA(166),XIMAX),(Z(18000),TPRE(1))

DO 310 IN=1,10

DO 310 JN=1,12

310 WRIT(IN,JN)=0.

QO2=0.

DELTm=10.

NITMAX=TMX

TMULT=TIMF

TIME=0.

TDEC=TMULT/100.

SLB=LB

LBNEW=SLB/SEGM+.01

JJZ=1

DO 7501 I=1,4

IF(LBNEW-JJZ)7501,7502,7501

7501 JJZ=2\*JJZ

LBNEW=1.

```

7502 WJZ=LBNEW
      NOSEG=LB/LBNEW
      IF(NOSEG-LB)7503,7504,7504
7503 IL=IB(1)
      IU=IE(1)
      DO 7505 I=IL,IU
      CONDB(I,1)=0.
7505 CONDB(I,LB+1)=0.
      IL=IB(NOSEG+1)
      J=LB+NOSEG+1
      IU=IE(J)
      DO 7506 I=IL,IU
      CONDB(I,NOSEG+1)=0.
7506 CONDB(I,J)=0.
7504 CONTINUE
      TIMF=0.
      JFLAG=0
      NLINE=0
      JCNT=TMACH
      IMAX=XIMAX
      KX=0
      NIT=0
      DELT=1.0
25  TERM3=SOLC*ALBE
      TIMS = TIMPS
      TIML = TIMPL
607  NIT=NIT+1
      IF(NIT-100)7010,7011,7011
7010 TIMF=TIMF+TDEC
7011 JTE=XMAT
      JCOL=MATC
      JRAD=MATR
      JCCON=ZA(79)+1.E-4
      JRCON=ZA(119)+1.E-4
      CALL PROP(TC,V1,CAPC,ALPHAC,EPSC,EPS2C,V6,JCOL)
      CALL PROP(TR,V1,CAPR,ALPHAR,EPSR,EPS2R,V6,JRAD)
      DO 600 J=1,NOSEG
        I=IB(J)
603  TXX = T(I,J)
      CALL PROP ( TXX,V1,V2,V3,V4,V5,V6,JCOL)
      XK(I,J) =V1
      CAP(I,J)=V2
      ALPH(I,J)=V3
      EPSC1(I,J)=V4
      EPSC2(I,J)=V5
      IF(I-IE(J))601,600,600
601  I =I+1
      GO TO 603
600  CONTINUE
      JL= LB+1
      JU=NOSEG+LB
      DO 602 J=JL,JU
        I=IB(J)
605  TXX = T(I,J)
      CALL PROP(TXX,V1,V2,V3,V4,V5,V6,JRAD)
      XK(I,J) =V1
      CAP(I,J)=V2
      ALPH(I,J)=V3
      EPSR1(I,J)=V4
      EPSR2(I,J)=V5
      IF(I-IE(J))604,602,602

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PRNT

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604 I=I+1
GO TO 605
602 CONTINUE
DO 606 I=2,IMAXP
TXX=TTE(I)
CALL PROP(TXX,V1,V2,V3,V4,V5,V6,JTE)
ALPHA(I)=V1
XKTE(I)=V2
RESIST(I)=V3
CAPTE(I)=V4
EPSI(I)=V5
606 CONTINUE
MAY=17
CALL PROP(TIME,TPLAN,F1C,F1R,F2C,F2R,DELTA,MAY)
DELTA=DELTA/57.296
CALL PROP(TC,V1,V2,CCR,V4,V5,V6,JCCON)
CALL PROP(TR,V1,V2,RCR,V4,V5,V6,JRCON)
TERM2=SIGMA*TPLAN*TPLAN*TPLAN*TPLAN
IF(DELTA-1.5708)810,810,811
810 TERMC=SOLC*COS(DELTA)
TERMR=.0
GOTO 812
811 TERMR=SOLC*COS(DELTA)
TERMC=.0
812 CONTINUE
21 EA=.0
EPSAR=.0
TAVR4=.0
ATOT=.0
AXT=.0
JJ=LB+1
11 II=IB(JJ)
12 ATOT=ATOT+A(II,JJ)
T4(II,JJ)=T(II,JJ)*T(II,JJ)*T(II,JJ)*T(II,JJ)
AXT=AXT+A(II,JJ)*T4(II,JJ)
EA=EA+A(II,JJ)*EPSR2(II,JJ)
IF(II-IE(JJ))101,100,100
101 II=II+1
GO TO 12
100 LP=LB+NOSEG
IF(JJ-LP)103,102,102
103 JJ=JJ+1
GO TO 11
102 TAVR4=AXT/ATOT
EPSAR=EA/ATOT
TAVR=SQRT(TAVR4)
TAVR=SQRT(TAVR)
20 EA=.0
TAVC4=.0
ATOT=.0
AXT=.0
EPSAC=.0
JJ=1
13 II=IB(JJ)
14 ATOT=ATOT+A(II,JJ)
T4(II,JJ)=T(II,JJ)*T(II,JJ)*T(II,JJ)*T(II,JJ)
AXT=AXT+A(II,JJ)*T4(II,JJ)
EA=EA+A(II,JJ)*EPSC2(II,JJ)
IF(II-IE(JJ))108,109,109
108 II=II+1
GO TO 14

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109 IF(JJ-NOSEG)110,111,111
110 JJ=JJ+1
GO TO 13
111 TAVC4=AXT/ATOT
EPSAC=EA/ATOT
TAVC=SQRT(TAVC4)
TAVC=SQRT(TAVC)
E=(TTE(2)-TR)* ALPHA(2)
DO 380 I=3,IMAXP
380 E=E+(TTE(I)-TTE(I-1))* .5*(ALPHA(I)+ALPHA(I-1))
E=E+(TC-TTE(IMAXP))* ALPHA(IMAXP)
RI=0.
DO 381 I=3,IMAXP
381 RI=RI+DELX*.5*(RESIST(I)+RESIST(I-1))
RI=1./((U-UO)*(V-VO))*(RI)+RCR+CCR
CURRNT=E/(RI+RE)
VOLT=CURRNT*RE
POWR=VOLT*CURRNT
801 CONTINUE
DELTS=DELT
TERM6=DELT/(TPLC*(U-UO)*(V-VO))
TERM4=(XKTE(IMAXP)*(U-UO)*(V-VO)*(TTE(IMAXP)-TC) )*.5/DELX
TERM5=.0
DO 226 J=1,NOSEG
I=IB(J)
TERM5= TERM5 +XK(I,J)*CONDI(I,J)*(T(I,J)-TC)
226 CONTINUE
TERM5=TERM5*WJZ
ALPC=ALPHA(IMAXP)
Q = CCR*CURRNT**2-ABS(ALPC )*CURRNT*TC+TERMC*ALPHAC*(U-UO)*(V-
1VO) +TERM3*F2C*ALPHAC*(U-UO)*(V-VO)+TERM2*F1C*(U-UO)*(V-VO)*EPSC
2-SIGMA*EPSC*TC**4*(U-UO)*(V-VO) +QO2
TPRE(1)=TC
TC= TC +TERM6/CAPC*(Q +TERM4 +TERM5)
IF(ABS(TC)-1.0E+05)5025,367,367
5025 CONTINUE
DTMAX =ABS(TC-TPRE(1))
TERM6=DELT/(TPLR*(U-UO)*(V-VO))
TERM4= 2.*XKTE(2)*(U-UO)*(V-VO)*(TTE(2)-TR) /DELX
TERM5=.0
LL=LB+1
LU=LB+NOSEG
DO 251 J=LL,LU
I=IB(J)
251 TERM5=TERM5 +XK(I,J)*CONDI(I,J)*(T(I,J)-TR)
TERM5=TERM5*WJZ
ALPR=ALPHA(2)
Q = RCR *CURRNT**2 +ABS(ALPR )*CURRNT*TR+TERMR*ALPHAR*(U-UO)*(V
1-VO) +TERM3*F2R*ALPHAR*(U-UO)*(V-VO)+TERM2*F1R*(U-UO)*(V-VO)*EPSR
2-SIGMA*EPSR*TR**4*(U-UO)*(V-VO) +QO2
TPRE(2)=TR
TR= TR+ TERM6/CAPR*(Q+TERM4+TERM5)
DTMAXP=ABS(TR-TPRE(2))
IF(DTMAXP-DTMAX)5005,5005,5006
5006 DTMAX=DTMAXP
5005 CONTINUE
TCP4=.5*(TAVC4+TAVR4)
LI=2
DO 7000 I=2,IMAXP
LI=LI+1
7000 TPRE(LI)=TTE(I)

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LI=2
QO2=0.
DO 200 I=2,IMAXP
QQ=1.
QR=1.
QY=2.
LI=LI+1
TXB=TPRE(LI-1)
TXA=TPRE(LI+1)
IF(I-2)201,201,202
201 ALPHA(I-1)=ALPHA(2)
QQ=2.
QY=3.
202 IF(I-IMAXP)204,203,203
203 TXA      =TPRE(1)
QR=2.
QY=3.
ALPHA(I+1)=ALPHA(IMAXP)
204 TI4=TTE(I)*TTE(I)*TTE(I)*TTE(I)
QO1=-SIGMA*EPSI(I)*2.*(U-UO+V-VO)*DELX*(TI4-TCP4)
Q=CURRNT**2*RESIST(I)*DELX/((U-UO)*(V-VO))
2-CURRNT*TTE(I)*(ALPHA(I)-ALPHA(I-1)) +QO1
RHS=-XKTE(I)*(U-UO)*(V-VO)/DELX*(-QQ*TXB-QR*TXA
1+QY*TPRE(LI))+Q
TTE(I)=TTE(I)+DELT/(CAPTE(I)*(U-UO)*(V-VO)*DELX)*RHS
5039 DTMAXP=ABS(TTE(I)-TPRE(LI))
IF(DTMAXP-DTMAX)5007,5007,5008
5008 DTMAX=DTMAXP
5007 QO2=QO2-QO1
QO2=QO2/2.
200 CONTINUE
LBJ=NOSEG
24 IC=1
10 IF(IC-1)112,113,112
112 J=LB +1
GO TO 4
113 J=1
4 IF(J-LBJ)115,114,115
115 JF=J+1
GO TO 116
114 IF(IC-1)118,117,118
118 JF=LB +1
GO TO 116
117 JF=1
116 IF(J-1)120,121,120
120 IF(J-LB -1)122,121,122
121 JB=LBJ
GO TO 2
122 JB=J-1
2 IL=IB(J)
IU=IE(J)
DO 1002 I=IL,IU
129 IF(IC-1)131,130,131
131 F3=1./(1./EPSR2(I,J)+1./EPSAC-1.)
TEPL=TPRE(2)
TERM1=TERMR
F1=F1R
F2=F2R
EPSP=EPSR1(I,J)
THPL =TPLR
TAV4=TAVC4

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GO TO 26
130 F3=1./(1./EPSC2(I,J)+1./EPSAR-1.)
    TEPL=TPRE(1)
    TERM1=TERMC
    TAV4 = TAVR4
    F1=F1C
    F2=F2C
    EPSP=EPSC1(I,J)
    THPL =TPLC
26 Q=TERM1*A(I,J)*ALPH(I,J) + ALPH(I,J) * TERM3*A(I,J)*F2 -SIGMA
1*A(I,J)*EPSP *T4(I,J)-SIGMA*A(I,J)*F3*(T4(I,J)-TAV4)
2+TERM2*A(I,J)*F1 *EPSP
    LI=LI+1
    TPRE(LI)=T(I,J)
    IF(I-IB(J))141,140,141
140 TERMO(I-1,J)=-CONDI(I,J)*(T(I,J)-TEPL)
141 IF(J-1)143,142,143
143 IF(J-LB -1)821,142,821
142 TERMF(I,JB) = -CONDB(I,J)*(T(I,J)-T(I,JB))
    TP(I)= T(I,J)
821 TERMO(I,J)=CONDI(I+1,J)*(T(I,J)-T(I+1,J))
    IF(J-LBJ)145,144,145
144 TERMF(I,J)=CONDB(I,JF)*(T(I,J)-TP(I))
    GO TO 2071
145 TERMF(I,J)=CONDB(I,JF)*(T(I,J)-T(I,JF))
2071 T(I,J)=T(I,J)+ DELT/CAP(I,J)*(Q+XK(I,J)*(-TERMO(I,J)-TERMF(I,J)
1+TERMO(I-1,J) + TERMF(I,JB)))/(A(I,J)*THPL)
5041 DTMAXP=ABS(TPRE(LI)-T(I,J))
    IF(DTMAXP-DTMAX)5009,5009,5010
5010 DTMAX=DTMAXP
5009 CONTINUE
1002 CONTINUE
1003 IF(J-LBJ)1004,147,147
1004 J=J+1
    GO TO 4
147 IF(IC-1)27,149,27
149 IC=0
    LBJ=LB+NSEG
    GO TO 10
27 TIME=TIME+DELT
    GOTO 365
5013 CONTINUE
5043 IF(NIT-NITMAX)1006,367,367
1006 CONTINUE
    JZ=MOD(NIT,NL )
    IF(JZ)5030,362,5030
5030 JY=MOD(NIT,NS)
    IF(JY)5031,364,5031
5031 CONTINUE
    IF(TIME-TIML )361,361,362
362 LONG=1
    NLINE=0
    TIML= TIML + TIMPL
    TIMS= TIMS + TIMPS
    GO TO 2001
361 IF(TIME-TIMS)5002,5002,364
364 LONG=0
    TIMS=TIMS + TIMPS
    GO TO 2001
365 CONTINUE
    IF(DTMAX-TIMF)5014,5014,5012

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5014 DELT=.9*TIME/DTMAX*DELT
GO TO 5013
5012 TIME=TIME-DELT
DELT=DELT*DELT
TIME=TIME+DELT
7042 DELTM=.9*TIME/DTMAX
TC=TPRE(1)+DELT*(TC-TPRE(1))
TR=TPRE(2)+DELT*(TR-TPRE(2))
LI=2
DO 5019 I=2,IMAXP
LI=LI+1
5019 TTE(I)=TPRE(LI)+DELT*(TTE(I)-TPRE(LI))
LL=1
LU=NOSEG
DO 5020 J=LL,LU
IL=IB(J)
IU=IE(J)
DO 5021 I=IL,IU
LI=LI+1
5021 T(I,J)=TPRE(LI)+DELT*(T(I,J)-TPRE(LI))
5020 CONTINUE
LL=LB+1
LU=LB+NOSEG
DO 5022 J=LL,LU
IL=IB(J)
IU=IE(J)
DO 5023 I=IL,IU
LI=LI+1
5023 T(I,J)=TPRE(LI)+DELT*(T(I,J)-TPRE(LI))
5022 CONTINUE
GO TO 5013
5002 IF (TIME-TIMAX) 366,366,367
366 IF (NIT - NITMAX) 607,367,367
367 CONTINUE
JFLAG=1
LONG=1
2001 QIC=0.
QRC=0.
QIR=0.
QRR=0.
QOC=0.
QOR=0.
QINC=TERMC*ALPHAC*(U-UO)*(V-VO)+TERM3*F2C*ALPHAC*(U-UO)*(V-VO)
1+TERM2*F1C*(U-UO)*(V-VO)*EPSC
QRAC=SIGMA*EPSC*TC**4*(U-UO)*(V-VO)
QIN=QINC
7900 QOUT=QRAC+CAPC*(U-UO)*(V-VO)*TPLC*(TC-TPRE(1))/DELTS
7902 QINR=TERMR*ALPHAR*(U-UO)*(V-VO)+TERM3*F2R*ALPHAR*(U-UO)
1*(V-VO)+TERM2*F1R*(U-UO)*(V-VO)*EPSR
QRAR=SIGMA*EPSR*TR**4*(U-UO)*(V-VO)
QIN=QIN+QINR
7903 QOUT=QOUT+QRAR+CAPR*(U-UO)*(V-VO)*TPLR*(TR-TPRE(2))/DELTS
7905 LI=2
DO 7906 I=2,IMAXP
LI=LI+1
7906 CONTINUE
DO 7909 J=1,NOSEG
IL=IB(J)
IU=IE(J)
DO 7910 I=IL,IU
LI=LI+1

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      QIC=QIC+TERMC*A(I,J)*ALPH(I,J)+TERM3*A(I,J)*F2C*ALPH(I,J)
1+TERM2*A(I,J)*F1C*EPSC1(I,J)
      QX=SIGMA*A(I,J)*EPSC1(I,J)*T(I,J)**4
7911  QOC=QOC+QX+CAP(I,J)*A(I,J)*TPLC*(T(I,J)-TPRE(LI))/DELTS
      QRC=QRC+QX
7910  CONTINUE
7909  CONTINUE
      QIC=QIC*WJZ
      QRC=QRC*WJZ
      QOC=QOC*WJZ
      QIN=QIN+QIC
      QINC=QINC+QIC
      QRAC=QRAC+QRC
      QOUT=QOUT+QOC
      LL=LB+1
      LU=LB+NSEG
      DO 7913 J=LL,LU
        IL=IB(J)
        IU=IE(J)
        DO 7914 I=IL,IU
          LI=LI+1
          QIR=QIR+TERMR*A(I,J)*ALPH(I,J)+ALPH(I,J)*TERM3*A(I,J)*F2R
1 +TERM2*A(I,J)*F1R*EPSR1(I,J)
          QY=SIGMA*A(I,J)*EPSR1(I,J)*T(I,J)**4
7915  QOR=QOR+QY+CAP(I,J)*A(I,J)*TPLR*(T(I,J)-TPRE(LI))/DELTS
          QRR=QRR+QY
7914  CONTINUE
7913  CONTINUE
          QOR=QOR*WJZ
          QRR=QRR*WJZ
          QIR=QIR*WJZ
          QIN=QIN+QIR
          QINR=QINR+QIR
          QRAR=QRAR+QRR
          QOUT=QOUT+QOR
          QAVG=.5*(QIN+QOUT+POWR)
          ERROR=(QIN-(QOUT+POWR))/QAVG
          EFF=100.*(POWR/QINC)
          KX=KX+1
8300  WRIT(KX,1)=TIME
      WRIT(KX,2)=DELT
      WRIT(KX,3)=NIT
      WRIT(KX,4)=POWR
      WRIT(KX,5)=CURRNT
      WRIT(KX,6)=ERROR
      WRIT(KX,7)=EFF
      WRIT(KX,8)=TC
      WRIT(KX,9)=TR
      WRIT(KX,10)=TAVR
      WRIT(KX,11)=TAVC
      IF(KX-10)8318,8301,8301
8318  IF(JFLAG)8302,8302,8301
8301  WRITE(IOUT,8303)(WRIT(KX,1),KX=1,10)
8303  FORMAT(1H,1X,4HTIME,12X,10F11.4)
      WRITE(IOUT,8305)(WRIT(KX,2),KX=1,10)
8305  FORMAT(1H,1X,14HTIME INCREMENT,2X,10F11.6)
      WRITE(IOUT,8306)(WRIT(KX,3),KX=1,10)
8306  FORMAT(1H,1X,13HITERATION NO.,3X,10F11.1)
      WRITE(IOUT,8307)(WRIT(KX,4),KX=1,10)
8307  FORMAT(1H,1X,5HPOWER,11X,10F11.5)
      WRITE(IOUT,8308)(WRIT(KX,5),KX=1,10)

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8308 FORMAT(1H ,1X,7HCURRENT,9X,10F11.5)
      WRITE(IOUT,8309)(WRIT(KX,6),KX=1,10)
8309 FORMAT(1H ,1X,5HERROR,11X,10F11.6)
      WRITE(IOUT,8310)(WRIT(KX,7),KX=1,10)
8310 FORMAT(1H ,1X,10HEFFICIENCY,6X,10F11.6)
      WRITE(IOUT,8312)
      WRITE(IOUT,8311)(WRIT(KX,8),KX=1,10)
8312 FORMAT(1H ,1X,12HTEMPERATURES)
8311 FORMAT(1H ,2X,12HHOT JUNCTION,3X,10F11.4)
      WRITE(IOUT,8313)(WRIT(KX,9),KX=1,10)
8313 FORMAT(1H ,2X,13HCOLD JUNCTION,2X,10F11.4)
      WRITE(IOUT,8314)(WRIT(KX,10),KX=1,10)
8314 FORMAT(1H ,2X,14HRADIATOR (AVG),1X,10F11.4)
      WRITE(IOUT,8315)(WRIT(KX,11),KX=1,10)
8315 FORMAT(1H ,2X,14HABSORBER (AVG),1X,10F11.4)
      KX=0
8302 IF(LONG)2019,2019,2008
2008 WRITE(IOUT,2009)
2009 FORMAT(1H1,52X,12HTEMPERATURES//)
      WRITE(IOUT,2011)
2011 FORMAT(3X,4HNODE)
      WRITE(IOUT,2012)
2012 FORMAT(3X,6HNUMBER,6X,1H0,9X,1H1,9X,1H2,9X,1H3,9X,1H4,9X,1H5,9X,1H
16,9X,1H7,9X,1H8,9X,1H9)
      L=1
      DO 2050 I=2,IMAXP
        TMPTR(L)=TTE(I)
2050 L=L+1
      IF(LBNEW-1)7600,7600,7601
7601 JJZ=LBNEW/2
      IF(JJZ-3)7602,7602,7603
7603 JJZ=3
7602 JJK=1
      LLR=NOSEG
      NONE=0
7608 LLS=NOSEG+1+NONE
      DO 7604 JJY=1,JJZ
        LLS=(LLS-1-NONE)*2+1+NONE
        DO 7605 J=JJK,LLR
          LL=IB(J)
          LU=IE(J)
          DO 7606 I=LL,LU
            LXY=LLS-J +NONE
7606 T(I,LXY)=T(I,J)
7605 CONTINUE
7604 LLR=(LLR-NONE)*2+NONE
      IF(JJK-1)7607,7607,7600
7607 LLR=LB+NOSEG
      JJK=LB+1
      NONE=LB
      GO TO 7608
7600 DO 2051 J=1,LB
      IA=IB(J)
      IC=IE(J)
      DO 2052 I=IA,IC
        TMPTR(L)=T(I,J)
2052 L=L+1
2051 CONTINUE
      TMPTR(L)=TC
      L=L+1
      LL=LB+1

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```
LU=2*LB
DO 2053 J=LL,LU
IA=IB(J)
IC=IE(J)
DO 2054 I=IA,IC
TMPTR(L)=T(I,J)
2054 L=L+1
2053 CONTINUE
TMPTR(L)=TR
LMAX=L
WRITE(IOUT,2014) (TMPTR(L),L=1,9)
2014 FORMAT(5X,2H 0,13X,9F10.2)
L=10
2016 WRITE(IOUT,2015)L,TMPTR(L),TMPTR(L+1),TMPTR(L+2),TMPTR(L+3),TMPTR(
1L+4),TMPTR(L+5),TMPTR(L+6),TMPTR(L+7),TMPTR(L+8),TMPTR(L+9)
2015 FORMAT(17,3X,10F10.2)
L=L+10
IF(L-LMAX)2016,2016,2019
2019 CONTINUE
IF(JFLAG)7018,7018,7019
7019 RETURN
7018 GO TO 5002
END
```